United States Department of Agriculture

## Natural

 Resources Conservation ServicePart 636 Structural Engineering National Engineering Handbook

## Chapter 52

## Structural Design of Flexible Conduits

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## Preface

Flexible conduits used on NRCS projects typically consist of corrugated metal pipe (CMP), various types of plastic pipe, steel pipe, or ductile iron pipe. The design of these conduits was completed by allowable fill height tables in various Conservation Practice Standards, guidance given in TR 77-Design and Installation of Flexible Conduits and the associated computer program, and multiple technical notes developed by NRCS staff.

NEH 636 chapter 52 updates the design procedure to current industry and government agency practice. Although symbols for conduit (pipe) design vary among types of materials and industry guidance, those used in chapter 52 are consistent within the document (see appendix 52A). Appendix 52B contains several design examples that were developed using the formulas and information in this chapter. A glossary of terms used within this chapter is included following the references and prior to the appendices.

## Acknowledgments

The technical guidance in this document is a compilation of flexible conduit design guidance from the American Society of Testing Materials (ASTM), American Association of State Highway Transportation Officials (AASHTO), other Federal agencies, trade organizations, pipe manufacturers, and other text. This version was prepared by Wade Anderson, design engineer, National Design, Construction, and Soil Mechanics Center, Natural Resources Conservation Service (NRCS), Fort Worth, Texas.

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## Chapter 52

## Structural Design of Flexible Conduits

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## Chapter 52

## Structural Design of Flexible Conduits

### 636.5200 Introduction

Pipe materials are generally considered to be rigid or flexible. A flexible pipe is one that will deflect at least 2 percent without structural distress (fig. 52-1). Materials that do not meet this criterion are generally considered rigid. Some pipe materials are described as semi-rigid based on their behavior and design procedures.

A flexible conduit derives its external load capacity from its flexibility. Under load, the pipe tends to deflect, developing soil support at the sides of the pipe. The ring deflection (fig. 52-1) relieves the pipe of the major portion of the vertical soil load, which is then transferred to the soil surrounding the pipe through the soil arching action over the pipe.

Flexible pipe materials consist of smooth-wall steel pipe, corrugated spiral rib or composite ribbed metal pipe (fig. 52-2), ductile iron pipe, and solid-wall, cor-rugated-wall, or profile-wall thermoplastic pipe (PVC, ABS, or PE) (fig. 52-3). Appendix 52B has design examples for various types of flexible pipes.

Figure 52-1 Deflected pipe


Figure 52-2 Corrugated metal pipe wall sections


Corrugated wall


Spiral rib

### 636.5201 Internal pressure design

Conduits used in pressure applications must withstand the internal working pressure. The internal pressure is resisted by tensile stress (hoop stress) in the conduit wall (fig. 52-4).

Figure 52-3 Plastic pipe sections


Solid wall


Corrugated wall


Profile wall

Figure 52-4 Internal pressure



Composite ribbed

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## (a) Plastic pipe

The internal pressure capacity of plastic pipe is given as a pressure rating for plastic pipe manufactured in accordance with ASTM standards and as a pressure class for pipe meeting AWWA standards.

The pressure capacity is time dependent and should be considered in the design of a pressure pipe system. The long-term strength (hydrostatic design basis) of plastic pipe governs the pressure capacity design; yet, plastic pipe is capable of withstanding higher shortterm surge pressures.

The pressure rating or pressure class for solid-wall plastic pipe may be determined by one of the following formulas:

Outside diameter controlled pipe:

$$
\begin{equation*}
\mathrm{PC}=\mathrm{PR}=\frac{2 \times \mathrm{HDS}}{\mathrm{SDR}-1} \tag{52-1}
\end{equation*}
$$

Inside diameter controlled pipe:

$$
\begin{equation*}
\mathrm{PC}=\mathrm{PR}=\frac{2 \times \mathrm{HDS}}{\mathrm{SIDR}+1} \tag{52-2}
\end{equation*}
$$

AWWA C900 pressure class pipe:

$$
\begin{equation*}
\mathrm{PC}=\frac{2 \times \mathrm{HDS}}{\mathrm{SDR}-1}-\mathrm{P}_{\text {surge }} \tag{52-3}
\end{equation*}
$$

where:
PR = pressure rating, $\mathrm{lb} / \mathrm{in}^{2}$
$\mathrm{PC}=$ pressure class, $\mathrm{lb} / \mathrm{in}^{2}$
$P_{\text {surge }}=$ surge pressure based on an instantaneous
velocity change of $2 \mathrm{ft} / \mathrm{s}, \mathrm{lb} / \mathrm{in}^{2}$
HDS $=$ hydrostatic design stress, $\mathrm{lb} / \mathrm{in}^{2}$
HDS $=$ HDB/FS
HDB = hydrostatic design basis, $\mathrm{lb} / \mathrm{in}^{2}$
FS = factor of safety
$=2.5$ (AWWA C900 pipe)
$=2.0$ (all others)
$\mathrm{SDR}=\mathrm{D}_{\mathrm{o}}$ dimension ratio
$\mathrm{SDR}=\mathrm{D}_{\mathrm{o}} / \mathrm{t}$
$\mathrm{D}_{\mathrm{o}}=$ pipe outside diameter, in
$\mathrm{t}=$ minimum wall thickness, in
$\mathrm{SIDR}=\mathrm{D}_{\mathrm{i}}$ dimension ratio
SIDR $=D_{i} / \mathrm{t}$
$D_{i}=$ pipe inside diameter, in

Pressure ratings or pressure class and pertinent dimensions for various plastic pipe materials are provided in appendix 52C. A complete description of HDB and HDS is available in ASTM D 2837.

The maximum design pressure for systems designed without a water hammer analysis should be limited to 72 percent of the pressure rating or pressure class of the pipe (ASAE, 1998, and ASTM 1176, 1993).

For plastic pipe systems subject to recurring or cyclic surge pressures, as described in 636.5202 , the operating pressure plus the cyclic surge pressure should not exceed the pressure rating or pressure class of the pipe. If the number of cycles expected throughout the design life of the project is determined, design criteria using the short-term pressure rating and the number of cycles to failure found in Uni-Bell (2001) or recommended by the manufacturer may be used in selection of the pipe.

For occasional or infrequent pressure surges, as described in 636.5202 , plastic pipe provides a higher short-term hoop strength. The pressure that corresponds to this elevated hoop stress is referred to as the quick-burst pressure or short-term strength (STS). A short-term pressure rating may be determined from the following equation:

$$
\begin{equation*}
\mathrm{STR}=\frac{\mathrm{STS}}{\mathrm{FS}} \tag{52-4}
\end{equation*}
$$

where:

$$
\begin{aligned}
& \text { STR = short-term pressure rating, } \mathrm{lb} / \mathrm{in}^{2} \\
& \text { STS = short-term strength (quick burst pressure), } \\
& \mathrm{lb} / \mathrm{in}^{2} \\
& =\frac{2 \times \text { STHS }}{\text { SDR }-1} \quad \text { (for outside diameter contı } \\
& =\frac{2 \times \text { STHS }}{\text { SIDR }+1} \quad \text { (for inside diameter contro } \\
& \text { where: } \\
& \text { STHS = short-term hoop strength, lb/in }{ }^{2} \text { (see } \\
& \text { appendix 52C) } \\
& \mathrm{SDR}=\mathrm{D}_{\mathrm{o}} \text { dimension ratio } \\
& \text { SIDR }=\mathrm{D}_{\mathrm{i}} \text { dimension ratio } \\
& \text { FS }=2.5 \text { (AWWA C900 pipe) } \\
& =2.0 \text { (all others) }
\end{aligned}
$$

The design operating pressure plus the infrequent surge pressure should not exceed the short-term pressure rating.

Corrugated plastic pipe and profile wall plastic pipe are often not pressure rated. Because of the limited allowable pressure for watertight joints of corrugated or profile wall plastic pipe, the maximum allowable pressure shall be 10.8 pounds per square inch ( $\mathrm{lb} / \mathrm{in}^{2}$ ) (25 feet).

The HDB is typically determined in a water environment of approximately 73 degrees Fahrenheit. As the operating temperature falls below 73 degrees Fahrenheit, the pressure capacity of plastic pipe increases. As the temperature of the environment or fluid increases, the pipe becomes more ductile. The pressure rating should be decreased by the factors shown in table $52-1$ or by using the HDB determined by ASTM D 2837 at the desired elevated temperature in the pressure rating (or pressure class) calculations.

Table 52-1 Temperature factors

| Temperature <br> ${ }^{\circ} \mathrm{F}$ | PVC <br> factor | ABS <br> factor | PE <br> factor |
| :--- | :--- | :---: | :---: |
| 73.4 | 1.00 | 1.00 | 1.00 |
| 80 | 0.88 | 0.94 | 0.92 |
| 90 | 0.75 | 0.84 | 0.81 |
| 100 | 0.62 | 0.68 | 0.70 |
| 110 | 0.50 | 0.56 | 0.65 |
| 120 | 0.40 | 0.49 | 0.60 |
| 130 | 0.30 | 0.44 | 0.55 |
| 140 | 0.22 | 0.40 | 0.50 |

Source: Uni-Bell, 2001; ASTM 1176, 1993; and Plastic Pipe Institute,
2003

## (b) Smooth wall steel and aluminum pipe

The pressure rating for steel and aluminum pipe shall be determined by the following formula:

$$
\begin{equation*}
\mathrm{PR}=\frac{2 \times \mathrm{S} \times \mathrm{t}}{\mathrm{D}_{\mathrm{o}}} \tag{52-5}
\end{equation*}
$$

where:

$$
\begin{aligned}
& \mathrm{PR}=\text { pressure rating, } \mathrm{lb} / \mathrm{in}^{2} \\
& \mathrm{~S}=\text { allowable stress, } \mathrm{lb} / \mathrm{in}^{2} \text { ( } 50 \% \text { of the yield } \\
& \text { strength of steel, } 7,500 \mathrm{lb} / \mathrm{in}^{2} \text { for aluminum) } \\
& \mathrm{t}=\text { wall thickness, in } \\
& \mathrm{D}_{\mathrm{o}}=\text { outside pipe diameter, in }
\end{aligned}
$$

ASTM A 1011 Structural steel

| Grade 30 | 15,000 |
| :--- | :--- |
| Grade 33 | 16,500 |
| Grade 36 | 18,000 |
| Grade 40 | 20,000 |
| Grade 45 | 22,500 |
| Grade 50 | 25,000 |
| Grade 55 | 27,500 |

ASTM A 53

| Grade A | 15,000 |
| :--- | :--- |
| Grade B | 17,500 |

ASTM A 135

| Grade A | 15,000 |
| :--- | :--- |
| Grade B | 17,500 |

ASTM A 139

| Grade A | 15,000 |
| :--- | :--- |
| Grade B | 17,500 |
| Grade C | 21,000 |
| Grade D | 23,000 |
| Grade E | 26,000 |

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The stress in a metal pipe may be allowed to increase from 50 percent of the yield strength to 75 percent for surge pressures. Therefore, the internal pipe pressure for working pressure plus surge pressure may be 1.5 times the pressure rating determined above.

## (c) Corrugated metal

The maximum allowable pressure should be limited to 20 feet of head for annular pipe and 30 feet of head for helical pipe with lock or continuously welded seams, annular ends, and watertight couplings.

Corrugated bands (fig. 52-5) and gaskets (fig. 52-6) are necessary when watertightness is required. The ends of helical pipe should be reformed so the pipe may be

Figure 52-5 Standard band types



Semi-corrugated



Flat


Hat
coupled. Flat bands with sleeve or O-ring type gaskets, or hat/channel with mastic bands (fig. 52-5) are not considered watertight joints since they are susceptible to pulling apart. Bands with annular corrugations and rod and lug connectors, a band angle connector (fig. $52-7$ ), or flanged connections are acceptable watertight couplings.

Figure 52-6 Standard corrugated pipe gaskets


O-ring gasket


Sleeve gasket


Strip gasket

Figure 52-7 Corrugated pipe watertight connectors


Band angle connector


Rod and lug

## (d) Ductile iron pipe

The net thickness for internal pressure (static pressure plus surge pressure) may be determined from the following formula:

$$
\begin{equation*}
\mathrm{t}=\left(\frac{\mathrm{P} \times \mathrm{D}_{\mathrm{o}}}{2 \times \mathrm{S}_{\mathrm{y}}}\right) \tag{52-6}
\end{equation*}
$$

where:
$\mathrm{t}=$ net pipe wall thickness, in
$\mathrm{P}=$ internal pressure, $\mathrm{lb} / \mathrm{in}^{2}$ $\mathrm{P}=2.0\left(\mathrm{P}_{\text {work }}+\mathrm{P}_{\text {surge }}\right)$ or static pressure
$\mathrm{P}_{\text {work }}=$ working pressure, $\mathrm{lb} / \mathrm{in}^{2}$
$\mathrm{P}_{\text {surge }}=$ maximum surge pressure, $\mathrm{lb} / \mathrm{in}^{2}$
$\mathrm{D}_{\mathrm{o}}=$ outside pipe diameter, in
$\mathrm{S}_{\mathrm{y}}=$ yield strength $\left(42,000 \mathrm{lb} / \mathrm{in}^{2}\right.$ for ductile iron $)$
The standard surge allowance for ductile iron pipe is $100 \mathrm{lb} / \mathrm{in}^{2}$. The pressure class designation signifies the allowable working pressure with a maximum surge pressure of $100 \mathrm{lb} / \mathrm{in}^{2}$. If the anticipated surge pressure is different from $100 \mathrm{lb} / \mathrm{in}^{2}$, the anticipated surge pressure should be used and the working pressure adjusted accordingly.

Once the net pipe wall thickness is determined, an 0.08 -inch service tolerance and the casting tolerance from appendix 52 F , table $52 \mathrm{~F}-1$, are added to calculate the thickness, from which the appropriate pressure class is chosen.

### 636.5202 Water hammer/ surge pressure

Water hammer (or surge pressure) occurs when the flow velocity in a pipe system is suddenly stopped or changed. When flow is suddenly changed, the mass inertia of the water is converted into a pressure wave or high static head on the pressure side of the pipeline. Some of the most common causes of water hammer are the opening and closing of valves, starting and stopping pumps, entrapped air, and poor pipe system layout.

For detailed surge analysis and to analyze flow changes other than instantaneous stoppage, a computer analysis is recommended. SURGE is one available computer program.

Surges may generally be divided into two categories: transient surges and cyclic surges. Transients are described as the intermediate conditions that exist in a system as it moves from one steady-state condition to another. Cyclic surging is a condition that recurs regularly with time. Surging of this type is often associated with the action of equipment, such as reciprocating pumps, pressure reducing valves, and float valves. Any piping material may eventually fatigue if exposed to continuous cyclic surging at sufficiently high frequency and stress.

Recurring surge pressures occur frequently and are inherent to the design and operation of the system (such as normal pump startup or shutdown and normal valve opening and closure).

Occasional surge pressures are caused by emergency operations and are usually the result of a malfunction, such as power failure or system component failure, which includes pump seize-up, valve-stem failure, and pressure-relief valve failure.

The pressure wave caused by the water hammer travels back and forth in the pipe getting progressively lower with each transition from end to end. The magnitude of the pressure change caused by the water hammer wave depends on the elastic properties of the

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pipe and liquid, as well as the magnitude and speed of the velocity change. The maximum surge pressure from water hammer is equal to:

$$
\begin{gather*}
\mathrm{H}_{\text {surge }}=\frac{\mathrm{a} \times \Delta \mathrm{V}}{\mathrm{~g}}  \tag{52-7}\\
\text { or } \\
\mathrm{P}_{\text {surge }}=\frac{\mathrm{a} \times \Delta \mathrm{V}}{\mathrm{~g}} \times \frac{\gamma_{\mathrm{w}}}{144}  \tag{52-8}\\
\text { or } \\
\mathrm{P}_{\text {surge }}=\frac{\mathrm{a} \times \Delta \mathrm{V}}{2.31 \times \mathrm{g}} \text { (for water) } \tag{52-9}
\end{gather*}
$$

where:

| $\mathrm{H}_{\text {surge }}=$ surge pressure, ft of water |
| :--- |
| $\mathrm{P}_{\text {surge }}=$ surge pressure, $\mathrm{lb} / \mathrm{in}^{2}$ |
| a |
| $\Delta \mathrm{V} \quad=$ velocity of the pressure wave, $\mathrm{ft} / \mathrm{s}$ |
| V |
| g |
| $\gamma_{\mathrm{w}}$ |$\quad=$ change in velocity of fluid, $\mathrm{ft} / \mathrm{s}$.

