United States Department of Agriculture

Natural Resources Conservation Service Part 636 Structural Engineering National Engineering Handbook

Chapter 52

Structural Design of Flexible Conduits

Issued June 2005

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Preface

Flexible conduits used on NRCS projects typically consist of corrugated metal pipe (CMP), various types of plastic pipe, steel pipe, or ductile iron pipe. The design of these conduits was completed by allowable fill height tables in various Conservation Practice Standards, guidance given in TR 77—Design and Installation of Flexible Conduits and the associated computer program, and multiple technical notes developed by NRCS staff.

NEH 636 chapter 52 updates the design procedure to current industry and government agency practice. Although symbols for conduit (pipe) design vary among types of materials and industry guidance, those used in chapter 52 are consistent within the document (see appendix 52A). Appendix 52B contains several design examples that were developed using the formulas and information in this chapter. A glossary of terms used within this chapter is included following the references and prior to the appendices.

Acknowledgments

The technical guidance in this document is a compilation of flexible conduit design guidance from the American Society of Testing Materials (ASTM), American Association of State Highway Transportation Officials (AASHTO), other Federal agencies, trade organizations, pipe manufacturers, and other text. This version was prepared by **Wade Anderson**, design engineer, National Design, Construction, and Soil Mechanics Center, Natural Resources Conservation Service (NRCS), Fort Worth, Texas.

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Special thanks also to National Cartography and Geospatial Center's Technical Publishing Team members: **Mary Mattinson**, for her guidance and editing, **Suzi Self** for her desktop publishing, review, and editing, and **Wendy Pierce** for the graphic illustrations.

Chapter 52

Structural Design of Flexible Conduits

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Chapter 52

Structural Design of Flexible Conduits

636.5200 Introduction

Pipe materials are generally considered to be rigid or flexible. A flexible pipe is one that will deflect at least 2 percent without structural distress (fig. 52–1). Materials that do not meet this criterion are generally considered rigid. Some pipe materials are described as semi-rigid based on their behavior and design procedures.

A flexible conduit derives its external load capacity from its flexibility. Under load, the pipe tends to deflect, developing soil support at the sides of the pipe. The ring deflection (fig. 52–1) relieves the pipe of the major portion of the vertical soil load, which is then transferred to the soil surrounding the pipe through the soil arching action over the pipe.

Flexible pipe materials consist of smooth-wall steel pipe, corrugated spiral rib or composite ribbed metal pipe (fig. 52–2), ductile iron pipe, and solid-wall, corrugated-wall, or profile-wall thermoplastic pipe (PVC, ABS, or PE) (fig. 52–3). Appendix 52B has design examples for various types of flexible pipes.



636.5201 Internal pressure design

Conduits used in pressure applications must withstand the internal working pressure. The internal pressure is resisted by tensile stress (hoop stress) in the conduit wall (fig. 52–4).







Composite ribbed

(a) Plastic pipe

The internal pressure capacity of plastic pipe is given as a pressure rating for plastic pipe manufactured in accordance with ASTM standards and as a pressure class for pipe meeting AWWA standards.

The pressure capacity is time dependent and should be considered in the design of a pressure pipe system. The long-term strength (hydrostatic design basis) of plastic pipe governs the pressure capacity design; yet, plastic pipe is capable of withstanding higher shortterm surge pressures.

The pressure rating or pressure class for solid-wall plastic pipe may be determined by one of the following formulas:

Outside diameter controlled pipe:

$$PC = PR = \frac{2 \times HDS}{SDR - 1}$$
(52-1)

Inside diameter controlled pipe:

$$PC = PR = \frac{2 \times HDS}{SIDR + 1}$$
(52-2)

AWWA C900 pressure class pipe:

$$PC = \frac{2 \times HDS}{SDR - 1} - P_{surge}$$
(52-3)

where:

 $PR = pressure rating, lb/in^2$ PC = pressure class, lb/in^2 P_{surge} = surge pressure based on an instantaneous velocity change of 2 ft/s, lb/in² HDS = hydrostatic design stress, lb/in² HDS = HDB/FSHDB = hydrostatic design basis, lb/in^2 \mathbf{FS} = factor of safety = 2.5 (AWWA C900 pipe) = 2.0 (all others) $SDR = D_0$ dimension ratio $SDR = D_o/t$ $D_0 = pipe$ outside diameter, in t = minimum wall thickness, in $SIDR = D_i$ dimension ratio $SIDR = D_i/t$ $D_i = pipe$ inside diameter, in

Pressure ratings or pressure class and pertinent dimensions for various plastic pipe materials are provided in appendix 52C. A complete description of HDB and HDS is available in ASTM D 2837.

The maximum design pressure for systems designed without a water hammer analysis should be limited to 72 percent of the pressure rating or pressure class of the pipe (ASAE, 1998, and ASTM 1176, 1993).

For plastic pipe systems subject to recurring or cyclic surge pressures, as described in 636.5202, the operating pressure plus the cyclic surge pressure should not exceed the pressure rating or pressure class of the pipe. If the number of cycles expected throughout the design life of the project is determined, design criteria using the short-term pressure rating and the number of cycles to failure found in Uni-Bell (2001) or recommended by the manufacturer may be used in selection of the pipe.

For occasional or infrequent pressure surges, as described in 636.5202, plastic pipe provides a higher short-term hoop strength. The pressure that corresponds to this elevated hoop stress is referred to as the quick-burst pressure or short-term strength (STS). A short-term pressure rating may be determined from the following equation:

$$STR = \frac{STS}{FS}$$
(52–4)

where:

STR = short-term pressure rating, lb/in²

$$\label{eq:STS} \begin{split} \text{STS} &= \text{short-term strength (quick burst pressure),} \\ & \text{lb/in}^2 \end{split}$$

$$= \frac{2 \times \text{STHS}}{\text{SDR} - 1} \quad \text{(for outside diameter contro})$$
$$= \frac{2 \times \text{STHS}}{\text{SIDR} + 1} \quad \text{(for inside diameter contro})$$

where:

STHS = short-term hoop strength, lb/in² (see appendix 52C)

 $SDR = D_0$ dimension ratio

 $SIDR = D_i$ dimension ratio

FS = 2.5 (AWWA C900 pipe)

= 2.0 (all others)

The design operating pressure plus the infrequent surge pressure should not exceed the short-term pressure rating.

Corrugated plastic pipe and profile wall plastic pipe are often not pressure rated. Because of the limited allowable pressure for watertight joints of corrugated or profile wall plastic pipe, the maximum allowable pressure shall be 10.8 pounds per square inch (lb/in²) (25 feet).

The HDB is typically determined in a water environment of approximately 73 degrees Fahrenheit. As the operating temperature falls below 73 degrees Fahrenheit, the pressure capacity of plastic pipe increases. As the temperature of the environment or fluid increases, the pipe becomes more ductile. The pressure rating should be decreased by the factors shown in table 52–1 or by using the HDB determined by ASTM D 2837 at the desired elevated temperature in the pressure rating (or pressure class) calculations.

Table 52–1	Temperature factors			
Temperature °F	PVC factor	ABS factor	PE factor	
73.4	1.00	1.00	1.00	
80	0.88	0.94	0.92	
90	0.75	0.84	0.81	
100	0.62	0.68	0.70	
110	0.50	0.56	0.65	
120	0.40	0.49	0.60	
130	0.30	0.44	0.55	
140	0.22	0.40	0.50	

Source: Uni-Bell, 2001; ASTM 1176, 1993; and Plastic Pipe Institute, 2003

(b) Smooth wall steel and aluminum pipe

The pressure rating for steel and aluminum pipe shall be determined by the following formula:

$$PR = \frac{2 \times S \times t}{D_o}$$
(52–5)

where:

 $PR = pressure rating, lb/in^2$

- S = allowable stress, lb/in² (50% of the yield strength of steel, 7,500 lb/in² for aluminum)
- t = wall thickness, in

 $D_0 =$ outside pipe diameter, in

Specification and grade of steel	Allowable stress 50% yield point lb/in ²
ASTM A 283	
Grade A	12,000
Grade B	13,500
Grade C	15,000
Grade D	16,500

ASTM A 1011 Structural steel

15,000
16,500
18,000
20,000
22,500
25,000
27,500
15,000
17,500
15,000
15,000 17,500
15,000 17,500
15,000 17,500 15,000
15,000 17,500 15,000 17,500
15,000 17,500 15,000 17,500 21,000
15,000 17,500 15,000 17,500 21,000 23,000

The stress in a metal pipe may be allowed to increase from 50 percent of the yield strength to 75 percent for surge pressures. Therefore, the internal pipe pressure for working pressure plus surge pressure may be 1.5 times the pressure rating determined above.

(c) Corrugated metal

The maximum allowable pressure should be limited to 20 feet of head for annular pipe and 30 feet of head for helical pipe with lock or continuously welded seams, annular ends, and watertight couplings.

Corrugated bands (fig. 52–5) and gaskets (fig. 52–6) are necessary when watertightness is required. The ends of helical pipe should be reformed so the pipe may be coupled. Flat bands with sleeve or O-ring type gaskets, or hat/channel with mastic bands (fig. 52–5) are not considered watertight joints since they are susceptible to pulling apart. Bands with annular corrugations and rod and lug connectors, a band angle connector (fig. 52–7), or flanged connections are acceptable watertight couplings.







Corrugated Sleeve gasket Universal





Band angle connector



Rod and lug

(210-VI-NEH, First Edition, June 2005)

(d) Ductile iron pipe

The net thickness for internal pressure (static pressure plus surge pressure) may be determined from the following formula:

$$t = \left(\frac{P \times D_{o}}{2 \times S_{y}}\right)$$
(52-6)

where:

t = net pipe wall thickness, in

 $P = internal pressure, lb/in^2$

$$P = 2.0 (P_{work}+P_{surge}) \text{ or static pressure}$$
$$P_{work} = \text{working pressure, lb/in}^2$$

 P_{surge} = maximum surge pressure, lb/in²

 $D_o = outside pipe diameter, in$

 S_v = yield strength (42,000 lb/in² for ductile iron)

The standard surge allowance for ductile iron pipe is 100 lb/in^2 . The pressure class designation signifies the allowable working pressure with a maximum surge pressure of 100 lb/in^2 . If the anticipated surge pressure is different from 100 lb/in^2 , the anticipated surge pressure should be used and the working pressure adjusted accordingly.

Once the net pipe wall thickness is determined, an 0.08-inch service tolerance and the casting tolerance from appendix 52F, table 52F–1, are added to calculate the thickness, from which the appropriate pressure class is chosen.

636.5202 Water hammer/ surge pressure

Water hammer (or surge pressure) occurs when the flow velocity in a pipe system is suddenly stopped or changed. When flow is suddenly changed, the mass inertia of the water is converted into a pressure wave or high static head on the pressure side of the pipeline. Some of the most common causes of water hammer are the opening and closing of valves, starting and stopping pumps, entrapped air, and poor pipe system layout.

For detailed surge analysis and to analyze flow changes other than instantaneous stoppage, a computer analysis is recommended. SURGE is one available computer program.

Surges may generally be divided into two categories: transient surges and cyclic surges. Transients are described as the intermediate conditions that exist in a system as it moves from one steady-state condition to another. Cyclic surging is a condition that recurs regularly with time. Surging of this type is often associated with the action of equipment, such as reciprocating pumps, pressure reducing valves, and float valves. Any piping material may eventually fatigue if exposed to continuous cyclic surging at sufficiently high frequency and stress.

Recurring surge pressures occur frequently and are inherent to the design and operation of the system (such as normal pump startup or shutdown and normal valve opening and closure).

Occasional surge pressures are caused by emergency operations and are usually the result of a malfunction, such as power failure or system component failure, which includes pump seize-up, valve-stem failure, and pressure-relief valve failure.

The pressure wave caused by the water hammer travels back and forth in the pipe getting progressively lower with each transition from end to end. The magnitude of the pressure change caused by the water hammer wave depends on the elastic properties of the pipe and liquid, as well as the magnitude and speed of the velocity change. The maximum surge pressure from water hammer is equal to:

$$H_{surge} = \frac{a \times \Delta V}{g}$$
(52–7)

$$P_{surge} = \frac{a \times \Delta V}{g} \times \frac{\gamma_w}{144}$$
(52–8)

or

or

$$P_{surge} = \frac{a \times \Delta V}{2.31 \times g} \quad (for water) \quad (52-9)$$

where:

- H_{surge} = surge pressure, ft of water P_{surge} = surge pressure, lb/in² a = velocity of the pressure wave, ft/s
- AV shange in velocity of the pressure wave, it's
- ΔV = change in velocity of fluid, ft/s g = acceleration due to gravity, 32.2 ft/s²
- g = acceleration due to gravity, 32.2 ft/ γ_{w} = unit weight of water, 62.4 lb/ft^3

 $\gamma_{\rm w}$ = unit weight of water, 62.4 lb/ft³

The maximum surge pressure results when the time required to stop or change the flow velocity is equal to or less than 2L/a such that:

$$\Gamma_{\rm CR} \le \frac{2L}{a} \tag{52-10}$$

where:

 T_{CR} = critical time, seconds

- L = distance within the pipeline that the pressure wave moves before it is reflected back by a boundary condition, ft
- a = velocity of the pressure wave, ft/s

The velocity of the pressure wave, a, may be expressed as:

$$a = \frac{12 \times \sqrt{\frac{K_{L}}{\rho}}}{\sqrt{1 + \frac{K_{L}}{E} \times \frac{D_{i}}{t}}}$$
or
(52-11)

$$a = \frac{12}{\sqrt{\frac{\gamma_{w}}{g} \left(\frac{1}{K_{L}} + \frac{D_{i}}{Et}\right)}}$$
(52-12)

or

$$a = \frac{4,720}{\sqrt{1 + \frac{K_L}{E} \times \frac{D_i}{t}}}$$
(for water) (52–13)

For SDR pipe, the velocity of the pressure wave may be expressed as:

$$a = \frac{12 \times \sqrt{\frac{K_{L}}{\rho}}}{\sqrt{1 + \frac{K_{L}(SDR - 2)}{E}}}$$
 (52–14)

$$a = \frac{12}{\sqrt{\frac{\gamma_{w}}{g} \left(\frac{1}{K_{L}} + \frac{\text{SDR} - 2}{E}\right)}}}$$
or
$$(52-15)$$

$$a = \frac{4,720}{\sqrt{1 + \frac{K_L(SDR - 2)}{E}}}$$
 (for water) (52–16)

where:

 K_{I} = bulk modulus of liquid, lb/in²

 $= 300,000 \text{ lb/in}^2 \text{ for water}$

E = modulus of elasticity of pipe material, lb/in² (as shown below)

SDR = standard dimension ratio

- ρ = density of fluid, slugs/ft³
 - = 1.93 slugs/ft³ for water
- γ_w = unit weight of water, 62.4 lb/ft³
- $g = acceleration due to gravity, 32.2 ft^2/s$
- D_i = internal diameter of the pipe, in

t = pipe wall thickness, in

Material	Modulus of elasticity* (lb/in ²)
Steel	29,000,000
Aluminum	10,000,000
Ductile Iron	24,000,000
PVC	400,000 (short term)
ABS	300,000 (short term)
Polyethylene	110,000 (short term)

*Short-term modulus of elasticity varies with the cell class of each plastic. Specific values may be obtained from the manufacturer.

636.5203 Loads on pipe

(a) Soil pressure

The soil pressure above flexible pipe is determined by the soil prism load theory (fig. 52–8). The soil pressure may be determined by the following equation:

$$P_{s} = \gamma_{s} \times h \tag{52-17}$$

where:

 $\rm P_{s}$ = pressure due to weight of soil at depth of h, $\rm lb/ft^{2}$

 $\gamma_{\rm s}$ = unit weight of soil, lb/ft³

h = height of ground surface above top of pipe, ft

When groundwater is above the top of the pipe, P_s may be reduced for buoyancy by the factor, R_w :

$$R_w =$$
 water buoyancy factor
= 1-0.33 h_w/h

where:

 $h = height of ground surface above top of pipe, ft h_w = height of water above top of pipe, ft$

The soil load per foot length of pipe may be determined by:

$$W_s = P_s \times \frac{D_o}{12}$$
(52–18)

Figure 52–8 Soil prism



where:

 W_s = soil load per linear foot of pipe, lb/ft D_o = outside diameter of pipe, in

(b) Wheel loading

Underground pipes may be subjected to vehicular loads. The use of actual wheel/track loads is recommended. The magnitude of the wheel load may be estimated from the following:

Load class	P_L , lb
Field equipment	10,000
H15	12,000
H20	16,000

The effect of wheel loads at the surface reduces significantly with depth. When the wheel load is large, such as 20,000 pounds, the possibility of a similar load within a distance equal to the depth of consideration should be evaluated using special analysis.

The pressure distribution is based on the stress distribution theory (fig. 52–9) and may be expressed as follows:

When
$$D_0 - t < 2.67h \times 12$$
:

$$W_{L} = \frac{0.48P_{L}I_{f}\left(\frac{D_{o}-t}{12}\right)^{2}}{2.67h^{3}} \left[\frac{2.67h}{\left(\frac{D_{o}-t}{12}\right)} - 0.5\right]$$
(52–19)

When
$$D_0 - t \ge 2.67h \times 12$$
:

$$W_{L} = \frac{0.64 P_{L} I_{f}}{h}$$
(52–20)

where:

- W_{L} = wheel load per linear foot of pipe, lb/ft
- P_{L} = wheel load at the surface, lb
- I_{f} = impact factor (as described below)
- h = height of ground surface above top of pipe, ft
- $D_0 =$ outside diameter of pipe, in
- t = pipe wall thickness, in

Depth of cover	Impact factor
< 1'0"	1.3
1'1'' - 2'0''	1.2
2'0'' - 2'11''	1.1
$\geq 3'0"$	1.0

The pressure on the pipe from the wheel load may be determined by:

$$P_{W} = \frac{W_{L}}{\left(\frac{D_{o}}{12}\right)}$$
(52–21)

where:

 P_w = pressure on pipe from wheel load, lb/ft²

 D_o = outside diameter of pipe, in

When the depth of fill is 2 feet or more, wheel loads may be considered as uniformly distributed over a square with sides equal to 1 3/4 times the depth of fill.

$$P_{w} = \frac{P_{L}}{(1.75h)^{2}}$$
(52–22)

(c) Vacuum pressure

Pipe may be subject to an effective external pressure because of an internal vacuum pressure, P_v Sudden valve closures, shutoff of a pump, or drainage from high points within the system often create a vacuum in pipelines. Siphons will all be subject to negative pressures.

Vacuum pressure should be incorporated into the design of buried and aboveground pipes as described in this chapter. The vacuum pressure may be intermittent (short term), for long durations, or continuously (long term).

The vacuum load per length of pipe may be determined by:

$$W_v = P_v \times \frac{D_i}{12}$$
(52–23)

where:

 W_v = vacuum load per linear foot of pipe, lb/ft

- P_v = internal vacuum pressure, lb/ft²
- D_i = inside pipe diameter, in





(a) D_0 -t < 2.67hx12



(b) $D_0-t \ge 2.67hx12$

(d) Hydrostatic pressure

Pipe may be subject to external hydrostatic pressure if it is below the water elevation. The hydrostatic pressure may be determined by the following equation:

$$P_{\rm G} = \gamma_{\rm w} \times h_{\rm w} \tag{52-24}$$

where:

$$P_{G}$$
 = external hydrostatic pressure, lb/ft²

 γ_w^- = unit weight of water, lb/ft^3

 h_w = height of water above top of pipe, ft

636.5204 Buried pipe design

The typical modes of failure of buried flexible pipe include wall crushing (stress), local buckling, or excessive deflection (fig. 52–10).

Excessive wall stress may lead to wall crushing if the compressive strength of the pipe wall is exceeded.

Buckling may occur because of insufficient pipe stiffness and may control design for pipes subject to internal vacuum, external hydrostatic pressure, or pipe embedded in loose or poorly compacted soil.

Deflection of flexible pipe is a performance limit to prevent cracking of liners, avoid reversal of curvature, limit bending stress and strain, and avoid pipe flattening. Deflection of a nonpressure flexible pipe increases with time after construction is complete. The time is a function of the embedment and surrounding soil density. The deflection continues to increase as long as the soil around the pipe continues to consolidate (increase in density). A deflection lag factor, D_L , was included in the modified Iowa equation to account for the increase in deflection with time. A D_L value of 1.0 to 1.5 is often recommended. A D_L value of 1.0 is often used when the soil load is estimated by the soil prism load as illustrated in figure 52–8. A D_L value of 1.5 has historically been used by the NRCS and is recommended as the factor to be applied to only the soil load.



(a) Plastic pipe

Plastic pipe materials consist of poly-vinyl chloride (PVC), acrylonitrile-butadiene-styrene (ABS), and polyethylene (PE). Each type of material is supplied in several grades as shown in appendix 52C.

Design of buried plastic pipe includes analyses of the wall crushing, buckling resistance, allowable longterm deflection, and allowable strain.

At a constant load, the plastic modulus of elasticity of the plastic pipe decreases with time. With any increase in load, the plastic reacts with the short-term modulus of elasticity. The ratio of the short-term to long-term modulus of elasticity varies from approximately 3 for PVC to 5 for PE. The short-term modulus of elasticity is recommended for conditions that change through time, such as deflection. The pipe-soil interaction that occurs as discrete events is similar to a new load (Chevron Chemical, 1998). The long-term modulus of elasticity is often recommended for buckling since the loads and reaction of the pipe are considered static.

(1) Wall crushing

The design pressure and ring compression thrust in the pipe wall is determined by:

$$P = P_s + P_w + P_v$$
 (52–25)

where:

 $P = pressure on pipe, lb/ft^2$

 P_s = pressure due to weight of soil, lb/ft²

 P_w = pressure on pipe due to wheel load, lb/ft²

 P_{v} = internal vacuum pressure, lb/ft²

$$\Gamma_{\rm pw} = \frac{P \times \frac{D_o}{12}}{2} \tag{52-26}$$

where:

 T_{pw} = thrust in pipe wall, lb/ft

 D_o^r = outside pipe diameter, in

The required wall cross-sectional area is determined by:

T

$$A_{pw} = \frac{\frac{1_{pw}}{12}}{\sigma}$$
(52–27)

where:

 A_{nw} = required wall area, in²/in

 T'_{pw} = thrust in pipe wall, lb/ft

= allowable long-term compressive stress, lb/in² (see appendix 52C, table 52C–1)

The area of a solid-wall pipe wall may be computed as:

$$A_{pw} = \frac{(D_o - D_i)}{2}$$
 or t (52–28)

where:

D_i = inside pipe diameter, in

t = pipe wall thickness, in

The average area of pipe wall for corrugated and profile wall pipe should be obtained from the manufacturer.

(2) Deflection

The Modified Iowa Equation may be transposed and rewritten to compute the percent deflection of each type of pipe. The properties of a pipe section are expressed as the standard dimension ratio (SDR) or standard inside dimension ratio (SIDR) for solid wall pipe, pipe stiffness (PS) for corrugated plastic pipe, and the ring stiffness constant (RSC) for profile wall pipe.

Solid-wall plastic pipe as:

$$\frac{\%\Delta X}{D} = \frac{\left(D_{L}P_{S} + P_{w} + P_{V}\right)\left(\frac{1}{144}\right)K(100)}{\left[\left(\frac{2E}{3(SDR - 1)^{3}}\right) + 0.061E'\right]}$$
(52–29)

or

$$\frac{\%\Delta X}{D} = \frac{\left(D_{L}P_{S} + P_{w} + P_{v}\right)\left(\frac{1}{144}\right)K(100)}{\left[\left(\frac{2E}{3(\text{SIDR}+1)^{3}}\right) + 0.061\text{E'}\right]}$$
(52–30)

Corrugated-plastic pipe as:

$$\frac{\%\Delta X}{D} = \frac{\left(D_{L}P_{S} + P_{w} + P_{v}\right)\left(\frac{1}{144}\right)K(100)}{\left[0.149PS + 0.061E'\right]}$$
(52–31)

Profile-wall pipe:

$$\frac{\%\Delta X}{D} = \frac{\left(D_{L}P_{S} + P_{w} + P_{v}\right)\left(\frac{1}{144}\right)K(100)}{\left[\left(\frac{1.24(RSC)}{D_{i}}\right) + 0.061E'\right]}$$
(52–32)

where:

%ΛΧ D = percent deflection = deflection lag factor (1.0 to 1.5) D_L Κ = bedding constant (0.1) P. = pressure on pipe from soil (lb/ft²) P_w = pressure on pipe from wheel load (lb/ft^2) P_v = internal vacuum pressure (lb/ft^2) E = modulus of elasticity of pipe material (as shown below) SDR = D_o dimension ratio $SDR = D_0/t$ D_{o} = pipe outside diameter, in = minimum wall thickness, in t. $SIDR = D_i$ dimension ratio $SIDR = D_i/t$ D_i = pipe inside diameter, in = minimum wall thickness, in t. \mathbf{PS} = pipe stiffness RSC = ring stiffness constant = inside pipe diameter, in D. \mathbf{E}' = modulus of soil reaction, lb/in² (see table 52-2)Material Modulus of elasticity* (lb/in^2) PVC 400,000 (short term)

ABS 300,000 (short term)

Polyethylene 110,000 (short term)

* Short-term modulus of elasticity varies with the cell class of each plastic. Specific values may be obtained from the manufacturer. The modulus of soil reaction, E', is an interactive modulus representing support of the soil in reaction to the lateral pipe deflection under load. Amster Howard of the Bureau of Reclamation (Howard, 1977) developed recommended E' values based on the soil prism load described above. The recommended values are provided in table 52–2.

The allowable deflections for plastic pipe typically are limited to 5 percent for a spillway/outlet conduit in embankment dam practice and 7.5 percent in water or liquid conveyance practice and drains in embankment dam practice.