The maximum surge pressure results when the time required to stop or change the flow velocity is equal to or less than 2L/a such that:

$$
\begin{equation*}
\mathrm{T}_{\mathrm{CR}} \leq \frac{2 \mathrm{~L}}{\mathrm{a}} \tag{52-10}
\end{equation*}
$$

where:
$\mathrm{T}_{\mathrm{CR}}=$ critical time, seconds
$\mathrm{L}=$ distance within the pipeline that the pressure wave moves before it is reflected back by a boundary condition, ft
$\mathrm{a}=$ velocity of the pressure wave, $\mathrm{ft} / \mathrm{s}$
The velocity of the pressure wave, a, may be expressed as:

$$
\begin{gather*}
a=\frac{12 \times \sqrt{\frac{\mathrm{K}_{\mathrm{L}}}{\rho}}}{\sqrt{1+\frac{\mathrm{K}_{\mathrm{L}}}{\mathrm{E}} \times \frac{\mathrm{D}_{\mathrm{i}}}{\mathrm{t}}}}  \tag{52-11}\\
\mathrm{a}=\frac{12}{\sqrt{\frac{\gamma_{\mathrm{w}}}{\mathrm{~g}}\left(\frac{1}{\mathrm{~K}_{\mathrm{L}}}+\frac{\mathrm{D}_{\mathrm{i}}}{\mathrm{Et}}\right)}}
\end{gather*}
$$

or

$$
\begin{equation*}
a=\frac{4,720}{\sqrt{1+\frac{\mathrm{K}_{\mathrm{L}}}{\mathrm{E}} \times \frac{\mathrm{D}_{\mathrm{i}}}{\mathrm{t}}}} \text { (for water) } \tag{52-13}
\end{equation*}
$$

For SDR pipe, the velocity of the pressure wave may be expressed as:

$$
\begin{equation*}
a=\frac{12 \times \sqrt{\frac{\mathrm{K}_{\mathrm{L}}}{\rho}}}{\sqrt{1+\frac{\mathrm{K}_{\mathrm{L}}(\mathrm{SDR}-2)}{\mathrm{E}}}} \tag{52-14}
\end{equation*}
$$

$$
\begin{equation*}
a=\frac{12}{\sqrt{\frac{\gamma_{\mathrm{w}}}{\mathrm{~g}}\left(\frac{1}{\mathrm{~K}_{\mathrm{L}}}+\frac{\mathrm{SDR}-2}{\mathrm{E}}\right)}} \tag{52-15}
\end{equation*}
$$

$$
\begin{equation*}
a=\frac{4,720}{\sqrt{1+\frac{\mathrm{K}_{\mathrm{L}}(\mathrm{SDR}-2)}{\mathrm{E}}}} \text { (for water) } \tag{52-16}
\end{equation*}
$$

where:
$\mathrm{K}_{\mathrm{L}}=$ bulk modulus of liquid, $\mathrm{lb} / \mathrm{in}^{2}$ $=300,000 \mathrm{lb} / \mathrm{in}^{2}$ for water
$\mathrm{E}=$ modulus of elasticity of pipe material, $\mathrm{lb} / \mathrm{in}^{2}$ (as shown below)
SDR = standard dimension ratio
$\rho \quad=$ density of fluid, slugs/ft ${ }^{3}$
$=1.93$ slugs $/ \mathrm{ft}^{3}$ for water
$\gamma_{\mathrm{w}}=$ unit weight of water, $62.4 \mathrm{lb} / \mathrm{ft}^{3}$
$\mathrm{g} \quad=$ acceleration due to gravity, $32.2 \mathrm{ft}^{2} / \mathrm{s}$
$\mathrm{D}_{\mathrm{i}}=$ internal diameter of the pipe, in
t = pipe wall thickness, in

| Material | Modulus of elasticity* <br> $\left(\mathrm{lb} / \mathrm{in}^{2}\right)$ |
| :--- | ---: |
| Steel | $29,000,000$ |
| Aluminum | $10,000,000$ |
| Ductile Iron | $24,000,000$ |
| PVC | 400,000 (short term) |
| ABS | 300,000 (short term) |
| Polyethylene | 110,000 (short term) |
| *Short-term modulus of elasticity varies with the cell <br> class of each plastic. Specific values may be obtained <br> from the manufacturer. |  |

### 636.5203 Loads on pipe

## (a) Soil pressure

The soil pressure above flexible pipe is determined by the soil prism load theory (fig. 52-8). The soil pressure may be determined by the following equation:

$$
\begin{equation*}
\mathrm{P}_{\mathrm{s}}=\gamma_{\mathrm{s}} \times \mathrm{h} \tag{52-17}
\end{equation*}
$$

where:
$P_{S}=$ pressure due to weight of soil at depth of $h$, $\mathrm{lb} / \mathrm{ft}^{2}$
$\gamma_{\mathrm{s}}=$ unit weight of soil, $\mathrm{lb} / \mathrm{ft}^{3}$
$\mathrm{h}=$ height of ground surface above top of pipe, ft
When groundwater is above the top of the pipe, $\mathrm{P}_{\mathrm{s}}$ may be reduced for buoyancy by the factor, $R_{w}$ :

$$
\begin{aligned}
\mathrm{R}_{\mathrm{w}} & =\text { water buoyancy factor } \\
& =1-0.33 \mathrm{~h}_{\mathrm{w}} / \mathrm{h}
\end{aligned}
$$

where:
$h=$ height of ground surface above top of pipe, ft
$h_{w}=$ height of water above top of pipe, ft
The soil load per foot length of pipe may be determined by:

$$
\begin{equation*}
\mathrm{W}_{\mathrm{s}}=\mathrm{P}_{\mathrm{s}} \times \frac{\mathrm{D}_{\mathrm{o}}}{12} \tag{52-18}
\end{equation*}
$$

Figure 52-8 Soil prism

where:
$\mathrm{W}_{\mathrm{s}}=$ soil load per linear foot of pipe, $\mathrm{lb} / \mathrm{ft}$
$D_{o}=$ outside diameter of pipe, in

## (b) Wheel loading

Underground pipes may be subjected to vehicular loads. The use of actual wheel/track loads is recommended. The magnitude of the wheel load may be estimated from the following:

| Load class | $\mathrm{P}_{\mathrm{L}}, \mathrm{lb}$ |
| :--- | :---: |
| Field equipment | 10,000 |
| H15 | 12,000 |
| H20 | 16,000 |

The effect of wheel loads at the surface reduces significantly with depth. When the wheel load is large, such as 20,000 pounds, the possibility of a similar load within a distance equal to the depth of consideration should be evaluated using special analysis.

The pressure distribution is based on the stress distribution theory (fig. 52-9) and may be expressed as follows:

When $\mathrm{D}_{\mathrm{o}}-\mathrm{t}<2.67 \mathrm{~h} \times 12$ :
$\mathrm{W}_{\mathrm{L}}=\frac{0.48 \mathrm{P}_{\mathrm{L}} \mathrm{I}_{\mathrm{f}}\left(\frac{\mathrm{D}_{\mathrm{o}}-\mathrm{t}}{12}\right)^{2}}{2.67 \mathrm{~h}^{3}}\left[\frac{2.67 \mathrm{~h}}{\left(\frac{\mathrm{D}_{\mathrm{o}}-\mathrm{t}}{12}\right)}-0.5\right]$

When $\mathrm{D}_{\mathrm{o}}-\mathrm{t} \geq 2.67 \mathrm{~h} \times 12$ :

$$
\begin{equation*}
\mathrm{W}_{\mathrm{L}}=\frac{0.64 \mathrm{P}_{\mathrm{L}} \mathrm{I}_{\mathrm{f}}}{\mathrm{~h}} \tag{52-20}
\end{equation*}
$$

where:
$\mathrm{W}_{\mathrm{L}}=$ wheel load per linear foot of pipe, lb/ft
$P_{L}=$ wheel load at the surface, lb
$\mathrm{I}_{\mathrm{f}}=$ impact factor (as described below)
$h=$ height of ground surface above top of pipe, ft
$D_{o}=$ outside diameter of pipe, in
t = pipe wall thickness, in

| Depth of cover | Impact factor |
| :--- | :---: |
| $<1^{\prime} 0^{\prime \prime}$ | 1.3 |
| $1^{\prime} 1^{\prime \prime}-2^{\prime} 0^{\prime \prime}$ | 1.2 |
| $2^{\prime} 0^{\prime \prime}-2^{\prime} 11^{\prime \prime}$ | 1.1 |
| $\geq 3^{\prime} 0^{\prime \prime}$ | 1.0 |

Figure 52-9 Load pressure distribution

(a) $\mathrm{D}_{\mathrm{o}}$-t $<2.67 \mathrm{~h} \times 12$


The pressure on the pipe from the wheel load may be determined by:

$$
\begin{equation*}
\mathrm{P}_{\mathrm{w}}=\frac{\mathrm{W}_{\mathrm{L}}}{\left(\frac{\mathrm{D}_{\mathrm{o}}}{12}\right)} \tag{52-21}
\end{equation*}
$$

where:
$\mathrm{P}_{\mathrm{w}}=$ pressure on pipe from wheel load, $\mathrm{lb} / \mathrm{ft}^{2}$
$D_{o}=$ outside diameter of pipe, in
When the depth of fill is 2 feet or more, wheel loads may be considered as uniformly distributed over a square with sides equal to $13 / 4$ times the depth of fill.

$$
\begin{equation*}
\mathrm{P}_{\mathrm{w}}=\frac{\mathrm{P}_{\mathrm{L}}}{(1.75 \mathrm{~h})^{2}} \tag{52-22}
\end{equation*}
$$

## (c) Vacuum pressure

Pipe may be subject to an effective external pressure because of an internal vacuum pressure, $\mathrm{P}_{\mathrm{v}}$ Sudden valve closures, shutoff of a pump, or drainage from high points within the system often create a vacuum in pipelines. Siphons will all be subject to negative pressures.

Vacuum pressure should be incorporated into the design of buried and aboveground pipes as described in this chapter. The vacuum pressure may be intermittent (short term), for long durations, or continuously (long term).

The vacuum load per length of pipe may be determined by:

$$
\begin{equation*}
\mathrm{W}_{\mathrm{v}}=\mathrm{P}_{\mathrm{v}} \times \frac{\mathrm{D}_{\mathrm{i}}}{12} \tag{52-23}
\end{equation*}
$$

where:
$\mathrm{W}_{\mathrm{v}}=$ vacuum load per linear foot of pipe, lb/ft
$\mathrm{P}_{\mathrm{v}}=$ internal vacuum pressure, $\mathrm{lb} / \mathrm{ft}^{2}$
$\mathrm{D}_{\mathrm{i}}=$ inside pipe diameter, in

## (d) Hydrostatic pressure

Pipe may be subject to external hydrostatic pressure if it is below the water elevation. The hydrostatic pressure may be determined by the following equation:

$$
\begin{equation*}
\mathrm{P}_{\mathrm{G}}=\gamma_{\mathrm{w}} \times \mathrm{h}_{\mathrm{w}} \tag{52-24}
\end{equation*}
$$

where:
$\mathrm{P}_{\mathrm{G}}=$ external hydrostatic pressure, $\mathrm{lb} / \mathrm{ft}^{2}$
$\gamma_{\mathrm{w}}=$ unit weight of water, $\mathrm{lb} / \mathrm{ft}^{3}$
$\mathrm{h}_{\mathrm{w}}=$ height of water above top of pipe, ft

### 636.5204 Buried pipe design

The typical modes of failure of buried flexible pipe include wall crushing (stress), local buckling, or excessive deflection (fig. 52-10).

Excessive wall stress may lead to wall crushing if the compressive strength of the pipe wall is exceeded.

Buckling may occur because of insufficient pipe stiffness and may control design for pipes subject to internal vacuum, external hydrostatic pressure, or pipe embedded in loose or poorly compacted soil.

Deflection of flexible pipe is a performance limit to prevent cracking of liners, avoid reversal of curvature, limit bending stress and strain, and avoid pipe flattening. Deflection of a nonpressure flexible pipe increases with time after construction is complete. The time is a function of the embedment and surrounding soil density. The deflection continues to increase as long as the soil around the pipe continues to consolidate (increase in density). A deflection lag factor, $\mathrm{D}_{\mathrm{L}}$, was included in the modified Iowa equation to account for the increase in deflection with time. $\mathrm{A}_{\mathrm{L}}$ value of 1.0 to 1.5 is often recommended. $\mathrm{A}_{\mathrm{L}}$ value of 1.0 is often used when the soil load is estimated by the soil prism load as illustrated in figure 52-8. A $\mathrm{D}_{\mathrm{L}}$ value of 1.5 has historically been used by the NRCS and is recommended as the factor to be applied to only the soil load.

Figure 52-10 Modes of failure


## (a) Plastic pipe

Plastic pipe materials consist of poly-vinyl chloride (PVC), acrylonitrile-butadiene-styrene (ABS), and polyethylene (PE). Each type of material is supplied in several grades as shown in appendix 52C.

Design of buried plastic pipe includes analyses of the wall crushing, buckling resistance, allowable longterm deflection, and allowable strain.

At a constant load, the plastic modulus of elasticity of the plastic pipe decreases with time. With any increase in load, the plastic reacts with the short-term modulus of elasticity. The ratio of the short-term to long-term modulus of elasticity varies from approximately 3 for PVC to 5 for PE. The short-term modulus of elasticity is recommended for conditions that change through time, such as deflection. The pipe-soil interaction that occurs as discrete events is similar to a new load (Chevron Chemical, 1998). The long-term modulus of elasticity is often recommended for buckling since the loads and reaction of the pipe are considered static.

## (1) Wall crushing

The design pressure and ring compression thrust in the pipe wall is determined by:

$$
\begin{equation*}
P=P_{s}+P_{w}+P_{v} \tag{52-25}
\end{equation*}
$$

where:
$\mathrm{P}=$ pressure on pipe, $\mathrm{lb} / \mathrm{ft}^{2}$
$P_{s}=$ pressure due to weight of soil, $\mathrm{lb} / \mathrm{ft}^{2}$
$\mathrm{P}_{\mathrm{w}}^{\mathrm{s}}=$ pressure on pipe due to wheel load, $\mathrm{lb} / \mathrm{ft}^{2}$
$\mathrm{P}_{\mathrm{v}}=$ internal vacuum pressure, $\mathrm{lb} / \mathrm{ft}^{2}$

$$
\begin{equation*}
\mathrm{T}_{\mathrm{pw}}=\frac{\mathrm{P} \times \frac{\mathrm{D}_{\mathrm{o}}}{12}}{2} \tag{52-26}
\end{equation*}
$$

where:
$\mathrm{T}_{\mathrm{pw}}=$ thrust in pipe wall, $\mathrm{lb} / \mathrm{ft}$
$\mathrm{D}_{\mathrm{o}}=$ outside pipe diameter, in

The required wall cross-sectional area is determined by:

$$
\begin{equation*}
\mathrm{A}_{\mathrm{pw}}=\frac{\frac{\mathrm{T}_{\mathrm{pw}}}{12}}{\sigma} \tag{52-27}
\end{equation*}
$$

where:

$$
\begin{aligned}
\mathrm{A}_{\mathrm{pw}}= & \text { required wall area, in }{ }^{2} / \mathrm{in} \\
\mathrm{~T}_{\mathrm{pw}}= & \text { thrust in pipe wall, } \mathrm{lb} / \mathrm{ft} \\
\sigma= & \text { allowable long-term compressive stress, } \\
& \left.\mathrm{lb} / \mathrm{in}^{2} \text { (see appendix } 52 \mathrm{C}, \text { table } 52 \mathrm{C}-1\right)
\end{aligned}
$$

The area of a solid-wall pipe wall may be computed as:

$$
\begin{equation*}
\mathrm{A}_{\mathrm{pw}}=\frac{\left(\mathrm{D}_{\mathrm{o}}-\mathrm{D}_{\mathrm{i}}\right)}{2} \text { or } \mathrm{t} \tag{52-28}
\end{equation*}
$$

where:
$\mathrm{A}_{\mathrm{pw}}=$ area of pipe wall, $\mathrm{in}^{2} / \mathrm{in}$
$\mathrm{D}_{\mathrm{o}}=$ outside pipe diameter, in
$\mathrm{D}_{\mathrm{i}}=$ inside pipe diameter, in
t = pipe wall thickness, in
The average area of pipe wall for corrugated and profile wall pipe should be obtained from the manufacturer.

## (2) Deflection

The Modified Iowa Equation may be transposed and rewritten to compute the percent deflection of each type of pipe. The properties of a pipe section are expressed as the standard dimension ratio (SDR) or standard inside dimension ratio (SIDR) for solid wall pipe, pipe stiffness (PS) for corrugated plastic pipe, and the ring stiffness constant (RSC) for profile wall pipe.

Solid-wall plastic pipe as:

$$
\frac{\% \Delta X}{D}=\frac{\left(\mathrm{D}_{\mathrm{L}} \mathrm{P}_{\mathrm{S}}+\mathrm{P}_{\mathrm{w}}+\mathrm{P}_{\mathrm{v}}\right)\left(\frac{1}{144}\right) \mathrm{K}(100)}{\left[\left(\frac{2 \mathrm{E}}{3(\mathrm{SDR}-1)^{3}}\right)+0.061 \mathrm{E}^{\prime}\right]}
$$

or

$$
\begin{equation*}
\frac{\% \Delta \mathrm{X}}{\mathrm{D}}=\frac{\left(\mathrm{D}_{\mathrm{L}} \mathrm{P}_{\mathrm{S}}+\mathrm{P}_{\mathrm{w}}+\mathrm{P}_{\mathrm{v}}\right)\left(\frac{1}{144}\right) \mathrm{K}(100)}{\left\lceil\left(\frac{2 \mathrm{E}}{3(\operatorname{SIDR}+1)^{3}}\right)+0.061 \mathrm{E}^{\prime}\right\rceil} \tag{52-30}
\end{equation*}
$$

Corrugated-plastic pipe as:

$$
\begin{equation*}
\frac{\% \Delta X}{D}=\frac{\left(D_{\mathrm{L}} \mathrm{P}_{\mathrm{S}}+\mathrm{P}_{\mathrm{w}}+\mathrm{P}_{\mathrm{V}}\right)\left(\frac{1}{144}\right) \mathrm{K}(100)}{\left[0.149 \mathrm{PS}+0.061 \mathrm{E}^{\prime}\right]} \tag{52-31}
\end{equation*}
$$

Profile-wall pipe:
$\frac{\% \Delta X}{D}=\frac{\left(\mathrm{D}_{\mathrm{L}} \mathrm{P}_{\mathrm{S}}+\mathrm{P}_{\mathrm{w}}+\mathrm{P}_{\mathrm{v}}\right)\left(\frac{1}{144}\right) \mathrm{K}(100)}{\left[\left(\frac{1.24(\mathrm{RSC})}{\mathrm{D}_{\mathrm{i}}}\right)+0.061 \mathrm{E}^{\prime}\right]}$
where:

| X |  |
| :---: | :---: |
| D | percent deflection |
| $\mathrm{D}_{\mathrm{L}}$ | $=$ deflection lag factor (1.0 to 1.5) |
| K | $=$ bedding constant (0.1) |
| $\mathrm{P}_{\text {s }}$ | $=$ pressure on pipe from soil ( $\mathrm{lb} / \mathrm{ft}^{2}$ ) |
| $\mathrm{P}_{\text {w }}$ | $=$ pressure on pipe from wheel load ( $\mathrm{lb} / \mathrm{ft}^{2}$ ) |
| $\mathrm{P}_{\mathrm{v}}$ | $=$ internal vacuum pressure ( $\mathrm{lb} / \mathrm{ft}^{2}$ ) |
| E | $=$ modulus of elasticity of pipe material (as shown below) |
| SDR |  |
| SIDR | $\begin{aligned} & =\mathrm{D}_{\mathrm{i}} \text { dimension ratio } \\ & \quad \text { SIDR }=\mathrm{D}_{\mathrm{i}} / \mathrm{t} \\ & \quad \mathrm{D}_{\mathrm{i}}=\text { pipe inside diameter, in } \\ & \mathrm{t} \quad=\text { minimum wall thickness, in } \end{aligned}$ |
| PS | $=$ pipe stiffness |
| RSC | $=$ ring stiffness constant |
| $\mathrm{D}_{\text {i }}$ | $=$ inside pipe diameter, in |
| $\mathrm{E}^{\prime}$ | $=\underset{52-2)}{\text { modulus }}$ of soil reaction, $\mathrm{lb} / \mathrm{in}^{2}$ (see table |


| Material | Modulus of elasticity* <br> (lb/in 2 ) |
| :--- | :---: |
| PVC | 400,000 (short term) |
| ABS | 300,000 (short term) |
| Polyethylene | 110,000 (short term) |
| * Short-term modulus of elasticity varies |  |
| with the cell class so each plastic. Specific <br> values may be obtained from the manufac- <br> turer. |  |

The modulus of soil reaction, $\mathrm{E}^{\prime}$, is an interactive modulus representing support of the soil in reaction to the lateral pipe deflection under load. Amster Howard of the Bureau of Reclamation (Howard, 1977) developed recommended $\mathrm{E}^{\prime}$ values based on the soil prism load described above. The recommended values are provided in table 52-2.

The allowable deflections for plastic pipe typically are limited to 5 percent for a spillway/outlet conduit in embankment dam practice and 7.5 percent in water or liquid conveyance practice and drains in embankment dam practice.

## (3) Wall buckling

Plastic pipe embedded in soil may buckle because of excessive loads and deformations. The total permanent pressure must be less than the allowable buckling pressure. The permanent load should consist of the soil pressure, groundwater pressure, and any internal long-term vacuum pressures. The allowable buckling pressure may be determined from:

$$
\begin{equation*}
\mathrm{q}_{\mathrm{a}}=\frac{1}{\mathrm{FS}}\left(32 \mathrm{R}_{\mathrm{w}} \mathrm{~B}^{\prime} \mathrm{E}^{\prime} \frac{\mathrm{E}_{\mathrm{long}} \mathrm{I}_{\mathrm{pw}}}{\mathrm{D}_{\mathrm{o}}{ }^{3}}\right)^{1 / 2} \tag{52-33}
\end{equation*}
$$

(Moser, 2001)
where:
$\mathrm{q}_{\mathrm{a}}=$ allowable buckling pressure, $\mathrm{lb} / \mathrm{in}^{2}$
FS = design factor of safety
$=2.5$ for $\left(\mathrm{h} /\left(\mathrm{D}_{\mathrm{o}} / 12\right)>2\right.$
$=3.0$ for $\left(\mathrm{h} /\left(\mathrm{D}_{\mathrm{o}} / 12\right)<2\right.$
where:
$h=$ height of ground surface above top of pipe, ft
$\mathrm{D}_{\mathrm{o}}=$ outside diameter of the pipe, in
$\mathrm{R}_{\mathrm{w}}=$ water buoyancy factor
$=1-0.33\left(\mathrm{~h}_{\mathrm{w}} / \mathrm{h}\right), 0<\mathrm{h}_{\mathrm{w}}<\mathrm{h}$
where:
$h$ = height of ground surface above top of pipe, ft
$h_{w}=$ height of water above top of pipe, ft
$B^{\prime}=$ empirical coefficient of elastic support

$$
=\frac{4\left(\mathrm{~h}^{2}+\left(\frac{\mathrm{D}_{0}}{12}\right) \mathrm{h}\right)}{1.5\left(2 \mathrm{~h}+\left(\frac{\mathrm{D}_{0}}{12}\right)\right)^{2}}
$$

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$\mathrm{E}_{\text {long }}=$ long term modulus of elasticity, $\mathrm{lb} / \mathrm{in}^{2}$
(see table below)
The long term modulus of elasticity is recommended if the pipe is subject to the pressure in the normal operations. If the pipe is subject to the pressure for short time periods and infrequently, the use of the short-term modulus of elasticity is acceptable.
$\mathrm{E}^{\prime}=$ modulus of soil reaction, $\mathrm{lb} / \mathrm{in}^{2}$ (table 52-2)
$\mathrm{I}_{\mathrm{pw}}=$ pipe wall moment of inertia

| $=\frac{\mathrm{t}^{3}}{12}, \mathrm{in}^{4} / \mathrm{in}$ <br> (for solid wall pipe) | where: <br> C | $=$ reduction factor for buckling pressure |
| :---: | :---: | :---: |
| where: | \% 4 X |  |
| $\mathrm{D}_{\mathrm{o}}=$ outside pipe diameter, in | D | $=$ percent deflection |

Pipes that are out-of-round or deflected increase in bending moment and have less allowable buckling pressure. The allowable buckling pressure should be reduced by the following factor:

$$
\begin{equation*}
\mathrm{C}=\left[\frac{\left(1-\frac{\% \Delta \mathrm{X}}{\mathrm{D}} \frac{1}{100}\right)}{\left(1+\frac{\% \Delta \mathrm{X}}{\mathrm{D}} \frac{1}{100}\right)^{2}}\right]^{3} \tag{52-34}
\end{equation*}
$$

where:
C = reduction factor for buckling pressure
\% $\Delta \mathrm{X}$
D = percent deflection

Material Modulus of elasticity* $\left(\mathrm{lb} / \mathrm{in}^{2}\right)$
PVC 140,000 (long term)
ABS 65,000 (long term)
Polyethylene 22,000 (long term)

* Long-term modulus of elasticity varies with the cell class of each plastic. Specific values may be obtained from the manufacturer.

Table 52-2 Average values of the modulus of soil reaction for the Modified Iowa Equation

| Soil type - pipe bedding material (Unified Soil Classification - ASTM D2487) | -------- E' for degree of compaction of bedding, $\mathrm{lb} / \mathrm{in}^{21 /}$ - -- -- -- |  |  |  |
| :---: | :---: | :---: | :---: | :---: |
|  | Dumped | Slight, <br> < 85\% proctor, <br> < 40\% relative density | Moderate, 85-95\% proctor, 40-70\% relative density | High, <br> $>95 \%$ proctor, <br> $>70 \%$ relative density |
| Fine-grained soil (LL>50) ${ }^{2 /}$ Soil with medium to high plasticity CH, MH, CH-MH | No data available, use $\mathrm{E}^{\prime}=0$ or consult with a geotechnical engineer |  |  |  |
| Fine-grained soil ( $L L<50$ ) soil with medium to no plasticity CL, ML, ML-CL, with less than $25 \%$ coarsegrained particles | 50 | 200 | 400 | 1,000 |
| Fine-grained soil ( $L L<50$ ) soil with medium to no plasticity CL, ML, ML-CL, with more than $25 \%$ coarsegrained particles. Coarse-grained soil with fines GM, GC, SM, SC contains more than $12 \%$ fines | 100 | 400 | 1,000 | 2,000 |
| Coarse-grained soil with little or no fines GW, GP, SW, SP contains less than $12 \%$ fines | 200 | 1,000 | 2,000 | 3,000 |
| Crushed rock | 1,000 | 3,000 | 3,000 | 3,000 |

[^0]
## (4) Strain

Total strain in a pipe wall can be caused by two actions: (1) flexure of the pipe as it deforms, and (2) hoop stress caused by internal or external pressure in the pipe wall. If a homogeneous wall is assumed and pressure concentrations are neglected, the formula follows:

Hoop strain:

$$
\begin{equation*}
\varepsilon_{\mathrm{h}}=\frac{\frac{\mathrm{P}}{144} \mathrm{D}}{2 \mathrm{~A}_{\mathrm{pw}} \mathrm{E}} \tag{52-35}
\end{equation*}
$$

For solid wall pipe, the equation becomes:

$$
\begin{equation*}
\varepsilon_{\mathrm{h}}=\frac{\frac{\mathrm{P}}{144} \mathrm{D}_{\mathrm{M}}}{2 \mathrm{tE}} \tag{52-36}
\end{equation*}
$$

where:
$\varepsilon_{\mathrm{h}}=$ maximum strain in pipe wall because of ring bending, in/in
P = pressure on/in pipe (may be internal and/or external pressure with the appropriate sign), $\mathrm{lb} / \mathrm{ft}^{2}$
$\mathrm{D}_{\mathrm{M}}=$ mean pipe diameter, in
$\mathrm{A}_{\mathrm{pw}}=$ area of pipe wall, $\mathrm{in}^{2} / \mathrm{in}$
$\mathrm{E}=$ modulus of elasticity of the pipe material, $\mathrm{lb} / \mathrm{in}^{2}$
t = pipe wall thickness, in
Maximum strains because of deflection or flexure may be determined by assuming the pipe remains an ellipse during deflections. The resulting equations are:

$$
\begin{align*}
& \begin{array}{l}
\varepsilon_{f}=\frac{t}{D_{M}}\left(\frac{\frac{3 \Delta Y}{D_{M}}}{1-2 \frac{\Delta Y}{D_{M}}}\right)=\frac{1}{\operatorname{SDR}}\left(\frac{\frac{3 \Delta Y}{D_{M}}}{1-2 \frac{\Delta Y}{D_{M}}}\right) \\
\text { or (solid wall pipe) } \\
\varepsilon_{f}=6 \frac{\mathrm{t}}{D_{M}} \frac{\Delta Y}{D_{M}} \\
\text { (corrugated or profile wall pipe) }
\end{array} \\
& \text { (52-38) } \\
& \begin{array}{l}
\text { where: } \\
\varepsilon_{f} \quad=\begin{array}{c}
\text { maximum strain in pipe wall because of ring } \\
\text { deflection, in/in }
\end{array} \\
\Delta Y \quad=\text { vertical decrease in diameter, in }
\end{array}
\end{align*}
$$

$$
\begin{aligned}
& \mathrm{D}_{\mathrm{M}}=\text { mean pipe diameter, in } \\
& \Delta \mathrm{Y} / \mathrm{D}_{\mathrm{M}}=\Delta \mathrm{X} / \mathrm{D}=\text { percent deflection ex- } \\
& \text { pressed as a decimal }
\end{aligned},
$$

In a buried pipeline, these strain components act simultaneously. The maximum combined strain in the pipe wall can be determined by summing both components.

$$
\begin{equation*}
\varepsilon=\varepsilon_{\mathrm{f}} \pm \varepsilon_{\mathrm{h}} \tag{52-39}
\end{equation*}
$$

where:
$\varepsilon=$ maximum combined strain in pipe wall, in/in
In calculating the maximum combined strain, the hoop strain, $\varepsilon_{\mathrm{h}}$, resulting from applied internal pressure, if any, should be added to the maximum strain due to deflection, $\varepsilon_{\mathrm{f}}$. If the hoop strain is due to external load or internal vacuum pressure, the ring hoop strain should be substracted to obtain the maximum combined strain, $\varepsilon$.

The maximum combined strain in the pipe should be limited to:

$$
\begin{equation*}
\varepsilon \leq \varepsilon_{\text {all }} \tag{52-40}
\end{equation*}
$$

where:
$\varepsilon_{\text {all }}=$ allowable strain for the pipe material
The allowable strain should be no more than 5 percent for polyethylene and ABS pipes.

The allowable deflection for PVC pipe limits strain in standard PVC pipes to an acceptable value. Therefore, computation of strain and comparison to an allowable strain limit is not required for PVC pipe.

In polyethylene pressure pipe with pressure near the pipe pressure rating, the strain may be limited by limiting the deflection to the values shown in table 52-3.

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## (b) Steel

Design of steel pipe includes an analysis of the deflection and the buckling pressure.

## (1) Deflection

The Modified Iowa Equation may be used to compute the deflection as:

$$
\begin{equation*}
\Delta \mathrm{X}=\left(\frac{\left(\mathrm{D}_{\mathrm{L}} \mathrm{~W}_{\mathrm{S}}+\mathrm{W}_{\mathrm{L}}+\mathrm{W}_{\mathrm{V}}\right)\left(\frac{1}{12}\right) \mathrm{Kr}^{3}}{\mathrm{EI}_{\mathrm{pw}}+0.061 \mathrm{E}^{\prime} \mathrm{r}^{3}}\right) \tag{52-41}
\end{equation*}
$$

where:

```
\(\Delta \mathrm{X}=\) deflection, in
\(\mathrm{D}_{\mathrm{L}} \quad=\) deflection lag factor (1.0 to 1.5)
\(\mathrm{W}_{\mathrm{s}} \quad=\) soil load per linear foot of pipe, \(\mathrm{lb} / \mathrm{ft}\)
\(\mathrm{W}_{\mathrm{L}}=\) wheel load per linear foot of pipe, lb/ft
\(\mathrm{W}_{\mathrm{v}}\) = vacuum load per linear foot of pipe, \(\mathrm{lb} / \mathrm{ft}\)
\(\mathrm{K} \quad=\) bedding constant (0.1)
r = radius of pipe, in
\(E I_{p w}=\) pipe wall stiffness, in-lb*
    where:
    \(\mathrm{E}=\) modulus of elasticity (29,000, \(000 \mathrm{lb} / \mathrm{in}^{2}\)
        for steel and 4,000,000 lb/in² for cement
        mortar)
        \(I_{p w}=\) pipe wall moment of inertia \(=\frac{t^{3}}{12}\), in \(^{4} / \mathrm{in}\)
        \(\mathrm{t}=\) wall thickness, in
\(\mathrm{E}^{\prime}=\) modulus of soil reaction, lb/in \({ }^{2}\) (table 52-2)
```

* Under load, the individual elements; i.e., mortar lining, steel shell, and mortar coating. work together as laminated rings $\left(E_{S} I_{S}+E_{I} I_{I}+E_{C} I_{C}\right)$ - shell, lining, coating). Structurally, the combined elements increase the moment of inertia of the pipe section, above the shell alone, thus increasing its ability to resist loads. The pipe wall stiffness EI of these individual elements is additive. (AWWA 1995)

Table 52-3
Safe deflection of polyethylene pressure pipe

| SDR | Safe deflection <br> as \% of diameter |
| ---: | :---: |
| 32.5 | 8.5 |
| 26.0 | 7.0 |
| 21.0 | 6.0 |
| 17.0 | 5.0 |
| 13.5 | 4.0 |
| 11.0 | 3.0 |
| 9.0 | 2.5 |

Source: ASTM F 714

The percent deflection may be determined by:

$$
\begin{equation*}
\% \frac{\Delta \mathrm{X}}{\mathrm{D}}=\frac{\Delta \mathrm{X}}{\mathrm{D}_{\mathrm{o}}} \times 100 \tag{52-42}
\end{equation*}
$$

Allowable deflections for various lining and coating systems are

| Steel pipe | $=5$ percent |
| :--- | ---: |
| Flexible lined and coated steel pipe | $=5$ percent |
| Mortar-lined and flexible coated steel pipe | $=3$ percent |
| Mortar-lined and coated steel pipe | $=2$ percent |

## (2) Buckling

Steel pipe embedded in soil may buckle because of excessive loads and deformations. The total permanent pressure must be less than the allowable buckling pressure. The permanent pressure should consist of the soil pressure, hydrostatic pressure, and any longterm vacuum pressure. The allowable buckling pressure may be determined from:

$$
\begin{equation*}
\mathrm{q}_{\mathrm{a}}=\frac{1}{\mathrm{FS}}\left(32 \mathrm{R}_{\mathrm{w}} \mathrm{~B}^{\prime} \mathrm{E}^{\prime} \frac{\mathrm{EI}_{\mathrm{pw}}}{\mathrm{D}_{\mathrm{o}}^{3}}\right)^{1 / 2} \tag{52-43}
\end{equation*}
$$

where:
$\mathrm{q}_{\mathrm{a}}=$ allowable buckling pressure, $\mathrm{lb} / \mathrm{in}^{2}$
$\mathrm{FS}=$ design factor of safety
$=2.5$ for $\left(\mathrm{h} /\left(\mathrm{D}_{\mathrm{o}} / 12\right)>2\right.$
$=3.0$ for $\left(\mathrm{h} /\left(\mathrm{D}_{\mathrm{o}} / 12\right)<2\right.$
where:
$h=$ height of ground surface above top of the pipe, ft
$D_{o}=$ outside diameter of the pipe, in
$\mathrm{R}_{\mathrm{w}}=$ water buoyancy factor
$=1-0.33\left(\mathrm{~h}_{\mathrm{w}} / \mathrm{h}\right), 0<\mathrm{h}_{\mathrm{w}}<\mathrm{h}$
where:
$h=$ height of ground surface above top of the pipe, ft
$h_{w}=$ height of water above top of pipe, ft
$B^{\prime}=$ empirical coefficient of elastic support
$=\frac{1}{1+4 \mathrm{e}^{(-0.065 \mathrm{~h})}}$
(AWWA, 1989)
where:
$h=$ height of ground service above top of pipe, ft
$\mathrm{E}^{\prime}=$ modulus of soil reaction, $\mathrm{lb} / \mathrm{in}^{2}$ (table 52-2)
$\mathrm{E}=$ modulus of elasticity, $\mathrm{lb} / \mathrm{in}^{2}(29,000,000$ for steel)

$$
\begin{aligned}
& \mathrm{I}_{\mathrm{pw}}=\text { transverse moment of inertia }=\frac{\mathrm{t}^{3}}{12}, \mathrm{in}^{4} / \text { in } \\
& \quad \mathrm{t}=\text { pipe wall thickness, in } \\
& \mathrm{D}_{\mathrm{o}}=\text { outside pipe diameter, in }
\end{aligned}
$$

## (c) Corrugated and spiral rib metal pipe

Design of corrugated and spiral rib metal pipe includes analysis of the wall strength, buckling strength, seam strength, and handling stiffness. Section properties of corrugated and spiral rib metal pipe are included in appendix 52D.