(3) Wall buckling

Plastic pipe embedded in soil may buckle because of excessive loads and deformations. The total permanent pressure must be less than the allowable buckling pressure. The permanent load should consist of the soil pressure, groundwater pressure, and any internal long-term vacuum pressures. The allowable buckling pressure may be determined from:

$$q_{a} = \frac{1}{FS} \left(32R_{w}B'E'\frac{E_{long}I_{pw}}{D_{o}^{3}} \right)^{1/2}$$
(52–33)
(Moser, 2001)

where:

 R_w

- $q_a = allowable buckling pressure, lb/in^2$
- FS = design factor of safety
 - $= 2.5 \text{ for } (h/(D_o/12) > 2)$
 - $= 3.0 \text{ for } (h/(D_0/12) < 2$
 - where:
 - h = height of ground surface above top of pipe, ft

 $D_0 =$ outside diameter of the pipe, in

wher

h = height of ground surface above top of pipe, ft

 h_w = height of water above top of pipe, ft B'= empirical coefficient of elastic support

$$\frac{4\left(h^2 + \left(\frac{D_o}{12}\right)h\right)}{1.5\left(2h + \left(\frac{D_o}{12}\right)\right)^2}$$

 E_{long} = long term modulus of elasticity, lb/in² (see table below)

The long term modulus of elasticity is recommended if the pipe is subject to the pressure in the normal operations. If the pipe is subject to the pressure for short time periods and infrequently, the use of the short-term modulus of elasticity is acceptable.

E' = modulus of soil reaction, lb/in² (table 52–2)

 I_{pw} = pipe wall moment of inertia

 $= \frac{\frac{t^3}{12}, \text{ in}^4 / \text{ in}}{(\text{for solid wall pipe})}$ where:

t = pipe wall thickness, in $D_0 = outside pipe diameter, in$

MaterialModulus of elasticity* (lb/in²)PVC140,000 (long term)ABS65,000 (long term)Polyethylene22,000 (long term)

* Long-term modulus of elasticity varies with the cell class of each plastic. Specific values may be obtained from the manufacturer. Pipes that are out-of-round or deflected increase in bending moment and have less allowable buckling pressure. The allowable buckling pressure should be reduced by the following factor:

$$C = \left[\frac{\left(1 - \frac{\%\Delta X}{D} \frac{1}{100}\right)}{\left(1 + \frac{\%\Delta X}{D} \frac{1}{100}\right)^{2}}\right]^{3}$$
(52–34)

where:

C = reduction factor for buckling pressure

%ΔX

D = percent deflection

Table 52–2 Average values of the modulus of soil reaction for the Modified Iowa Equation

Soil type _ pipe hedding material	\mathbf{F}' for degree of compaction of hedding $\frac{1}{2}$				
(Unified Soil Classification – ASTM D2487)	Dumped	Slight, < 85% proctor, < 40% relative density	Moderate, 85-95% proctor, 40-70% relative density	High, > 95% proctor, > 70% relative density	
Fine-grained soil (LL>50) ^{2/} Soil with medium to high plasticity CH, MH, CH-MH	No data a geotechni	No data available, use E' = 0 or consult with a geotechnical engineer			
Fine-grained soil (LL<50) soil with medium to no plasticity CL, ML, ML-CL, with less than 25% coarse-grained particles	50	200	400	1,000	
Fine-grained soil (LL<50) soil with medium to no plasticity CL, ML, ML-CL, with more than 25% coarse- grained particles. Coarse-grained soil with fines GM, GC, SM, SC contains more than 12% fines	100	400	1,000	2,000	
Coarse-grained soil with little or no fines GW, GP, SW, SP contains less than 12% fines	200	1,000	2,000	3,000	
Crushed rock	1,000	3,000	3,000	3,000	

1/ Source ASCE Journal of Geotechnical Engineering Division, January 1977

^{2/} LL = liquid limit

(4) Strain

Total strain in a pipe wall can be caused by two actions: (1) flexure of the pipe as it deforms, and (2) hoop stress caused by internal or external pressure in the pipe wall. If a homogeneous wall is assumed and pressure concentrations are neglected, the formula follows:

Hoop strain:

$$\varepsilon_{\rm h} = \frac{\frac{\rm P}{144} \rm D}{2 \rm A_{\rm pw} \rm E}$$
(52–35)

For solid wall pipe, the equation becomes:

$$\varepsilon_{\rm h} = \frac{\frac{\rm P}{144} \rm D_{\rm M}}{2 \rm tE} \tag{52-36}$$

where:

- ϵ_h = maximum strain in pipe wall because of ring bending, in/in
- $\begin{array}{ll} P & = \mbox{pressure on/in pipe (may be internal and/or} \\ & \mbox{external pressure with the appropriate sign),} \\ & \mbox{lb/ft}^2 \end{array}$

 D_{M} = mean pipe diameter, in

- A_{pw} = area of pipe wall, in²/in
- $E^{'}$ = modulus of elasticity of the pipe material, lb/in²
- t = pipe wall thickness, in

Maximum strains because of deflection or flexure may be determined by assuming the pipe remains an ellipse during deflections. The resulting equations are:

$$\begin{aligned} \epsilon_{\rm f} &= \frac{t}{D_{\rm M}} \left(\frac{\frac{3\Delta Y}{D_{\rm M}}}{1 - 2\frac{\Delta Y}{D_{\rm M}}} \right) = \frac{1}{\rm SDR} \left(\frac{\frac{3\Delta Y}{D_{\rm M}}}{1 - 2\frac{\Delta Y}{D_{\rm M}}} \right) & \text{(solid wall pipe)} \\ & \text{(52-37)} \end{aligned}$$

$$\epsilon_{\rm f} &= 6\frac{t}{D_{\rm M}} \frac{\Delta Y}{D_{\rm M}} & \text{(corrugated or profile wall pipe)} \\ & \text{(52-38)} \end{aligned}$$

where:

- ϵ_{f} = maximum strain in pipe wall because of ring deflection, in/in
- ΔY = vertical decrease in diameter, in

In a buried pipeline, these strain components act simultaneously. The maximum combined strain in the pipe wall can be determined by summing both components.

$$\varepsilon = \varepsilon_{\rm f} \pm \varepsilon_{\rm h} \tag{52-39}$$

where:

 ϵ = maximum combined strain in pipe wall, in/in

In calculating the maximum combined strain, the hoop strain, ε_h , resulting from applied internal pressure, if any, should be added to the maximum strain due to deflection, ε_f . If the hoop strain is due to external load or internal vacuum pressure, the ring hoop strain should be substracted to obtain the maximum combined strain, ε .

The maximum combined strain in the pipe should be limited to:

$$\varepsilon \le \varepsilon_{all}$$
 (52–40)

where:

 ϵ_{all} = allowable strain for the pipe material

The allowable strain should be no more than 5 percent for polyethylene and ABS pipes.

The allowable deflection for PVC pipe limits strain in standard PVC pipes to an acceptable value. Therefore, computation of strain and comparison to an allowable strain limit is not required for PVC pipe.

In polyethylene pressure pipe with pressure near the pipe pressure rating, the strain may be limited by limiting the deflection to the values shown in table 52–3.

(b) Steel

Design of steel pipe includes an analysis of the deflection and the buckling pressure.

(1) Deflection

The Modified Iowa Equation may be used to compute the deflection as:

$$\Delta \mathbf{X} = \left(\frac{\left(\mathbf{D}_{\mathrm{L}} \mathbf{W}_{\mathrm{S}} + \mathbf{W}_{\mathrm{L}} + \mathbf{W}_{\mathrm{v}} \right) \left(\frac{1}{12} \right) \mathbf{K} \mathbf{r}^{3}}{\mathbf{E} \mathbf{I}_{\mathrm{pw}} + 0.061 \mathbf{E'} \mathbf{r}^{3}} \right)$$
(52–41)

where:

- $\Delta X = deflection, in$
- D_{L} = deflection lag factor (1.0 to 1.5)
- W_s = soil load per linear foot of pipe, lb/ft
- W_L = wheel load per linear foot of pipe, lb/ft
- W_v = vacuum load per linear foot of pipe, lb/ft
- K = bedding constant (0.1)
- r = radius of pipe, in

 EI_{pw} = pipe wall stiffness, in-lb*

- where: E = modulus of
- E = modulus of elasticity (29,000,000 lb/in² for steel and 4,000,000 lb/in² for cement mortar)

$$I_{pw}$$
 = pipe wall moment of inertia = $\frac{t^3}{12}$, in⁴/in

t = wall thickness, in

E' = modulus of soil reaction, lb/in² (table 52–2)

* Under load, the individual elements; i.e., mortar lining, steel shell, and mortar coating. work together as laminated rings $(E_SI_S + E_II_I + E_CI_C)$ – shell, lining, coating). Structurally, the combined elements increase the moment of inertia of the pipe section, above the shell alone, thus increasing its ability to resist loads. The pipe wall stiffness EI of these individual elements is additive. (AWWA 1995)

Table 52–3	Safe deflection of po	lyethylene pressure pipe
------------	-----------------------	--------------------------

SDR	Safe deflection as % of diameter	
32.5	8.5	
26.0	7.0	
21.0	6.0	
17.0	5.0	
13.5	4.0	
11.0	3.0	
9.0	2.5	

Source: ASTM F 714

The percent deflection may be determined by:

$$\% \frac{\Delta X}{D} = \frac{\Delta X}{D_o} \times 100$$
(52–42)

Allowable deflections for various lining and coating systems are:

Steel pipe	=	5 percent
Flexible lined and coated steel pipe	=	$5 \mathrm{percent}$
Mortar-lined and flexible coated steel pipe	=	$3 \mathrm{percent}$
Mortar-lined and coated steel pipe	=	2 percent

(2) Buckling

Steel pipe embedded in soil may buckle because of excessive loads and deformations. The total permanent pressure must be less than the allowable buckling pressure. The permanent pressure should consist of the soil pressure, hydrostatic pressure, and any longterm vacuum pressure. The allowable buckling pressure may be determined from:

$$q_{a} = \frac{1}{FS} \left(32R_{w}B'E'\frac{EI_{pw}}{D_{o}^{3}} \right)^{1/2}$$
(52-43)

where:

 q_a = allowable buckling pressure, lb/in^2

FS = design factor of safety

- $= 2.5 \text{ for } (h/(D_0/12) > 2$
- = 3.0 for (h/(D_o/12) < 2 where:
 - h = height of ground surface above top of the pipe, ft
- $D_0 =$ outside diameter of the pipe, in

 R_w = water buoyancy factor

- = 1–0.33(h_w/h), 0<h_w<h where:
 - h = height of ground surface above top of the pipe, ft
- h_w = height of water above top of pipe, ft
- B' = empirical coefficient of elastic support

$$= \frac{1}{1+4e^{(-0.065h)}}$$
 (AWWA, 1989)

where:

- $\label{eq:h} \begin{array}{l} h \ = \ height \ of \ ground \ service \ above \ top \ of \ pipe, \\ ft \end{array}$
- E' = modulus of soil reaction, lb/in² (table 52–2)
- E = modulus of elasticity, lb/in² (29,000,000 for steel)

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$$I_{pw}$$
 = transverse moment of inertia = $\frac{t^3}{12}$, in⁴/in
t = pipe wall thickness, in

 $D_o = outside pipe diameter, in$

(c) Corrugated and spiral rib metal pipe

Design of corrugated and spiral rib metal pipe includes analysis of the wall strength, buckling strength, seam strength, and handling stiffness. Section properties of corrugated and spiral rib metal pipe are included in appendix 52D.

The strength requirements may be determined by either the allowable stress design (ASD) method or the load and resistance factor design (LRFD) method. Both methods are presented in ASTM B 790 for corrugated aluminum pipe and ASTM A 796 for corrugated steel pipe. The ASD method is presented next.

(1) Thrust

The design pressure and ring compression thrust in the pipe wall are determined by:

$$P = P_{s} + P_{w} + P_{v}$$
(52–44)

where:

 $P = design pressure, lb/ft^2$

 $\rm P_s~$ = pressure due to weight of soil, lb/ft^2 ~

 $P_{\rm w}$ = pressure on pipe due to wheel load, lb/ft^2

 P_v = internal vacuum pressure, lb/ft²

$$T_{pw} = \frac{P \times \left(\frac{D_i}{12}\right)}{2}$$
(52-45)

where:

 $\begin{array}{l} T_{pw} = thrust \mbox{ in pipe wall, lb/ft} \\ D_i = \mbox{ inside pipe diameter, in} \end{array}$

The required wall cross-sectional area is determined by:

$$A_{s} = \frac{T_{pw} \times (FS)}{f_{y}}$$

where:

 A_s = required area of section, in²/ft T_{nw} = thrust in pipe wall, lb/ft $\begin{array}{l} \mathrm{FS} &= \mathrm{safety\ factor,\ 2.0\ for\ wall\ area} \\ f_{\mathrm{y}} &= \mathrm{minimum\ yield\ strength,\ lb/in^2} \\ &\quad 33,000\ lb/in^2\ for\ steel \end{array}$

24,000 lb/in² for aluminum 20,000 lb/in² for aluminum alloy 3004-H32

(2) Buckling

The selected corrugated pipe section with the required wall area shall be checked for possible buckling. If the critical buckling stress, f_c , is less than the minimum yield stress, f_y , the required wall area must be recalculated using f_c instead of f_y .

 $D_i < \frac{r}{k} \sqrt{\frac{24E}{f_u}}$

When:

$$f_{\rm c} = f_{\rm u} - \frac{f_{\rm u}^2}{48\rm E} \left(\frac{\rm kD_{\rm i}}{\rm r}\right)^2$$
 (52–48)

(52-47)

$$D_{i} \ge \frac{r}{k} \sqrt{\frac{24E}{f_{u}}}$$
(52-49)

$$f_{\rm c} = \frac{12E}{\left(\frac{\rm kD_i}{\rm r}\right)^2} \tag{52-50}$$

where:

When:

- $D_i = inside pipe diameter, in$
- r = radius of gyration of corrugation, in
- $\label{eq:k} \begin{array}{l} \mbox{= soil stiffness factor = 0.22 for good fill material} \\ \mbox{compacted to 90\% of standard density based on} \\ \mbox{ASTM D 698 or } \phi > 15^{\circ} \end{array}$
 - = 0.44 for soils with $\phi < 15^{\circ}$ (Contech, 2001)
- E = modulus of elasticity of pipe material, lb/in²
- $f_{\rm u}$ = minimum tensile strength of material, lb/in²
 - 45,000 lb/in² for steel 34,000 lb/in² for aluminum
 - $27,000 \text{ lb/in}^2$ for aluminum alloy 3004-H32
- f_c = critical buckling stress, lb/in²

(52-46)

(3) Seam strength

For pipe fabricated with longitudinal seams (riveted, spot-welded, or bolted), the seam strength shall be sufficient to develop the thrust in the pipe wall. The required seam strength shall be:

$$SS = T_{pw} \times FS \tag{52-51}$$

where:

SS = required seam strength, lb/ft T_{pw} = thrust in pipe wall, lb/ft

FS = safety factor, 3.0 for seam strength

Since helical lockseam and welded-seam pipe do not have longitudinal seams, seam strength criteria are not valid for these types of corrugated pipe.

(4) Flexibility factor

The metal pipe must have sufficient stiffness to withstand temporary loads that occur during shipping, handling, and installation. Relationships referred to as the flexibility factor have been developed that relate the required pipe wall stiffness to the pipe diameter. The flexibility factor is determined as:

$$FF = \frac{D_i^2}{EI_{pw}}$$
(52–52)

where:

FF = flexibility factor, in/lb

 D_i = inside diameter of the pipe, in

E = modulus of elasticity of pipe material, lb/in²

 I_{pw} = moment of inertia of pipe wall, in⁴/in

The flexibility factor shall not exceed the allowable flexibility factors in appendix 52E.

(d) Ductile iron

The required wall thickness for ductile iron pipe under external load is based on two design considerations: ring bending stress and ring deflection. Thicknesses for standard pressure classes are provided in appendix 52F.

(1) Ring bending stress

The design ring bending stress, f, of 48,000 pounds per square inch provides a factor of safety of at least 1.5 on the minimum ring yield strength and 2.0 on the

ultimate ring strength. The pressure due to soil, wheel, and vacuum loads required to develop a bending stress of 48,000 pounds per square inch at the pipe invert may be determined by:

$$P_{bs} = \frac{f}{3\left(\frac{D_{o}}{t}\right)\left(\frac{D_{o}}{t}-1\right)\left[\frac{K_{b} - \frac{K_{x}}{8E}}{E'\left(\frac{D_{o}}{t}-1\right)^{3} + 0.732}\right]}$$
(52-53)

where:

- P_{bs} = pressure to develop maximum ring bending stress, lb/in²
- = design maximum bending stress (48,000 f lb/in²)
- D_o = outside diameter of pipe, in
- = net pipe wall thickness
 - $= t_n service$ allowance casting tolerance where:

 t_n = nominal thickness from appendix 52F service allowance = 0.08 in (AWWA, 2002) casting tolerance from appendix 52F

- $K_{\rm b}$ = bending moment coefficient (table 52–4)
- K_{x} = deflection coefficient (table 52–4)
- = modulus of elasticity $(24,000,000 \text{ lb/in}^2)$ Ε
- = modulus of soil reaction, lb/in^2 (table 52–4) \mathbf{E}'

The total pressure on the buried pipe is:

$$P = P_s + P_w + P_v$$
 (52–54)

where:

 $P = design pressure, lb/ft^2$

 P_s = pressure from weight of soil, lb/ft²

 P_w = pressure on pipe because of wheel load, lb/ft² P_v = internal vacuum pressure, lb/ft²

The total pressure on the buried pipe, P, must be less than the design pressure to develop the maximum ring bending stress, P_{bs}:

$$P \le P_{bs} \times 144$$
 (52–55)

where:

 $P = design pressure, lb/ft^2$

P_{bs} = pressure to develop ring bending stress, lb/in²

(2) Ring deflection

Maximum allowable ring deflection for unlined ductile iron pipe is 5 percent of the outside diameter. The maximum allowable ring deflection for cement-mortarlined ductile iron pipe is 3 percent of the outside diameter. Research has shown that 3 percent deflection provides a safety factor of at least 2.0 against failure of the cement-mortar lining. The following equation may be used to determine the allowable design pressure at the allowable deflection:

$$P_{bs} = \frac{f}{3\left(\frac{D_{o}}{t}\right)\left(\frac{D_{o}}{t}-1\right)\left[K_{b} - \frac{K_{x}}{\frac{8E}{E'\left(\frac{D_{o}}{t}-1\right)^{3}} + 0.732}\right]}$$
(52-56)

where:

 $P_{rd}~$ = pressure to develop allowable ring deflection, $$lb/in^2$$

 ΔX

- D = percent deflection
 - = 5% (0.05) for unlined pipe
 - = 3% (0.03) for mortar-line pipe
- K_x = deflection coefficient (see table 52–4)
- D_0^{*} = outside diameter of pipe, in
- t_1 = minimum manufacturing thickness, in

$$(t_n - casting tolerance)$$

- t_n = nominal pipe wall thickness from appendix 52F
- E = modulus of elasticity $(24,000,000 \text{ lb/in}^2)$
- E' = modulus of soil reaction, lb/in² (table 52-4)

The total pressure on the buried pipe, P, must be less than the design pressure to develop acceptable deflection, $\rm P_{rd}\!:$

$$P \le P_{rd} \times 144 \tag{52-57}$$

where:

 $P = design pressure, lb/ft^2$

 P_{rd} =pressure to develop ring deflection, lb/in²

A required net thickness, t, is determined using both the ring bending stress and allowable deflection equations above. The larger of the two net thicknesses, t, is selected. The nominal thickness is determined by adding the service allowance and casting tolerance. The nominal thickness is typically specified.

Although backfill around the pipe should be well compacted, design values of laying condition type 3 (table 52–4) are recommended for ductile iron pipes used in embankments for dams and ponds.

Table 52–4 Design values for standard laying conditions

Laying Condition	Description	E' psi ^B	Bedding Angle	K _b	K _x
Type 1	Flat bottom trench C loose backfill. D	150	30	0.235	0.108
Type 2	Flat bottom trench C Backfill lightly consolidated to centerline of pipe.	300	45	0.210	0.105
Type 3	Pipe bedded in 4 inch (102 mm) minimum loose soil ^{E} Backfill lightly consolidated to top of pipe.	400	60	0.189	0.103
Type 4	Pipe bedded in sand, gravel, or crushed stone to depth of 1/8 pipe diameter. 4 inch (102 mm minimum. Backfill compacted to top of pipe. (Approximately 80 percent standard proctor, AASHTO T-99.)) 500	90	0.157	0.096
	Pipe bedded in compacted granular material to centerline of pipe, 4 inch (102 mm) minimum under pipe. Compacted granular or select _E material to top of pipe. (Approximately 90 percent standard proctor, AASHTO T-99.)	700	150	0.128	0.085

 ${\rm Type}\ 5$

A Consideration of the pipe-zone embedment conditions included in this table may be influenced by factors other than pipe strength. For additional information see ANSI/AWWA C600. Standard for installation of Ductile-Iron Water Mains and their Appurtenances. 1 $lb/in^2 = 6.894757$ kPa.

В

CFlat-bottom is defined as undisturbed earth.

DFor pipe 14 inch (350 mm) and larger, consideration should be given to use of laying conditions other than Type 1. ELoose soil or select material is defined as native soil excavated from the trench free of rocks, foreign materials, and frozen earth.

636.5205 Expansion and contraction

All pipe products expand and contract with changes in temperature. Approximate coefficients of thermal expansion for pipe materials is presented in table 52-5. Buried pipe used in NRCS applications will not typically experience significant changes in temperature, and thermal stress or dimension change will be minimal. However, changes in the ambient temperature prior to backfilling around the pipe may lead to excessive expansion or contraction. Therefore, the backfill should be placed as construction progresses.

Unrestrained pipe will experience a length change with changing temperature. The length may be estimated by:

$$\Delta L = L_{ur} \alpha \Delta T \tag{52-58}$$

where:

 $\begin{array}{l} \Delta L = change \ in \ length, \ in \\ L_{ur} = length \ of \ unrestrained \ pipe, \ in \\ \alpha = coefficient \ of \ thermal \ expansion, \ in/in/^{\circ}F \\ \Delta T = change \ in \ temperature, \ ^{\circ}F \end{array}$

A pipe restrained or anchored at both ends will experience a change in stress with changing temperature because of expansion and contraction. The longitudinal stress in the pipe wall caused by temperature changes may be estimated by:

$$S_{EC} = E\alpha\Delta T$$
 (52–59)

where:

 S_{EC} = stress due to temperature change, lb/in² E = short term modulus of elasticity, lb/in²

 α = coefficient of thermal expansion, in/in/°F

 ΔT = change in temperature, °F

The modulus of elasticity of plastic pipe is a function of the temperature. Since the temperature change does not occur rapidly, the average temperature is recommended for use in determining the appropriate modulus of elasticity. The modulus of elasticity should be adjusted for temperature by the factors shown in table 52–1. Various pipe joints that allow some movement because of expansion and contraction are available. Gasketed pipe joints (such as bell and spigots) for plastic, steel, or ductile iron pipe and expansion joints for steel pipe allow some movement at the joint. The allowable movement at the joint should be obtained for the particular joint and compared to the length change caused by a change in temperature. Welded steel or plastic pipes or solvent cemented plastic pipes do not allow movement at the joint.

Coefficients of thermal expansion
Coefficient (in/in/°F)
3.0x10 ⁻⁵
$1.2 \mathrm{x} 10^{-4}$
$5.5 \mathrm{x} 10^{-5}$
$1.3 x 10^{-5}$
$5.8 \mathrm{x} 10^{-6}$
$6.5 \mathrm{x10^{-6}}$

Source: AWWA, 2002

636.5206 Aboveground pipe design

Aboveground applications frequently require noncontinuous support. These applications include pipe support from a saddle, rack, or stand supported by an adequate foundation or suspended from an overhead structure (figs. 52–11, 52–12, and 52–13). The equations shown apply to uniformly loaded and simply supported pipe. Lower bending moment and deflection will result for continuous rigidly joined and multiple span pipe.

(a) Bending stress

The maximum bending stress in the pipe wall of an unsupported pipe is:

$$S_{b} = \frac{MD_{o}}{2I}$$
(52–60)

where:

 S_{b} = bending stress, lb/in²

- M = bending moment, in-lb
- I = moment of inertia, in^4

$$= \frac{\pi}{64} \left(D_{o}^{4} - D_{i}^{4} \right), \text{ in}^{4} \text{ (plastic or ductile iron pipe)}$$
$$= \frac{\pi}{8} \left(D_{o}^{3} t \right) \text{ (steel pipe)}$$
$$D_{o} = \text{outside pipe diameter, in}$$
$$D_{i} = \text{ inside pipe diameter, in}$$

t = pipe wall thickness, in

The moment for an end-supported simple beam with a single span may be calculated by:

$$M = \frac{w L_{span}^{2}}{8}$$
(52–61)

where:

- M = bending moment, in-lb
- w = load of pipe filled with liquid, lb/in

 $L_{span} = span length, in$

The above two equations may be combined to determine the bending stress at center span of the pipe or an allowable support spacing of a uniformly loaded, simply supported pipe.

$$S_{b} = \frac{0.0625 w L_{span}^{2} D_{o}}{I}$$
 (52–62)
and

$$L_{span} = 4.0 \sqrt{\frac{S_{ball}I}{wD_o}}$$
(52–63)

where:

 $S_{\rm b}$ = bending stress, lb/in²

 S_{ball} = allowable bending stress, lb/in^2

- (50% of yield strength for steel, 48,000 lb/in² for ductile iron, and 7,500 lb/in² for aluminum)
 - = HDS = HDB/FS for plastic
 - HDS = hydrostatic design stress
 - HDB = hydrostatic design basis
 - FS = factor of safety (2.5 for AWWA C900 pipe, 2.0 for others)

Figure 52–11 Pipeline hanger



Figure 52–12 Pipeline support



52-20

= load of pipe filled with liquid, lb/in W L_{span} = span length, in = moment of inertia, in⁴ Ι

$$= \frac{\pi}{64} (D_{o}^{4} - D_{i}^{4}), \text{ in}^{4} \text{ (plastic and ductile iron pipe)}$$

$$=\frac{\pi}{8} \left(D_o^{3} t \right)$$
 (steel and aluminum pipe)
D_o = outside pipe diameter, in

- D_i = inside pipe diameter, in
- t = pipe wall thickness, in

(b) Deflection

The length of the span between pipe supports shall be such that the deflection between supports is limited to an acceptable value. A maximum deflection of 1/360 of the span is recommended for steel pipe, 1/120 for ductile iron pipe, 0.5 percent of span for PVC pipe, and 1-inch for other plastic pipe. The maximum theoretical deflection for a uniformly loaded, simply supported pipe may be determined by:

$$y = 0.0130 \left(\frac{WL_{span}^3}{E_{long}I} \right)$$
(52–64)
or

$$y = \frac{0.0130 \text{wL}_{\text{span}}^4}{\text{E}_{\text{long}}\text{I}}$$
 (52–65)

where:

- У = maximum deflection at center of span, in
- W = total load on span, lb = weight of pipe filled with liquid, lb/in w
- $L_{span} = span length, in$
- E_{long}^{j} = long-term modulus of elasticity, lb/in² (see below)
- = transverse moment of inertia Ι

$$=\frac{\pi}{64} \left(D_{o}^{4} - D_{i}^{4} \right), \text{ in}^{4}$$
 (plastic or ductile iron pipe)

$$=\frac{\pi}{8} \left(D_o^{3} t \right) \quad \text{(steel pipe)}$$

 $D_o = outside diameter, in D_i = inside diameter, in$

- t = pipe wall thickness, in, in⁴



Note: This equation for I does not apply to corrugated, ribbed, or profile wall pipe. The appropriate values should be obtained from ASTM specifications or the manufacturer.

Material	Modulus of elasticity (lb/in)*		
Steel	29,000,000		
Aluminum	10,000,000		
Ductile iron	24,000,000		
PVC	140,000 (long term)		
ABS	65,000 (long term)		
Polyethylene	22,000 (long term)		

* Long-term modulus of elasticity varies with the cell class of each plastic. Specific values may be obtained from the manufacturer.

(c) Hoop stress

The hoop stress caused by internal pressure may be estimated by:

$$S_{p} = \frac{P \times D_{i}}{2 \times t}$$

(52-66)

where

 S_p = stress from internal pressure, lb/in²

 P^{r} = pressure in the pipe, lb/in²

 D_i = inside diameter of the pipe, in

t = pipe wall thickness, in

(d) Localized stress at supports

An unstiffened pipe resting in saddle supports has high local stresses, longitudinal and circumferential, adjacent to the tips of the saddles. The localized stresses are less for a larger saddle angle (β) than for a small angle, and are practically independent of the thickness of the saddle (saddle dimension parallel to the pipe axis). Saddle angles of 90 degrees to 120 degrees are recommended. Ductile iron pipe research shows that little benefit is gained by increasing the saddle angle above 120 degrees, yet the maximum stress increases

rapidly with saddle angle less than 90 degrees. For a pipe that fits the saddle well, the maximum longitudinal or circumferential localized stress probably does not exceed

$$S_{l} = k_{support} \frac{R_{support}}{t^{2}} ln \left(\frac{R_{o}}{t}\right)$$
 (Roark, 1975) (52–67)

where:

$$\begin{split} S_{l} &= \text{local stress at the saddle, lb/in}^{2} \\ R_{\text{support}} &= \text{total saddle reaction, lb} \\ &= \frac{wL_{\text{span}}}{2} \quad (\text{single span}) \\ &= wL_{\text{span}} \quad (\text{multiple span}) \\ &\text{where:} \\ &w &= \text{weight of pipe filled with liquid, } \\ && \text{lb/in} \\ && L_{\text{span}} = \text{span length, in} \\ R_{o} &= \text{outside radius of pipe, in} \\ t &= \text{pipe wall thickness, in} \\ k_{\text{support}} &= \text{coefficient} \\ &= 0.02 - 0.00012 \ (\beta-90) \\ &= 0.03 - 0.00017 (\beta-90) \ (\text{ductile iron pipe}) \\ && (\text{DIPRA, 2001}) \\ && \beta = \text{saddle angle, degrees} \end{split}$$

Theories and data differ on the importance of the saddle support width. Some test data indicate little effect on the maximum local stress when the support width is a minimum of:

$$b = \sqrt{2D_o t}$$
(52–68)

where:

- b = saddle width, in
- D_{o} = outside diameter of pipe, in
- t = pipe wall thickness, in
 - = net pipe wall thickness, in (ductile iron)

Some polyethylene pipe manufacturers recommend the support width be at least equal to the outside pipe diameter.

(e) Total stress at the saddle support

The total stress at the saddle is a combination of the longitudinal stresses in the pipe wall. In the case of a pipe with internal pressure, the Poisson ratio effect of the hoop stress, which produces a lateral tension, must be added to determine the total beam stress in the pipe wall (Barnard, 1948). The total stress may be computed as

$$S_{\rm T} = vS_{\rm p} + S_{\rm b} + S_{\rm l} + S_{\rm EC}$$
 (52–69)

where:

- S_{T} = total stress at the saddle, lb/in²
- = Poisson's ratio (0.30 for steel and ductile iron, 0.33 for aluminum, 0.38 for PVC, 0.40 for PE, 0.50 for ABS)
- $\begin{array}{ll} S_p & = hoop \; stress \; from \; internal \; pressure, \; lb/in^2 \\ S_b & = bending \; stress, \; lb/in^2 \end{array}$

 S_1 = local stress at saddle, lb/in²

 S_{EC} = stress from expansion and contraction (if restrained), lb/in²

The total stress must be less than the allowable stress.

$$\mathbf{S}_{\mathrm{T}} < \mathbf{S}_{\mathrm{all}} \tag{52-70}$$

where:

 S_{T} = total stress at saddle, lb/in²

 S_{all} = allowable stress, lb/in² (50% of yield strength for steel, 48,000 lb/in² for ductile iron, and 7,500lb/in² for aluminum)

$$= \frac{\text{HDB} \times \text{T}_{f}}{\text{FS}} \text{ for plastic}$$

HDB = hydrostatic design basis T_{f} = temperature factor from table 52–1. \vec{FS} = factor of safety (2.5 for AWWA C900 pipe, 2.0 for others)

(f) Buckling

For aboveground pipe subject to external hydrostatic pressure or internal vacuum pressure, the critical collapse pressure may be determined by the following equations:

$$P_{CR} = \frac{3EI_{pw}}{\left(1 - v^2\right)r^3} \quad \text{for all pipe}$$
 (52–71)

$$P_{CR} = \frac{0.447PS}{\left(1 - v^2\right)}$$
 for corrugated plastic pipe (52–72)

$$P_{CR} = \frac{2E}{(1 - v^2)} \left(\frac{1}{SDR - 1}\right)^3$$
(52-73)

or

$$P_{CR} = \frac{2E}{(1 - v^2)} \left(\frac{1}{SIDR + 1}\right)^3$$
 for solid-wall pipe (52–74)

where:

- = critical external collapse pressure, lb/in² P_{CR}
- Ε = modulus of elasticity, lb/in² The long-term modulus of elasticity is recommended if the pipe is subject to the presssure in the normal operations. If the pipe is subject to the pressure for short time periods and infrequently, the use of the short-term modulus of elasticity is acceptable.

= pipe wall moment of inertia, in⁴ I_{pw}

= Poisson's ratio (0.30 for steel and ductile iron, 0.33 for aluminum, 0.38 for PVC, 0.40 for PE, 0.50 for ABS)

$$PS = pipe stiffness, lb/in^2$$

$$SDR = D_0$$
 dimension ratio

$$SIDR = D_i$$
 dimension ratio

636.5207 Thrust block design

The internal pressure of a pipe acts perpendicular to any plane with a force equal to the pressure, P, times the area of the pipe, A. The radial forces within the pipe are balanced by the tension in the pipe wall. The axial components of pressure through a straight section are balanced by the same pressure in the opposite direction. An unbalanced thrust force will exist in other configurations (fig. 52–14).

The internal pressure used in thrust block design is the working pressure for a pumped system or static pressure head in a gravity system.

Abrupt changes in pipeline grade, horizontal alignment, or reduction in pipe size normally require an anchor or thrust blocks (fig. 52–15) to absorb any axial thrust of the pipeline. Thrust control may also be needed at the end of the pipeline and at inline control valves.

Thrust blocks and anchors must be large enough to withstand the forces tending to move the pipe, including those of momentum and pressure, as well as forces from expansion and contraction.

The positioning of the thrust blocks must consider whether connections adjacent to the thrust block are capable of movement, as well as the anticipated direction of movement.

The vector sum of the pressure forces is shown as T, a thrust force, for various configurations in figure 52–14. The area of the thrust block may be determined by the following:

$$A_{\rm T} = \frac{\rm T}{\rm q_{all}} \tag{52-75}$$

where:

 A_{T} = area of thrust block required, ft²

T = thrust force, lb

 q_{all} = allowable soil bearing pressure, lb/ft²

If adequate soil tests are not available, the soil pressure may be estimated from table 52–6.

Natural soil material	Depth of	f cover to ce	enter of thru	st block
	2 ft	3 ft lb	4 ft /ft ²	5 ft
Sound bedrock	8,000	10,000	10,000	10,000
Dense sand and gravel mixture (assumed $\emptyset = 40^{\circ}$)	1,200	1,800	2,400	3,000
Dense fine to coarse sand (assumed $\emptyset = 35^{\circ}$)	800	1,200	1,650	2,100
Silt and clay mixture (assumed $\emptyset = 25^{\circ}$)	500	700	950	1,200
Soft clay and organic soils (assumed $\emptyset = 10^\circ$)	200	300	400	500

Figure 52–14 Thrust forces



Figure 52–15 Thrust block types





Plan view



Plan view



Ø

Plan view

Plan view

(52 - 76)

636.5208 Longitudinal bending

Flexible plastic pipe is often installed in conditions that require longitudinal bending. Steel, corrugated metal pipe, and ductile iron pipe will withstand minimal longitudinal bending. Controlled longitudinal bending of the pipe within acceptable limits can be accommodated by the flexibility of the pipe itself. Additional longitudinal deviation must be accomplished by joint deflection or the use of special fittings. Joint deflection limits may be obtained from the manufacturer. Acceptable bending may be expressed in terms of the minimum bending radius calculated by:

R_b = minimum bending radius, in

E = short-term modulus of elasticity, lb/in^2

 $R_{b} = \frac{ED_{o}}{2S_{ball}}$

 $D_0 =$ outside pipe diameter, in

 S_{ball} = allowable bending stress, lb/in²

 $=\frac{\text{HDB} \times \text{T}_{f}}{\text{FS}} \text{ (nonpressure or gasketed pressure plastic pipe)}$

$$=\frac{\left(\mathrm{HDB}-\frac{\mathrm{HDB}}{2}\right)\times\mathrm{T}_{f}}{\mathrm{FS}}$$

HDB = hydrostatic design basis

 T_{f} = temperature factor from table 52–1.

FS = factor of safety (2.5 for AWWA C900 pipe, 2.0 for others)

= S_{all} – S_p (for steel, aluminum, corrugated metal, and ductile iron pipe) where:

- $$\begin{split} S_{all} &= allowable \, stress, \, lb/in^2 \, (50\% \, of \, yield \\ strength \, for \, steel, \, 48,000 \, lb/in^2 \, for \\ ductile \, iron, \, and \, 7,500 \, lb/in^2 \, for \\ aluminum) \end{split}$$
- $S_p = stress caused by internal pressure, lb/in²$

$$=\frac{\text{PD}_o}{2\text{t}}$$

where:

P = maximum working pressure or static pressure, lb/in²

 $D_0 =$ outside pipe diameter, in

- t = pipe wall thickness, in
 - net pipe wall thickness, in (for ductile iron)

Some bending may be accomplished by axial joint deflection in gasketed pipe joints. The amount of joint deflection may be obtained from the pipe manufacturer. Solvent cemented or welded joints do not allow joint deflection.

636.5209 References

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Glossary

Buckling. Failure by lateral or torsional instability of a structural member, occurring with stresses below the yield strength.

Collapse pressure (critical buckling pressure). The negative pressure at which the pipe collapses caused by water column separation from valve closure, sudden air evacuation, surge pressures, or other causes.

Critical time. Longest elapsed time before final flow stoppage that will still allow the maximum pressure surge to occur.

Deflection. The decrease in the vertical diameter of a pipe due to load, divided by the nominal diameter, expressed as a percent.

Gage. Reference system for thickness of metal sheets or wire.

Hoop stress. The tensile stress in the wall of the pipe in the circumferential orientation due to internal hydrostatic pressure.

Hydrostatic design basis. One of a series of established stress values specified in ASTM D 2837 for a plastic compound obtained by categorizing the longterm hydrostatic strength determined in accordance with Test Method D 2837.

Hydrostatic design stress. The recommended maximum hoop stress that can be applied continuously with a high degrees of certainty that failure of the pipe will not occur.

Modulus of soil reaction, E prime (E'). Measure of the stiffness of the embedment material that surrounds the pipe.

Out-of-roundness. The allowed difference between the maximum measured diameter and the minimum measured diameter (stated as an absolute deviation).

Pipe stiffness. For plastic pipe, a term to describe the stiffness of the pipe from a parallel plate test, which defines the pipe's resistance to load.

Pressure rating or pressure class. The maximum internal water pressure that can be exerted continuously in a pipe without damage at a specific temperature (73 $^{\circ}$ F).

Standard dimension ratio (SDR) or dimension ratio (DR). A specific ratio of the average specified outside diameter to the minimum specified wall thickness for outside diameter-controlled plastic pipe.

Standard inside dimension ratio (SIDR). A specific ratio of the average specified inside diameter to the minimum specified wall thickness for inside diametercontrolled plastic pipe.

Static head. The height of water above any plane or reference point.

Static pressure. The internal pressure when no flow is occurring in the pipe.

Surge pressure. The maximum pressure increase greater than working pressure (sometimes called water hammer) that is anticipated in the system as a result of change in velocity in the water. Some causes of surge include the opening and closing (full or partial) of valves, starting and stopping of pumps, changes in reservoir elevation, liquid column separation, and entrapped air.

Thrust in a pipe wall. The circumferential compressive force in the conduit walls, per unit length of pipe.

Total system pressure. The sum of working pressure plus surge pressure.

Water hammer. A pressure surge in a pipeline caused by a sudden change in water velocity. Typical causes include the sudden starting or stopping of a pump, sudden valve movement, or air movement in a pipeline. The surge may damage or destroy pipelines and pumps if severe.

Working pressure. The maximum anticipated sustained operating pressure for the system.

Appendix 52A

Symbols Used in NEH 636, Chapter 52

Α	pipe cross section area, in 2	PS	pipe stiffness
Anw	area of pipe wall, in ² /in	PCR	critical collapse pressure, lb/in ²
A _s ^p	area of section, in ² /ft		(aboveground pipe)
Å	area of thrust block required, ft ²	PG	external hydrostatic pressure, lb/ft ²
a	velocity of pressure wave, ft/s	P	wheel load at the surface, lb
B '	empirical coefficient of elastic support	Ps	pressure on pipe from weight of soil, lb/ft ²
b	aboveground pipe saddle width, in	P _v	internal vacuum pressure, lb/ft ²
С	reduction factor for buckling pressure	Pw	pressure on pipe from wheel load, lb/ft ²
DL	deflection lag factor	P _{bs}	pressure to develop maximum ring bending,
$\mathbf{D}_{\mathbf{M}}^{-}$	mean pipe diameter, in		lb/in ² (ductile iron pipe)
D	outside pipe diameter, in	P _{rd}	pressure to develop allowable ring
Di	inside pipe diameter, in		deflection, lb/in ² (ductile iron pipe)
Ε	modulus of elasticity of pipe material, lb/in ²	P _{surge}	surge pressure, lb/in ²
E _{long}	long-term modulus of elasticity, lb/in ²	Pwork	working pressure, lb/in ²
E'	modulus of soil reaction, lb/in ²	$\mathbf{q}_{\mathbf{a}}$	allowable buckling pressure, lb/in ²
EI	pipe wall stiffness, in-lb	$\mathbf{q}_{\mathbf{all}}$	allowable soil bearing pressure of the soil,
e	base of neutral logs, 2.71828		lb/ft^2
FF	flexibility factor, in/lb	RSC	ring stiffness constant
FS	factor of safety	R _b	minimum bending radius, in
f	design maximum bending stress, lb/in ²	R _o	outside radius of pipe, in
$f_{ m c}$	critical buckling stress, lb/in ² (CMP)	R _{support}	aboveground pipe support reaction, lb
f_{u}	minimum tensile strength of material, lb/in ²	$\mathbf{R}_{\mathbf{w}}$	water buoyancy factor
	(CMP)	r	radius of gyration of corrugation, in
f_{y}	minimum yield strength, lb/in ² (CMP)	S	allowable stress, lb/in ²
g	acceleration of gravity, 32.2 ft/s^2	SDR	standard dimension ratio (same as DR)
H	maximum working pressure, ft	SIDR	standard inside diameter dimension ratio
HDB	hydrostatic design basis, lb/in ²	~~	(same as IDR)
HDS	hydrostatic design stress, lb/in ²	SS	required seam strength, lbf/ft
H _{surge}	surge pressure, ft of water	STHS	short-term hoop strength, lb/m ²
h	height of ground surface above top of pipe, it	STR	short-term pressure rating, lb/in ²
n _w	neight of water above top of pipe, it	818	short-term strength (quick burst pressure), $\frac{1}{2}$
I	moment of inertia, III^{\pm}	C	1D/10 ²
I pw	pipe wall moment of inertia, in [*]		allowable stress, ID/In ²
I _f V	Impact factor	s s	allowable handing stress, lb/in ²
N V	bedding constant		allowable behaling stress, 10/11-
к к	bulk modulus of liquid lb/in ²	S _{EC}	lb/in ²
м _L	deflection coefficient	S	10/10
м _х k	soil stiffness factor = 0.22 for good fill mate	s S	stress from internal pressure lb/in^2
ĸ	rial compacted to 90% of standard density	S	total stress at saddle support lb/in^2
	hased on ASTM D 698	ST S	vield strength lb/in ²
k	coefficient for saddle support	т Т	thrust force lb
Support	distance within the nineline that the pressure	Т	thrust in nine wall lh/ft
	wave moves before it is reflected back by a	трw Т	critical time second
	boundary condition ft	\mathbf{T}_{cR}	temperature factor
L	span length, in	-r t	wall thickness, in
span L	length of unrestrained pipe, in	ť	net pipe wall thickness, in (ductile iron pipe)
-ur M	bending moment. in-lb	t.	minimum manufacturing thickness. in (duc-
P	pressure in or on pipe, lb/ft ²	-1	tile iron pipe)
PC	pressure class, lb/in ²	t.,	nominal pipe wall thickness. in (ductile iron
PR	pressure rating, lb/in ²	п	pipe)

W	total load on span, lb
WL	wheel load per linear foot of pipe, lb/ft
Ws	soil load per linear foot of pipe, lb/ft
Wv	vacuum load per linear foot of pipe, lb/ft
w	load of pipe filled with liquid, lb/in
У	maximum deflection at the center of span, in
α	coefficient of thermal expansion, in/in/°F
β	saddle angle, degrees
∆ /D	percent deflection expressed as a decimal
$\Delta \mathbf{T}$	change in temperature, °F
$\Delta \mathbf{V}$	change in velocity of fluid, ft/s
$\Delta \mathbf{Y}$	vertical change in diameter, in
8	maximum combined strain in pipe wall be-
	cause of ring bending
ϵ_{all}	allowable strain in pipe wall
ε _h	strain in the pipe wall caused by hoop stress
$\epsilon_{\mathbf{f}}$	strain in the pipe wall caused by bending or
	flexure
ε _h	strain in the pipe wall caused by hoop stress
γ_{s}	unit weight of soil, lb/ft ³
$\gamma_{\mathbf{w}}$	unit weight of water, 62.4 lb/ft^3
ρ	density of fluid, slugs/ft ³
θ	angle of pipe bed, degrees
σ	allowable long-term compressive stress,
	lb/in ²
ν	Poisson's ratio
$\Delta \mathbf{L}$	change in length, in
$\Delta X/D$	percent deflection expressed as a decimal
$\Delta \mathbf{y}$	Vertical decrease in diameter, in
%∆ X/D	percent deflection

Design Example 1 — Plastic Pipe



Design example 1—Plastic pipe (continued)

Solution:	A.	Dimension ratio (SDR for outside diameter controlled pipe) for maximum pressure (in- cluding surge pressure Water pressure—From table 52C–3 of appendix 52–C, an SDR of 26 is required since a 12-inch PVC 2116 pipe with SDR of 26 has a working pressure of 125 lb/in ² , which is greater than the 110 lb/in ² maximum pressure (including surge pressure).
	В.	External soil and wheel loads From equation 52–17, soil pressure is
		$P_s = \gamma_s \times h$
		$= 120 \times 10 = 1,200 \text{ lb/ft}^2$
		Wheel loading: From table 52C–3 of appendix 52–C, 12-inch PVC pipe with a SDR of 26 has a thickness, t=0.49 in. From equations 52–19 and 52–21: Since $D_o - t < 2.67h \times 12$ $12.75 - 0.49 < 2.67 (10) \times 12$ 12.26 < 320.4
		$W_{L} = \frac{0.48P_{L}I_{f}\left(\frac{D_{o}-t}{12}\right)^{2}}{2.67h^{3}} \left[\frac{2.67h}{\left(\frac{D_{o}-t}{12}\right)} - 0.5\right]$
		$W_{L} = \frac{0.48(10,000)(1.0)\left(\frac{12.75 - 0.49}{12}\right)^{2}}{2.67(10)^{3}} \left[\frac{2.67(10)}{\left(\frac{12.75 - 0.49}{12}\right)} - 0.5\right]$
		$W_L = 48 \text{ lb} / \text{ft of pipe}$
		$P_{w} = \frac{W_{L}}{\left(\frac{D_{o}}{12}\right)} = \frac{48}{12.75/12} = 45 \text{ lb}/\text{ft}^{2}$
		Design pressure: $P = P_0 + P_{yy} + P$
		$= 1,200 + 45 + 0 = 1,245 \text{ lb/ft}^2$
		From equation 52–26:
		Thrust: $T_{pw} = \frac{P \times \left(\frac{D_{o}}{12}\right)}{2}$
		$T_{pw} = \frac{1,245 \times \frac{12.75}{12}}{2}$
		$T_{pw} = 661 \text{ lb/ft of pipe}$

Design example 1—Plastic pipe (continued)

C. Required wall area for external load
From equation 52–27:

$$A_{pw} = \left(\frac{\frac{1}{12}}{\sigma}\right)$$

$$A_{pw} = \left(\frac{661}{12}\right)$$

$$A_{pw} = \left(\frac{661}{12}\right)$$

$$A_{pw} = \left(\frac{661}{12}\right)$$

$$A_{pw} = \left(\frac{60}{12}\right)$$

$$G = 1,600 \text{ lb/in}^2 \text{ from appendix 52C, table 52C, table 52C-1}$$

$$A_{pw} = 0.034 \text{ in}^2/\text{in}$$
Wall area of 12-inch pipe with SDR of 26 using equation 52–28:

$$A_{pw} = \left(\frac{D_o - D_i}{2}\right)$$
or t

$$A_{pw} = t$$

$$A_{pw} = t$$

$$A_{pw} = 0.49 \text{ in}^2/\text{in} > 0.034 \text{ in}^2/\text{in} \quad \text{O.K.}$$
D. Deflection
From equation 52–29, percent deflection for solid wall pipe is

$$\frac{96\Delta X}{D} = \frac{\left(D_i P_s + P_t + P_v\right) \left(\frac{1}{144}\right) K(100)}{\left[\left(\frac{2E}{144}\right) - \frac{1}{144}\right]} = \frac{15.(1,200) + 45 + 0!}{(0.11)(100)!}$$

$$\frac{96\Delta X}{D} = \frac{\left(1.5(1,200) + 45 + 0!}{\left[\left(\frac{2(400,000)}{3(26 - 1)^{3'}}\right) + 0.061(200)\right]}\right]}$$

$$\frac{96\Delta X}{D} = 4.38\% < \left(\frac{\%\Delta X}{D}\right)_{allowable} = 7.5\%$$
O.K.

E. Allowable buckling pressure From equation 52–33:

$$q_{a} = \frac{1}{FS} \left(32R_{w}B'E'\frac{E_{long}I_{pw}}{D_{o}^{3}} \right)^{1/2}$$

Design example 1—Plastic pipe (continued)

where:
$$\frac{h}{\left(\frac{D_{o}}{12}\right)} = \frac{10}{\left(\frac{11.75}{12}\right)} = 9.41 > 2$$

FS = 2.5
R_w = 1.0
B' = $\frac{4\left(h^{2} + \frac{D_{o}}{12}h\right)}{1.5\left(2h + \frac{D_{o}}{12}\right)^{2}} = \frac{4\left(120^{2} + \left(\frac{12.75}{12}\right)10\right)}{1.5\left(2(10) + \frac{12.75}{12}\right)^{2}} = 0.66$
I_{pw} = $\frac{t^{3}}{12} = \frac{(0.49)^{3}}{12} = 0.0098 \text{ in}^{4} / \text{ in}$

From equation 52–34, the reduction factor for the allowable buckling pressure from the deflection of the pipe is

$$C = \left[\frac{\left(1 - \frac{\%\Delta X}{D} \frac{1}{100}\right)}{\left(1 + \frac{\%\Delta X}{D} \frac{1}{100}\right)^2} \right]^3$$
$$= \left[\frac{\left(1 - 4.38 \frac{1}{100}\right)}{\left(1 + 4.38 \frac{1}{100}\right)^2} \right]^3$$
$$= 0.676$$

The reduced allowable buckling pressure is

$$q_a C = 3,045 \times 0.676$$

= 2,058 lb/ft² > 1,245 lb/ft² O.K.

F. Strain

The allowable deflection for PVC pipe limits strain in PVC pipes. Therefore, computation of strain and comparison to an allowable strain limit is not required for PVC pipe.

Conclusion: A PVC pipe of PVC 2116 plastic with a DR of 26 should be installed and the backfill at least slightly compacted.