The strength requirements may be determined by either the allowable stress design (ASD) method or the load and resistance factor design (LRFD) method. Both methods are presented in ASTM B 790 for corrugated aluminum pipe and ASTM A 796 for corrugated steel pipe. The ASD method is presented next.

## (1) Thrust

The design pressure and ring compression thrust in the pipe wall are determined by:

$$
\begin{equation*}
\mathrm{P}=\mathrm{P}_{\mathrm{s}}+\mathrm{P}_{\mathrm{w}}+\mathrm{P}_{\mathrm{v}} \tag{52-44}
\end{equation*}
$$

where:
$\mathrm{P}=$ design pressure, $\mathrm{lb} / \mathrm{ft}^{2}$
$\mathrm{P}_{\mathrm{s}}=$ pressure due to weight of soil, $\mathrm{lb} / \mathrm{ft}^{2}$
$\mathrm{P}_{\mathrm{w}}=$ pressure on pipe due to wheel load, $\mathrm{lb} / \mathrm{ft}^{2}$
$\mathrm{P}_{\mathrm{v}}=$ internal vacuum pressure, $\mathrm{lb} / \mathrm{ft}^{2}$

$$
\begin{equation*}
\mathrm{T}_{\mathrm{pw}}=\frac{\mathrm{P} \times\left(\frac{\mathrm{D}_{\mathrm{i}}}{12}\right)}{2} \tag{52-45}
\end{equation*}
$$

where:
$\mathrm{T}_{\mathrm{pw}}=$ thrust in pipe wall, lb/ft
$D_{i}=$ inside pipe diameter, in
The required wall cross-sectional area is determined by:

$$
\begin{equation*}
\mathrm{A}_{\mathrm{s}}=\frac{\mathrm{T}_{\mathrm{pw}} \times(\mathrm{FS})}{f_{\mathrm{y}}} \tag{52-46}
\end{equation*}
$$

where:
$\mathrm{A}_{\mathrm{s}}=$ required area of section, $\mathrm{in}^{2} / \mathrm{ft}$
$\mathrm{T}_{\mathrm{pw}}=$ thrust in pipe wall, lb/ft

$$
\begin{aligned}
\mathrm{FS}= & \text { safety factor, } 2.0 \text { for wall area } \\
f_{\mathrm{y}}= & \text { minimum yield strength, } \mathrm{lb} / \mathrm{in}^{2} \\
& 33,000 \mathrm{lb} / \mathrm{in}^{2} \text { for steel } \\
& 24,000 \mathrm{lb} / \mathrm{in}^{2} \text { for aluminum } \\
& 20,000 \mathrm{lb} / \mathrm{in}^{2} \text { for aluminum alloy } 3004-\mathrm{H} 32
\end{aligned}
$$

## (2) Buckling

The selected corrugated pipe section with the required wall area shall be checked for possible buckling. If the critical buckling stress, $f_{\mathrm{c}}$, is less than the minimum yield stress, $f_{y}$, the required wall area must be recalculated using $f_{\mathrm{c}}$ instead of $f_{\mathrm{y}}$

When:

$$
\begin{equation*}
\mathrm{D}_{\mathrm{i}}<\frac{\mathrm{r}}{\mathrm{k}} \sqrt{\frac{24 \mathrm{E}}{f_{\mathrm{u}}}} \tag{52-47}
\end{equation*}
$$

$$
\begin{gather*}
f_{\mathrm{c}}=f_{\mathrm{u}}-\frac{f_{\mathrm{u}}{ }^{2}}{48 \mathrm{E}}\left(\frac{\mathrm{kD}_{\mathrm{i}}}{\mathrm{r}}\right)^{2}  \tag{52-48}\\
\mathrm{D}_{\mathrm{i}} \geq \frac{\mathrm{r}}{\mathrm{k}} \sqrt{\frac{24 \mathrm{E}}{f_{\mathrm{u}}}} \tag{52-49}
\end{gather*}
$$

where:
$\mathrm{D}_{\mathrm{i}}=$ inside pipe diameter, in
$r=$ radius of gyration of corrugation, in
$\mathrm{k}=$ soil stiffness factor $=0.22$ for good fill material
compacted to $90 \%$ of standard density based on
ASTM D 698 or $\phi>15^{\circ}$
$=0.44$ for soils with $\phi<15^{\circ}$ (Contech, 2001)
$\mathrm{E}=$ modulus of elasticity of pipe material, $\mathrm{lb} / \mathrm{in}^{2}$
$f_{\mathrm{u}}=$ minimum tensile strength of material, $\mathrm{lb} / \mathrm{in}^{2}$ $45,000 \mathrm{lb} / \mathrm{in}^{2}$ for steel $34,000 \mathrm{lb} / \mathrm{in}^{2}$ for aluminum
$27,000 \mathrm{lb} / \mathrm{in}^{2}$ for aluminum alloy $3004-\mathrm{H} 32$
$f_{\mathrm{c}}=$ critical buckling stress, $\mathrm{lb} / \mathrm{in}^{2}$
When:

$$
\begin{equation*}
f_{\mathrm{c}}=\frac{12 \mathrm{E}}{\left(\frac{\mathrm{kD}_{\mathrm{i}}}{\mathrm{r}}\right)^{2}} \tag{52-50}
\end{equation*}
$$

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## (3) Seam strength

For pipe fabricated with longitudinal seams (riveted, spot-welded, or bolted), the seam strength shall be sufficient to develop the thrust in the pipe wall. The required seam strength shall be:

$$
\begin{equation*}
\mathrm{SS}=\mathrm{T}_{\mathrm{pw}} \times \mathrm{FS} \tag{52-51}
\end{equation*}
$$

where:
$\mathrm{SS}=$ required seam strength, $\mathrm{lb} / \mathrm{ft}$
$\mathrm{T}_{\mathrm{pw}}=$ thrust in pipe wall, lb/ft
FS = safety factor, 3.0 for seam strength
Since helical lockseam and welded-seam pipe do not have longitudinal seams, seam strength criteria are not valid for these types of corrugated pipe.

## (4) Flexibility factor

The metal pipe must have sufficient stiffness to withstand temporary loads that occur during shipping, handling, and installation. Relationships referred to as the flexibility factor have been developed that relate the required pipe wall stiffness to the pipe diameter. The flexibility factor is determined as:

$$
\begin{equation*}
\mathrm{FF}=\frac{\mathrm{D}_{\mathrm{i}}^{2}}{\mathrm{EI}_{\mathrm{pw}}} \tag{52-52}
\end{equation*}
$$

where:
$\mathrm{FF}=$ flexibility factor, $\mathrm{in} / \mathrm{lb}$
$\mathrm{D}_{\mathrm{i}}=$ inside diameter of the pipe, in
$\mathrm{E}=$ modulus of elasticity of pipe material, $\mathrm{lb} / \mathrm{in}^{2}$
$\mathrm{I}_{\mathrm{pw}}=$ moment of inertia of pipe wall, in ${ }^{4} / \mathrm{in}$
The flexibility factor shall not exceed the allowable flexibility factors in appendix 52E.

## (d) Ductile iron

The required wall thickness for ductile iron pipe under external load is based on two design considerations: ring bending stress and ring deflection. Thicknesses for standard pressure classes are provided in appendix 52F.

## (1) Ring bending stress

The design ring bending stress, $f$, of 48,000 pounds per square inch provides a factor of safety of at least 1.5 on the minimum ring yield strength and 2.0 on the
ultimate ring strength. The pressure due to soil, wheel, and vacuum loads required to develop a bending stress of 48,000 pounds per square inch at the pipe invert may be determined by:

$$
\begin{equation*}
\mathrm{P}_{\mathrm{bs}}=\frac{f}{3\left(\frac{\mathrm{D}_{\mathrm{o}}}{\mathrm{t}}\right)\left(\frac{\mathrm{D}_{\mathrm{o}}}{\mathrm{t}}-1\right)\left[\mathrm{K}_{\mathrm{b}}-\frac{\mathrm{K}_{\mathrm{x}}}{\frac{8 \mathrm{E}}{\mathrm{E}^{\prime}\left(\frac{\mathrm{D}_{\mathrm{o}}}{\mathrm{t}}-1\right)^{3}}+0.732}\right]} \tag{52-53}
\end{equation*}
$$

where:

$$
\begin{aligned}
& P_{b s}=\text { pressure to develop maximum ring bending } \\
& \text { stress, lb/in }{ }^{2} \\
& f=\text { design maximum bending stress }(48,000 \\
& \mathrm{lb} / \mathrm{in}^{2} \text { ) } \\
& D_{o}=\text { outside diameter of pipe, in } \\
& \mathrm{t}=\text { net pipe wall thickness } \\
& =\mathrm{t}_{\mathrm{n}} \text { - service allowance }- \text { casting tolerance } \\
& \text { where: } \\
& \mathrm{t}_{\mathrm{n}}=\text { nominal thickness from appendix } 52 \mathrm{~F} \\
& \text { service allowance }=0.08 \text { in (AWWA, 2002) } \\
& \text { casting tolerance from appendix } 52 \mathrm{~F} \\
& \mathrm{~K}_{\mathrm{b}}=\text { bending moment coefficient (table 52-4) } \\
& \mathrm{K}_{\mathrm{x}}=\text { deflection coefficient (table 52-4) } \\
& \mathrm{E}=\text { modulus of elasticity ( } 24,000,000 \mathrm{lb} / \mathrm{in}^{2} \text { ) } \\
& \mathrm{E}^{\prime}=\text { modulus of soil reaction, } \mathrm{lb} / \mathrm{in}^{2} \text { (table 52-4) }
\end{aligned}
$$

The total pressure on the buried pipe is:

$$
\begin{equation*}
P=P_{S}+P_{w}+P_{v} \tag{52-54}
\end{equation*}
$$

where:
$\mathrm{P}=$ design pressure, $\mathrm{lb} / \mathrm{ft}^{2}$
$\mathrm{P}_{\mathrm{s}}=$ pressure from weight of soil, $\mathrm{lb} / \mathrm{ft}^{2}$
$\mathrm{P}_{\mathrm{w}}=$ pressure on pipe because of wheel load, lb/ft ${ }^{2}$
$\mathrm{P}_{\mathrm{v}}=$ internal vacuum pressure, $\mathrm{lb} / \mathrm{ft}^{2}$

The total pressure on the buried pipe, P , must be less than the design pressure to develop the maximum ring bending stress, $\mathrm{P}_{\mathrm{bs}}$ :

$$
\begin{equation*}
\mathrm{P} \leq \mathrm{P}_{\mathrm{bs}} \times 144 \tag{52-55}
\end{equation*}
$$

where:
$\mathrm{P}=$ design pressure, $\mathrm{lb} / \mathrm{ft}^{2}$
$\mathrm{P}_{\mathrm{bs}}=$ pressure to develop ring bending stress, $\mathrm{lb} / \mathrm{in}^{2}$

## (2) Ring deflection

Maximum allowable ring deflection for unlined ductile iron pipe is 5 percent of the outside diameter. The maximum allowable ring deflection for cement-mortarlined ductile iron pipe is 3 percent of the outside diameter. Research has shown that 3 percent deflection provides a safety factor of at least 2.0 against failure of the cement-mortar lining. The following equation may be used to determine the allowable design pressure at the allowable deflection:

$$
\begin{equation*}
\mathrm{P}_{\mathrm{bs}}=\frac{f}{3\left(\frac{\mathrm{D}_{\mathrm{o}}}{\mathrm{t}}\right)\left(\frac{\mathrm{D}_{\mathrm{o}}}{\mathrm{t}}-1\right)\left[\mathrm{K}_{\mathrm{b}}-\frac{8 \mathrm{~K}}{\frac{8 \mathrm{x}}{\mathrm{E}^{\prime}\left(\frac{\mathrm{D}_{\mathrm{o}}}{\mathrm{t}}-1\right)^{3}}+0.732}\right]} \tag{52-56}
\end{equation*}
$$

where:
$P_{r d}=$ pressure to develop allowable ring deflection, $\mathrm{lb} / \mathrm{in}^{2}$

$$
\frac{\Delta \mathrm{X}}{\mathrm{D}}
$$

$=$ percent deflection
$=5 \%(0.05)$ for unlined pipe
$=3 \%$ ( 0.03 ) for mortar-line pipe
$\mathrm{K}_{\mathrm{x}}=$ deflection coefficient (see table 52-4)
$\mathrm{D}_{\mathrm{o}}=$ outside diameter of pipe, in
$\mathrm{t}_{1}=$ minimum manufacturing thickness, in ( $\mathrm{t}_{\mathrm{n}}$ - casting tolerance) $\mathrm{t}_{\mathrm{n}}=$ nominal pipe wall thickness from appendix 52 F
$\mathrm{E}=$ modulus of elasticity $\left(24,000,000 \mathrm{lb} / \mathrm{in}^{2}\right)$
$\mathrm{E}^{\prime}=$ modulus of soil reaction, $\mathrm{lb} / \mathrm{in}^{2}$ (table 52-4)

The total pressure on the buried pipe, P , must be less than the design pressure to develop acceptable deflection, $\mathrm{P}_{\mathrm{rd}}$ :

$$
\begin{equation*}
\mathrm{P} \leq \mathrm{P}_{\mathrm{rd}} \times 144 \tag{52-57}
\end{equation*}
$$

where:
$\mathrm{P}=$ design pressure, $\mathrm{lb} / \mathrm{ft}^{2}$
$\mathrm{P}_{\mathrm{rd}}=$ pressure to develop ring deflection, $\mathrm{lb} / \mathrm{in}^{2}$
A required net thickness, t , is determined using both the ring bending stress and allowable deflection equations above. The larger of the two net thicknesses, t , is selected. The nominal thickness is determined by adding the service allowance and casting tolerance. The nominal thickness is typically specified.

Although backfill around the pipe should be well compacted, design values of laying condition type 3 (table $52-4)$ are recommended for ductile iron pipes used in embankments for dams and ponds.

Table 52-4 Design values for standard laying conditions

| Laying Condition | Description | $\mathbf{E}^{\prime} \mathrm{psi}^{\text {B }}$ | $\begin{gathered} \text { Bedding } \\ \text { Angle } \end{gathered}$ | $\boldsymbol{K}_{\boldsymbol{b}}$ | $\boldsymbol{K}_{\boldsymbol{x}}$ |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  | Flat bottom trench ${ }^{C}$ loose backfill. ${ }^{D}$ | 150 | 30 | 0.235 | 0.108 |
| Type 1 |  |  |  |  |  |
|  | Flat bottom trench ${ }^{C}$ Backfill lightly consolidated to centerline of pipe. | 300 | 45 | 0.210 | 0.105 |



Pipe bedded in 4 inch ( 102 mm ) minimum loose soil ${ }^{E}$ Backfill lightly consolidated to top of pipe.

400
$60 \quad 0.189$
0.103


Pipe bedded in sand, gravel, or crushed stone to depth of $1 / 8$ pipe diameter. 4 inch ( 102 mm ) minimum. Backfill compacted to top of pipe. (Approximately 80 percent standard proctor, AASHTO T-99.)

90
0.157
0.096


Pipe bedded in compacted granular material to centerline of pipe, 4 inch ( 102 mm ) minimum under pipe. Compacted granular or select ${ }_{E}$ material to top of pipe. (Approximately 90 $\begin{array}{llllll}\text { percent standard proctor, AASHTO T-99.) } & 700 & 150 & 0.128 & 0.085\end{array}$

Type 5

A Consideration of the pipe-zone embedment conditions included in this table may be influenced by factors other than pipe strength. For additional information see ANSI/AWWA C600. Standard for installation of Ductile-Iron Water Mains and their Appurtenances.
B $\quad 1 \mathrm{lb} / \mathrm{in}^{2}=6.894757 \mathrm{kPa}$.
${ }_{D}$ Flat-bottom is defined as undisturbed earth.
$D \quad$ For pipe 14 inch ( 350 mm ) and larger, consideration should be given to use of laying conditions other than Type 1.
$E$ Loose soil or select material is defined as native soil excavated from the trench free of rocks, foreign materials, and frozen earth.

### 636.5205 Expansion and contraction

All pipe products expand and contract with changes in temperature. Approximate coefficients of thermal expansion for pipe materials is presented in table 52-5. Buried pipe used in NRCS applications will not typically experience significant changes in temperature, and thermal stress or dimension change will be minimal. However, changes in the ambient temperature prior to backfilling around the pipe may lead to excessive expansion or contraction. Therefore, the backfill should be placed as construction progresses.

Unrestrained pipe will experience a length change with changing temperature. The length may be estimated by:

$$
\begin{equation*}
\Delta \mathrm{L}=\mathrm{L}_{\mathrm{ur}} \alpha \Delta \mathrm{~T} \tag{52-58}
\end{equation*}
$$

where:
$\Delta \mathrm{L}=$ change in length, in
$\mathrm{L}_{\mathrm{ur}}=$ length of unrestrained pipe, in
$\alpha=$ coefficient of thermal expansion, in/in/ ${ }^{\circ} \mathrm{F}$
$\Delta \mathrm{T}=$ change in temperature, ${ }^{\circ} \mathrm{F}$
A pipe restrained or anchored at both ends will experience a change in stress with changing temperature because of expansion and contraction. The longitudinal stress in the pipe wall caused by temperature changes may be estimated by:

$$
\begin{equation*}
\mathrm{S}_{\mathrm{EC}}=\mathrm{E} \alpha \Delta \mathrm{~T} \tag{52-59}
\end{equation*}
$$

where:
$\mathrm{S}_{\mathrm{EC}}=$ stress due to temperature change, $\mathrm{lb} / \mathrm{in}^{2}$
$\mathrm{E}=$ short term modulus of elasticity, lb/in ${ }^{2}$
$\alpha=$ coefficient of thermal expansion, in/in/ ${ }^{\circ} \mathrm{F}$
$\Delta \mathrm{T}=$ change in temperature, ${ }^{\circ} \mathrm{F}$
The modulus of elasticity of plastic pipe is a function of the temperature. Since the temperature change does not occur rapidly, the average temperature is recommended for use in determining the appropriate modulus of elasticity. The modulus of elasticity should be adjusted for temperature by the factors shown in table $52-1$.

Various pipe joints that allow some movement because of expansion and contraction are available. Gasketed pipe joints (such as bell and spigots) for plastic, steel, or ductile iron pipe and expansion joints for steel pipe allow some movement at the joint. The allowable movement at the joint should be obtained for the particular joint and compared to the length change caused by a change in temperature. Welded steel or plastic pipes or solvent cemented plastic pipes do not allow movement at the joint.

Table 52-5 Coefficients of thermal expansion

| Pipe material | Coefficient <br> $\left(\mathrm{in} / \mathrm{in} /{ }^{\circ} \mathrm{F}\right)$ |
| :--- | :--- |
| PVC | $3.0 \times 10^{-5}$ |
| HDPE | $1.2 \times 10^{-4}$ |
| ABS | $5.5 \times 10^{-5}$ |
| Aluminum | $1.3 \times 10^{-5}$ |
| Ductile Iron | $5.8 \times 10^{-6}$ |
| Steel | $6.5 \times 10^{-6}$ |

[^1]
### 636.5206 Aboveground pipe design

Aboveground applications frequently require noncontinuous support. These applications include pipe support from a saddle, rack, or stand supported by an adequate foundation or suspended from an overhead structure (figs. 52-11, 52-12, and 52-13). The equations shown apply to uniformly loaded and simply supported pipe. Lower bending moment and deflection will result for continuous rigidly joined and multiple span pipe.

## (a) Bending stress

The maximum bending stress in the pipe wall of an unsupported pipe is:

$$
\begin{equation*}
\mathrm{S}_{\mathrm{b}}=\frac{\mathrm{MD}_{\mathrm{o}}}{2 \mathrm{I}} \tag{52-60}
\end{equation*}
$$

where:
$\mathrm{S}_{\mathrm{b}}=$ bending stress, $\mathrm{lb} / \mathrm{in}^{2}$
$\mathrm{M}=$ bending moment, $\mathrm{in}-\mathrm{lb}$
I = moment of inertia, $\mathrm{in}^{4}$

$$
\begin{aligned}
& =\frac{\pi}{64}\left(\mathrm{D}_{\mathrm{o}}{ }^{4}-\mathrm{D}_{\mathrm{i}}{ }^{4}\right), \mathrm{in}^{4} \quad \text { (plastic or ductile iron pipe) } \\
& =\frac{\pi}{8}\left(\mathrm{D}_{\mathrm{o}}{ }^{3} \mathrm{t}\right) \quad \text { (steel pipe) } \\
& \mathrm{D}_{\mathrm{o}}=\text { outside pipe diameter, in } \\
& \mathrm{D}_{\mathrm{i}}=\text { inside pipe diameter, in } \\
& \mathrm{t} \quad=\text { pipe wall thickness, in }
\end{aligned}
$$

The moment for an end-supported simple beam with a single span may be calculated by:

$$
\begin{equation*}
\mathrm{M}=\frac{\mathrm{wL}_{\text {span }}{ }^{2}}{8} \tag{52-61}
\end{equation*}
$$

where:
M = bending moment, in-lb
$\mathrm{w} \quad=$ load of pipe filled with liquid, $\mathrm{lb} / \mathrm{in}$
$\mathrm{L}_{\text {span }}=$ span length, in

The above two equations may be combined to determine the bending stress at center span of the pipe or an allowable support spacing of a uniformly loaded, simply supported pipe.

$$
\begin{gather*}
\mathrm{S}_{\mathrm{b}}=\frac{0.0625 \mathrm{wL}_{\text {span }}{ }^{2} \mathrm{D}_{\mathrm{o}}}{\mathrm{I}}  \tag{52-62}\\
\text { and } \\
\mathrm{L}_{\text {span }}=4.0 \sqrt{\frac{\mathrm{~S}_{\text {ball }} \mathrm{I}}{\mathrm{wD}_{\mathrm{o}}}} \tag{52-63}
\end{gather*}
$$

where:

$$
\begin{aligned}
& \mathrm{S}_{\mathrm{b}}= \text { bending stress, } \mathrm{lb} / \mathrm{in}^{2} \\
& \mathrm{~S}_{\mathrm{ball}}= \text { allowable bending stress, } \mathrm{lb} / \mathrm{in}^{2} \\
&\left(50 \% \text { of yield strength for steel, } 48,000 \mathrm{lb} / \mathrm{in}^{2}\right. \text { for } \\
&\text { ductile iron, and } \left.7,500 \mathrm{lb} / \mathrm{in}^{2} \text { for aluminum }\right) \\
&= \text { HDS }=\text { HDB/FS for plastic } \\
& \mathrm{HDS}=\text { hydrostatic design stress } \\
& \mathrm{HDB}= \text { hydrostatic design basis } \\
& \mathrm{FS}=\text { factor of safety }(2.5 \text { for AWWA C } 900 \\
&\text { pipe, } 2.0 \text { for others })
\end{aligned}
$$

Figure 52-11 Pipeline hanger


Figure 52-12 Pipeline support


$$
\begin{aligned}
& \mathrm{w} \quad= \text { load of pipe filled with liquid, lb/in } \\
& \mathrm{L}_{\text {span }}= \text { span length, in } \\
&= \text { moment of inertia, in }{ }^{4} \\
&= \frac{\pi}{64}\left(\mathrm{D}_{\mathrm{o}}{ }^{4}-\mathrm{D}_{\mathrm{i}}{ }^{4}\right), \mathrm{in}^{4} \quad \text { (plastic and ductile iron } \\
& \text { pipe) } \\
&= \frac{\pi}{8}\left(\mathrm{D}_{\mathrm{o}}{ }^{3} \mathrm{t}\right) \quad \text { (steel and aluminum pipe) } \\
& \mathrm{D}_{\mathrm{o}}=\text { outside pipe diameter, in } \\
& \mathrm{D}_{\mathrm{i}}=\text { inside pipe diameter, in } \\
& \mathrm{t}=\text { pipe wall thickness, in }
\end{aligned}
$$

## (b) Deflection

The length of the span between pipe supports shall be such that the deflection between supports is limited to an acceptable value. A maximum deflection of $1 / 360$ of the span is recommended for steel pipe, $1 / 120$ for ductile iron pipe, 0.5 percent of span for PVC pipe, and 1 -inch for other plastic pipe. The maximum theoretical deflection for a uniformly loaded, simply supported pipe may be determined by:

$$
\begin{equation*}
\mathrm{y}=0.0130\left(\frac{\mathrm{WL}_{\text {span }}^{3}}{\mathrm{E}_{\text {long }} \mathrm{I}}\right) \tag{52-64}
\end{equation*}
$$

or

$$
\begin{equation*}
\mathrm{y}=\frac{0.0130 \mathrm{wL}_{\text {span }}^{4}}{\mathrm{E}_{\text {long }} \mathrm{I}} \tag{52-65}
\end{equation*}
$$

where:
y = maximum deflection at center of span, in
W = total load on span, lb
$\mathrm{w}=$ weight of pipe filled with liquid, $\mathrm{lb} / \mathrm{in}$
$\mathrm{L}_{\text {span }}=$ span length, in
$\mathrm{E}_{\text {long }}=$ long-term modulus of elasticity, $\mathrm{lb} / \mathrm{in}^{2}$ (see below)
I = transverse moment of inertia

$$
=\frac{\pi}{64}\left(\mathrm{D}_{\mathrm{o}}^{4}-\mathrm{D}_{\mathrm{i}}^{4}\right), \mathrm{in}^{4}
$$

(plastic or ductile iron
pipe)
$=\frac{\pi}{8}\left(\mathrm{D}_{\mathrm{o}}{ }^{3} \mathrm{t}\right)$
(steel pipe)
$\mathrm{D}_{\mathrm{o}}=$ outside diameter, in
$\mathrm{D}_{\mathrm{i}}=$ inside diameter, in
$\mathrm{t}=$ pipe wall thickness, in, $\mathrm{in}^{4}$

Figure 52-13 Typical saddle details


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Note: This equation for I does not apply to corrugated, ribbed, or profile wall pipe. The appropriate values should be obtained from ASTM specifications or the manufacturer.

| Material | Modulus of elasticity (lb/in)* |
| :--- | ---: |
| Steel | $29,000,000$ |
| Aluminum | $10,000,000$ |
| Ductile iron | $24,000,000$ |
| PVC | 140,000 (long term) |
| ABS | 65,000 (long term) |
| Polyethylene | 22,000 (long term) |
| * Long-term modulus of elasticity varies with the |  |
| cell class of each plastic. Specific values may be <br> obtained from the manufacturer. |  |

## (c) Hoop stress

The hoop stress caused by internal pressure may be estimated by:

$$
\begin{equation*}
S_{p}=\frac{P \times D_{i}}{2 \times t} \tag{52-66}
\end{equation*}
$$

where
$\mathrm{S}_{\mathrm{p}}=$ stress from internal pressure, lb/in ${ }^{2}$
$\mathrm{P}=$ pressure in the pipe, $\mathrm{lb} / \mathrm{in}^{2}$
$\mathrm{D}_{\mathrm{i}}=$ inside diameter of the pipe, in
t = pipe wall thickness, in

## (d) Localized stress at supports

An unstiffened pipe resting in saddle supports has high local stresses, longitudinal and circumferential, adjacent to the tips of the saddles. The localized stresses are less for a larger saddle angle ( $\beta$ ) than for a small angle, and are practically independent of the thickness of the saddle (saddle dimension parallel to the pipe axis). Saddle angles of 90 degrees to 120 degrees are recommended. Ductile iron pipe research shows that little benefit is gained by increasing the saddle angle above 120 degrees, yet the maximum stress increases
rapidly with saddle angle less than 90 degrees. For a pipe that fits the saddle well, the maximum longitudinal or circumferential localized stress probably does not exceed

$$
\mathrm{S}_{1}=\mathrm{k}_{\text {support }} \frac{\mathrm{R}_{\text {support }}}{\mathrm{t}^{2}} \ln \left(\frac{\mathrm{R}_{\mathrm{o}}}{\mathrm{t}}\right)
$$

(Roark, 1975)
(52-67)
where:

$$
\begin{aligned}
& \mathrm{S}_{\mathrm{l}}=\text { local stress at the saddle, lb/in }{ }^{2} \\
& \mathrm{R}_{\text {support }}= \text { total saddle reaction, lb } \\
&= \frac{\mathrm{wL}_{\text {span }}}{2} \quad \text { (single span) } \\
&= \mathrm{wL}_{\text {span }} \quad \text { (multiple span) } \\
& \text { where: } \\
& \mathrm{w} \quad=\text { weight of pipe filled with liquid, } \\
& \quad \text { lb/in } \\
& \mathrm{L}_{\text {span }}=\text { span length, in } \\
&= \text { outside radius of pipe, in } \\
& \mathrm{R}_{\mathrm{o}} \text { pipe wall thickness, in } \\
& \mathrm{t} \mathrm{k}_{\text {support }}= \\
& \text { coefficient } \\
&= 0.02-0.00012(\beta-90) \\
&= 0.03-0.00017(\beta-90) \text { (ductile iron pipe) }
\end{aligned}
$$

(DIPRA, 2001)
$\beta=$ saddle angle, degrees
Theories and data differ on the importance of the saddle support width. Some test data indicate little effect on the maximum local stress when the support width is a minimum of:

$$
\begin{equation*}
\mathrm{b}=\sqrt{2 \mathrm{D}_{\mathrm{o}} \mathrm{t}} \tag{52-68}
\end{equation*}
$$

where:

$$
\begin{aligned}
\mathrm{b} & =\text { saddle width, in } \\
\mathrm{D}_{\mathrm{o}} & =\text { outside diameter of pipe, in } \\
\mathrm{t} & =\text { pipe wall thickness, in } \\
& =\text { net pipe wall thickness, in (ductile iron) }
\end{aligned}
$$

Some polyethylene pipe manufacturers recommend the support width be at least equal to the outside pipe diameter.

## (e) Total stress at the saddle support

The total stress at the saddle is a combination of the longitudinal stresses in the pipe wall. In the case of a pipe with internal pressure, the Poisson ratio effect of the hoop stress, which produces a lateral tension, must be added to determine the total beam stress in the pipe wall (Barnard, 1948). The total stress may be computed as

$$
\begin{equation*}
\mathrm{S}_{\mathrm{T}}=\mathrm{vS} \mathrm{~S}_{\mathrm{p}}+\mathrm{S}_{\mathrm{b}}+\mathrm{S}_{\mathrm{l}}+\mathrm{S}_{\mathrm{EC}} \tag{52-69}
\end{equation*}
$$

where:
$\mathrm{S}_{\mathrm{T}}=$ total stress at the saddle, $\mathrm{lb} / \mathrm{in}^{2}$
$\mathrm{v}=$ Poisson's ratio
( 0.30 for steel and ductile iron, 0.33 for aluminum, 0.38 for PVC, 0.40 for PE, 0.50 for ABS)
$\mathrm{S}_{\mathrm{p}}=$ hoop stress from internal pressure, $\mathrm{lb} / \mathrm{in}^{2}$
$\mathrm{S}_{\mathrm{b}}=$ bending stress, lb/in ${ }^{2}$
$\mathrm{S}_{\mathrm{l}}=$ local stress at saddle, $\mathrm{lb} / \mathrm{in}^{2}$
$\mathrm{S}_{\mathrm{EC}}=$ stress from expansion and contraction (if restrained), lb/in ${ }^{2}$

The total stress must be less than the allowable stress.

$$
\begin{equation*}
\mathrm{S}_{\mathrm{T}}<\mathrm{S}_{\mathrm{all}} \tag{52-70}
\end{equation*}
$$

where:
$\mathrm{S}_{\mathrm{T}}=$ total stress at saddle, $\mathrm{lb} / \mathrm{in}^{2}$
$\mathrm{S}_{\text {all }}=$ allowable stress, $\mathrm{lb} / \mathrm{in}^{2}$ (50\% of yield strength for steel, $48,000 \mathrm{lb} / \mathrm{in}^{2}$ for ductile iron, and 7,500 $\mathrm{lb} / \mathrm{in}^{2}$ for aluminum)

$$
=\frac{\mathrm{HDB} \times \mathrm{T}_{f}}{\mathrm{FS}} \text { for plastic }
$$

HDB = hydrostatic design basis
$\mathrm{T}_{f}=$ temperature factor from table 52-1.
FS = factor of safety ( 2.5 for AWWA
C900 pipe, 2.0 for others)