Design Example 2 — Steel Pipe



Design example 2—Steel pipe (continued)

B. External loads From equations 52–17 and 52–18, soil pressure is
$$\begin{split} P_{s} &= \gamma_{s} \times h \\ &= 120 \times 15 = 1,800 \text{ lb/ft}^{2} \\ W_{s} &= P_{s} \times \frac{D_{o}}{12} \\ &= 1,800 \times \frac{24}{12} = 3,600 \text{ lb/ft} = 300 \text{ lb/in of pipe} \\ \end{split}$$
From equation 52–19 and 52–21, wheel loading is calculated using the following: Since $D_{o} - t < 2.67h \times 12 \\ 24 - 0.125 < 2.67 (15) \times 12 \\ 23.875 < 480 \\ W_{L} &= \frac{0.48P_{L}I_{J} \left(\frac{D_{o} - t}{12}\right)^{2}}{2.67h^{3}} \left[\frac{2.67h}{\frac{D_{o} - t}{12}} - 0.5\right] \\ W_{L} &= \frac{0.48(10,000)(1.0)\left(\frac{24 - 0.125}{12}\right)}{2.67(15)^{3}} \left[\frac{2.67(15)}{\frac{24 - 0.125}{12}} - 0.5\right] \\ W_{L} &= 41.4 \text{ lb/ft of pipe} \\ \end{aligned}$

$$W = W_{s} + W_{L} + W_{v}$$

= 3,600 + 41 + 0 = 3,641 lb / ft
$$P_{w} = \frac{W_{L}}{\frac{D_{o}}{12}} = \frac{3,641}{\left(\frac{24}{12}\right)} = 1,820 \text{ lb / ft}^{2}$$

Design example 2—Steel pipe (continued)

C. Deflection of the steel pipe
From equation 52–41

$$\Delta X = \left(\frac{\left(D_L W_S + W_L + W_V \right) \left(\frac{1}{12} \right) Kr^3}{EI_{pw} + 0.061E'r^3} \right) \text{ and }$$

$$I_{pw} = \frac{t^3}{12} = \frac{0.125}{12}$$

$$= .000162 \text{ in}^4 / \text{ in }$$

$$\Delta X = \left(\frac{\left[1.5(300) + 3.4 + 0 \right] \left(\frac{1}{12} \right) (0.1) \left(\frac{24}{2} \right)^3}{(29,000,000) (.000162) + 0.061 (50) \left(\frac{24}{2} \right)^3} \right)$$

$$= 7.8 \text{ in }$$

Percent deflection:

$$\% \frac{\Delta X}{D} = \frac{\Delta X}{D_0} \times 100 = \frac{7.8}{24} \times 100 = 32.7\% > 5\%$$
 for unlined pipe

Since the deflection is excessive, try a wall thickness, t, of 3/16 in $t^3 = 0.1875^3$

$$I_{pw} = \frac{t}{12} = \frac{0.1873}{12} = 0.000549 \text{ in}^4 / \text{ in}$$

$$\Delta X = \left[\frac{\left[1.5(3,600) + 41 + 0 \right] \left(\frac{1}{12} \right) (0.1) \left(\frac{24}{2} \right)^3}{(29,000,000) (0.000549) + 0.061 (50) \left(\frac{24}{2} \right)^3} \right]$$

= 3.69 in

$$\% \frac{\Delta X}{D_o} = \frac{\Delta X}{D_o} \times 100$$
$$= \frac{3.69}{24} \times 100$$
$$= 15.4 > 5\% \qquad \text{for an unlined pipe}$$

a

Design example 2—Steel pipe (continued)

Since the deflection is still excessive, try a wall thickness, t of 5/16 in

$$I_{pw} = \frac{t^{2}}{12}$$

$$= 0.00254 \text{ in}^{4} / \text{in}$$

$$\Delta X = \begin{bmatrix} (1.5(3,600) + 4.1 + 0)(\frac{1}{12})(0.1)(\frac{24}{2})^{3} \\ (29,000,000)(0.00254) + 0.061(50)(\frac{24}{2})^{3} \end{bmatrix}$$

$$= 0.99 \text{ in}$$

$$\Re \frac{\Delta X}{D_{o}} = \frac{\Delta X}{D_{o}} \times 100$$

$$= \frac{0.99}{24} \times 100 \quad \text{for an unlined pipe, therefore } t = \frac{5}{16} \text{ is OK}$$
D. Allowable buckling pressure
From equation 52-43:

$$q_{a} = \frac{1}{\text{FS}} \left(32\text{R}_{w}\text{B}\text{Fe}' \frac{\text{EI}_{pw}}{D_{o}^{3}} \right)^{1/2}$$
where:

$$\frac{h}{(\frac{D_{o}}{12})} = \frac{15}{12} = 7.5 \ge 2, \text{ so FS. 2.5}$$

$$R_{w} = 1.0$$

$$B' = \frac{1}{1+4e^{(-0.005h)}} = \frac{1}{1+4e^{(-0.005h)}} = 0.398$$

$$q_{a} = \frac{1}{2.5} \left(32(1.0)(0.398)(50)(\frac{(29,000,000)(0.00254)}{(24)^{3}} \right)^{1/2}$$

$$= 23.3 \text{ Ib} / \text{in}^{2} = 3.355 \text{ Ib} / \text{fr}^{2} > 1,820 \text{ Ib} / \text{fr}^{2} \text{ O.K}$$
Conclusion: The 24-inch steel pipe should be made of ASTM A 53, grade A steel or stronger with minimum wall thickness of 5/16 inch.

Design Example 3—Corrugated Metal Pipe



Design example 3—Corrugated metal pipe (continued)

From equation 52-19 and 52-21, and assuming t=0.060 inch,
since
$$D_{0}-t < 2.67 h \times 12$$

 $12-0.060 < 2.67 (3) \times 12$
 $11.94 < 96.1$
wheel loading is

$$W_{L} = \frac{0.48P_{L}I_{1}\left(\frac{D_{n}-t}{12}\right)^{2}}{2.67(h^{3})} \left[\frac{2.67(h)}{(\frac{D_{n}-t}{12})} - 0.5\right]$$

$$W_{L} = \frac{0.48(16,000)(1.0)\left(\frac{12-0.060}{12}\right)^{2}}{2.67(3)^{2}} \left[\frac{2.67(3)}{(\frac{12-0.060}{12})} - 0.5\right]$$

$$= 796 \text{ lb/ft of pipe}$$

$$P_{w} = \frac{W_{v}}{(\frac{D_{0}}{2})} = \frac{796}{12}}{\frac{12}{12}} = 796 \text{ lb/ft}^{2}$$
Design pressure: $P = P_{8} + P_{w} + P_{*}$
 $= 360 + 796 + 0 = 1,156 \text{ lb/ft}^{2}$
B. Thrust
From equation 52-45:

$$T_{pw} = \frac{P \times \frac{D_{1}}{12}}{2}$$

$$T_{pw} = 578 \text{ lb/ft of pipe}$$
C. Required cross-sectional area from equation 52-46

$$A_{s} = \frac{T_{pw} \times FS}{f_{s}}$$

$$A_{s} = \frac{578 \times 2}{24,000}$$
A. = 0.048 in ²/ft < 0.775 in ²/ft for a 16 gage (0.060 in thick) $2\frac{2}{3} \times \frac{1}{2}$ corrugations O.K.

Design example 3—Corrugated metal pipe (continued)

D. Buckling From equation 52-47 and 52-48: Since $\mathbf{D_i} < \frac{\mathbf{r}}{\mathbf{k}} \sqrt{\frac{24\mathbf{E}}{f_u}} = \frac{0.1712}{0.22} \sqrt{\frac{(24)10,000,000}{34,000}} = 65$ $f_{\rm c} = f_{\rm u} - \frac{{f_{\rm u}}^2}{48{\rm E}} \left(\frac{{\rm kD_i}}{{\rm r}}\right)^2$ $= 34,000 - \frac{34,000^2}{48(10,000,000)} \left(\frac{(0.22)12}{0.1712}\right)^2$ $= 33,427 \text{ lb}/\text{in}^2 > f_v \text{ of } 24,000 \text{ lb}/\text{in}^2 \text{ so wall area is O.K.}$ E. Seam strength From section 636.5204(c)(3), if helical lockseam or welded-seam (for steel) pipe is used, this criterion does not apply. For riveted corrugated pipe, using equation 52–51, $SS = T_{pw} \times FS$ $SS = 578 \times 3.0 = 1,734 \text{ lbf/ft} < 9,000 \text{ lb/ft}$ for single rivets (from appendix 52D, table 52D-4) F. Flexibility factor From section 636.5204(c)(4): $FF = \frac{D_i^2}{EI_{pw}} = \frac{12^2}{(10,000,000).001892}$ = 0.0076 < 0.031 from appendix 52-E **Conclusion:** A 12-inch diameter, 16-gage, corrugated aluminum pipe with 2 2/3 x 1/2 corrugation is acceptable.

Design Example 4 — Ductile Iron Pipe



Design example 4—Ductile iron pipe (continued)



Design example 4—Ductile iron pipe (continued)

From equation 52–56:

$$P_{rd} = \frac{\Delta X}{12K_{x}} \left[\frac{8E}{\left(\frac{D_{o}}{t_{1}} - 1\right)^{3}} + 0.732E' \right]$$

$$P_{rd} = \frac{.05}{12(0.103)} \left[\frac{8(24,000,000)}{\left(\frac{24}{0.36} - 1\right)^{3}} + 0.732(400) \right]$$

$$P_{rd} = 39 \text{ lb / in}^{2} = 39 \times 144 = 5,655 \text{ lb / ft}^{2}$$

$$P \le P_{d} \times 144$$

$$2,400 < 5,655 \text{ lb / ft}^{2}$$
O.K.

Design Example 5 — Thrust Block



Design example 5—Thrust block (continued)

Solution:	A.	Thrust force on the pipe bend From figure 52–14, the thrust force on a bend may be estimated by:
		$T = 2PA \sin\left(\frac{\theta}{2}\right) = 2 \times 50 \times \frac{\pi \times 12^2}{4} \sin\left(\frac{90}{2}\right) = 7,993 \text{ lb}$
	B.	Allowable soil bearing pressure The depth to the center of the thrust block is $h + \frac{D_o}{2} = 4 + \frac{12}{12 \times 2} = 4.5 \text{ ft}$
		From table 52–6 the allowable bearing capacity for silty clay soil at a depth of 4 feet is 950 lb/ft^2 and $1,200 \text{ lb/ft}^2$ at 5. The allowable bearing capacity at 4.5 feet may be determined by an average.
		$\frac{950+1,200}{2} = 1,075 \text{ lb/ft}^2$
	C.	Area of thrust block required From equation 52–75, the area of the thrust block required is: $A_{T} = \frac{T}{q_{T}} = \frac{7,993}{1.075} = 7.43 \text{ ft}^{2}$
Conclusion:	Th wo	e thrust block should be a minimum of 7.43 square feet. A block 2.75 feet by 2.75 feet buld be sufficient to resist the thrust force at the 90 degree bends.

Design Example 6 — Longitudinal Bending



Design example 6—Longitudinal bending (continued)

	B. Minimum bending radius From equation 52–76, the minimum bending radius is
	$R_{b} = \frac{ED_{o}}{2S_{ball}}$
	From appendix 52C, table 52C–5, the $\mathrm{D_o}$ = 8.625 inches
	$R_{\rm b} = \frac{110,000 \times 8.625}{2 \times 400} = 1,185 \text{ in} = 98.8 \text{ ft}$
Conclusion:	The minimum longitudinal bending radius of the HDPE pipe made of PE3408 material with an SDR of 21 is 99 feet.

Design Example 7 — Aboveground Pipe



Design example 7—Aboveground pipe (continued)

$$I = \frac{\pi}{64} (D_o^4 - D_i^4) = \frac{\pi}{64} \left[12.24^4 - (12.24 - 2(0.299))^4 \right] = 199 \text{ in}^4$$
$$y = \frac{0.0130 \times w \times L^4}{E_{\text{long}} \times I} = \frac{0.0130 \times 4.25 \times (10 \times 12)^4}{140,000 \times 199} = 0.41 \text{ in}$$

The maximum recommended deflection for PVC pipe is 0.50 percent of the span:

 $0.005 \times (10 \times 12) = 0.60$ in<0.41 in

B. Hoop stress from internal pressure
 From equation 52–66, the hoop stress from internal pressure is

$$S_{p} = \frac{P \times D_{i}}{2 \times t} = \frac{60 \times (12.24 - 2(0.299))}{2 \times 0.299} = 1,168 \text{ lb/in}^{2}$$

C. Bending stress caused by unsupported length From equation 52–62, the bending stress caused by unsupported length is

$$S_{b} = \frac{0.0625 \text{wL}_{\text{span}}{}^{2}\text{D}_{o}}{\text{I}} = \frac{0.0625 \times 4.25 \times \left[\left(10 \times 12\right)^{2} \times 12.24\right]}{199} = 235 \text{ lb/in}^{2}$$

D. Localized stress at the saddle From equation 52–67, the localized stress at the saddle is

$$S_1 = k_{support} \frac{R_{support}}{t^2} ln\left(\frac{R_o}{t}\right)$$

 $k_{\rm support} = 0.02 - 0.00012 \left(\beta - 90\right) = 0.02 - 0.00012 \left(120 - 90\right) = 0.0164$

$$R_{support} = \frac{wL_{span}}{2} = \frac{4.25 \times (10 \times 12)}{2} = 255 \text{ lb}$$

$$S_1 = 0.0164 \frac{255}{0.299^2} \ln\left(\frac{12.24}{2}\right) = 141 \text{ lb/in}^2$$

E. Stress caused by temperature change From section 636.5205 and table 52–5:

$$S_{EC} = E\alpha\Delta T = 400,000 \times .00003 \times 30 = 360 \text{ lb/in}^2$$

Design example 7—Aboveground pipe (continued)

	F. Total stress at the saddle support from equation 52–69 $S_{T} = vS_{p} + S_{b} + S_{1} + S_{EC} = 0.38 \times 1,168 + 235 + 141 + 360 = 1,179 \text{ lb/in}^{2}$	
	G. Allowable stress in the pipe wall From section 636.5206(e), allowable stress in the pipe wall is	
	$S_{all} = {HDB \times T_f \over FS} = {2,500 \times 1 \over 2} = 1,250 \text{ lb/in}^2$	
	$1,250 \text{ lb/in}^2 > 1,179 \text{ lb/in}^2$ O.K.	
Conclusion:	A PVC pipe of PVC 2112 with SDR of 41 will span 10 feet with an acceptable allowable deflection and allowable stress in the pipe wall.	

Design Example 8 — Plastic Pipe Siphon



Design example 8—Plastic pipe siphon (continued)

B. Percent deflection of the pipe from equation 52–29

$$\frac{\%\Delta X}{D} = \frac{(D_L P_s + P_L + P_v) \left(\frac{1}{144}\right) k(100)}{\left[\left(\frac{2E}{3(SDR - 1)^3}\right) + 0.061E'\right]}$$

$$\frac{\%\Delta X}{D} = \frac{(1.5 \times 200 + 0 + 1,008) \left(\frac{1}{144}\right) (0.1) (100)}{\left[\left(\frac{2(400,000)}{3(51 - 1)^3}\right) + 0.061(200)\right]}$$

$$\frac{\%\Delta X}{D} = 6.33\%$$

C. Allowable buckling pressure From equation 52–33

$$q_{a} = \frac{1}{FS} \left(32R_{w}B'E'\frac{E_{Long}I_{pw}}{D_{o}^{3}} \right)^{\frac{1}{2}}$$

where:

$$\begin{pmatrix} \frac{h}{\left(\frac{D_{o}}{12}\right)^{2}} \\ = \frac{2}{\frac{12}{10.2}} = 2.4 \ge 2 \text{ so F.S. } 2.5 \\ R_{w} = 1.0 \\ 4 \left(h^{2} + \left(\frac{D_{o}}{2}\right)h\right) = 4 \left[2^{2} + \left(\frac{10.2}{2}\right)2\right]$$

$$B' = \frac{I \left(12^{\circ} + \left(12^{\circ}\right)^{2}\right)}{1.5 \left(2h + \frac{D_{o}}{12}\right)^{2}} = \frac{I \left[12^{\circ} + \left(12^{\circ}\right)^{2}\right]}{1.5 \left(2^{2} + 10.2\right)^{2}} = 0.640$$
$$I_{pw} = \frac{t^{3}}{12} = \frac{0.2^{3}}{12} = 0.00067 \text{ in}^{4}$$

$$q_{a} = \frac{1}{2.5} \left[32(1.0)(0.646)(200) \frac{(140,000)(0.00067)}{(10.20)^{3}} \right]^{\frac{1}{2}}$$

= 7.64 lb/in²

Design example 8—Plastic pipe siphon (continued)

D. Reduced allowable buckling pressure From equation 52–34, the reduction factor for the allowable buckling pressure caused by deflection of the pipe is

$$C = \left[\frac{\left(1 - \frac{\%\Delta X}{D} \frac{1}{100}\right)}{\left(1 + \frac{\%\Delta X}{D} \frac{1}{100}\right)^2}\right]^3$$
$$= \left[\frac{\left(1 - 6.33 \frac{1}{100}\right)}{\left(1 + 6.33 \frac{1}{100}\right)^2}\right]^3$$
$$= 0.57$$

The reduced allowable buckling pressure caused by the deflected pipe is

$$\begin{split} q_{a}C &= 7.64 \times 0.57 \\ &= 4.4 ~ lb/in^{2} < P_{s} + P_{v} \\ P_{s} + P_{v} &= 1.38 + 7 \\ P_{s} + P_{v} &= 8.38 ~ lb/in^{2} > 4.4 ~ lb/in^{2} \end{split} \qquad \text{Not O.K.} \end{split}$$

The PVC PIP with an SDR of 51 and backfilled as assumed does not provide adequate resistance to buckling. A higher quality backfill or pipe with a lower SDR (thicker wall) should be investigated. Try an SDR of 41 with t = 0.299 inch, from table 52C–2.

B1. Percent deflection of the SDR 41 pipe From equation 52–29

$$\frac{\%\Delta X}{D} = \frac{(1.5 \times 200 + 0 + 1,008) \left(\frac{1}{144s}\right) (0.1) (100)}{\left[\left(\frac{2(400,000)}{3(41-1)^3} \right) + 0.061 (200) \right]}$$
$$\frac{\%\Delta X}{D} = 5.54\%$$

Design example 8—Plastic pipe siphon (continued)

C1. Allowable buckling pressure of SDR 41 pipe $I_{pw} = \frac{t^3}{12}$ $=\frac{0.299^3}{12}$ $= 0.00223 \text{ in}^4$ $q_{a} = \frac{1}{2.5} \left[32(1.0)(0.646)(200) \frac{(140,000)(0.00223)}{(10.20)^{3}} \right]^{\frac{1}{2}}$ $= 13.9 \text{ lb/in}^2$ D1. Reduced allowable buckling pressure of SDR 41 pipe The reduction factor for the allowable buckling pressure from the deflection of the pipe is $C = \left| \frac{\left(1 - 5.54 \frac{1}{100} \right)}{\left(1 + 5.54 \frac{1}{100} \right)^2} \right| = 0.61$ The reduced allowable buckling pressure caused by the deflected pipe is $q_aC = 13.95 \times 0.61$ $= 8.5 \text{ lb/in}^2 > P_s + P_v$ $P_{s} + P_{y} = 1.38 + 7$ $= 8.38 \text{ lb/in}^2 < 8.5$ O.K. **Conclusion:** The PVC PIP with an SDR of 51 and backfilled as assumed does not provide adequate resistance to buckling. Using a lower SDR of 41 provides adequate resistance to buckling.

Design Example 9-Plastic Pipe During Construction

Problem: An 18-inch diameter HDPE pipe will be installed as an outlet pipe in an earthen dam. Heavy construction equipment with wheel loads up to 16,000 pounds will be allowed to traverse the pipe once 2 feet of fill has been placed over the top of the pipe. The top of the pipe will be 10 feet below the top of the completed dam. The dam will be constructed of an SC material compacted to 90 percent of the standard Proctor density.



Assumptions: 1. The pipe is outside diameter controlled.

- 2. The HDPE pipe will be PE 3408 with a Hydrostatic Design Basis of 1,600 lb/in² (see app. 52C)
- 3. Assume unit weight of soil = 120 lb/ft^3
- 4. Since the pipe will be installed in an embankment dam of SC soil, $E' = 400 \text{ lb/in}^2$.
- 5. The allowable deflection is 5 percent.

Determine: A. External soil and wheel load during construction

- B. Required wall area for external load during construction
- C. Deflection during construction
- D. Allowable buckling during construction
- E. Strain during construction
- F. External soil load upon completion of the dam
- G. Required wall area for completed external load
- H. Deflection upon completion of the dam
- I. Allowable buckling upon completion of the dam
- J. Strain upon completion of the dam

Design example 9—Plastic pipe during construction (continued)

Solution:
A. External soli and wheel load during construction
From equation 52-17, the soil pressure due to 2 feet of soil is

$$P_{c} = \gamma_{c} \times h$$

 $= 120 \times 2$
 $= 240 \text{ lb/ft}^{2}$
Wheel loading: From table 52C-5 of appendix 52C, an 18-inch PE pipe with a SDR of 17 has a
thickness, t = 1.059 in. From section 636.5203 (b) and equations 52-19 and 52-21:
 $D_{o} = t < 2.67h \times 12$
Since $18 \cdot 1.161 < 2.67 (2) \times 12$
 $16.84 < 64.1$
 $W_{L} = \frac{0.48P_{L}I_{L}\left(\frac{D_{a} - t}{12}\right)^{2} \left[\frac{2.67h}{\left(\frac{D_{a} - t}{12}\right)^{-}} = 0.5\right]$
Since the depth of cover is 2.0, the I_t is 1.2.
 $W_{L} = \frac{0.48(16,000)(1.2)\left(\frac{18 - 1.059}{12}\right)^{2} \left[\frac{2.67(2)}{\left(\frac{18 - 1.059}{12}\right)} = 0.5\right]$
 $W_{L} = 2,822 \text{ lbs/ft of pipe}$
 $P_{w} = \frac{W_{L}}{\frac{D_{a}}{12}}$
 $= 2,822 \text{ lbs/ft of pipe}$
 $P_{w} = \frac{W_{L}}{\frac{D_{a}}{12}} = 1,881 \text{ lb/ft}^{2}$
Design Pressure :
 $P = P_{s} + P_{w} + P_{v}$
 $= 240 + 1,881 + 0 = 2,121 \text{ lb/ft}^{2}$
Design example 9—Plastic pipe during construction (continued)

From equation 52-26:
Thrust:

$$T_{pw} = \frac{P \times \left(\frac{D_{o}}{12}\right)}{2}$$

$$T_{pw} = \frac{2,121 \times \frac{18}{12}}{2}$$

$$T_{pw} = 1,591 \text{ lb/ft of pipe}$$
B. Required wall area for external load during construction
From equation 52-27:

$$A_{pw} = \frac{\frac{T_{pw}}{12}}{\sigma}$$

$$A_{pw} = \frac{\frac{1,591}{2}}{800}, \sigma = 800 \text{ lb/in}^{2} \text{ from appendix 52C, table 52C-1}$$

$$A_{pw} = 0.166 \text{ in}^{2}/\text{in}$$
Area of an 18-inch pipe with SDR of 17 using equation 52-28

$$\begin{split} A_{pw} &= \frac{(D_o - D_i)}{2} \text{ or } t \\ A_{pw} &= t \\ A_{pw} &= 1.059 \text{ in}^2/\text{in} > 0.166 \text{ in}^2/\text{in} \quad \text{O.K.} \end{split}$$

C. Deflection during construction:

From equation 52-29, the percent deflection for solid wall pipe is:

$$\frac{\%\Delta X}{D} = \frac{\left(D_{L}P_{S} + P_{W} + P_{V}\right)\left(\frac{1}{144}\right)K(100)}{\left[\left(\frac{2E}{3(SDR - 1)^{3}}\right) + 0.061E'\right]}$$
$$\frac{\%\Delta X}{D} = \frac{\left(1.5(240) + 1881 + 0\right)\left(\frac{1}{144}\right)(0.1)(100)}{\left[\left(\frac{2(110,000)}{3(17 - 1)^{3}}\right) + 0.061(400)\right]}$$
$$\frac{\%\Delta X}{D} = 3.67\% < \left(\frac{\%\Delta}{D}\right)_{allowable} = 5\% \quad O.K.$$

Design example 9—Plastic pipe during construction—(continued)

D.	Allowable buckling pressure during construction:
	From equation 52-33 using the short-term modulus of elasticity since the wheel loads are short and intermittent:
	$\mathbf{q}_{a} = \frac{1}{FS} \left(32\mathbf{R}_{w}\mathbf{B'E'} \frac{\mathbf{EI}}{\mathbf{D}^{3}} \right)^{1/2}$
	where: $\frac{h}{\left(\frac{D_{o}}{12}\right)} = \frac{2}{\left(\frac{18}{12}\right)} = 1.3 < 2$
	F.S.=3.0
	Rw=1.0
	$B' = \frac{4\left(h^2 + \left(\frac{D_0}{12}\right)h\right)}{1.5\left(2h + \frac{D_o}{12}\right)^2} = \frac{4\left(2^2 + \left(\frac{18}{12}\right)2\right)}{1.5\left(2*2 + \frac{18}{12}\right)^2} = 0.617$
	$I_{pw} = \frac{t^3}{12} = \frac{1.059^3}{12} = 0.099 \text{ in}^4/\text{in}$
	$q_{a} = \frac{1}{3.0} \left(32(1.0)(0.617)(400)\frac{(110,000)(0.099)}{(18)^{3}} \right)^{\frac{1}{2}}$
	$= 40.5 \text{ lb/in}^2 = 5.829 \text{ lb/ft}^2$

From equation 52-34, the reduction factor for the allowable buckling pressure from the deflection of the pipe is:

$$C = \left[\frac{\left(1 - \frac{\%\Delta X}{D} \frac{1}{100}\right)}{\left(1 + \frac{\%\Delta X}{D} \frac{1}{100}\right)^2} \right]^3$$
$$C = \left[\frac{\left(1 - 3.67 \frac{1}{100}\right)}{\left(1 + 3.67 \frac{1}{100}\right)^2} \right]^3$$
$$C = 0.72$$

The reduced allowable buckling pressure is

$$q_a C = 5,829 \times 0.72 = 4,197 \text{ lb/ft}^2 > P = 2,121 \text{ lb/ft}^2$$
 O.K.