## (f) Buckling

For aboveground pipe subject to external hydrostatic pressure or internal vacuum pressure, the critical collapse pressure may be determined by the following equations:

$$
\begin{align*}
& \mathrm{P}_{\mathrm{CR}}=\frac{3 \mathrm{EI}_{\mathrm{pw}}}{\left(1-\mathrm{v}^{2}\right) \mathrm{r}^{3}} \text { for all pipe }  \tag{52-71}\\
& \mathrm{P}_{\mathrm{CR}}=\frac{0.447 \mathrm{PS}}{\left(1-\mathrm{v}^{2}\right)} \text { for corrugated plastic pipe }  \tag{52-72}\\
& \mathrm{P}_{\mathrm{CR}}=\frac{2 \mathrm{E}}{\left(1-\mathrm{v}^{2}\right)}\left(\frac{1}{\mathrm{SDR}-1}\right)^{3} \tag{52-73}
\end{align*}
$$

or

$$
\begin{equation*}
P_{\mathrm{CR}}=\frac{2 \mathrm{E}}{\left(1-\mathrm{v}^{2}\right)}\left(\frac{1}{\operatorname{SIDR}+1}\right)^{3} \text { for solid-wall pipe } \tag{52-74}
\end{equation*}
$$

where:

| $\mathrm{P}_{\mathrm{CR}}=$ | critical external collapse pressure, $\mathrm{lb} / \mathrm{in}^{2}$ |
| ---: | :--- |
| $\mathrm{E}=$ | modulus of elasticity, lb/in ${ }^{2}$ |
|  | The long-term modulus of elasticity is |
|  | recommended if the pipe is subject to the |
|  | presssure in the normal operations. If the |
|  | pipe is subject to the pressure for short |
|  | time periods and infrequently, the use of the |
|  | short-term modulus of elasticity is accept- |
|  | able. |
| $=$ | pipe wall moment of inertia, in ${ }^{4}$ |
| $=$ | Poisson's ratio (0.30 for steel and ductile |
|  | iron, 0.33 for aluminum, 0.38 for PVC, 0.40 |
| I |  |
| $\mathrm{v}=$ | for PE, 0.50 for ABS $)$ |
| $\mathrm{r}=$ | mean pipe radius, in |
| $\mathrm{PS}=$ | pipe stiffness, lb/in ${ }^{2}$ |
| $\mathrm{SDR}=$ | $\mathrm{D}_{\mathrm{o}}$ dimension ratio |
| $\mathrm{SIDR}=$ | $\mathrm{D}_{\mathrm{i}}$ dimension ratio |

$\mathrm{C}_{\text {n }}$ The long-term modulus of elasticity is recommended if the pipe is subject to the presssure in the normal operations. If the pipe is subject to the pressure for short time periods and infrequently, the use of the short-term modulus of elasticity is acceptable.
$\mathrm{I}_{\mathrm{pw}}=$ pipe wall moment of inertia, in ${ }^{4}$
$\mathrm{v}=$ Poisson's ratio ( 0.30 for steel and ductile iron, 0.33 for aluminum, 0.38 for PVC, 0.40 for PE, 0.50 for ABS)
$\mathrm{r} \quad=$ mean pipe radius, in
$\mathrm{PS}=$ pipe stiffness, $\mathrm{lb} / \mathrm{in}^{2}$
$\mathrm{SDR}=\mathrm{D}_{\mathrm{o}}$ dimension ratio
$\mathrm{SIDR}=\mathrm{D}_{\mathrm{i}}$ dimension ratio

### 636.5207 Thrust block design

The internal pressure of a pipe acts perpendicular to any plane with a force equal to the pressure, $P$, times the area of the pipe, A. The radial forces within the pipe are balanced by the tension in the pipe wall. The axial components of pressure through a straight section are balanced by the same pressure in the opposite direction. An unbalanced thrust force will exist in other configurations (fig. 52-14).

The internal pressure used in thrust block design is the working pressure for a pumped system or static pressure head in a gravity system.

Abrupt changes in pipeline grade, horizontal alignment, or reduction in pipe size normally require an anchor or thrust blocks (fig. 52-15) to absorb any axial thrust of the pipeline. Thrust control may also be needed at the end of the pipeline and at inline control valves.

Thrust blocks and anchors must be large enough to withstand the forces tending to move the pipe, including those of momentum and pressure, as well as forces from expansion and contraction.

The positioning of the thrust blocks must consider whether connections adjacent to the thrust block are capable of movement, as well as the anticipated direction of movement.

The vector sum of the pressure forces is shown as T , a thrust force, for various configurations in figure $52-14$. The area of the thrust block may be determined by the following:

$$
\begin{equation*}
\mathrm{A}_{\mathrm{T}}=\frac{\mathrm{T}}{\mathrm{q}_{\text {all }}} \tag{52-75}
\end{equation*}
$$

where:
$\mathrm{A}_{\mathrm{T}}=$ area of thrust block required, $\mathrm{ft}^{2}$
$\mathrm{~T}=$ thrust force, lb
$\mathrm{q}_{\text {all }}=$ allowable soil bearing pressure, $\mathrm{lb} / \mathrm{ft}^{2}$

If adequate soil tests are not available, the soil pressure may be estimated from table 52-6.

Table 52-6 Allowable soil bearing pressure

| Natural soil material | Depth of cover to center of thrust block2 ft3 ft4 ft5 ft |  |  |  |
| :---: | :---: | :---: | :---: | :---: |
| Sound bedrock | 8,000 | 10,000 | 10,000 | 10,000 |
| Dense sand and gravel mixture (assumed $\emptyset=40^{\circ}$ ) | 1,200 | 1,800 | 2,400 | 3,000 |
| Dense fine to coarse sand (assumed $\emptyset=35^{\circ}$ ) | 800 | 1,200 | 1,650 | 2,100 |
| Silt and clay mixture (assumed $\emptyset=25^{\circ}$ ) | 500 | 700 | 950 | 1,200 |
| Soft clay and organic soils (assumed $\emptyset=10^{\circ}$ ) | 200 | 300 | 400 | 500 |

Figure 52-14 Thrust forces


Tee


Closed valve


Wye


Figure 52-15 Thrust block types


### 636.5208 Longitudinal bending

Flexible plastic pipe is often installed in conditions that require longitudinal bending. Steel, corrugated metal pipe, and ductile iron pipe will withstand minimal longitudinal bending. Controlled longitudinal bending of the pipe within acceptable limits can be accommodated by the flexibility of the pipe itself. Additional longitudinal deviation must be accomplished by joint deflection or the use of special fittings. Joint deflection limits may be obtained from the manufacturer. Acceptable bending may be expressed in terms of the minimum bending radius calculated by:

$$
\begin{equation*}
\mathrm{R}_{\mathrm{b}}=\frac{\mathrm{ED}_{\mathrm{o}}}{2 \mathrm{~S}_{\text {ball }}} \tag{52-76}
\end{equation*}
$$

where:
$R_{b}=$ minimum bending radius, in
E = short-term modulus of elasticity, $\mathrm{lb} / \mathrm{in}^{2}$
$\mathrm{D}_{\mathrm{o}}=$ outside pipe diameter, in
$\mathrm{S}_{\text {ball }}=$ allowable bending stress, $\mathrm{lb} / \mathrm{in}^{2}$

$$
\left.\begin{array}{l}
=\frac{\mathrm{HDB} \times \mathrm{T}_{f}}{\mathrm{FS}} \text { (nonpressure or gasketed pressure } \\
\text { plastic pipe) }
\end{array}\right] \begin{aligned}
& \mathrm{FS}
\end{aligned}
$$

HDB = hydrostatic design basis

$$
\mathrm{T}_{f}=\text { temperature factor from table 52-1. }
$$

$$
\text { FS = factor of safety ( } 2.5 \text { for AWWA }
$$

$$
\text { C900 pipe, } 2.0 \text { for others) }
$$

$$
=\mathrm{S}_{\text {all }}-\mathrm{S}_{\mathrm{p}} \text { (for steel, aluminum, corrugated }
$$ metal, and ductile iron pipe)

where:
$\mathrm{S}_{\text {all }}=$ allowable stress, $\mathrm{lb} / \mathrm{in}^{2}$ ( $50 \%$ of yield strength for steel, $48,000 \mathrm{lb} / \mathrm{in}^{2}$ for ductile iron, and $7,500 \mathrm{lb} / \mathrm{in}^{2}$ for aluminum)
$\mathrm{S}_{\mathrm{p}}=$ stress caused by internal pressure, $\mathrm{lb} / \mathrm{in}^{2}$
$=\frac{\mathrm{PD}_{o}}{2 \mathrm{t}}$
where:
$\mathrm{P}=$ maximum working pressure or static pressure, $\mathrm{lb} / \mathrm{in}^{2}$
$\mathrm{D}_{\mathrm{o}}=$ outside pipe diameter, in
$t=$ pipe wall thickness, in
$=$ net pipe wall thickness, in (for ductile iron)

Some bending may be accomplished by axial joint deflection in gasketed pipe joints. The amount of joint deflection may be obtained from the pipe manufacturer. Solvent cemented or welded joints do not allow joint deflection.

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### 636.5209 References

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## Glossary

Buckling. Failure by lateral or torsional instability of a structural member, occurring with stresses below the yield strength.

## Collapse pressure (critical buckling pressure).

The negative pressure at which the pipe collapses caused by water column separation from valve closure, sudden air evacuation, surge pressures, or other causes.

Critical time. Longest elapsed time before final flow stoppage that will still allow the maximum pressure surge to occur.

Deflection. The decrease in the vertical diameter of a pipe due to load, divided by the nominal diameter, expressed as a percent.

Gage. Reference system for thickness of metal sheets or wire.

Hoop stress. The tensile stress in the wall of the pipe in the circumferential orientation due to internal hydrostatic pressure.

Hydrostatic design basis. One of a series of established stress values specified in ASTM D 2837 for a plastic compound obtained by categorizing the longterm hydrostatic strength determined in accordance with Test Method D 2837.

Hydrostatic design stress. The recommended maximum hoop stress that can be applied continuously with a high degrees of certainty that failure of the pipe will not occur.

Modulus of soil reaction, $\mathbf{E}$ prime ( $\mathbf{E}^{\prime}$ ). Measure of the stiffness of the embedment material that surrounds the pipe.

Out-of-roundness. The allowed difference between the maximum measured diameter and the minimum measured diameter (stated as an absolute deviation).

Pipe stiffness. For plastic pipe, a term to describe the stiffness of the pipe from a parallel plate test, which defines the pipe's resistance to load.

Pressure rating or pressure class. The maximum internal water pressure that can be exerted continuously in a pipe without damage at a specific temperature ( $73^{\circ} \mathrm{F}$ ).

Standard dimension ratio (SDR) or dimension ratio (DR). A specific ratio of the average specified outside diameter to the minimum specified wall thickness for outside diameter-controlled plastic pipe.

Standard inside dimension ratio (SIDR). A specific ratio of the average specified inside diameter to the minimum specified wall thickness for inside diametercontrolled plastic pipe.

Static head. The height of water above any plane or reference point.

Static pressure. The internal pressure when no flow is occurring in the pipe.

Surge pressure. The maximum pressure increase greater than working pressure (sometimes called water hammer) that is anticipated in the system as a result of change in velocity in the water. Some causes of surge include the opening and closing (full or partial) of valves, starting and stopping of pumps, changes in reservoir elevation, liquid column separation, and entrapped air.

Thrust in a pipe wall. The circumferential compressive force in the conduit walls, per unit length of pipe.

Total system pressure. The sum of working pressure plus surge pressure.

Water hammer. A pressure surge in a pipeline caused by a sudden change in water velocity. Typical causes include the sudden starting or stopping of a pump, sudden valve movement, or air movement in a pipeline. The surge may damage or destroy pipelines and pumps if severe.

Working pressure. The maximum anticipated sustained operating pressure for the system.

## Appendix 52A

## Symbols Used in NEH 636, Chapter 52

| A | pipe cross section area, $\mathrm{in}^{2}$ | PS | pipe stiffness |
| :---: | :---: | :---: | :---: |
| $\mathrm{A}_{\mathrm{pw}}$ | area of pipe wall, $\mathrm{in}^{2} / \mathrm{in}$ | PCR | critical collapse pressure, $\mathrm{lb} / \mathrm{in}^{2}$ |
| $\mathrm{A}_{\text {s }}$ | area of section, $\mathrm{in}^{2} / \mathrm{ft}$ |  | (aboveground pipe) |
| $\mathrm{A}_{\text {T }}$ | area of thrust block required, $\mathrm{ft}^{2}$ | $\mathbf{P}_{\text {G }}$ | external hydrostatic pressure, $\mathrm{lb} / \mathrm{ft}^{2}$ |
| a | velocity of pressure wave, $\mathrm{ft} / \mathrm{s}$ | $\mathbf{P}_{\text {L }}$ | wheel load at the surface, lb |
| B' | empirical coefficient of elastic support | $\mathbf{P}_{\text {S }}$ | pressure on pipe from weight of soil, $\mathrm{lb} / \mathrm{ft}^{2}$ |
| b | aboveground pipe saddle width, in | $\mathbf{P}_{\text {V }}$ | internal vacuum pressure, $\mathrm{lb} / \mathrm{ft}^{2}$ |
| C | reduction factor for buckling pressure | $\mathrm{P}_{\text {W }}$ | pressure on pipe from wheel load, $\mathrm{lb} / \mathrm{ft}^{2}$ |
| $\mathrm{D}_{\mathrm{L}}$ | deflection lag factor | $\mathrm{P}_{\text {bs }}$ | pressure to develop maximum ring bending, |
| $\mathrm{D}_{\mathrm{M}}$ | mean pipe diameter, in |  | $\mathrm{lb} / \mathrm{in}^{2}$ (ductile iron pipe) |
| $\mathrm{D}_{0}$ | outside pipe diameter, in | $\mathbf{P r d}_{\text {rd }}$ | pressure to develop allowable ring |
| $\mathrm{D}_{1}$ | inside pipe diameter, in |  | deflection, $\mathrm{lb} / \mathrm{in}^{2}$ (ductile iron pipe) |
| E | modulus of elasticity of pipe material, $\mathrm{lb} / \mathrm{in}^{2}$ | $\mathbf{P}_{\text {surge }}$ | surge pressure, $\mathrm{lb} / \mathrm{in}^{2}$ |
| $\mathbf{E}_{\text {long }}$ | long-term modulus of elasticity, $\mathrm{lb} / \mathrm{in}^{2}$ | $\mathrm{P}_{\text {work }}$ | working pressure, $\mathrm{lb} / \mathrm{in}^{2}$ |
| $\mathbf{E}^{\prime}$ | modulus of soil reaction, $\mathrm{lb} / \mathrm{in}^{2}$ | $\mathbf{q}_{\text {a }}$ | allowable buckling pressure, $\mathrm{lb} / \mathrm{in}^{2}$ |
| EI | pipe wall stiffness, in-lb | $\mathrm{q}_{\text {all }}$ | allowable soil bearing pressure of the soil, |
| e | base of neutral logs, 2.71828 |  | $\mathrm{lb} / \mathrm{ft}^{2}$ |
| FF | flexibility factor, in/lb | RSC | ring stiffness constant |
| FS | factor of safety | $\mathbf{R}_{\text {b }}$ | minimum bending radius, in |
| $f$ | design maximum bending stress, $\mathrm{lb} / \mathrm{in}^{2}$ | $\mathbf{R}_{\text {o }}$ | outside radius of pipe, in |
| $f_{\text {c }}$ | critical buckling stress, lb/in ${ }^{2}$ (CMP) | $\mathbf{R}_{\text {support }}$ | aboveground pipe support reaction, lb |
| $f_{\text {u }}$ | minimum tensile strength of material, $\mathrm{lb} / \mathrm{in}^{2}$ (CMP) | $\mathbf{R}_{\mathbf{w}}$ | water buoyancy factor radius of gyration of corrugation, in |
| $\boldsymbol{f}_{\text {y }}$ | minimum yield strength, $\mathrm{lb} / \mathrm{in}^{2}$ (CMP) | S | allowable stress, lb/in ${ }^{2}$ |
| g | acceleration of gravity, $32.2 \mathrm{ft} / \mathrm{s}^{2}$ | SDR | standard dimension ratio (same as DR) |
| H | maximum working pressure, ft | SIDR | standard inside diameter dimension ratio |
| HDB | hydrostatic design basis, lb/in ${ }^{2}$ |  | (same as IDR) |
| HDS | hydrostatic design stress, $\mathrm{lb} / \mathrm{in}^{2}$ | SS | required seam strength, lbf/ft |
| $\mathrm{H}_{\text {surge }}$ | surge pressure, ft of water | STHS | short-term hoop strength, lb/in ${ }^{2}$ |
| $h^{\text {surge }}$ | height of ground surface above top of pipe, ft | STR | short-term pressure rating, lb/in ${ }^{2}$ |
| $\mathrm{h}_{\text {w }}$ | height of water above top of pipe, ft moment of inertia, in ${ }^{4}$ | STS | short-term strength (quick burst pressure), $\mathrm{lb} / \mathrm{in}^{2}$ |
| $\mathrm{I}_{\mathrm{pw}}$ | pipe wall moment of inertia, in ${ }^{4}$ | $\mathbf{S}_{\text {all }}$ | allowable stress, $\mathrm{lb} / \mathrm{in}^{2}$ |
| $\mathrm{I}_{\mathrm{f}}$ | impact factor | $\mathrm{S}_{\mathrm{b}}$ | bending stress, lb/in ${ }^{2}$ |
| K | bedding constant | $\mathbf{S}_{\text {ball }}$ | allowable bending stress, $\mathrm{lb} / \mathrm{in}^{2}$ |
| $\mathbf{K}_{\text {b }}$ | bending moment coefficient | $\mathbf{S}_{\mathbf{E C}}$ | stress because of expansion or contraction, |
| $\mathbf{K}_{\mathbf{L}}$ | bulk modulus of liquid, $\mathrm{lb} / \mathrm{in}^{2}$ |  | $\mathrm{lb} / \mathrm{in}^{2}$ |
| $\mathbf{K}_{\mathrm{x}}$ | deflection coefficient | $\mathrm{S}_{1}$ | local stress at a saddle support, $\mathrm{lb} / \mathrm{in}^{2}$ |
| k | soil stiffness factor $=0.22$ for good fill mate- | $\mathbf{S}_{\mathbf{P}}$ | stress from internal pressure, $\mathrm{lb} / \mathrm{in}^{2}$ |
|  | rial compacted to $90 \%$ of standard density | $\mathbf{S}_{\text {T }}$ | total stress at saddle support, lb/in ${ }^{2}$ |
|  | based on ASTM D 698 | $\mathrm{S}_{\mathrm{y}}$ | yield strength, lb/in ${ }^{2}$ |
| $\mathbf{k}_{\text {support }}$ | coefficient for saddle support | T | thrust force, lb |
| L | distance within the pipeline that the pressure | $\mathrm{T}_{\text {PW }}$ | thrust in pipe wall, lb/ft |
|  | wave moves before it is reflected back by a | $\mathrm{T}_{\text {CR }}$ | critical time, second |
|  | boundary condition, ft | $\mathrm{T}_{\mathrm{f}}$ | temperature factor |
| $\mathbf{L}_{\text {span }}$ | span length, in | t | wall thickness, in |
| $\mathbf{L}_{\mathbf{u r}}$ | length of unrestrained pipe, in | t | net pipe wall thickness, in (ductile iron pipe) |
| M | bending moment, in-lb | $\mathrm{t}_{1}$ | minimum manufacturing thickness, in (duc- |
| P | pressure in or on pipe, $\mathrm{lb} / \mathrm{ft}^{2}$ |  | tile iron pipe) |
| PC | pressure class, lb/in ${ }^{2}$ | $\mathrm{t}_{\mathrm{n}}$ | nominal pipe wall thickness, in (ductile iron |
| PR | pressure rating, lb/in ${ }^{2}$ |  | pipe) |