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Design example 9—Plastic pipe during construction (continued)

E. Strain during construction From equation 52-36, the hoop strain due to external load is:

$$\epsilon_{\rm h} = \frac{\frac{P}{144}D_{\rm M}}{2tE} = \frac{\frac{P}{144} \times (18 - 1.059)}{2 \times 1.059 \times 110,000} = 0.011 \text{ in/in}$$

From equation 52-37, the maximum strain due to ring bending is:

$$\varepsilon_{\rm f} = \frac{1}{\rm SDR} \left(\frac{\frac{3\Delta Y}{D_{\rm M}}}{1 - 2\frac{\Delta Y}{D_{\rm M}}} \right) = \frac{1}{17} \left(\frac{3 \times 0.0367}{1 - 2(0.0367)} \right) = 0.007 \text{ in/in}$$

From equation 52-37, the combined strain is:

$$\varepsilon = \varepsilon_{\rm f} \pm \varepsilon_{\rm h}$$

-

Since the hoop strain is due to external load it is subtracted.

 $\epsilon = 0.007 - 0.0011 = 0.006$ $\epsilon=0.006<\epsilon_{\rm all}=5\%=0.05$ O.K.

F. External soil load upon completion of the dam From equation 52-17, the soil pressure due to 10 feet of soil is

$$\begin{split} P_{s} &= \gamma_{s} \times h \\ &= 120 \times 10 = 1,200 \ lb/ft^{2} \end{split}$$

Design Pressure :

$$\begin{split} P &= P_{\rm s} + P_{\rm w} + P_{\rm v} \\ &= 1,200 + 0 + 0 = 1,200 ~ \text{lb/ft}^2 \end{split}$$

/ \

From equation 52-26:

Thrust:
$$T_{pw} = \frac{P \times \left(\frac{D_o}{12}\right)}{2}$$

 $T_{pw} = \frac{1,200 \times \frac{18}{12}}{2}$
 $T_{pw} = 900$ lb/ft of pipe

Design example 9—Plastic pipe during construction (continued)

G. Required wall area for completed external load:
From equation 52-27:

$$A_{pw} = \frac{\frac{T_{pw}}{12}}{\sigma}$$

$$A_{pw} = \frac{\frac{900}{12}}{800}, \sigma = 800 \text{ lb/in}^2 \text{ frm appendix 52C, table 52C-1}$$

$$A_{pw} = 0.094 \text{ in}^2/\text{in}$$

Area of an 18-inch pipe with SDR of 17 using equation 52-28:

$$A_{pw} = \frac{(D_o - D_i)}{2}$$

$$A_{pw} = t$$

$$A_{pw} = 1.059 \text{ in}^2/\text{in} > 0.094 \text{ in}^2/\text{in}$$
O.K.

H. Deflection upon completion of the dam:

From equation 52-29, the percent deflection for solid wall pipe is:

$$\frac{\%\Delta X}{D} = \frac{\left(D_{L}P_{S} + P_{W} + P_{V}\right)\left(\frac{1}{144}\right)K(100)}{\left[\left(\frac{2E}{3(SDR - 1)^{3}}\right) + 0.061E'\right]}$$
$$\frac{\%\Delta X}{D} = \frac{\left(1.5(1200) + 0 + 0\right)\left(\frac{1}{144}\right)(0.1)(100)}{\left[\left(\frac{2(110,000)}{3(17 - 1)^{3}}\right) + 0.061(400)\right]}$$
$$\frac{\%\Delta X}{D} = 2.95\% < \left(\frac{\%\Delta X}{D}\right)_{\text{allowable}} = 5\% \quad \text{O.K.}$$

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Design example 9—Plastic pipe during construction (continued)

I. Allowable buckling pressure upon completion of the dam:

From equation 52-33 using the long term modulus of elasticity since the soil load is a permanent load.

$$\mathbf{q}_{a} = \frac{1}{\mathrm{FS}} \left(32 \mathbf{R}_{w} \mathbf{B'E'} \frac{\mathbf{EI}}{\mathbf{D}^{3}} \right)^{1/2}$$

where:

$$\begin{split} \frac{h}{\left(\frac{D_o}{12}\right)} &= \frac{10}{\left(\frac{18}{12}\right)} = 6.66 \ge 2\\ F.S. &= 2.5\\ Rw &= 1.0\\ B' &= \frac{4\left(h^2 + \left(\frac{D_0}{12}\right)h\right)}{1.5\left(2h + \frac{D_o}{12}\right)^2} = \frac{4\left(10^2 + \left(\frac{18}{12}\right)10\right)}{1.5\left(2*10 + \frac{18}{12}\right)^2} = 0.663\\ I_{pw} &= \frac{t^3}{12} = \frac{1.059^3}{12} = 0.099 \text{ in}^4/\text{in}\\ q_a &= \frac{1}{2.5}\left(32(1.0)(0.663)(400)\frac{(22,000)(0.099)}{(18)^3}\right)^{1/2}\\ &= 22.5 \text{ lb/in}^2 = 3,242 \text{ lb/ft}^2 \end{split}$$

From equation 52-34, the reduction factor for the allowable buckling pressure from the deflection of the pipe is:

$$C = \left[\frac{\left(1 - \frac{\%\Delta X}{D} \frac{1}{100}\right)}{\left(1 + \frac{\%\Delta X}{D} \frac{1}{100}\right)^2} \right]^3$$
$$C = \left[\frac{\left(1 - 2.95 \frac{1}{100}\right)}{\left(1 + 2.95 \frac{1}{100}\right)^2} \right]^3$$
$$C = 0.77$$

The reduced allowable buckling pressure is

$$q_aC = 3,242 \times 0.77 = 2,496 \text{ lb/ft}^2 > P = 1,200 \text{ lb/ft}^2$$
 O.K.

Design example 9—Plastic pipe during construction (continued)

J. Strain upon completion of the dam

From equation 52-36, the hoop strain due to external load is:

$$\epsilon_{\rm h} = \frac{\frac{P}{144} D_{\rm M}}{2tE} = \frac{\frac{1,200}{144} \times (18 - 1.059)}{2 \times 1.059 \times 110,000} = 0.0006 \text{ in/in}$$

From equation 52-37, the maximum strain due to ring bending is:

$$\varepsilon_{\rm f} = \frac{1}{\rm SDR} \left(\frac{\frac{3\Delta Y}{D_{\rm M}}}{1 - 2\frac{\Delta Y}{D_{\rm M}}} \right) = \frac{1}{17} = 0.005 \text{ in/in}$$

From equation 52-37, the combined strain is:

$$\varepsilon = \varepsilon_f \pm \varepsilon_h$$

Since the hoop strain is due to external load it is subtracted.

$$\begin{split} \epsilon &= 0.005 - 0.0006 = 0.005 \\ \epsilon &= 0.005 < \epsilon_{\rm all} = 5\% = 0.05 \end{split} \tag{C.K.}$$

Conclusion: An HDPE pipe of PE3408 with an SDR of 17 is acceptable for both the construction loads and final soil loads. NOTE: The construction loads are the most critical.

Appendix 52C

Material Properties, Pressure Ratings, and Pipe Dimensions for Plastic Pipe

(*Note:* The source of the information in this appendix is subject to periodic updating. The source documents should be referenced for any updated information.)

Table 52C-1 Hydrostatic design basic plastic pipe Plastic pipe	s, allowable long-ter	m compressive st	ress, short-term h	oop strength, and designation of
Plastic pipe material	Hydrostatic design basis	Allowable long-term compressive stress	Short-term hoop strength	Designation
	(lb/in ²)	(lb/in ²)	(lb/in ²)	
PVC Type I, Grade 1 (12454-B)	4,000	2,000	6,400	PVC1120
PVC Type I, Grade 2 (12454-C)	4,000	2,000	6,400	PVC1220
PVC Type II, Grade 1 (14333-D)	4,000	2,000	6,400	PVC2120
PVC Type II, Grade 1 (14333-D)	3,200	1,600	5,000	PVC2116
PVC Type II, Grade 1 (14333-D)	2,500	1,250	5,000	PVC2112
PVC Type II, Grade 1 (14333-D)	2,000	1,000	5,000	PVC2110
ABS Type 1, Grade 2	1,600	800	3,300	ABS1208
ABS Type 1, Grade 2	2,000	1,000	5,240	ABS1210
ABS Type 2, Grade 1	2,700	1,350	6,600	ABS2112
ABS Type 1, Grade 3	3,200	1,600	6,000	ABS1316
PE Grade P 14	800	400	1,250	PE1404
PE Grade P 23	1,000	500	2,000	PE2305
PE Grade P 23	1,260	630	2,520	PE2306
PE Grade P 24	1,260	630	2,520	PE2406
PE Grade P 33	1,260	630	2,520	PE3306
PE Grade P 34	1,260	630	2,520	PE3406
PE Grade P 34	1,600	800	3,200	PE3408

Source: ASTM D 1527, D 1785, D 2104, D 2239, D 2241, D 2282, and D 3035.

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 Table 52C-2
 PVC plastic irrigation pipe (PIP)

Nominal	SDR/		- PVC pressu	re rating (lb/i	in ²)	 Wall th	Dime	ension and to	olerance	
(in)	head	1120 1220	2116	2112	2110	minimum (in)	tolerance (in)	average (in)	± tole	rance max &
									OD (in)	min (in)
4	$50 \mathrm{ft}$	22				0.065	+0.020	4.134	0.009	0.050
	81	50	40	30	25	0.065	+0.020			
	51	80	63	50	40	0.081	+0.020			
	41	100	80	63	50	0.101	+0.020			
	32.5	125	100	80	63	0.127	+0.020			
	26	160	125	100	80	0.159	+0.020			
6	$50 \mathrm{ft}$	22				0.070	+0.020	6.140	0.011	0.050
	100 ft	44				0.070	+0.020			
	81	50	40	30	25	0.076	+0.020			
	51	80	63	50	40	0.120	+0.020			
	41	100	80	63	50	0.150	+0.020			
	32.5	125	100	80	63	0.189	+0.023			
	26	160	125	100	80	0.236	+0.028			
8	50 ft.	22				0.080	+0.020	8.160	0.015	0.075
	100 ft	44				0.087	+0.020			
	81	50	40	30	25	0.101	+0.020			
	51	80	63	50	40	0.160	+0.020			
	41	100	80	63	50	0.199	+0.024			
	32.5	125	100	80	63	0.251	+0.031			
	26	160	125	100	80	0.314	+0.038			
10	$50 \mathrm{~ft}$	22				0.100	+0.020	10.200	0.015	0.075
	100 ft	44				0.109	+0.020			
	81	50	40	30	25	0.126	+0.020			
	51	80	63	50	40	0.200	+0.024			
	41	100	80	63	50	0.240	+0.030			
	32.5	125	100	80	63	0.314	+0.038			
	26	160	125	100	80	0.392	+0.047			
12	$50 \mathrm{~ft}$	22				0.120	+0.020	12.240	0.015	0.070
	100 ft	44				0.131	+0.020			
	81	50	40	30	25	0.151	+0.020			
	51	80	63	50	40	0.240	+0.029			
	41	100	80	63	50	0.299	+0.036			
	32.5	125	100	80	63	0.377	+0.045			
	26	160	125	100	80	0.471	+0.056			
14	51	80	63	50	40	0.280	+0.034	14.280	0.015	0.075
	41	100	80	63	50	0.348	+0.042			
	32.5	125	100	80	63	0.439	+0.053			
	26	160	125	100	80	0.549	+0.066			

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Nominal	SDR/		- PVC pressu	re rating (lb/i	n ²)	Wall th	Dime	ension and to	olerance	ter
(in)	head	1120 1220	2116	2112	2110	minimum (in)	tolerance (in)	average (in)	± tole average OD (in)	rance max & min (in)
15	50 ft	22				0.150	+0.020	15.3	0.016	0.075
	100 ft	44				0.164	+0.020			
	81	50	40	30	25	0.189	+0.023			
	51	80	63	50	40	0.300	+0.042			
	41	100	80	63	50	0.373	+0.052			
	32.5	125	100	80	63	0.437	+0.052			
	32.5	125	100	80	63	0.471	+0.056			
	26	160	125	100	80	0.588	+0.070			
	21	200	160	125	100	0.728	+0.087			
18	100 ft	44				0.200	+0.024	18.701	0.020	0.075
	81	50	40	30	25	0.231	+0.028			
	51	80	63	50	40	0.366	+0.051			
	41	100	80	63	50	0.456	+0.064			
	32.5	125	100	80	63	0.534	+0.064			
	32.5	125	100	80	63	0.575	+0.069			
	26	160	125	100	80	0.719	+0.086			
21	100 ft	44				0.236	+0.028	22.047	0.025	0.075
	81	50	40	30	25	0.272	+0.033			
	51	80	63	50	40	0.432	+0.060			
	41	100	80	63	50	0.538	+0.075			
	32.5	125	100	80	63	0.630	+0.076			
	32.5	125	100	80	63	0.678	+0.081			
	26	160	125	100	80	0.848	+0.102			
24	100 ft	44				0.266	+0.032	24.803	0.032	0.075
	81	50	40	30	25	0.306	+0.037			
	51	80	63	50	40	0.486	+0.068			
	41	100	80	63	50	0.605	+0.085			
	32.5	100	80	63	50	0.709	+0.085			
	32.5	125	100	80	63	0.763	+0.092			
	26	160	125	100	80	0.954	0.115			
27	51	80	63	50	40	0.548	+0.077	27.953	0.038	0.075
	41	100	80	63	50	0.682	+0.095			
	32.5	125	100	80	63	0.799	+0.096			
	32.5	125	100	80	63	0.860	+0.103			
	26	160	125	100	80	1.075	+0.129			

Table 52C-2 PVC plastic irrigation pipe (PIP)—Continued

Source: ASTM D 2241 and ASAE S376.2

Note: PIP pipe sizes in the source documents were developed from Soil Conservation Service Practice Standards 430DD and 430EE, whic have been rescinded.

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 Table 52C-3
 PVC and ABS thermoplastic pipe (SDR-PR)-(IPS) (nonthreaded)

Nominal pipe size (in)	SDR	PVC 1120 1220 2120	pressure Mate 2116	rating (lb/ erial 2112	/in ²) 2110	Wall thi min. (in)	- Dimensio ickness) toler-	on and tol Out average (in)	erance tside diam ±tole avg OD	eter rance max &	AB 1316	S pressur Mate 2112	e rating (ll erial 1210	b/in ²) 1208
		2120							(111)					
1/8	13.5	315	250	200	160	0.060	+0.020	0.405	0.004	0.008	250	200	160	125
1/4	13.5	315	250	200	160	0.060	+0.020	0.540	0.004	0.008	250	200	160	125
3/8	13.5	315	250	200	160	0.060	+0.020	0.675	0.004	0.008	250	200	160	125
1/2	17	250	200	160	125	0.060	+0.020	0.840	0.004	0.008	200	160	125	100
	13.5	315	250	200	160	0.062	+0.020			0.008	250	200	160	125
3/4	21	200	160	125	100	0.060	+0.020	1.050	0.004	0.015	160	125	100	80
	17	250	200	160	125	0.062	+0.020			0.010	200	160	125	100
	13.5	315	250	200	160	0.078	+0.020			0.010	250	200	160	125
1	26	160	125	100	80	0.060	+0.020	1.315	0.005	0.015	125	100	80	
	21	200	160	125	100	0.063	+0.020			0.015	160	125	100	80
	17	250	200	160	125	0.077	+0.020			0.010	200	160	125	100
	13.5	315	250	200	160	0.097	+0.020			0.010	250	200	160	125
1 1/4	32.5	125	100	80	63	0.060	+0.020	1.660	0.005	0.015				
	26	160	125	100	80	0.064	+0.020			0.015	125	100	80	
	21	200	160	125	100	0.079	+0.020			0.015	160	125	100	80
	17	250	200	160	125	0.098	+0.020			0.012	200	160	125	100
	13.5	315	250	200	160	0.123	+0.020			0.012	250	200	160	125
1 1/2	32.5	125	100	80	63	0.060	+0.020	1.900	0.006	0.030				
	26	160	125	100	80	0.073	+0.020			0.030	125	100	80	
	21	200	160	125	100	0.090	+0.020			0.030	160	125	100	80
	17	250	200	160	125	0.112	+0.020			0.012	200	160	125	100
	13.5	315	250	200	160	0.141	+0.020			0.012	250	200	160	125
2	32.5	125	100	80	63	0.073	+0.020	2.375	0.006	0.030				
	26	160	125	100	80	0.091	+0.020			0.030	125	100	80	
	21	200	160	125	100	0.113	+0.020			0.030	160	125	100	80
	17	250	200	160	125	0.140	+0.020			0.012	200	160	125	100
	13.5	315	250	200	160	0.176	+0.020			0.012	250	200	160	125
$2 \ 1/2$	32.5	125	100	80	63	0.088	+0.020	2.875	0.007	0.030				
	26	160	125	100	80	0.110	+0.020			0.030	125	100	80	
	21	200	160	125	100	0.137	+0.020			0.030	160	125	100	80
	17	250	200	160	125	0.169	+0.020			0.015	200	160	125	100
	13.5	315	250	200	160	0.213	+0.026			0.015	250	200	160	125
3	41	100	80	63	50	0.085	+0.020	3.500	0.008	0.030				
	32.5	125	100	80	63	0.108	+0.020			0.030				
	26	160	125	100	80	0.135	+0.020			0.030	125	100	80	
	21	200	160	125	100	0.167	+0.020			0.030	160	125	100	80
	17	250	200	160	125	0.206	+0.025			0.015	200	160	125	100
	13.5	315	250	200	160	0.259	+0.031			0.015	250	200	160	125

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 Table 52C-3
 PVC and ABS thermoplastic pipe (SDR-PR)-(IPS) (nonthreaded)—Continued

Nominal pipe size	SDR	PVC	pressure	rating (lb/ erial	/in ²)	Wall thi	- Dimensi ickness	on and to Ou	lerance tside diam	neter	AB	S pressur Mate	e rating (1 erial	b/in ²)
(in)		1120 1220 2120	2116	2112	2110	min. (in)	toler- ance (in)	average (in)	±tole avg OD (in)	erance max & min (in)	1316	2112	1210	1208
$3\frac{1}{2}$	41	100	80	63	50	0.098	+0.020	4.000	0.008	0.050				
	32.5	125	100	80	63	0.123	+0.020			0.050				
	26	160	125	100	80	0.154	+0.020			0.050	125	100	80	
	21	200	160	125	100	0.190	+0.023			0.050	160	125	100	80
	17	250	200	160	125	0.235	+0.028			0.015	200	160	125	100
	13.5	315	250	200	160	0.296	+0.036			0.015	250	200	160	125
4	64	63	50			0.070	+0.020	4.500	0.009	0.050				
	41	100	80	63	50	0.110	+0.020			0.050				
	32.5	125	100	80	63	0.138	+0.020			0.050				
	26	160	125	100	80	0.173	+0.020			0.050	125	100	80	
	21	200	160	125	100	0.214	+0.026			0.050	160	125	100	80
	17	250	200	160	125	0.265	+0.032			0.015	200	160	125	100
	13.5	315	250	200	160	0.333	+0.040			0.015	250	200	160	125
5	64	63	50			0.087	+0.020	5.563	0.010	0.050				
	41	100	80	63	50	0.136	+0.020			0.050				
	32.5	125	100	80	63	0.171	+0.021			0.050				
	26	160	125	100	80	0.214	+0.027			0.050	125	100	80	
	21	200	160	125	100	0.265	+0.032			0.050	160	125	100	80
	17	250	200	160	125	0.327	+0.039			0.030	200	160	125	100
	13.5	315	250	200	160	0.412	+0.049			0.030	250	200	160	125
6	64	63	50			0.104	+0.020	6.625	0.011	0.050				
	41	100	80	63	50	0.162	+0.020			0.050				
	32.5	125	100	80	63	0.204	+0.024			0.050				
	26	160	125	100	80	0.255	+0.031			0.050	125	100	80	
	21	200	160	125	100	0.316	+0.038			0.050	160	125	100	80
	17	250	200	160	125	0.390	+0.047			0.035	200	160	125	100
	13.5	315	250	200	160	0.491	+0.059			0.035	250	200	160	125
8	64	63	50			0.135	+0.020	8.625	0.015	0.075				
	41	100	80	63	50	0.210	+0.025			0.075				
	32.5	125	100	80	63	0.265	+0.032			0.075				
	26	160	125	100	80	0.332	+0.040			0.075	125	100	80	
	21	200	160	125	100	0.410	+0.049			0.045	160	125	100	80
	17	250	200	160	125	0.508	+0.061			0.045				
10	64	63	50	<u></u>		0.168	+0.020	10.750	0.015	0.075				
	41	100	80	63	50	0.262	+0.031			0.075				
	32.5	125	100	80	63	0.331	+0.040			0.075	105	100	00	
	26	160	125	100	80	0.413	+0.050			0.075	125	100	80	00
	21	200	100	125	100	0.511	+0.001			0.050	100	125	100	80
	17	250	200	160	125	0.632	+0.076			0.050				

Table 52C-3 PVC and ABS thermoplastic pipe (SDR-PR)-(IPS) (nonthreaded)—Continued

Nominal pipe size	SDR	PVC	pressure Mat	rating (lb erial	/in ²)	Wall th	- Dimensio	on and tol Out	lerance tside dian		ABS	S pressur Mate	e rating (lt rial	0/in ²)
(in)		1120 1220 2120	2116	2112	2110	min. (in)) toler- ance (in)	average (in)	±tole avg OD (in)	erance max & min (in)	1316	2112	1210	1208
12	64 41	63 100	50 80	63	50	0.199	+0.024	12.750	0.015	0.075				
	32.5	125	100	80	63	0.392	+0.037 +0.047			0.075				
	26	160	125	100	80	0.552	+0.041 +0.059			0.075	125	100	80	
	21	200	160	125	100	0.606	+0.073			0.075	160	125	100	80
	17	250	200	160	125	0.750	+0.090			0.060	100		100	00
14	41	100	80	63	50	0.341	+0.048	14.000	0.015	0.100				
	32.5	125	100	80	63	0.430	+0.052			0.100				
	26	160	125	100	80	0.538	+0.064			0.100				
	21	200	160	125	100	0.666	+0.080			0.100				
	17	250	200	160	125	0.823	+0.099			0.075				
16	41	100	80	63	50	0.390	+0.055	16.000	0.019	0.160				
	32.5	125	100	80	63	0.492	+0.059			0.160				
	26	160	125	100	80	0.615	+0.074			0.160				
	21	200	160	125	100	0.762	+0.091			0.160				
	17	250	200	160	125	0.941	+0.113			0.080				
18	41	100	80	63	50	0.439	+0.061	18.000	0.019	0.180				
	32.5	125	100	80	63	0.554	+0.066			0.180				
	26	160	125	100	80	0.692	+0.083			0.180				
	21	200	160	125	100	0.857	+0.103			0.180				
	17	250	200	160	125	1.059	+0.127			0.090				
20	41	100	80	63	50	0.488	+0.068	20.000	0.023	0.200				
	32.5	125	100	80	63	0.615	+0.074			0.200				
	26	160	125	100	80	0.769	+0.092			0.200				
	21	200	160	125	100	0.952	+0.114			0.200				
	17	250	200	160	125	1.176	+0.141			0.100				
24	41	100	80	63	50	0.585	+0.082	24.000	0.031	0.240				
	32.5	125	100	80	63	0.738	+0.088			0.240				
	26	160	125	100	80	0.923	+0.111			0.240				
	21	200	160	125	100	1.143	+0.137			0.240				
	17	250	200	100	125	1.412	+0.169			0.120				
30	41	100	80	63	50	0.732	+0.123	30.000	0.041	0.300				
	32.5	125	100	80	63	1.108	+0.133			0.300				
	26	160	125	100	80	1.385	+0.166			0.300				
	21	200	160	125	100	1.714	+0.205			0.300				
	17	250	200	160	125	2.118	+0.254			0.150				

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 Table 52C-3
 PVC and ABS thermoplastic pipe (SDR-PR)-(IPS) (nonthreaded)—Continued

Nominal pipe size	SDR	PVC	pressure Mate	rating (lb. erial	/in ²)	Wall thickness Outside diameter					ABS pressure rating (lb/in ²) Material 1316 2112 1210 1208			
(in)		1120 1220 2120	2116	2112	2110	min. (in)	toler- ance (in)	average (in)	±tole avg OD (in)	erance max & min (in)	1316	2112	1210	1208
36	41	100	80	63	50	0.878	+0.123	36.000	0.050	0.360				
	32.5	125	100	80	63	1.108	+0.133			0.360				
	26	160	125	100	80	1.385	+0.166			0.360				
	21	200	160	125	100	1.714	+0.205			0.360				
	17	250	200	160	125	2.118	+0.254			0.180				

Source: ASTM D 2241 and D 2282.

 Table 52C-4
 Polyethylene plastic pipe (SIDR-PR)–I.D. controlled (nonthreaded)

Nominal pipe size (in)	SIDR	PE p 3408	oressure ra Mate 3306	ating (lb/i erial 2305	n ²) 1404	Wall th	Dimensio ickness	on and toler Ins average	rance side diamet ±tole	er erance	
()		0100	3406 2306 2406	2000	1101	min. (in)	toler- ance (in)	(in)	(in)	_ (in)	
1/2	19	80				0.060	+0.020	0.622	0.010	0.010	
	15	100	80			0.060	+0.020				
	11.5	125	100	80		0.060	+0.020				
	9	160	125	100	80	0.069	+0.020				
	7	200	160	125	100	0.089	+0.020				
	5.3	250	200	160	125	0.117	+0.020				
3/4	19	80				0.060	+0.020	0.824	0.010	0.015	
	15	100	80			0.060	+0.020				
	11.5	125	100	80		0.072	+0.020				
	9	160	125	100	80	0.092	+0.020				
	7	200	160	125	100	0.118	+0.020				
	5.3	250	200	160	125	0.155	+0.020				
1	19	80				0.060	+0.020	1.049	0.010	0.020	
	15	100	80			0.070	+0.020				
	11.5	125	100	80		0.091	+0.020				
	9	160	125	100	80	0.117	+0.020				
	7	200	160	125	100	0.150	+0.020				
	5.3	250	200	160	125	0.198	+0.024				
1 1/4	19	80				0.073	+0.020	1.380	0.010	0.020	
	15	100	80			0.092	+0.020				
	11.5	125	100	80		0.120	+0.020				
	9	160	125	100	80	0.153	+0.020				
	7	200	160	125	100	0.197	+0.024				
	5.3	250	200	160	125	0.260	+0.031				
1 1/2	19	80				0.085	+0.020	0.230	0.015	0.020	
	15	100	80			0.107	+0.020				
	11.5	125	100	80		0.140	+0.020				
	9	160	125	100	80	0.179	+0.020				
	7	200	160	125	100	0.230	+0.028				
	5.3	250	200	160	125	0.304	+0.036				
2	19	80				0.109	+0.020	2.067	0.015	0.020	
	15	100	80			0.138	+0.020				
	11.5	125	100	80		0.180	+0.022				
	9	160	125	100	80	0.230	+0.028				
	7	200	160	125	100	0.295	+0.035				
	5.3	250	200	160	125	0.390	+0.047				
$2\frac{1}{2}$	19	80				0.130	+0.020	2.469	0.015	0.025	
	15	100	80			0.165	+0.020				
	11.5	125	100	80		0.215	+0.025				

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 Table 52C-4
 Polyethylene plastic pipe (SIDR-PR)–I.D. controlled (nonthreaded)—Continued

Nominal pipe size (in)	SIDR	PE pressure rating (lb/in ²)			Wall th	ion and tolerance					
(in)		3408	3306 3406 2306 2406	2305	1404	min. (in)	toler- ance (in)	average (in)	±tole + (in)	- (in)	
3	19 15	80 100	80			$0.161 \\ 0.205$	+0.020 +0.020	3.068	0.015	0.030	
	11.5	125	100	80		0.267	+0.032				
4	19 15 11.5	80 100 125	80 100	80		$0.212 \\ 0.268 \\ 0.350$	+0.025 +0.032 +0.042	4.026	0.015	0.035	
6	19 15 11.5	80 100 125	80 100	80		$\begin{array}{c} 0.319 \\ 0.404 \\ 0.527 \end{array}$	+0.038 +0.048 +0.063	6.065	0.020	0.035	

Source: ASTM D 2239

 Table 52C-5
 Polyethylene plastic pipe (SDR-PR)–O.D. controlled (IPS) (nonthreaded)

Nominal pipe size	SDR	PE p	oressure ra Mate	ating (lb/ii erial	n ²)	Wall th	Dimensio ickness	on and toler Out	rance side diame	ter	
(in)		3408	3306	2305	1404			average	±tole	erance	
			$3406 \\ 2306 \\ 2406$			min. (in)	toler- ance (in)	(in)	(in)	_ (in)	
1/2	32.5	51	40	32	25	0.062	0.020	0.840	0.004	0.004	
	26	64	50	40	32	0.062	0.020				
	21	80	63	50	40	0.062	0.020				
	17	100	79	63	50	0.062	0.020				
	15.5	110	87	69	55	0.062	0.020				
	13.5	128	100	80	64	0.062	0.020				
	11	160	126	100	80	0.076	0.020				
	9.3	193	152	120	96	0.090	0.020				
	9	200	158	125	100	0.093	0.020				
	7	267	210	167	133	0.120	0.020				
3/4	32.5	51	40	32	25	0.062	0.020	1.050	0.004	0.004	
	26	64	50	40	32	0.062	0.020				
	21	80	63	50	40	0.062	0.020				
	17	100	79	63	50	0.062	0.020				
	15.5	110	87	69	55	0.068	0.020				
	13.5	128	100	80	64	0.078	0.020				
	11	160	126	100	80	0.095	0.020				
	9.3	193	152	120	96	0.113	0.020				
	9	200	158	125	100	0.117	0.020				
	7	267	210	167	133	0.150	0.020				
1	32.5	51	40	32	25	0.062	0.020	1.315	0.005	0.005	
	26	64	50	40	32	0.062	0.020				
	21	80	63	50	40	0.063	0.020				
	17	100	79	63	50	0.077	0.020				
	15.5	110	87	69	55	0.084	0.020				
	13.5	128	100	80	64	0.097	0.020				
	11	160	126	100	80	0.120	0.020				
	9.3	193	152	120	96	0.141	0.020				
	9	200	158	125	100	0.146	0.020				
	7	267	210	167	133	0.188	0.023				
1 1/4	32.5	51	40	32	25	0.062	0.020	1.660	0.005	0.005	
	26	64	50	40	32	0.064	0.020				
	21	80	63	50	40	0.079	0.020				
	17	100	79	63	50	0.098	0.020				
	15.5	110	87	69	55	0.107	0.020				
	13.5	128	100	80	64	0.123	0.020				
	11	160	126	100	80	0.151	0.020				
	9.3	193	152	120	96	0.178	0.021				
	9	200	158	125	100	0.184	0.022				
	7	267	210	167	133	0.237	0.028				

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 Table 52C-5
 Polyethylene plastic pipe (SDR-PR)-O.D. controlled (IPS) (nonthreaded)—Continued

Nominal pipe size	SDR	PE p	oressure ra	ating (lb/in erial	n ²)	Wall th	Dimensio	on and toler Outs	ance side diame	ter	
(in)		3408	3306 3406 2306	2305	1404	min. (in)	toler- ance (in)	average (in)	±tole + (in)	rance – (in)	
			2406								
1 1/2	32.5	51	40	32	25	0.062	0.020	1.900	0.006	0.006	
1 1/ 2	26	64	50	40	32	0.073	0.020	1.000	0.000	0.000	
	21	80	63	50	40	0.090	0.020				
	17	100	79	63	50	0.112	0.020				
	15.5	110	87	69	55	0.123	0.020				
	13.5	128	100	80	64	0.120	0.020				
	11	160	126	100	80	0.173	0.021				
	93	193	152	120	96	0.204	0.021				
	9	200	158	125	100	0.201	0.025				
	7	267	210	167	133	0.211 0.271	0.020				
	•	201	210	101	100	0.211	0.000				
2	32.5	51	40	32	25	0.073	0.020	2.375	0.006	0.006	
	26	64	50	40	32	0.091	0.020				
	21	80	63	50	40	0.113	0.020				
	17	100	79	63	50	0.140	0.020				
	15.5	110	87	69	55	0.153	0.020				
	13.5	128	100	80	64	0.176	0.021				
	11	160	126	100	80	0.216	0.026				
	9.3	193	152	120	96	0.255	0.031				
	9	200	158	125	100	0.264	0.032				
	7	267	210	167	133	0.339	0.041				
3	32.5	51	40	32	25	0.108	0.020	3.500	0.008	0.008	
	26	64	50	40	32	0.135	0.020				
	21	80	63	50	40	0.167	0.020				
	17	100	79	63	50	0.206	0.025				
	15.5	110	87	69	55	0.226	0.027				
	13.5	128	100	80	64	0.259	0.031				
	11	160	126	100	80	0.318	0.038				
	9.3	193	152	120	96	0.376	0.045				
	9	200	158	125	100	0.389	0.047				
	7	267	210	167	133	0.500	0.060				
4	32.5	51	40	32	25	0 138	0.020	4 500	0.009	0.009	
•	26	64	50	40	32	0.100	0.020	1.000	0.000	0.000	
	21	80	63	50	40	0.214	0.021				
	17	100	79	63	50	0.265	0.032				
	15.5	110	87	69	55	0.290	0.035				
	13.5	128	100	80	64	0.333	0.035				
	11	160	126	100	80	0.409	0.049				
	93	193	152	120	96	0 484	0.058				
	9	200	158	125	100	0.500	0.060				
	7	267	210	167	133	0.643	0.000				
	•	201	1 10	T O I	100	0.010	0.011				

 Table 52C-5
 Polyethylene plastic pipe (SDR-PR)-O.D. controlled (IPS) (nonthreaded)—Continued

Nominal pipe size	SDR	PE p	oressure ra Mate	ating (lb/ii erial	n ²)	Wall th	Dimensio	on and toler Outs	ance side diame	ter	
(in)		3408	3306 3406 2306 2406	2305	1404	min. (in)	toler- ance (in)	average (in)	±tole + (in)	in)	
5	32.5	51	40	32	25	0 171	0.021				
0	26	64	50	40	32	0.214	0.021				
	21	80	63	50	40	0.211 0.265	0.020				
	17	100	79	63	50	0.200 0.327	0.032				
	15.5	110	87	69	55	0.359	0.055				
	13.5	128	100	80	64	0.000	0.049				
	10.0	160	100	100	80	0.412	0.043				
	0.2	100	152	120	06	0.500	0.001				
	9.5 0	200	152	120	100	0.556	0.072				
	9 7	$200 \\ 267$	210	167	100	0.018	0.074				
	1	207	210	107	155	0.795	0.095				
6	32.5	51	40	32	25	0.204	0.024	6.625	0.011	0.011	
	26	64	50	40	32	0.255	0.031				
	21	80	63	50	40	0.315	0.038				
	17	100	79	63	50	0.390	0.047				
	15.5	110	87	69	55	0.427	0.051				
	13.5	128	100	80	64	0.491	0.059				
	11	160	126	100	80	0.602	0.072				
	9.3	193	152	120	96	0.712	0.085				
	9	200	158	125	100	0.736	0.088				
	7	267	210	167	133	0.946	0.114				
0	22.5	51	40	<u> </u>	25	0.265	0 022	9 695	0.012	0.019	
0	04.0 96	64	40 50		20 20	0.200	0.052	0.020	0.015	0.015	
	20 91	04 00	00 69	40 50	32 40	0.332	0.040				
	41 17	100	70	00 69	40 50	0.411	0.049				
	16	100	19	05 60	50 55	0.507	0.001				
	10.0	110	ð/ 100	09	99 64	0.000	0.007				
	13.5	128	100	80	04	0.639	0.077				
	11	100	120	100	80	0.784	0.094				
	9.3	193	152	120	96	0.927	0.111				
	9	200	158	125	100	0.958	0.115				
	7	267	210	167	133	1.232	0.147				
10	32.5	51	40	32	25	0.331	0.040	10.750	0.015	0.015	
	26	64	50	40	32	0.413	0.050				
	21	80	63	50	40	0.512	0.061				
	17	100	79	63	50	0.632	0.076				
	15.5	110	87	69	55	0.694	0.083				
	13.5	128	100	80	64	0.796	0.096				
	11	160	126	100	80	0.977	0.117				
	9.3	193	152	120	96	1.156	0.139				
	9	200	158	125	100	1.194	0.143				
	7	267	210	167	133	1.536	0 184				

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 Table 52C-5
 Polyethylene plastic pipe (SDR-PR)-O.D. controlled (IPS) (nonthreaded)—Continued

Nominal	SDR	PE p	oressure ra	ating (lb/i	n ²)		Dimensi	on and toler	ance	tor	
(in)		3408	Mate 3306	2305	1404	wan th	ICKIIESS	average	tol€±tol€	erance	
			$3406 \\ 2306 \\ 2406$		-	min. (in)	toler- ance (in)	(in)	+ (in)	_ (in)	
12	32.5	51	40	32	25	0.392	0.047	12.750	0.017	0.017	
	26	64	50	40	32	0.490	0.059				
	21	80	63	50	40	0.607	0.073				
	17	100	79	63	50	0.750	0.090				
	15.5	110	87	69	55	0.823	0.099				
	13.5	128	100	80	64	0.944	0.113				
	11	160	126	100	80	1.159	0.139				
	9.3	193	152	120	96	1.371	0.165				
	9	200	158	125	100	1.417	0.170				
	7	267	210	167	133	1.821	0.219				
14	32.5	51	40	32	25	0.431	0.052	14.000	0.063	0.063	
	26	64	50	40	32	0.538	0.065				
	21	80	63	50	40	0.667	0.080				
	17	100	79	63	50	0.824	0.099				
	15.5	110	87	69	55	0.903	0.108				
	13.5	128	100	80	64	1.037	0.124				
	11	160	126	100	80	1.273	0.153				
	9.3	193	152	120	96	1.505	0.181				
	9	200	158	125	100	1.556	0.187				
	7	267	210	167	133	2.000	0.240				
16	32.5	51	40	32	25	0.492	0.059	16.000	0.072	0.072	
	26	64	50	40	32	0.615	0.074				
	21	80	63	50	40	0.762	0.091				
	17	100	79	63	50	0.941	0.113				
	15.5	110	87	69	55	1.032	0.124				
	13.5	128	100	80	64	1.185	0.142				
	11	160	126	100	80	1.455	0.175				
	9.3	193	152	120	96	1.720	0.206				
	9	200	158	125	100	1.778	0.213				
	7	267	210	167	133	2.286	0.274				
18	32.5	51	40	32	25	0.554	0.066	18.000	0.081	0.081	
	26	64	50	40	32	0.692	0.083				
	21	80	63	50	40	0.857	0.103				
	17	100	79	63	50	1.059	0.127				
	15.5	110	87	69	55	1.161	0.139				
	13.5	128	100	80	64	1.333	0.160				
	11	160	126	100	80	1.636	0.196				
	9.3	193	152	120	96	1.935	0.232				
	9	200	158	125	100	2.000	0.240				
	7	267	210	167	133	2.571	0.309				

 Table 52C-5
 Polyethylene plastic pipe (SDR-PR)-O.D. controlled (IPS) (nonthreaded)—Continued

Nominal pipe size	SDR	PE p	pressure ra	ating (lb/i erial	n ²)	Wall th	Dimensi iickness	on and toler Outs	ance	ter	
(in)		3408	3306 3406 2306 2406	2305	1404	min. (in)	toler- ance (in)	average (in)	±tole + (in)	(in)	
20	32.5	51	40	32	25	0.615	0.074	20.000	0.090	0.090	
	26	64	50	40	32	0.769	0.092				
	21	80	63	50	40	0.952	0.114				
	17	100	79	63	50	1.176	0.141				
	15.5	110	87	69	55	1.290	0.155				
	13.5	128	100	80	64	1.481	0.178				
	11	160	126	100	80	1.818	0.218				
	9.3	193	152	120	96	2.151	0.258				
	9	200	158	125	100	2.222	0.267				
	7	267	210	167	133	2.857	0.343				
22	32.5	51	40	32	25	0.677	0.089	22.000	0.099	0.099	
	26	64	50	40	32	0.846	0.102				
	21	80	63	50	40	1.048	0.126				
	17	100	79	63	50	1.294	0.155				
	15.5	110	87	69	55	1.419	0.170				
	13.5	128	100	80	64	1.630	0.196				
	11	160	126	100	80	2.000	0.240				
	9.3	193	152	120	96	2.366	0.284				
	9	200	158	125	100	2.444	0.293				
	7	267	210	167	133	3.143	0.377				
24	32.5	51	40	32	25	0.738	0.089	24.000	0.108	0.108	
	26	64	50	40	32	0.923	0.111				
	21	80	63	50	40	1.143	0.137				
	17	100	79	63	50	1.412	0.169				
	15.5	110	87	69	55	1.548	0.186				
	13.5	128	100	80	64	1.778	0.213				
	11	160	126	100	80	2.182	0.262				
	9.3	193	152	120	96	2.581	0.310				
	9	200	158	125	100	2.667	0.320				
	7	267	210	167	133	3.429	0.411				

Source: ASTM D 3035

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Table 52C-6aPVC schedule 40, 80, and 120 and ABS schedule 40, and 80 plastic pipe (unthreaded)

Nominal pipe size	Sch.	PVC	pressure	rating (lb/ erial	/in ²)	Wall thi	- Dimensi ickness	on and to Ou	lerance tside dian	neter	AB	S pressur Mate	e rating (1) erial	o/in ²)
(in)		1120 1220 2120	2116	2112	2110	min. (in)) toler- ance (in)	average (in)	±tole avg OD (in)	erance max & min (in)	1316	2112	1210	1208
1/8	$\begin{array}{c} 40\\ 80 \end{array}$	810 1230	$650 \\ 980$	$500 \\ 770$	400 610	$0.068 \\ 0.095$	+0.020 +0.020	0.405	0.004	0.008	650 980	500	400	320
1/4	40 80	780 1130	620 900	$\begin{array}{c} 490 \\ 710 \end{array}$	390 570	$0.088 \\ 0.119$	+0.020 +0.020	0.540	0.004	0.008	620 900	490	390	310
3/8	$\begin{array}{c} 40\\ 80 \end{array}$	620 920	500 730	$390 \\ 570$	310 460	$0.091 \\ 0.126$	+0.020 +0.020	0.675	0.004	0.008	500 730	390	310	250
1/2	40 80 120	$600 \\ 850 \\ 1010$	480 680 810	370 530 630	$300 \\ 420 \\ 510$	$0.109 \\ 0.147 \\ 0.170$	+0.020 +0.020 +0.020	0.840	0.004	0.008	480 680	$\begin{array}{c} 370\\ 530 \end{array}$	$\begin{array}{c} 300\\ 420 \end{array}$	240 340
3/4	40 80 120	$480 \\ 690 \\ 770$	390 550 620	$300 \\ 430 \\ 480$	240 340 390	$\begin{array}{c} 0.113 \\ 0.154 \\ 0.170 \end{array}$	+0.020 +0.020 +0.020	1.050	0.004	0.010	390 550	$\begin{array}{c} 300\\ 430 \end{array}$	$\begin{array}{c} 240\\ 340\end{array}$	190 280
1	40 80 120	450 630 720	360 500 570	$280 \\ 390 \\ 450$	220 320 360	$\begin{array}{c} 0.133 \\ 0.179 \\ 0.200 \end{array}$	+0.020 +0.021 +0.024	1.315	0.005	0.010	360 500	280 390	220 320	180 250
1 1/4	40 80 120	370 520 600	290 420 480	230 320 370	180 260 300	$0.140 \\ 0.191 \\ 0.215$	+0.020 +0.023 +0.026	1.660	0.005	0.012	290 420	230 330	180 260	150 210
1 1/2	40 80 120	$330 \\ 470 \\ 540$	$260 \\ 380 \\ 430$	210 290 340	$170 \\ 240 \\ 270$	$0.145 \\ 0.200 \\ 0.225$	+0.020 +0.024 +0.027	1.900	0.006	0.012	260 380	210 290	$\begin{array}{c} 170 \\ 240 \end{array}$	130 190
2	40 80 120	$280 \\ 400 \\ 470$	220 320 380	$170 \\ 250 \\ 290$	140 200 240	$\begin{array}{c} 0.154 \\ 0.218 \\ 0.250 \end{array}$	+0.020 +0.026 +0.030	2.375	0.006	0.012	220 320	$\begin{array}{c} 170 \\ 250 \end{array}$	140 200	110 160
2 1/2	40 80 120	$300 \\ 420 \\ 470$	$240 \\ 340 \\ 370$	190 260 290	150 210 230	$\begin{array}{c} 0.203 \\ 0.276 \\ 0.300 \end{array}$	+0.024 +0.033 +0.036	2.875	0.007	0.015	$\begin{array}{c} 240\\ 340\end{array}$	190 270	$\begin{array}{c} 150 \\ 210 \end{array}$	120 170
3	40 80 120	$260 \\ 370 \\ 440$	$210 \\ 300 \\ 360$	160 230 280	130 190 220	$\begin{array}{c} 0.216 \\ 0.300 \\ 0.350 \end{array}$	+0.026 +0.036 +0.042	3.500	0.008	0.015	210 300	160 230	130 190	100 150
3 1/2	40 80 120	240 350 380	190 280 310	$150 \\ 220 \\ 240$	120 170 190	$\begin{array}{c} 0.226 \\ 0.318 \\ 0.350 \end{array}$	+0.027 +0.038 +0.042	4.000	0.008	$\begin{array}{c} 0.050 \\ 0.015 \\ 0.015 \end{array}$	190 280	$\begin{array}{c} 150 \\ 220 \end{array}$	120 170	90 140
4	40 80 120	220 320 430	$180 \\ 260 \\ 340$	$140 \\ 200 \\ 270$	110 160 220	$0.237 \\ 0.337 \\ 0.437$	+0.028 +0.040 +0.052	4.500	0.009	$0.050 \\ 0.015 \\ 0.015$	180 260	140 200	110 160	90 130

Table 52C-6a PVC schedule 40, 80, and 120 and ABS schedule 40, and 80 plastic pipe (unthreaded)—Continued

Nominal	Sch.	PVC	pressure	rating (lb	/in ²)	Wall thi	- Dimensi	on and tol	erance		AB	S pressure	e rating (ll	o/in ²)
(in)		1120 1220 2120	2116	2112	2110	min. (in)	toler- ance (in)	average (in)	±tole avg OD (in)	erance max & min (in)	1316	2112	1210	1208
5	40 80 120	190 290 400	160 230 320	120 180 250	100 140 200	$0.258 \\ 0.375 \\ 0.500$	+0.031 +0.045 +0.060	5.563	0.010	$0.050 \\ 0.030 \\ 0.030$	160 230	120 180	100 140	80 120
6	$40 \\ 80 \\ 120$	180 280 370	$140 \\ 220 \\ 300$	110 170 230	90 140 190	$\begin{array}{c} 0.280 \\ 0.432 \\ 0.562 \end{array}$	+0.034 +0.052 +0.067	6.625	0.011	$0.050 \\ 0.035 \\ 0.035$	140 220	$\begin{array}{c} 110\\170 \end{array}$	90 140	70 110
8	40 80 120	160 250 380	120 200 290	$100 \\ 150 \\ 230$	80 120 180	$\begin{array}{c} 0.322 \\ 0.500 \\ 0.718 \end{array}$	+0.039 +0.060 +0.086	8.625	0.015	$\begin{array}{c} 0.075 \\ 0.075 \\ 0.045 \end{array}$	120 200	$\begin{array}{c} 100 \\ 150 \end{array}$	80 120	60 100
10	40 80 120	$140 \\ 230 \\ 370$	110 190 290	90 150 230	70 120 180	$\begin{array}{c} 0.365 \\ 0.593 \\ 0.843 \end{array}$	+0.044 +0.071 +0.101	10.750	0.015	$\begin{array}{c} 0.075 \\ 0.075 \\ 0.050 \end{array}$	110 190	90 150	70 120	60 90
12	40 80 120	130 230 340	110 180 270	$80 \\ 140 \\ 210$	$70 \\ 110 \\ 170$	$\begin{array}{c} 0.406 \\ 0.687 \\ 1.000 \end{array}$	+0.049 +0.082 +0.120	12.750	0.015	$\begin{array}{c} 0.075 \\ 0.075 \\ 0.060 \end{array}$	110 180	80 140	70 110	50 90
14	40 80	130 220	100 180	80 140	$\begin{array}{c} 60\\110\end{array}$	$0.437 \\ 0.750$	+0.053 +0.090	14.000	0.015	0.100				
16	40 80	130 220	100 180	80 140	$\begin{array}{c} 60\\110\end{array}$	$0.500 \\ 0.843$	+0.060 +0.101	16.000	0.019	0.160				
18	40 80	130 220	100 180	80 140	$\begin{array}{c} 60\\110\end{array}$	$0.562 \\ 0.937$	+0.067 +0.112	18.000	0.019	0.180				
20	40 80	120 220	$\begin{array}{c} 100 \\ 170 \end{array}$	80 140	$\begin{array}{c} 60\\110\end{array}$	$0.593 \\ 1.031$	+0.071 +0.124	20.000	0.023	0.200				
24	$\begin{array}{c} 40\\ 80 \end{array}$	120 210	90 170	70 130	$\begin{array}{c} 60\\110\end{array}$	$0.687 \\ 1.218$	+0.082 +0.146	24.000	0.031	0.240				

Source: ASTM D 1785 for PVC and D 1527 for ABS.