| W | total load on span, lb |
| :---: | :---: |
| $\mathrm{W}_{\text {L }}$ | wheel load per linear foot of pipe, lb/ft |
| $\mathrm{W}_{\text {S }}$ | soil load per linear foot of pipe, $\mathrm{lb} / \mathrm{ft}$ |
| $\mathrm{W}_{\mathrm{V}}$ | vacuum load per linear foot of pipe, lb/ft |
| w | load of pipe filled with liquid, $\mathrm{lb} / \mathrm{in}$ |
| y | maximum deflection at the center of span, in |
| $\alpha$ | coefficient of thermal expansion, $\mathrm{in} / \mathrm{in} /{ }^{\circ} \mathrm{F}$ |
| $\beta$ | saddle angle, degrees |
| $\Delta / \mathrm{D}$ | percent deflection expressed as a decimal |
| $\Delta \mathrm{T}$ | change in temperature, ${ }^{\circ} \mathrm{F}$ |
| $\Delta \mathrm{V}$ | change in velocity of fluid, $\mathrm{ft} / \mathrm{s}$ |
| $\Delta \mathbf{Y}$ | vertical change in diameter, in |
| $\varepsilon$ | maximum combined strain in pipe wall because of ring bending |
| $\varepsilon_{\text {all }}$ | allowable strain in pipe wall |
| $\varepsilon_{\text {h }}$ | strain in the pipe wall caused by hoop stress |
| $\varepsilon_{\text {f }}$ | strain in the pipe wall caused by bending or flexure |
| $\varepsilon_{\text {h }}$ | strain in the pipe wall caused by hoop stress |
| $\gamma_{\text {s }}$ | unit weight of soil, $\mathrm{lb} / \mathrm{ft}^{3}$ |
| $\gamma_{\text {w }}$ | unit weight of water, $62.4 \mathrm{lb} / \mathrm{ft}^{3}$ |
| $\rho$ | density of fluid, slugs//ft ${ }^{3}$ |
| $\theta$ | angle of pipe bed, degrees |
| $\sigma$ | allowable long-term compressive stress, $\mathrm{lb} / \mathrm{in}^{2}$ |
| $v$ | Poisson's ratio |
| $\Delta \mathbf{L}$ | change in length, in |
| $\Delta \mathbf{X} / \mathrm{D}$ | percent deflection expressed as a decimal |
| $\Delta \mathrm{y}$ | Vertical decrease in diameter, in |
| \% $\Delta \mathbf{X} / \mathbf{D}$ | percent deflection |

## Design Example 1 - Plastic Pipe

Problem: 12-inch diameter PVC pipe will be installed for an irrigation pipeline system. The pipe will be buried under 10 feet of soil. The maximum pressure (including surge pressure) in the pipe will be 110 pounds per square inch. The pipe is subject to farm equipment with wheel loads of 10,000 pounds. The excavation will be backfilled and minimally compacted with CL soils that have less than 25 percent coarse particles.


Assumptions: 1. The pipe is outside diameter controlled.
2. The PVC pipe will be PVC 2116 with a hydrostatic design basis of 3,200 pounds per square inch (see appendix 52C)
3. Assume unit weight of soil $=120 \mathrm{lb} / \mathrm{ft}^{3}$
4. Slightly compacted CL soils, $\mathrm{E}^{\prime}=200 \mathrm{lb} / \mathrm{in}^{2}$

Determine: A. Dimension ratio (SDR for outside diameter controlled pipe) for the maximum pressure (including surge pressure)
B. External soil and wheel loads
C. Required wall area for external load
D. Deflection
E. Allowable buckling
F. Strain

## Design example 1-Plastic pipe (continued)

Solution: A. Dimension ratio (SDR for outside diameter controlled pipe) for maximum pressure (including surge pressure
Water pressure-From table 52C-3 of appendix 52-C, an SDR of 26 is required since a 12-inch PVC 2116 pipe with SDR of 26 has a working pressure of $125 \mathrm{lb} / \mathrm{in}^{2}$, which is greater than the $110 \mathrm{lb} / \mathrm{in}^{2}$ maximum pressure (including surge pressure).
B. External soil and wheel loads

From equation 52-17, soil pressure is

$$
\begin{aligned}
\mathrm{P}_{\mathrm{s}} & =\gamma_{\mathrm{s}} \times \mathrm{h} \\
& =120 \times 10=1,200 \mathrm{lb} / \mathrm{ft}^{2}
\end{aligned}
$$

Wheel loading: From table 52C-3 of appendix 52-C, 12-inch PVC pipe with a SDR of 26 has a thickness, $\mathrm{t}=0.49 \mathrm{in}$. From equations 52-19 and 52-21:
Since $D_{o}-t<2.67 \mathrm{~h} \times 12$

$$
\begin{aligned}
& 12.75-0.49<2.67(10) \times 12 \\
& 12.26<320.4 \\
& \mathrm{~W}_{\mathrm{L}}=\frac{0.48 \mathrm{P}_{\mathrm{L}} \mathrm{I}_{\mathrm{f}}\left(\frac{\mathrm{D}_{\mathrm{o}}-\mathrm{t}}{12}\right)^{2}}{2.67 \mathrm{~h}^{3}}\left[\frac{2.67 \mathrm{~h}}{\left(\frac{\mathrm{D}_{\mathrm{o}}-\mathrm{t}}{12}\right)}-0.5\right] \\
& \mathrm{W}_{\mathrm{L}}=\frac{0.48(10,000)(1.0)\left(\frac{12.75-0.49}{12}\right)^{2}}{2.67(10)^{3}}\left[\frac{2.67(10)}{\left(\frac{12.75-0.49}{12}\right)}-0.5\right] \\
& \mathrm{W}_{\mathrm{L}}=48 \mathrm{lb} / \mathrm{ft} \text { of pipe } \\
& \mathrm{P}_{\mathrm{w}}=\frac{\mathrm{W}_{\mathrm{L}}}{\left(\frac{\mathrm{D}_{\mathrm{o}}}{12}\right)}=\frac{48}{12.75 / 12}=45 \mathrm{lb} / \mathrm{ft}^{2}
\end{aligned}
$$

Design pressure: $P=P_{S}+P_{w}+P_{v}$

$$
=1,200+45+0=1,245 \mathrm{lb} / \mathrm{ft}^{2}
$$

From equation 52-26:
Thrust:

$$
\begin{aligned}
& \text { st: } \\
& \mathrm{T}_{\mathrm{pw}}=\frac{\mathrm{P} \times\left(\frac{\mathrm{D}_{\mathrm{o}}}{12}\right)}{2} \\
& \mathrm{~T}_{\mathrm{pw}}=\frac{1,245 \times \frac{12.75}{12}}{2} \\
& \mathrm{~T}_{\mathrm{pw}}=661 \mathrm{lb} / \mathrm{ft} \text { of pipe }
\end{aligned}
$$

Design example 1—Plastic pipe (continued)
C. Required wall area for external load

From equation 52-27:
$\mathrm{A}_{\mathrm{pw}}=\frac{\left(\frac{\mathrm{T}_{\mathrm{pw}}}{12}\right)}{\sigma}$
$\mathrm{A}_{\mathrm{pw}}=\frac{\left(\frac{661}{12}\right)}{1,600}$,
, $\sigma=1,600 \mathrm{lb} / \mathrm{in}^{2}$ from appendix 52 C , table 52 C , table $52 \mathrm{C}-1$
$\mathrm{A}_{\mathrm{pw}}=0.034 \mathrm{in}^{2} / \mathrm{in}$
Wall area of 12 -inch pipe with SDR of 26 using equation 52-28:
$A_{p w}=\frac{\left(D_{o}-D_{i}\right)}{2}$ or $t$
$\mathrm{A}_{\mathrm{pw}}=\mathrm{t}$
$\mathrm{A}_{\mathrm{pw}}=0.49 \mathrm{in}^{2} / \mathrm{in}>0.034 \mathrm{in}^{2} / \mathrm{in} \quad$ O.K.
D. Deflection

From equation 52-29, percent deflection for solid wall pipe is

$$
\begin{aligned}
& \frac{\% \Delta X}{\mathrm{D}}=\frac{\left(\mathrm{D}_{\mathrm{L}} \mathrm{P}_{\mathrm{S}}+\mathrm{P}_{\mathrm{L}}+\mathrm{P}_{\mathrm{V}}\right)\left(\frac{1}{144}\right) \mathrm{K}(100)}{\left[\left(\frac{2 \mathrm{E}}{\sim}\right)+0.061 \mathrm{E}^{\prime}\right]} \\
& \frac{\% \Delta \mathrm{X}}{\mathrm{D}}=\frac{\underline{[1.5(1,200)+45+0]}}{144}(0.1)(100) \\
& {\left[\left(\frac{2(400,000)}{3(26-1)^{3^{\prime}}}\right)+0.061(200)\right]} \\
& \frac{\% \Delta \mathrm{X}}{\mathrm{D}}=4.38 \%<\left(\frac{\% \Delta \mathrm{X}}{\mathrm{D}}\right)_{\text {allowable }}=7.5 \% \\
& \text { O.K. }
\end{aligned}
$$

E. Allowable buckling pressure

From equation 52-33:

$$
\mathrm{q}_{\mathrm{a}}=\frac{1}{\mathrm{FS}}\left(32 \mathrm{R}_{\mathrm{w}} \mathrm{~B}^{\prime} \mathrm{E}^{\prime} \frac{\mathrm{E}_{\mathrm{long}} \mathrm{I}_{\mathrm{pw}}}{\mathrm{D}_{\mathrm{o}}{ }^{3}}\right)^{1 / 2}
$$

## Design example 1—Plastic pipe (continued)

where:

$$
\begin{aligned}
& \frac{\mathrm{h}}{\left(\frac{\mathrm{D}_{\mathrm{o}}}{12}\right)}=\frac{10}{\left(\frac{11.75}{12}\right)}=9.41>2 \\
& \mathrm{FS}=2.5 \\
& \mathrm{R}_{\mathrm{w}}=1.0 \\
& \mathrm{~B}^{\prime}=\frac{4\left(\mathrm{~h}^{2}+\frac{\mathrm{D}_{\mathrm{o}}}{12} \mathrm{~h}\right)}{1.5\left(2 \mathrm{~h}+\frac{\mathrm{D}_{\mathrm{o}}}{12}\right)^{2}}=\frac{4\left(120^{2}+\left(\frac{12.75}{12}\right) 10\right)}{1.5\left(2(10)+\frac{12.75}{12}\right)^{2}}=0.66 \\
& \mathrm{I}_{\mathrm{pw}}=\frac{\mathrm{t}^{3}}{12}=\frac{(0.49)^{3}}{12}=0.0098 \mathrm{in}^{4} / \mathrm{in}
\end{aligned}
$$

From equation 52-34, the reduction factor for the allowable buckling pressure from the deflection of the pipe is

$$
\begin{aligned}
\mathrm{C} & =\left[\frac{\left(1-\frac{\% \Delta \mathrm{X}}{\mathrm{D}} \frac{1}{100}\right)}{\left(1+\frac{\% \Delta \mathrm{X}}{\mathrm{D}} \frac{1}{100}\right)^{2}}\right]^{3} \\
& =\left[\frac{\left(1-4.38 \frac{1}{100}\right)}{\left(1+4.38 \frac{1}{100}\right)^{2}}\right]^{3} \\
& =0.676
\end{aligned}
$$

The reduced allowable buckling pressure is

$$
\begin{aligned}
\mathrm{q}_{\mathrm{a}} \mathrm{C} & =3,045 \times 0.676 \\
& =2,058 \mathrm{lb} / \mathrm{ft}^{2}>1,245 \mathrm{lb} / \mathrm{ft}^{2} \quad \text { O.K. }
\end{aligned}
$$

## F. Strain

The allowable deflection for PVC pipe limits strain in PVC pipes. Therefore, computation of strain and comparison to an allowable strain limit is not required for PVC pipe.

Conclusion: A PVC pipe of PVC 2116 plastic with a DR of 26 should be installed and the backfill at least slightly compacted.

## Design Example 2 - Steel Pipe

Problem: A 24-inch-diameter steel pipe will be installed for an irrigation pipeline system. The pipe will be buried under 15 feet of soil. The maximum pressure (including surge pressure) in the pipe will be 150 pounds per square inch. The pipe is subject to farm equipment with wheel loads of 10,000 pounds. The excavation will be backfilled with dumped CL soils with minimal coarse particles.


## Assumptions:

1. The pipe is ASTM A-139 Grade A Steel, with a design stress at 50 percent of the yield stress of $15,000 \mathrm{lb} / \mathrm{in}^{2}$
2. Assume unit weight of soil $=120 \mathrm{lb} / \mathrm{ft}^{3}$
3. $\mathrm{E}^{\prime}=50 \mathrm{lb} / \mathrm{in}^{2}$

Determine: A. Required wall thickness of the pipe for the internal pressure
B. External soil and wheel loads
C. Deflection
D. Allowable buckling

Solution: A. Internal pressure-The working pressure rating equation can be revised to compute the required thickness. From equation $52-5$ :

$$
\begin{aligned}
P R & =\frac{2 \times S \times t}{D_{o}} \\
t & =\frac{P \times D_{o}}{2 \times S} \\
t & =\frac{150 \times 24}{2 \times 15,000}=0.12 \mathrm{in}
\end{aligned}
$$

Use $1 / 8$ - or 0.125 -inch thick steel.