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Fable 52C-6b	PE schedule 40	and 80 plastic	pipe (unthreade	ed)
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Nominal pipe size	Sch.	PE p	ressure (lb/in ²	rating)		D2104	D	imension	and toleran	ce	D2447		PE pr	essure ra (lb/in²)	ting
(in)		2306 2406 3306 3406	Materia 2305	al 1404	Insid average (in)	e diamet ±tole + (in)	er erance – (in)	Wall min. (in)	thickness tolerance (in)	Out: average (in)	side diam ±toler + (in)	neter rance – (in)	2306 2406 3306 3406	N 2305	Material 1404
1/2	40 80	190	150	120	0.622	0.010	0.010	$\begin{array}{c} 0.109 \\ 0.147 \end{array}$	+0.020 +0.020	0.840	0.004	0.004	188 267	149 212	119 170
3/4	40 80	150	120	100	0.824	0.010	0.015	$\begin{array}{c} 0.113 \\ 0.154 \end{array}$	+0.020 +0.020	1.050	0.004	0.004	$\begin{array}{c} 152 \\ 217 \end{array}$	120 172	96 137
1	40 80	140	110	90	1.049	0.010	0.020	$\begin{array}{c} 0.133 \\ 0.179 \end{array}$	+0.020 +0.021	1.315	0.005	0.005	$142 \\ 199$	113 158	90 126
1 1/4	40 80	120	90	70	1.380	0.010	0.020	$\begin{array}{c} 0.140\\ 0.191 \end{array}$	+0.020 +0.023	1.660	0.005	0.005	$\begin{array}{c} 116 \\ 164 \end{array}$	92 130	74 104
1 1/2	40 80	100	80	70	1.610	0.015	0.020	$\begin{array}{c} 0.145\\ 0.200 \end{array}$	+0.020 +0.024	1.900	0.006	0.006	$\begin{array}{c} 104 \\ 148 \end{array}$	83 118	66 94
2	40 80	90	70	60	2.067	0.015	0.020	$\begin{array}{c} 0.154 \\ 0.218 \end{array}$	+0.020 +0.026	2.375	0.006	0.006	87 127	$\begin{array}{c} 69\\ 101 \end{array}$	55 81
2 1/2	40 80	100	80	60	2.469	0.015	0.025	$0.203 \\ 0.276$	+0.024 +0.033	2.875	0.007	0.007	96 134	76 106	61 85
3	40 80	80	70	50	3.068	0.015	0.030	$\begin{array}{c} 0.216\\ 0.300 \end{array}$	+0.026 +0.036	3.500	0.008	0.008	83 118	$\begin{array}{c} 66\\ 94 \end{array}$	53 75
3 1/2	40 80							$\begin{array}{c} 0.226\\ 0.318\end{array}$	+0.027 +0.038	4.000	0.008	0.008	75 109	60 86	50 69
4	40 80	70	60	NPR	4.026	0.015	0.035	$0.237 \\ 0.337$	+0.028 +0.040	4.500	0.009	0.009	70 102	55 81	NPR 65
5	40 80							$0.258 \\ 0.375$	$^{+0.031}_{+0.045}$	5.563	0.010	0.010	$\begin{array}{c} 61 \\ 91 \end{array}$	50 72	NPR 58
6	40 80	60	NPR	NPR	6.065	0.020	0.035	$\begin{array}{c} 0.280\\ 0.432\end{array}$	+0.034 +0.052	6.625	0.011	0.011	55 88	NPR 70	NPR 56
8	40							0.322	+0.039	8.625	0.015	0.015	50	NPR	NPR
10	40							0.365	+0.044	10.750	0.015	0.015	NPR	NPR	NPR
12	40							0.406	+0.049	12.750	0.015	0.015	NPR	NPR	NPR

Source: ASTM D 2104 for inside diameter controlled and D 2447 for outside diameter controlled. NPR: Not Pressure Rated

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 Table 52C-7
 Polyethylene plastic tubing

Nominal pipe size (in)	SDR	Pressure 3408	e rating (lb, Material 3306 3406 2306 2406	⁄in²) 2305	Wall thick min. (in)	Dimen mess toler- ance (in)	sion and toler average (in)	rance Outs ±tolera avg. OD (in)	side diameter ance max. & min. (in)	
1/2	7.3 9 11	200 160	160	160	0.086 0.069 0.062	+0.010 +0.010 +0.010	0.625	0.004	0.015	
5/8	7.3 9 11	200 160	160	160	$\begin{array}{c} 0.103 \\ 0.083 \\ 0.068 \end{array}$	+0.010 +0.010 +0.010	0.750	0.004	0.015	
3/4	7.3 9 11	200 160	160	160	$0.120 \\ 0.097 \\ 0.080$	+0.012 +0.010 +0.010	0.875	0.004	0.015	
1	7.3 9 11	200 160	160	160	$\begin{array}{c} 0.154 \\ 0.125 \\ 0.102 \end{array}$	+0.015 +0.012 +0.010	1.125	0.005	0.015	
1 1/4	7.3 9 11	200 160	160	160	$\begin{array}{c} 0.188 \\ 0.153 \\ 0.125 \end{array}$	+0.019 +0.015 +0.012	1.375	0.005	0.015	
1 1/2	7.3 9 11	200 160	160	160	$\begin{array}{c} 0.233 \\ 0.181 \\ 0.148 \end{array}$	+0.022 +0.018 +0.015	1.625	0.006	0.015	
2	7.3 9 11	200 160	160	160	$\begin{array}{c} 0.291 \\ 0.236 \\ 0.193 \end{array}$	+0.029 +0.024 +0.019	2.125	0.006	0.015	

Source: ASTM D 2737

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Table 52C–8

8 PVC plastic pipe dimensions, pressure classes, SDR, and tolerancesfor iron pipe sizes

Nominal pipe size	Pressure class	SDR	Outside d average	liameter (in) tolerance	Minimum wal minimum	l thickness (in) tolerance
(1n)	(lb/ln^2)					
4	100	25	4.80	0.009	0.192	0.023
	150	18			0.267	0.032
	200	14			0.343	0.041
6	100	25	6.90	0.011	0.276	0.033
	150	18			0.383	0.046
	200	14			0.493	0.059
8	100	25	9.05	0.015	0.362	0.043
	150	18			0.503	0.060
	200	14			0.646	0.078
10	100	25	11.10	0.015	0.444	0.053
	150	18			0.617	0.074
	200	14			0.793	0.095
12	100	25	13.20	0.015	0.528	0.063
	150	18			0.733	0.088
	200	14			0.943	0.113

Source: AWWA C900

Hydrostatic Design Stress (HDS) = $1,600 \text{ lb/in}^2$

Table 520-5 I Oryentylette pipe, instue matteret bas	Table 52U-9	Polyethyle	ne pipe, inside	e diameter	based
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Nominal pipe size (in)	SIDR	Pressur Mate 2406 3406	re class rial 3408	Insic minimum (in)	Dime le diamete tole: – (in)	ension and t er rance + (in)	tolerance Wall thi minimum	ckness tolerance
0.5	9	125	160	0.622	0.010	0.010	0.069	+0.020
	7	160	200				0.089	+0.020
	5.3	200					0.117	+0.020
0.75	11.5		125	0.824	0.015	0.010	0.072	+0.020
	9	125	160				0.092	+0.020
	7	160	200				0.118	+0.020
	5.3	200					0.155	+0.020
1	11.5		125	1.049	0.020	0.010	0.091	+0.020
	9	125	160				0.117	+0.020
	7	160	200				0.150	+0.020
	5.3	200					0.198	+0.024

Table 52C-9 Polyethylene pipe, inside diameter based—Continued

Nominal	SIDR	Pressu	re class	Insi	Dim de diameté	ension and	tolerance	ckness	
(in)		2406 3406	3408	minimum (in)	tole - (in)	rance + (in)	minimum	tolerance	
1.25	11.5		125	1.380	0.020	0.010	0.120	+0.020	
	9	125	160				0.153	+0.020	
	7	160	200				0.197	+0.024	
	5.3	200					0.260	+0.031	
1.5	11.5		125				0.140	+0.020	
	9	125	160				0.179	+0.020	
	7	160	200				0.230	+0.028	
	5.3	200					0.304	+0.036	
2	19		80	2.067	0.020	0.015	0.109	+0.020	
	15	80	100				0.138	+0.020	
	11.5	100	125				0.180	+0.022	
	9	125	160				0.230	+0.028	
	7	160	200				0.295	+0.035	
	5.3	200					0.390	+0.047	
2.5	19		80	2.469	0.025	0.015	0.130	+0.020	
	15	80	100				0.165	+0.020	
	11.5	100	125				0.215	+0.025	
	9	125	160				0.272	+0.033	
	7	160	200				0.353	+0.042	
	5.3	200					0.466	+0.056	
3	19		80	3.068	0.030	0.015	0.161	+0.020	
	15	80	100				0.205	+0.020	
	11.5	100	125				0.267	+0.032	
	9	125	160				0.341	+0.041	
	7	160	200				0.438	+0.053	
	5.3	200					0.579	+0.069	

Source: AWWA C 901

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Nominal pipe size (in)	SDR	Pressur Mate 2406	re class erial 3408	Out minimum	Dime side diam toler	nsion and te eter	olerance Wall th minimum	ickness tolerance	
		5400		(III)	- (III)	+ (III)			
0.5	11	125	160	0.840	0.004	0.004	0.076	+0.020	
	9	160	200				0.093	+0.020	
0.75	13.5		125	1.050	0.004	0.004	0.078	+0.020	
	11	125	160				0.095	+0.020	
	9	160	200				0.117	+0.020	
1	13.5		125	1.315	0.005	0.005	0.097	+0.020	
	11	125	160				0.119	+0.020	
	9	160	200				0.146	+0.020	
1.25	13.5		125	1.660	0.005	0.005	0.123	+0.020	
	11	125	160				0.151	+0.020	
	9	160	200				0.184	+0.022	
1.5	13.5		125	1.900	0.006	0.006	0.141	+0.020	
	11	125	160				0.173	+0.021	
	9	160	200				0.211	+0.025	
2	21		80	2.375	0.006	0.006	0.113	+0.020	
	17	80	100				0.140	+0.020	
	13.5	100	125				0.176	+0.021	
	11	125	160				0.216	+0.026	
	9	160	200				0.264	+0.032	
3	21		80	3.500	0.008	0.008	0.167	+0.020	
	17	80	100				0.206	+0.025	
	13.5	100	125				0.259	+0.031	
	11	125	160				0.318	+0.038	
	9	160	200				0.389	+0.047	

Table 52C-10 Polyethylene pipe, outside diameter based

Source: AWWA C 901

Nominal pipe size	SDR	Pressure rating	Outside d	- Dimension and liameter (in)	tolerance - Wall thick	xness (in)	
(m)		(10/1n²)	average	loierance (–/+)	mmum	tolerance	
14	41	100	14.000	0.015	0.341	+0.048	
	32.5	125	11000	01010	0.430	+0.052	
	26	160			0.538	+0.064	
	21	200			0.666	+0.080	
16	41	100	16.000	0.019	0.390	+0.055	
	32.5	125			0.492	+0.059	
	26	160			0.615	+0.074	
	21	200			0.762	+0.091	
18	41	100	18.000	0.019	0.439	+0.061	
	32.5	125			0.554	+0.066	
	26	160			0.692	+0.083	
	21	200			0.857	+0.103	
20	41	100	20.000	0.023	0.488	+0.068	
20	32.5	125	20.000	0.020	0.615	+0.074	
	26	160			0.769	+0.092	
	21	200			0.952	+0.114	
24	<i>4</i> 1	100	24 000	0.031	0 585	+0.082	
27	32.5	125	21.000	0.001	0.565	+0.082	
	26	160			0.190	+0.111	
	21	200			1.143	+0.137	
20	41	100	30,000	0.041	0 732	+0.102	
90	41 20 5	100	50.000	0.041	0.734	+0.102	
	54.9 96	120			0.920 1 154	+0.111	
	20 91	200			1.104	+0.130	
	41	200			1.420	+0.171	
36	41	100	36.000	0.050	0.878	+0.123	
	32.5	125			1.108	+0.133	
	26	160			1.385	+0.166	
	21	200			1.714	+0.205	

Table 52C-11 PVC plastic pipe, iron pipe size (IPS) outside diameter

Source: AWWA C 905

PVC material cell class 12454-B as defined by ASTM D 1784 with hydrostatic design basis of 4,000 pounds per square inch.

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Nominal	SDR	Pressure	Outsida	- Dimension and	tolerance		 	
(in)		(lb/in ²)	average	tolerance (-/+)	minimum	tolerance		
14	41	100	15.300	0.015	0.373	+0.052		
	32.5	125			0.471	+0.056		
	25	165			0.612	+0.073		
	21	200			0.729	+0.088		
	18	235			0.850	+0.102		
	14	305			1.093	+0.131		
16	41	100	17.400	0.020	0.424	+0.059		
	32.5	125			0.535	+0.064		
	25	165			0.696	+0.084		
	21	200			0.829	+0.100		
	18	235			0.967	+0.116		
	14	305			1.243	+0.149		
18	51	80	19.500	0.020	0.382	+0.053		
	41	100			0.476	+0.067		
	32.5	125			0.600	+0.072		
	25	165			0.780	+0.094		
	21	200			0.929	+0.111		
	18	235			1.083	+0.130		
	14	305			1.393	+0.167		
20	51	80	21.600	0.025	0.424	+0.059		
	41	100			0.527	+0.074		
	32.5	125			0.665	+0.080		
	25	165			0.864	+0.104		
	21	200			1.029	+0.123		
	18	235			1.200	+0.144		
24	51	80	25.800	0.030	0.506	+0.071		
	41	100			0.629	+0.088		
	32.5	125			0.794	+0.095		
	25	165			1.032	+0.124		
	21	200			1.229	+0.147		
	18	235			1.433	+0.172		
30	51	80	32.000	0.040	0.627	+0.088		
	41	100			0.780	+0.109		
	32.5	125			0.985	+0.118		
	25	165			1.280	+0.154		
	21	200			1.524	+0.183		
	18	235			1 778	+0.213		

Nominal pipe size	SDR	Pressure rating	Outside	Dimension and diameter (in)	tolerance Wall thick	ness (in)
(in)		(lb/in ²)	average	tolerance (-/+)	minimum	tolerance
36	51	80	38.300	0.050	0.751	+0.105
	41	100			0.934	+0.131
	32.5	125			1.178	+0.141
	25	165			1.532	+0.184
	21	200			1.824	+0.219
42	51	80	44.500	0.060	0.872	+0.122
	41	100			1.085	+0.152
	32.5	125			1.369	+0.164
	25	165			1.780	+0.214
48	51		50.800	0.075	0.996	+0.139
	41				1.239	+0.173
	32.5				1.563	+0.188
	25				2.032	+0.244

Source: AWWA C 905 PVC material Cell class 12454-B as defined by ASTM D 1784 with hydrostatic design basis of 4,000 pounds per square inch.

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Nominal pipe size	SDR	Pressure class		Dime	nsion and tole	erance Wall thickness	
(in)		2406 3406	3408	minimum (in)	tolerance (-/+)	wan unckness minimum (in)	
4	32.5	40	51	4.5	0.020	0.138	
	26	50	64			0.173	
	21	63	80			0.214	
	17	78	100			0.265	
	15.5	86	110			0.290	
	13.5	100	128			0.333	
	11	125	160			0.409	
	9.3	151	193			0.482	
	9	156	200			0.500	
	7.3	198	254			0.616	
5	32.5	40	51	5.563	0.025	0.171	
	26	50	64			0.214	
	21	63	80			0.265	
	17	78	100			0.327	
	15.5	86	110			0.359	
	13.5	100	128			0.412	
	11	125	160			0.506	
	9.3	151	193			0.598	
	9	156	200			0.618	
	7.3	198	254			0.762	
6	32.5	40	51	6.625	0.030	0.204	
	26	50	64			0.255	
	21	63	80			0.316	
	17	78	100			0.390	
	15.5	86	110			0.427	
	13.5	100	128			0.491	
	11	125	160			0.602	
	9.3	151	193			0.710	
	9	156	200			0.736	
	7.3	198	254			0.908	
7	32.5	40	51	7.125	0.034	0.220	
	26	50	64			0.274	
	21	63	80			0.340	
	17	78	100			0.420	
	15.5	86	110			0.460	
	13.5	100	128			0.528	
	11	125	160			0.648	
	9.3	151	193			0.766	
	9	156	200			0.792	
	7.3	198	254			0.976	

$Table \ 52C{-}13 \quad {\rm Polyethylene \ pipe, \ iron \ pipe \ size \ outside \ diameter}$

Table 52C-13 Polyethylene pipe, iron pipe size outside diameter—Cont
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Nominal pipe size	SDR	Pressu	re class erial	Dim Outside	ension and tole diameter	erance Wall thickness	
(in)		2406	3408	minimum	tolerance	minimum	
		3406		(in)	(-/+)	(in)	
8	32.5	40	51	8.625	0.039	0.265	
	26	50	64			0.332	
	21	63	80			0.411	
	17	78	100			0.507	
	15.5	86	110			0.556	
	13.5	100	128			0.639	
	11	125	160			0.784	
	9.3	151	193			0.927	
	9	156	200			0.958	
	7.3	198	254			1.182	
10	20.5	40	51	10.75	0.049	0.991	
10	54.9 96	40	01 64	10.75	0.040	0.351	
	20 01	00 C2	04			0.413	
	21	63 70	80			0.512	
	17	78	100			0.632	
	15.5	86	110			0.694	
	13.5	100	128			0.796	
	11	125	160			0.977	
	9.3	151	193			1.156	
	9	156	200			1.194	
	7.3	198	254			1.473	
12	32.5	40	51	12.75	0.057	0.392	
	26	50	64			0.490	
	21	63	80			0.607	
	17	78	100			0.750	
	15.5	86	110			0.823	
	13.5	100	128			0.944	
	11	125	160			1.159	
	9.3	151	193			1.371	
	9	156	200			1.417	
	7.3	198	254			1.747	
13	325	40	51	13 375	0.060	0 412	
20	26	50	64	201010	0.000	0.515	
	21	63	80			0.638	
	17	78	100			0.788	
	15.5	86	110			0.863	
	13.5	100	128			0.991	
	11	125	160			1.216	
	9.3	151	193			1.438	
	9	156	200			1.486	
	7.3	198	254			1.832	

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Nominal	SDR	Pressure class		Dime	ension and tole	rance	
pipe size (in)		Mat 2406 3406	erial 3408	Outside minimum (in)	diameter tolerance (-/+)	Wall thickness minimum (in)	
14	32.5	40	51	14.000	0.063	0.431	
	26	50	64			0.538	
	21	63	80			0.667	
	17	78	100			0.824	
	15.5	86	110			0.903	
	13.5	100	128			1.037	
	11	125	160			1.273	
	9.3	151	193			1.505	
	9	156	200			1.556	
	7.3	198	254			1.918	
16	32.5	40	51	16.000	0.072	0.492	
	26	50	64			0.615	
	21	63	80			0.762	
	17	78	100			0.941	
	15.5	86	110			1.032	
	13.5	100	128			1.185	
	11	125	160			1.455	
	9.3	151	193			1.720	
	9	156	200			1.778	
	7.3	198	254			2.192	
18	32.5	40	51	18.000	0.081	0.554	
	26	50	64			0.692	
	21	63	80			0.857	
	17	78	100			1.059	
	15.5	86	110			1.161	
	13.5	100	128			1.333	
	11	125	160			1.636	
	9.3	151	193			1.935	
	9	156	200			2.000	
	7.3	198	254			2.466	
20	32.5	40	51	20.000	0.090	0.615	
	26	50	64			0.769	
	21	63	80			0.952	
	17	78	100			1.176	
	15.5	86	110			1.290	
	13.5	100	128			1.481	
	11	125	160			1.818	
	9.3	151	193			2.151	
	9	156	200			2.222	
	7.3	198	254			2.740	

Table 52C-13 Polyethylene pipe, iron pipe size outside diameter—Continued

Table 52C-13 Polyethylene pipe, iron pipe size outside diameter—Cont
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Nominal pipe size (in)	SDR	Pressur	Pressure class		ension and tole diameter	erance Wall thickness	
		2406	2406 3408		tolerance	minimum	
		3406		(in)	(-/+)	(in)	
21.5	32.5	40	51	21.500	0.097	0.662	
	26	50	64			0.827	
	21	63	80			1.024	
	17	78	100			1.265	
	15.5	86	110			1.387	
	13.5	100	128			1.593	
	11	125	160			1.955	
	9.3	151	193			2.312	
	9	156	200			2.389	
	7.3	198	254			2.945	
22	32.5	40	51	22.000	0.099	0.677	
	26	50	64			0.846	
	21	63	80			1.048	
	17	78	100			1.294	
	15.5	86	110			1.419	
	13.5	100	128			1.630	
	11	125	160			2.000	
	9.3	151	193			2.366	
	9	156	200			2 444	
	7.3	198	254			3.014	
24	32.5	40	51	24.000	0.108	0.738	
	26	50	64			0.923	
	21	63	80			1.143	
	17	78	100			1.412	
	15.5	86	110			1.548	
	13.5	100	128			1.778	
	11	125	160			2.182	
	9.3	151	193			2.581	
	9	156	200			2.667	
	7.3	198	254			3.288	
26	32.5	40	51	26.000	0.117	0.800	
	26	50	64			1.000	
	21	63	80			1.238	
	17	78	100			1.529	
	15.5	86	110			1.677	
	13.5	100	128			1.926	
	11	125	160			2.364	
	9.3	151	193			2.796	
	9	156	200			2.889	
	7.3	198	254			3.562	

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Nominal	SDR	Pressure class		Dime	ension and tole	rance	
pipe size (in)		Mat 2406 3406	erial 3408	Outside minimum (in)	diameter tolerance (–/+)	Wall thickness minimum (in)	
28	32.5	40	51	28.000	0.126	0.862	
	26	50	64			1.077	
	21	63	80			1.333	
	17	78	100			1.647	
	15.5	86	110			1.806	
	13.5	100	128			2.074	
	11	125	160			2.545	
	9.3	151	193			3.011	
	9	156	200			3.111	
	7.3	198	254			3.836	
32	32.5	40	51	32.000	0.144	0.985	
	26	50	64			1.231	
	21	63	80			1.524	
	17	78	100			1.882	
	15.5	86	110			2.065	
	13.5	100	128			2.370	
	11	125	160			2.909	
	9.3	151	193			3.441	
	9	156	200			3.566	
	7.3	198	254			4.384	
34	32.5	40	51	34.000	0.153	1.046	
	26	50	64			1.308	
	21	63	80			1.619	
	17	78	100			2.000	
	15.5	86	110			2.194	
	13.5	100	128			2.519	
	11	125	160			3.091	
	9.3	151	193			3.656	
	9	156	200			3.778	
	7.3	198	254			4.658	
36	32.5	40	51	36.000	0.162	1.108	
	26	50	64			1.385	
	21	63	80			1.714	
	17	78	100			2.118	
	15.5	86	110			2.323	
	13.5	100	128			2.667	
	11	125	160			3.273	
	9.3	151	193			3.871	
	9	156	200			4.000	
	7.3	198	254			4.932	

Table 52C-13 Polyethylene pipe, iron pipe size outside diameter—Continued

Table 52C-13 Polyethylene pipe, iron pipe size outside diameter—Continue	ed
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Nominal	SDR	Pressur	Pressure class		ension and tole	erance	
pipe size		Mate 2406	erial 3408	Outside	diameter tolerance	Wall thickness minimum	
()		3406	0100	(in)	(-/+)	(in)	
42	32.5	40	51	42.000	0.189	1.292	
	26	50	64	1_1000	0.200	1.615	
	21	63	80			2.000	
	17	78	100			2.471	
	15.5	86	110			2.710	
	13.5	100	128			3.111	
	11	125	160			3.818	
	93	151	193			4 516	
	9	156	200			4 667	
	7.3	198	254			5.753	
48	32.5	40	51	48.000	0.216	1.477	
	26	50	64			1.846	
	21	63	80			2.286	
	17	78	100			2.824	
	15.5	86	110			3.097	
	13.5	100	128			3.556	
	11	125	160			4.364	
	9.3	151	193			5.161	
	9	156	200			5.333	
	7.3	198	254			6.575	
54	32.5	40	51	54.000	0.243	1.662	
	26	50	64			2.077	
	21	63	80			2.571	
	17	78	100			3.177	
	15.5	86	110			3.484	
	13.5	100	128			4 000	
	11	125	160			4 909	
	93	151	193			5 807	
	9	156	200			6,000	
	7.3	198	254			7.397	
63	32.5	40	51	63.000	0.284	1.938	
	26	50	64			2.423	
	21	63	80			3.000	
	17	78	100			3.706	
	15.5	86	110			4.065	
	13.5	100	128			4.667	
	11	125	160			5.727	
	9.3	151	193			6.774	
	9	156	200			7.000	
	7.3	198	254			8.630	

Source: AWWA C 906
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Nominal	SDR	Pressure class		Dime	ension and tole	rance	
pipe size (in)		Mat 2406 3406	erial 3408	Outside minimum (in)	tolerance (-/+)	Wall thickness minimum (in)	
		10	~ 1	4.000	0.022	0.140	
4	32.5	40	51	4.800	0.022	0.148	
	26	50	64			0.185	
	21	63	80			0.229	
	17	78	100			0.282	
	15.5	86	110			0.310	
	13.5	100	128			0.356	
	11	125	160			0.436	
	9.3	151	193			0.516	
	9	156	200			0.533	
	7.3	198	254			0.658	
6	32.5	40	51	6.900	0.031	0.212	
	26	50	64			0.265	
	21	63	80			0.329	
	17	78	100			0.406	
	15.5	86	110			0.445	
	13.5	100	128			0.511	
	11	125	160			0.627	
	9.3	151	193			0.742	
	9	156	200			0.787	
	7.3	198	254			0.945	
8	32.5	40	51	9.050	0.041	0.278	
0	26	50	64	0.000	0.011	0.348	
	21	63	80			0.431	
	17	78	100			0.532	
	15.5	86	110			0.584	
	13.5	100	128			0.670	
	11	125	160			0.823	
	93	151	193			0.973	
	9	156	200			1 006	
	5 7.3	198	254			1.240	
10	99 F	40	F 1	11 100	0.050	0.949	
10	34.9 96	40	01 C4	11.100	0.090	0.427	
	20 01	00	04			0.427	
	21	63	80			0.629	
	17	78	100			0.653	
	15.5	86	110			0.716	
	13.5	100	128			0.822	
	11	125	160			1.009	
	9.3	151	193			1.194	
	9	156	200			1.233	
	7.3	198	254			1.521	

$Table \ 52C{-}14 \quad {\rm Polyethylene \ pipe, \ ductile \ iron \ pipe \ size \ outside \ diameter}$

Table 52C-14 Polyethylene pipe, ductile iron pipe size outside diameter—Continued

Nominal pipe size	SDR)R Pressure class		Dim Outside	ension and tole diameter	erance Wall thickness	
(in)		2406	3408	minimum	tolerance	minimum	
		3406		(in)	(-/+)	(in)	
12	32.5	40	51	13.200	0.059	0.406	
	26	50	64			0.508	
	21	63	80			0.629	
	17	78	100			0.776	
	15.5	86	110			0.852	
	13.5	100	128			0.978	
	11	125	160			1.200	
	9.3	151	193			1.419	
	9	156	200			1.467	
	7.3	198	254			1.808	
14	32.5	40	51	15 300	0.069	0 471	
	26	50	64	201000	0.000	0.588	
	21	63	80			0.729	
	17	78	100			0.900	
	15.5	86	110			0.987	
	13.5	100	128			1 133	
	10.0	125	160			1.199	
	03	151	100			1.645	
	0	156	200			1.045	
	9 7.3	198	250			2.096	
10	00 5	40	F 1	17 400	0.070	0 505	
10	32.5 90	40	01 C4	17.400	0.078	0.030	
	26	50 C0	64 00			0.669	
	21	63 70	80			0.829	
	17	78	100			1.024	
	15.5	80	110			1.123	
	13.5	100	128			1.289	
	11	125	100			1.582	
	9.3	151	193			1.871	
	9	156	200			1.933	
	7.3	198	254			2.384	
18	32.5	40	51	19.500	0.088	0.600	
	26	50	64			0.750	
	21	63	80			0.929	
	17	78	100			1.147	
	15.5	86	110			1.258	
	13.5	100	128			1.444	
	11	125	160			1.773	
	9.3	151	193			2.097	
	9	156	200			2.167	
	7.3	198	254			2.671	

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Nominal	SDR	Pressure class		Dime	ension and tole	erance	
(in)		Mat 2406 3406	erial 3408	Outside minimum (in)	tolerance (-/+)	Wall thickness minimum (in)	
20	32.5	40	51	21.600	0.097	0.665	
	26	50	64			0.831	
	21	63	80			1.029	
	17	78	100			1.271	
	15.5	86	110			1.394	
	13.5	100	128			1.600	
	11	125	160			1.964	
	9.3	151	193			2.323	
	9	156	200			2.400	
	7.3	198	254			2.959	
24	32.5	40	51	25.800	0.116	0.794	
	26	50	64			0.992	
	21	63	80			1.229	
	17	78	100			1.518	
	15.5	86	110			1.665	
	13.5	100	128			1.911	
	11	125	160			2.345	
	9.3	151	193			2.774	
	9	156	200			2.867	
	7.3	198	254			3.534	
30	32.5	40	51	32.000	0.144	0.985	
	26	50	64			1.231	
	21	63	80			1.524	
	17	78	100			1.882	
	15.5	86	110			2.065	
	13.5	100	128			2.370	
	11	125	160			2.909	
	9.3	151	193			3.441	
	9	156	200			3.556	
	7.3	198	254			4.384	
36	32.5	40	51	38.300	0.172	1.178	
	26	50	64			1.473	
	21	63	80			1.824	
	17	78	100			2.253	
	15.5	86	110			2.471	
	13.5	100	128			2.837	
	11	125	160			3.482	
	9.3	151	193			4.118	
	9	156	200			4.256	
	7.3	198	254			5.247	

$Table \ 52C{-}14 \quad {\rm Polyethylene \ pipe, \ ductile \ iron \ pipe \ size \ outside \ diameter}{-} Continued$

Table 52C–14	Polyethylene pi	pe, ductile i	ron pipe si	ize outside	diameter—	Continued
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Nominal	SDR	DR Pressure class		Dime	ension and tole	erance Wall thickness	
(in)		2406 3406	3408	minimum (in)	tolerance (-/+)	minimum (in)	
42	32.5	40	51	44.500	0.200	1.369	
	26	50	64			1.712	
	21	63	80			2.119	
	17	78	100			2.618	
	15.5	86	110			2.871	
	13.5	100	128			3.296	
	11	125	160			4.046	
	9.3	151	193			4.785	
	9	156	200			4.944	
	7.3	198	254			6.096	
48	32.5	40	51	50.800	0.229	1.563	
	26	50	64			1.954	
	21	63	80			2.419	
	17	78	100			2.988	
	15.5	86	110			3.277	
	13.5	100	128			3.763	
	11	125	160			4.618	
	9.3	151	193			5.462	
	9	156	200			5.644	
	7.3	198	254			6.959	
54	32.5	40	51	57.100	0.257	1.757	
	26	50	64			2.196	
	21	63	80			2.719	
	17	78	100			3.359	
	15.5	86	110			3.684	
	13.5	100	128			4.230	
	11	125	160			5.191	
	9.3	151	193			6.140	
	9	156	200			6.344	
	7.3	198	254			7.822	

Source: AWWA C 906.

Table 52C-15Type PSM PVC pipe

Nominal pipe size (in)	Outside di average	iameter (in) tolerance	SDR 41	- Minimum wa SDR 35	ll thickness (i SDR 26	n) SDR 23.5	
4	4.215	0.009		0.120	0.162	0.178	
6	6.275	0.011	0.153	0.180	0.241	0.265	
8	8.400	0.012	0.205	0.240	0.323		
9	9.440	0.014	0.230				
10	10.500	0.015	0.256	0.300	0.404		
12	12.500	0.018	0.305	0.360	0.481		
15	15.300	0.023	0.375	0.437	0.588		

Source: ASTM D 3034

PSM is not an abbreviation, but rather an arbitrary designation for a product having Note: certain dimensions.

Table 52C-16 PVC large-diameter plastic pipe

Nominal pipe size (in)	Outside d average	iameter (in) tolerance	Minimum wal cell class 12454	ll thickness (in) cell class 12364	Minimum pipe stiffness (lb/in²)	
18	18.701	0.028	0.536	0.499	46	
21	22.047	0.033	0.632	0.588	46	
24	24.803	0.037	0.711	0.661	46	
27	27.953	0.042	0.801	0.745	46	
30	31.946	0.047	0.903	0.840	46	
30*	32.000	0.040	0.917	0.853	46	
33	35.433	0.053	1.016	0.945	46	
36	39.370	0.059	1.129	1.050	46	
36*	38.300	0.050	1.098	1.021	46	
42	44.500	0.060	1.276	1.187	46	
48	50.800	0.075	1.456	1.355	46	

Source: ASTM F 679 * Cast iron pipe size

Cast iron pipe size

Table 52C-17 Smooth wall PVC plastic underdrain pipe

Nominal pipe size (in)	Outside di average	ameter (in) tolerance	Minimum wal PS28	inimum wall thickness (in) PS28 PS46		
4	4.215	0.009	0.103	0.120		
6	6.275	0.011	0.153	0.180		
8	8.400	0.012	0.205	0.240		
Source:	ASTM F 758					

Note: PS = pipe stiffness

Table 52C-18Type PS46 and PS115 PVC plastic pipe

Nominal	Pipe	Outside di	ameter (in)	r	T_{-1} Wall thickness (in)				
(in)	(lb/in ²)	average	tolerance	est. avg.	minimum	est. avg.	minimum	est. avg.	minimum
4	$46.000 \\ 115.000$	4.215	0.009	$0.114 \\ 0.152$	$\begin{array}{c} 0.107 \\ 0.143 \end{array}$	$\begin{array}{c} 0.111 \\ 0.148 \end{array}$	$0.104 \\ 0.139$	$\begin{array}{c} 0.108 \\ 0.144 \end{array}$	$0.102 \\ 0.135$
6	$46.000 \\ 115.000$	6.275	0.011	$0.170 \\ 0.226$	$\begin{array}{c} 0.160\\ 0.214\end{array}$	$0.165 \\ 0.220$	$0.155 \\ 0.207$	$\begin{array}{c} 0.161 \\ 0.215 \end{array}$	$0.151 \\ 0.202$
8	$46.000 \\ 115.000$	8.400	0.012	$0.227 \\ 0.302$	$0.213 \\ 0.284$	$0.221 \\ 0.294$	$0.208 \\ 0.276$	$0.216 \\ 0.287$	$0.203 \\ 0.270$
10	$46.000 \\ 115.000$	10.500	0.015	$0.284 \\ 0.378$	$0.267 \\ 0.355$	$0.276 \\ 0.363$	$0.259 \\ 0.341$	$0.270 \\ 0.359$	$0.254 \\ 0.337$
12	$46.000 \\ 115.000$	12.500	0.018	$0.338 \\ 0.450$	$0.318 \\ 0.423$	$0.329 \\ 0.438$	$\begin{array}{c} 0.309 \\ 0.414 \end{array}$	$\begin{array}{c} 0.321\\ 0.428\end{array}$	$0.302 \\ 0.402$
15	$46.000 \\ 115.000$	15.300	0.023	$\begin{array}{c} 0.414 \\ 0.548 \end{array}$	$0.389 \\ 0.515$	$0.403 \\ 0.536$	$0.379 \\ 0.504$	$0.393 \\ 0.523$	$0.369 \\ 0.492$
18	$46.000 \\ 115.000$	18.700	0.028	$0.507 \\ 0.673$	$0.477 \\ 0.633$	$0.494 \\ 0.655$	$\begin{array}{c} 0.464 \\ 0.616 \end{array}$	$\begin{array}{c} 0.482\\ 0.640\end{array}$	$0.452 \\ 0.602$

Source: ASTM F 789

Made with material that has modulus of 440,000 to 480,000 lb/in². Made with material that has modulus of 480,000 to 520,000 lb/in². T-1:

T-2:

Made with material that has modulus of 520,000 to 560,000 lb/in². T-3:

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Table 52C–19	Open and dual wall	PVC profile plastic pipe	dimensions and tolerances
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Nominal	Inside diameter (in)		Minim	um wall thick	ickness in waterway (in) dual wall		
(in)	minimitum	tolerance	PS 10	PS 46	PS 10	PS 46	
4	3.939	0.034		0.030		0.022	
6	5.875	0.049		0.045		0.025	
8	7.863	0.053		0.060		0.035	
10	9.825	0.067		0.070		0.045	
12	11.687	0.085		0.085		0.058	
15	14.303	0.116		0.105		0.077	
18	17.510	0.195	0.040	0.130	0.070	0.084	
21	20.656	0.200	0.085	0.160	0.070	0.095	
24	23.412	0.204	0.105	0.180	0.070	0.110	
27	26.371	0.209	0.115	0.205	0.070	0.120	
30	29.388	0.220	0.130	0.235	0.085	0.130	
33	32.405	0.227	0.150	0.260	0.095	0.150	
36	35.370	0.235	0.165	0.290	0.105	0.155	
39	38.380	0.245	0.195	0.315	0.120	0.200	
42	41.370	0.255	0.215	0.345	0.130	0.200	
45	44.365	0.265	0.225	0.370	0.145	0.200	
48	47.355	0.285	0.230	0.400	0.160	0.200	

Table 52C–20	PVC corrugated pipe with smooth interior dimensions and tolerances
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Nominal	Pipe	Outside dia	ameter (in)	Inside di	ameter (in)	Minimu	ım wall thickn	ess (in)
pipe size (in)	stiffness (lb/in ²)	average	tolerance	average	tolerance	inner wall	outer wall	at valley
4	46	4.300	0.009	3.950	0.011	0.022	0.018	0.028
6	46	6.420	0.011	5.909	0.015	0.025	0.022	0.032
8	46 115	8.600	0.012	7.881	0.018	$0.035 \\ 0.037$	$0.030 \\ 0.050$	$\begin{array}{c} 0.045\\ 0.048\end{array}$
10	46 115	10.786	0.015	9.846	0.021	$\begin{array}{c} 0.045\\ 0.046\end{array}$	$0.036 \\ 0.052$	$\begin{array}{c} 0.055\\ 0.065\end{array}$
12	46 115	12.795	0.018	11.715	0.028	$0.058 \\ 0.070$	$0.049 \\ 0.068$	$\begin{array}{c} 0.072\\ 0.091 \end{array}$
15	46 115	15.658	0.023	14.338	0.035	$0.077 \\ 0.092$	$0.055 \\ 0.088$	$0.092 \\ 0.118$
18	46	19.152	0.028	17.552	0.042	0.084	0.067	0.103
21	46	22.630	0.033	20.705	0.049	0.095	0.073	0.110
24	46	25.580	0.039	23.469	0.057	0.110	0.085	0.123
27	46	28.860	0.049	26.440	0.069	0.120	0.091	0.137
30	46	32.150	0.059	29.469	0.081	0.130	0.105	0.147
36	46	38.740	0.079	35.475	0.105	0.150	0.125	0.171

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Nominal	Inside dia	ameter (in)	Minimum	wall thicknes	ss in pipe wate	erway (in)	Min. bell
(in)	average	tolerance	1650 40	1650 05	100	NSC 100	(in)
18	18.00	0.38	0.18	0.18	0.18	0.22	0.7
21	21.00	0.38	0.18	0.18	0.18	0.24	0.7
24	24.00	0.38	0.18	0.18	0.22	0.24	0.7
27	27.00	0.38	0.18	0.18	0.24	0.24	0.7
30	30.00	0.38	0.18	0.22	0.24	0.26	0.7
33	33.00	0.38	0.18	0.24	0.24	0.30	0.95
36	36.00	0.38	0.18	0.24	0.26	0.30	1.05
42	42.00	0.42	0.24	0.24	0.30	0.38	1.15
48	48.00	0.48	0.24	0.26	0.30	0.38	1.25
54	54.00	0.54	0.24	0.30	0.38	0.42	1.25
60	60.00	0.60	0.26	0.30	0.38	0.52	1.3
66	66.00	0.66	0.30	0.38	0.42	0.67	1.3
72	72.00	0.72	0.30	0.38	0.42	0.90	1.3
78	78.00	0.78	0.30	0.38	0.52	0.90	1.35
84	84.00	0.84	0.38	0.42	0.67	0.90	1.35
90	90.00	0.90	0.38	0.42	0.90	0.95	1.35
96	96.00	0.96	0.38	0.52	0.90	0.95	1.35
108	108.00	1.08	0.42	0.67	0.90	0.95	1.35
120	120.00	1.20	0.52	0.67	0.90	0.95	1.35

 $Table \ 52C{-}21 \quad {\rm Open \ profile \ polyethylene \ pipe \ dimensions \ and \ tolerances}$

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$\label{eq:table 52C-22} Table \ 52C-22 \quad {\rm Closed \ profile \ polyethylene \ pipe \ dimensions \ and \ tolerances}$

Nominal pipe size (in)	Inside dia average	meter (in) tolerance	Min. wall thickness in pipe waterway (in)	Min. bell thickness (in)
10	10.00	0.38	0.18	0.5
12	12.00	0.38	0.18	0.5
15	15.00	0.38	0.18	0.5
18	18.00	0.38	0.18	0.5
21	21.00	0.38	0.18	0.5
24	24.00	0.38	0.18	0.5
27	27.00	0.38	0.18	0.5
30	30.00	0.38	0.18	0.5
33	33.00	0.38	0.18	0.5
36	36.00	0.38	0.18	0.5
40	40.00	0.38	0.18	0.5
42	42.00	0.42	0.18	0.5
48	48.00	0.48	0.18	0.5
54	54.00	0.54	0.18	0.5
60	60.00	0.60	0.18	0.6
66	66.00	0.66	0.18	0.6
72	72.00	0.72	0.18	0.6
78	78.00	0.78	0.18	0.6
84	84.00	0.84	0.18	0.7
90	90.00	0.90	0.18	0.7
96	96.00	0.96	0.18	0.7
108	108.00	1.08	0.18	0.7
120	120.00	1.20	0.18	0.8

Appendix 52D

Selection Properties of Corrugated and Spiral Rib Metal Pipe

Gage	Specified	1-1	/2" x 1/4" Corrugatio	on	2-2	2/3" x 1/2" Corruga	ation
	thickness (galvanized) (in)	Area of section, A _s (in²/ft)	Moment of I, inertia (in ⁴ /in)	Radius of gyration, r (in)	Area of section, A _s (in²/ft)	Moment of I, inertia (in ⁴ /in)	Radius of gyration, r (in)
20	0.040	0.456	0.000253	0.0816	0.465	.001122	0.1702
18	0.052	0.608	0.000343	0.0824	0.619	.001500	0.1707
16	0.064	0.761	0.000439	0.0832	0.775	.001892	0.1712
14	0.079	0.950	0.000566	0.0846	0.968	.002392	0.1721
12	0.109	1.333	0.000857	0.0879	1.356	.003425	0.1741
10	0.138	1.712	0.001205	0.0919	1.744	.004533	0.1766
8	0.168	2.098	0.001635	0.0967	2.133	.005725	0.1795

Table 52D-1 Section properties of corrugated steel pipe

Gage	Specified	;	3" x 1" Corrugation	1		5" x 1" Corrugation		
	thickness (galvanized) (in)	Area of section, A _s (in²/ft)	Moment of I, inertia (Ix10 ⁻³ in ⁴ /in)	Radius of gyration, r (in)	Area of section, A _s (in²/ft)	Moment of I, inertia (I x 10 ⁻³ in ⁴ /in)	Radius of gyration, r (in)	
18	0.052	0.711	0.006892	0.3410				
16	0.064	0.890	0.008658	0.3417	0.794	.008850	0.3657	
14	0.079	1.113	0.010883	0.3427	0.992	.011092	0.3663	
12	0.109	1.560	0.015458	0.3448	1.390	.015550	0.3677	
10	0.138	2.008	0.020175	0.3472	1.788	.020317	0.3693	
8	0.168	2.458	0.025083	0.3499	2.186	.025092	0.3711	

Source: ASTM A 796

AASHTO Standard Specifications for Highway Bridges

 Table 52D-2
 Ultimate longitudinal seam strength of riveted or spot welded corrugated steel pipe

Gage	Specified thickness (galvanized)	5/16" 1 2 2/3"	rivets x 1/2"	Seam stren 2 2/3"	gth (lb/ft of s - 3/8" rivets - x 1/2"	eam) 3 x 1" and 5 x 1"		7/16" rivets 3 x 1" and 5 x 1"
	(in)	single	double	single	double	double		double
16	.064	16,700	21,600			28,700		
14	0.079	18,200	29,800			35,700		
12	0.109			23,400	46,800		53,000	
10	0.138			24,500	49,000		63,700	
8	0.168			25,600	51,300		70,700	

Source: ASTM A 796.

Gage	Specified	1-1,	/2" x 1/4" Corrugatio	on	2-2	2/3" x 1/2" Corrugat	ion
C	thickness	Area of section, A _s	Moment of I, inertia	Radius of gyration, r	Area of section, A _s	Moment of I, inertia	Radius of gyration, r
	(in)	(in²/ft)	(in ⁴ /in)	(in)	(in²/ft)	(in 4/in)	(in)
18	0.048	0.608	0.000344	0.0824			
16	0.060	0.761	0.000349	0.0832	0.775	0.001892	0.1712
14	0.075		—	_	0.968	0.002392	0.1721
12	0.105			_	1.356	0.003425	0.1741
10	0.135	_	_	—	1.745	0.004533	0.1766
8	0.164		_	_	2.130	0.005725	0.1795

Table 52D-3 Section properties of corrugated aluminum pipe

Gage	Specified thickness (in)	Area of section, A _s (in²/ft)	3" x 1" Corrugation Moment of I, inertia (in ^{4/} in)	Radius of gyration, r (in)	Area of section, A _s (in ² /ft)	6" x 1" Co Effective area (in ² /ft)	orrugation Moment of I, inertia (in ⁴ /in)	Radius of gyration, r (in)
16	0.060	0.890	0.008659	0.3417	0.775	0.387	0.008505	0.3629
14	0.075	1.118	0.010883	0.3427	0.968	0.484	0.010631	0.3630
12	0.105	1.560	0.015459	0.3448	1.356	0.678	0.014340	0.3636
10	0.135	2.008	0.020183	0.3472	1.744	0.872	0.019319	0.3646
8	0.164	2.458	0.025091	0.3499	2.133	1.066	0.02376	0.3656

Source: ASTM B 790

AASHTO Standard Specification for Highway Bridges

Table 52D-4 Ultimate longitudinal seam strength of riveted corrugated aluminum pipe

Gage	Specified thickness	5/16 ir 2 2/3 x	1 rivets x 1/2 in	Seam stren; 2 2/3 x	gth (lb/ft of - 3/8 in rive 1/2 in	seam) ts 3 x 1 in and		1/2 in rivets 3 x 1 in and
	(in)	single	double	single	double	5 x 1 in double		5 x lin double
16	0.064	9,000	14,000			16,500		
14	0.075	9,000	18,000			20,500		
12	0.105			15,600	31,500		28,000	
10	0.135			16,200	33,000		42,000	
8	0.164			16,800	34,000		54,500	

Table 52D–5	Section	properties	of spiral	rib steel	pipe
		* *	*		* *

Gage	Specified	3	8/4" x 3/4" x 7-	1/2"	3/	/4" x 1" x 11-1/2	2"	3	/4" x 1" x 8-1/2	2"
	thickness (galvanized) (in)	$\begin{array}{c} {\rm Area \ of} \\ {\rm section, \ A_s} \\ {\rm (in^2/ft)} \end{array}$	Moment of I, inertia (in ⁴ /in)	Radius of gyration, r (in)	$\begin{array}{c} {\rm Area \ of} \\ {\rm section, \ A_s} \\ {\rm (in^2/ft)} \end{array}$	Moment of I, inertia (in ⁴ /in)	Radius of gyration, r (in)	Area of section, A _s (in²/ft)	Moment of I, inertia (in ⁴ /in)	Radius of gyration, r (in)
16	0.064	0.509	0.002821	0.258	0.374	0.00458	0.383	0.499	0.005979	0.379
14	0.079	0.712	0.003701	0.25	0.524	0.00608	0.373	0.694	0.007913	0.37
12	0.109	1.184	0.005537	0.237	0.883	0.00926	0.355	1.149	0.011983	0.354
10	0.138	1.717	0.007433	0.228						

Source: ASTM A 796

Table 52D-6Section properites of spiral rib aluminum pipe

Gage	Specified thickness	Area of	3/4" x 3/4" x 7- Moment	1/2" Radius of	Area of	4" x 1" x 11-1/ Moment of	2" Radius of
	(in)	(in ² /ft)	(in ⁴ /in)	(in)	(in²/ft)	(in 4/in)	(in)
16	0.06	0.415	0.002558	0.272	0.312	0.00408	0.396
14	0.075	0.569	0.003372	0.267	0.427	0.00545	0.391
12	0.105	0.914	0.005073	0.258	0.697	0.00839	0.38
10	0.135	1.29	0.006826	0.252	1.009	0.01148	0.369

Appendix 52E

Allowable Flexibility Factors of Corrugated and Spiral Rib Metal Pipe

Depth of corruga-	Material thickness	In trench Embankment					
tion (in)	(in)	steel	aluminum	steel	aluminum		
1/4	0.060	0.043	0.031	0.043	0.031		
	0.075	0.043	0.061	0.043	0.061		
	others	0.043	0.092	0.043	0.092		
1/2	0.060	0.060	0.031	0.043	0.031		
	0.075	0.060	0.061	0.043	0.061		
	others	0.060	0.092	0.043	0.092		
1	all	0.060	0.060	0.033	0.060		
2	all	0.020	—	0.020	—		
2 1/2	all	_	0.025	_	0.025		
5 1/2	all	0.020	—	0.020	—		

Table 52E-1 Flexibility factor for corrugated metal pipe

Source: ASTM A 796 and B 790

Table 52E–2 F	'lexibility f	actor for s	spiral	rib m	etal	pipe
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Profile (in)	In trench w/compacted		Flexibility factor (in/lbf) In trench w/o compacted		Embankment	
	steel	aluminum	steel	aluminum	steel	aluminum
3/4 x 3/4 x 7-1/2	$0.367 \ \mathrm{I}^{1/3}$	0.600 I ^{1/3}	$0.263 \ I^{1/3}$	$0.420 \ \mathrm{I}^{1/3}$	$0.217 \ \mathrm{I}^{1/3}$	0.340 I ^{1/3}
3/4 x 1 x 8-1/2	$0.262 \ \mathrm{I}^{1/3}$		$0.163 \mathrm{I}^{1/3}$		$0.140 \ I^{1/3}$	
3/4 x 1 x 11-1/2	$0.220 \ I^{1/3}$	$0.310 \ I^{1/3}$	$0.163 \ I^{1/3}$	$0.215 \ \mathrm{I}^{1/3}$	$0.140 \ I^{1/3}$	$0.175 \ \mathrm{I}^{1/3}$

Source: ASTM A 796 and B 790

Appendix 52F

Nominal Thickness for Standard Pressure Classes of Ductile Iron Pipe

 Table 52F-1
 Nominal thickness for standard pressure classes of ductile iron pipe and allowances for casting tolerance

Size, in	Outside diameter,	Nominal thickness, in (mm)						
	In (nun)	150	200	250	300	350	in (mm)	
3	3.96 (100.6)	_	—	_	_	0.25 (6.4)	0.05 (1.3)	
4	4.80 (121.9)		—		_	0.25 (6.4)	0.05 (1.3)	
6	6.90 (175.3)					0.25 (6.4)	0.05 (1.3)	
8	9.05 (229.9)		—		_	0.25 (6.4)	0.05 (1.3)	
10	11.10 (281.9)		—		_	0.26 (6.6)	0.06 (1.5)	
12	13.20 (335.3)					0.28 (7.1)	0.06 (1.5)	
14	15.30 (388.6)	_	—	0.28 (7.1)	0.30 (7.6)	0.31 (7.9)	0.07 (1.8)	
16	17.40 (442.0)			0.30 (7.6)	0.32 (8.1)	0.34 (8.6)	0.07 (1.8)	
18	19.50 (495.3)			0.31 (7.9)	0.34 (8.6)	0.36 (9.1)	0.07 (1.8)	
20	21.60 (548.6)			0.33 (8.4)	0.36 (9.1)	0.38 (9.7)	0.07 (1.8)	
24	25.80 (655.3)		0.33 (8.4)	0.37 (9.4)	0.40 (10.2)	0.43 (10.9)	0.07 (1.8)	
30	32.00 (812.8)	0.34 (8.6)	0.38 (9.7)	0.42 (10.7)	0.45 (11.4)	0.49 (12.4)	0.07 (1.8)	
36	38.30 (972.8)	0.38 (9.7)	0.42 (10.7)	0.47 (11.9)	0.51 (12.9)	0.56 (14.2)	0.07 (1.8)	
42	44.50 (1,130.3)	0.41 (10.4)	0.47 (11.9)	0.52 (13.2)	0.57 (14.5)	0.63 (16.0)	0.07 (1.8)	
48	50.80 (1,290.3)	0.46 (11.7)	0.52 (13.2)	0.58 (14.7)	0.64 (16.3)	0.70 (17.8)	0.08 (2.0)	
54	57.56 (1,450.3)	0.51 (12.9)	0.58 (14.7)	0.65 (16.5)	0.72 (18.3)	0.79 (20.1)	0.09 (2.3)	
60	61.61 (1,564.9)	0.54 (13.7)	0.61 (15.5)	0.68 (17.3)	0.76 (19.3)	0.83 (21.1)	0.09 (2.3)	
64	65.67 (1,668.0)	0.56 (14.2)	0.64 (16.3)	0.72 (18.3)	0.80 (20.3)	0.87 (22.1)	0.09 (2.3)	