Design example 2-Steel pipe (continued)

## B. External loads

From equations $52-17$ and $52-18$, soil pressure is

$$
\begin{aligned}
\mathrm{P}_{\mathrm{s}} & =\gamma_{\mathrm{s}} \times \mathrm{h} \\
& =120 \times 15=1,800 \mathrm{lb} / \mathrm{ft}^{2} \\
\mathrm{~W}_{\mathrm{s}} & =\mathrm{P}_{\mathrm{s}} \times \frac{\mathrm{D}_{\mathrm{o}}}{12} \\
& =1,800 \times \frac{24}{12}=3,600 \mathrm{lb} / \mathrm{ft}=300 \mathrm{lb} / \text { in of pipe }
\end{aligned}
$$

From equation 52-19 and 52-21, wheel loading is calculated using the following: Since $\quad D_{o}-t<2.67 \mathrm{~h} \times 12$

$$
\begin{aligned}
& 24-0.125<2.67(15) \times 12 \\
& 23.875<480
\end{aligned}
$$

$$
\mathrm{W}_{\mathrm{L}}=\frac{0.48 \mathrm{P}_{\mathrm{L}} \mathrm{I}_{\mathrm{f}}\left(\frac{\mathrm{D}_{\mathrm{o}}-\mathrm{t}}{12}\right)^{2}}{2.67 \mathrm{~h}^{3}}\left[\frac{2.67 \mathrm{~h}}{\frac{\mathrm{D}_{\mathrm{o}}-\mathrm{t}}{12}}-0.5\right]
$$

$$
\mathrm{W}_{\mathrm{L}}=\frac{0.48(10,000)(1.0)\left(\frac{24-0.125}{12}\right)}{2.67(15)^{3}}\left[\frac{2.67(15)}{\frac{24-0.125}{12}}-0.5\right]
$$

$$
\mathrm{W}_{\mathrm{L}}=41.4 \mathrm{lb} / \mathrm{ft} \text { of pipe }
$$

Design load:

$$
\begin{aligned}
\mathrm{W} & =\mathrm{W}_{\mathrm{s}}+\mathrm{W}_{\mathrm{L}}+\mathrm{W}_{\mathrm{V}} \\
& =3,600+41+0=3,641 \mathrm{lb} / \mathrm{ft} \\
\mathrm{P}_{\mathrm{w}} & =\frac{\mathrm{W}_{\mathrm{L}}}{\frac{\mathrm{D}_{\mathrm{o}}}{12}}=\frac{3,641}{\left(\frac{24}{12}\right)}=1,820 \mathrm{lb} / \mathrm{ft}^{2}
\end{aligned}
$$

## Design example 2-Steel pipe (continued)

C. Deflection of the steel pipe

From equation 52-41

$$
\begin{aligned}
\Delta \mathrm{X} & =\left(\frac{\left(\mathrm{D}_{\mathrm{L}} \mathrm{~W}_{\mathrm{s}}+\mathrm{W}_{\mathrm{L}}+\mathrm{W}_{\mathrm{V}}\right)\left(\frac{1}{12}\right) \mathrm{Kr}^{3}}{\mathrm{EI}_{\mathrm{pw}}+0.061 \mathrm{E}^{\prime} \mathrm{r}^{3}}\right) \text { and } \\
\mathrm{I}_{\mathrm{pw}} & =\frac{\mathrm{t}^{3}}{12}=\frac{0.125}{12} \\
& =.000162 \mathrm{in}^{4} / \mathrm{in} \\
\Delta \mathrm{X} & =\left(\frac{[1.5(300)+3.4+0]\left(\frac{1}{12}\right)(0.1)\left(\frac{24}{2}\right)^{3}}{(29,000,000)(.000162)+0.061(50)\left(\frac{24}{2}\right)^{3}}\right) \\
& =7.8 \text { in }
\end{aligned}
$$

Percent deflection:

$$
\% \frac{\Delta \mathrm{X}}{\mathrm{D}}=\frac{\Delta \mathrm{X}}{\mathrm{D}_{0}} \times 100=\frac{7.8}{24} \times 100=32.7 \%>5 \% \text { for unlined pipe }
$$

Since the deflection is excessive, try a wall thickness, $t$, of $3 / 16$ in

$$
\begin{aligned}
& \mathrm{I}_{\mathrm{pw}}=\frac{\mathrm{t}^{3}}{12}=\frac{0.1875^{3}}{12}=0.000549 \mathrm{in}^{4} / \mathrm{in} \\
& \begin{aligned}
\Delta \mathrm{X} & =\left[\frac{[1.5(3,600)+41+0]\left(\frac{1}{12}\right)(0.1)\left(\frac{24}{2}\right)^{3}}{(29,000,000)(0.000549)+0.061(50)\left(\frac{24}{2}\right)^{3}}\right] \\
& =3.69 \mathrm{in}
\end{aligned} \\
& \begin{aligned}
\% \frac{\Delta \mathrm{X}}{\mathrm{D}_{\mathrm{o}}} & =\frac{\Delta \mathrm{X}}{\mathrm{D}_{\mathrm{o}}} \times 100 \\
& =\frac{3.69}{24} \times 100 \\
& =15.4>5 \% \quad \text { for an unlined pipe }
\end{aligned}
\end{aligned}
$$

## Design example 2-Steel pipe (continued)

Since the deflection is still excessive, try a wall thickness, $t$ of $5 / 16$ in

$$
\begin{aligned}
\mathrm{I}_{\mathrm{pw}} & =\frac{\mathrm{t}^{3}}{12} \\
& =\frac{0.3125^{3}}{12} \\
& =0.00254 \mathrm{in}^{4} / \mathrm{in} \\
\Delta \mathrm{X} & =\left[\frac{(1.5(3,600)+4.1+0)\left(\frac{1}{12}\right)(0.1)\left(\frac{24}{2}\right)^{3}}{(29,000,000)(0.00254)+0.061(50)\left(\frac{24}{2}\right)^{3}}\right] \\
& =0.99 \mathrm{in} \\
\begin{aligned}
\% & \frac{\Delta \mathrm{X}}{\mathrm{D}_{\mathrm{o}}}
\end{aligned} & =\frac{\Delta \mathrm{X}}{\mathrm{D}_{\mathrm{o}}} \times 100 \quad \text { for an unlined pipe, therefore } \mathrm{t}=\frac{5}{16} \text { is } \mathrm{OK} \\
& =\frac{0.99}{24} \times 100 \quad \\
& =4.1<5 \%
\end{aligned}
$$

D. Allowable buckling pressure

From equation 52-43:

$$
\mathrm{q}_{\mathrm{a}}=\frac{1}{\mathrm{FS}}\left(32 \mathrm{R}_{\mathrm{w}} \mathrm{~B}^{\prime} \mathrm{E}^{\prime} \frac{E I_{\mathrm{pw}}}{\mathrm{D}_{\mathrm{o}}^{3}}\right)^{1 / 2}
$$

where:

$$
\begin{aligned}
\frac{\mathrm{h}}{\left(\frac{\mathrm{D}_{\mathrm{o}}}{12}\right)} & =\frac{15}{12}=7.5 \geq 2, \text { so F.S. } 2.5 \\
\mathrm{R}_{\mathrm{w}} & =1.0 \\
\mathrm{~B}^{\prime} & =\frac{1}{1+4 \mathrm{e}^{(-0.065 \mathrm{hh})}}=\frac{1}{1+4 \mathrm{e}^{(-0.065 \times 15)}}=0.398 \\
\mathrm{q}_{\mathrm{a}} & =\frac{1}{2.5}\left(32(1.0)(0.398)(50) \frac{(29,000,000)(0.00254)}{(24)^{3}}\right)^{1 / 2} \\
& =23.3 \mathrm{lb} / \mathrm{in}^{2}=3,355 \mathrm{lb} / \mathrm{ft}^{2}>1,820 \mathrm{lb} / \mathrm{ft}^{2} \quad \text { O.K. }
\end{aligned}
$$

Conclusion: The 24-inch steel pipe should be made of ASTM A 53, grade A steel or stronger with a minimum wall thickness of $5 / 16$ inch.

## Design Example 3-Corrugated Metal Pipe

Problem: A 12-inch corrugated aluminum pipe will be installed as outlet pipe in an earthen dam. The top of the pipe will be 3 feet below the top of the dam. The dam will be constructed of an SC material compacted to 90 percent of standard Proctor. Heavy traffic with wheel loads up to 16,000 pounds will cross the embankment.


Assumptions: 1. The pipe is made of aluminum with a minimum yield stress of $24,000 \mathrm{lb} / \mathrm{in}^{2}$
2. Assume unit weight of soil $=120 \mathrm{lb} / \mathrm{ft}^{3}$
3. $\mathrm{E}^{\prime}=400 \mathrm{lb} / \mathrm{in}^{2}$
4. Assume $\mathrm{D}_{\mathrm{o}}$ and $\mathrm{D}_{\mathrm{i}}=12$ in

Determine: A. External soil and wheel loading
B. Thrust
C. Required cross sectional area of $22 / 3$ by $1 / 2$ corrugated pipe
D. Check buckling
E. Check seam strength
F. Check flexibility factor

Solution: A. External loads
From equation 52-17, soil pressure is

$$
\begin{aligned}
& \mathrm{P}_{\mathrm{s}}=\gamma_{\mathrm{s}} \times \mathrm{h} \\
& 120 \times 3=360 \mathrm{lb} / \mathrm{ft}^{2}
\end{aligned}
$$

## Design example 3-Corrugated metal pipe (continued)

From equation 52-19 and 52-21, and assuming t=0.060 inch, since $\quad D_{o}-\mathrm{t}<2.67 \mathrm{~h} \times 12$

$$
\begin{aligned}
& 12-0.060<2.67(3) \times 12 \\
& 11.94<96.1
\end{aligned}
$$

wheel loading is

$$
\begin{aligned}
& \mathrm{W}_{\mathrm{L}}=\frac{0.48 \mathrm{P}_{\mathrm{L}} \mathrm{I}_{f}\left(\frac{\mathrm{D}_{\mathrm{o}}-\mathrm{t}}{12}\right)^{2}}{2.67 \mathrm{~h}^{3}}\left[\frac{2.67(\mathrm{~h})}{\left(\frac{\mathrm{D}_{\mathrm{o}}-\mathrm{t}}{12}\right)}-0.5\right] \\
& \mathrm{W}_{\mathrm{L}}=\frac{0.48(16,000)(1.0)\left(\frac{12-0.060}{12}\right)^{2}}{2.67(3)^{3}}\left[\frac{2.67(3)}{\left(\frac{12-0.060}{12}\right)}-0.5\right]
\end{aligned}
$$

$$
=796 \mathrm{lb} / \mathrm{ft} \text { of pipe }
$$

$$
\mathrm{P}_{\mathrm{w}}=\frac{\mathrm{W}_{\mathrm{L}}}{\left(\frac{\mathrm{D}_{\mathrm{o}}}{12}\right)}=\frac{796}{\frac{12}{12}}=796 \mathrm{lb} / \mathrm{ft}^{2}
$$

Design pressure: $P=P_{S}+P_{w}+P_{v}$

$$
=360+796+0=1,156 \mathrm{lb} / \mathrm{ft}^{2}
$$

B. Thrust

From equation 52-45:

$$
\begin{aligned}
\mathrm{T}_{\mathrm{pw}} & =\frac{\mathrm{P} \times \frac{\mathrm{D}_{\mathrm{i}}}{12}}{2} \\
\mathrm{~T}_{\mathrm{pw}} & =\frac{1,156 \times \frac{12}{12}}{2} \\
\mathrm{~T}_{\mathrm{pw}} & =578 \mathrm{lb} / \mathrm{ft} \text { of pipe }
\end{aligned}
$$

C. Required cross-sectional area from equation 52-46
$\mathrm{A}_{\mathrm{s}}=\frac{\mathrm{T}_{\mathrm{pw}} \times \mathrm{FS}}{f_{\mathrm{y}}}$
$\mathrm{A}_{\mathrm{s}}=\frac{578 \times 2}{24,000}$
$\mathrm{A}_{\mathrm{s}}=0.048 \mathrm{in}^{2} / \mathrm{ft}<0.775 \mathrm{in}^{2} /$ ft for a 16 gage $\left(0.060\right.$ in thick) $2 \frac{2}{3} \times \frac{1}{2}$ corrugations O.K.

## Design example 3-Corrugated metal pipe (continued)

D. Buckling

From equation 52-47 and 52-48:
Since

$$
\begin{aligned}
\mathrm{D}_{\mathrm{i}} & <\frac{\mathrm{r}}{\mathrm{k}} \sqrt{\frac{24 \mathrm{E}}{f_{\mathrm{u}}}}=\frac{0.1712}{0.22} \sqrt{\frac{(24) 10,000,000}{34,000}}=65 \\
f_{\mathrm{c}} & =f_{\mathrm{u}}-\frac{f_{\mathrm{u}}^{2}}{48 \mathrm{E}}\left(\frac{\mathrm{kD}_{\mathrm{i}}}{\mathrm{r}}\right)^{2} \\
& =34,000-\frac{34,000^{2}}{48(10,000,000)}\left(\frac{(0.22) 12}{0.1712}\right)^{2} \\
& =33,427 \mathrm{lb} / \mathrm{in}^{2}>f_{\mathrm{y}} \text { of } 24,000 \mathrm{lb} / \mathrm{in}^{2} \text { so wall area is O.K. }
\end{aligned}
$$

E. Seam strength

From section 636.5204(c)(3), if helical lockseam or welded-seam (for steel) pipe is used, this criterion does not apply. For riveted corrugated pipe, using equation 52-51,

$$
\begin{aligned}
\mathrm{SS}= & \mathrm{T}_{\mathrm{pw}} \times \mathrm{FS} \\
\mathrm{SS}= & 578 \times 3.0=1,734 \mathrm{lbf} / \mathrm{ft}<9,000 \mathrm{lb} / \mathrm{ft} \text { for single rivets } \\
& (\text { from appendix } 52 \mathrm{D}, \text { table } 52 \mathrm{D}-4)
\end{aligned}
$$

F. Flexibility factor

From section 636.5204(c)(4):

$$
\begin{aligned}
\mathrm{FF} & =\frac{\mathrm{D}_{\mathrm{i}}{ }^{2}}{\mathrm{EI}_{\mathrm{pw}}}=\frac{12^{2}}{(10,000,000) \cdot 001892} \\
& =0.0076<0.031 \text { from appendix } 52-\mathrm{E}
\end{aligned}
$$

Conclusion: A 12-inch diameter, 16-gage, corrugated aluminum pipe with $22 / 3 \times 1 / 2$ corrugation is acceptable.

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## Design Example 4 - Ductile Iron Pipe

Problem: A 24-inch ductile iron pipe will be installed as the primary outlet pipe in an earthen dam. The top of the pipe will be 20 feet below the top of the dam. The dam will be constructed of an SC material compacted to 90 percent of the standard Proctor density.


Assumptions: 1. Assume unit weight of soil $=120 \mathrm{lb} / \mathrm{ft}^{3}$.
2. Since the pipe will be installed in an embankment dam of SC soils, the design values for laying condition 3 will be used, $\mathrm{E}^{\prime}=400 \mathrm{lb} / \mathrm{in}^{2}, \mathrm{~K}_{\mathrm{b}}=0.189$, and $\mathrm{K}_{\mathrm{x}}=0.103$.
3. A nominal pipe thickness of 0.33 inch will be assumed since this is the minimum pipe thickness for 24 -inch pipe as shown in appendix 52 F .
4. The allowable ring deflection is 5 percent.

Determine: A. External soil load
B. Check ring bending stress
C. Check ring deflection

Solution: A. External loads
From equation 52-17, soil pressure is

$$
\begin{aligned}
& \mathrm{P}_{\mathrm{s}}=\gamma_{\mathrm{s}} \times \mathrm{h} \\
& 120 \times 20=2,400 \mathrm{lb} / \mathrm{ft}^{2}
\end{aligned}
$$

From equation 52-18, design pressure is

$$
\begin{aligned}
\mathrm{P} & =\mathrm{P}_{\mathrm{S}}+\mathrm{P}_{\mathrm{W}}+\mathrm{P}_{\mathrm{v}} \\
& =2,400+0+0=2,400 \mathrm{lb} / \mathrm{ft}^{2}
\end{aligned}
$$

B. Ring bending stress
$\mathrm{t}_{\mathrm{n}}=$ nominal thickness from appendix 52 F - service allowance - casting tolerance $=0.33-0.08-0.07=0.18 \mathrm{in}$.

## Design example 4-Ductile iron pipe (continued)

From equation 52-53:


$$
\begin{aligned}
\mathrm{P}_{\mathrm{bs}} & =11.43 \mathrm{lb} / \mathrm{in}^{2} \\
& =11.43 \times 144 \\
& =1,645 \mathrm{lb} / \mathrm{ft}^{2}
\end{aligned}
$$

Since $2,400 \mathrm{lb} / \mathrm{ft}^{2}>1,645 \mathrm{lb} / \mathrm{ft}^{2}$, a thicker pipe wall is required. Assume a nominal pipe wall thickness of 0.43 in .

$$
\left.\begin{array}{rl}
\mathrm{t}_{\mathrm{n}} & =0.43-0.08-0.07=0.28 \mathrm{in} \\
\mathrm{P}_{\mathrm{bs}} & =\frac{48,000}{} \\
3\left(\frac{24}{0.28}\right)\left(\frac{24}{0.28}-1\right)\left[0.189-\frac{0.103}{\frac{8(24,000,000)}{400\left(\frac{24}{0.28}-1\right)^{3}}+0.732}\right] \\
\mathrm{P}_{\mathrm{bs}} & =18.16 \mathrm{lb} / \mathrm{in}^{2} \\
& =18.16 \times 144 \\
& =2,615 \mathrm{lb} / \mathrm{ft}^{2}
\end{array}\right]
$$

C. Ring deflection
$\mathrm{t}_{1}=$ nominal thickness from appendix $52-\mathrm{F}\left(\mathrm{t}_{\mathrm{n}}\right)$ - casting tolerance

$$
=0.43-0.07
$$

$$
=0.36 \text { in }
$$

Design example 4-Ductile iron pipe (continued)
From equation 52-56:

$$
\begin{aligned}
& \mathrm{P}_{\mathrm{rd}}=\frac{\frac{\Delta \mathrm{X}}{\mathrm{D}_{\mathrm{o}}}}{12 \mathrm{~K}_{\mathrm{x}}}\left[\frac{8 \mathrm{E}}{\left(\frac{\mathrm{D}_{\mathrm{o}}}{\mathrm{t}_{1}}-1\right)^{3}}+0.732 \mathrm{E}\right] \\
& \mathrm{P}_{\mathrm{rd}}=\frac{.05}{12(0.103)}\left[\frac{8(24,000,000)}{\left(\frac{24}{0.36}-1\right)^{3}}+0.732(400)\right] \\
& \mathrm{P}_{\mathrm{rd}}=39 \mathrm{lb} / \mathrm{in}^{2}=39 \times 144=5,655 \mathrm{lb} / \mathrm{ft}^{2} \\
& \mathrm{P} \leq \mathrm{P}_{\mathrm{d}} \times 144 \\
& 2,400<5,655 \mathrm{lb} / \mathrm{ft}^{2}
\end{aligned}
$$

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## Design Example 5 - Thrust Block



Assumptions: 1. The allowable bearing capacity will be estimated.
2. The center of the thrust block will be at the centerline of the pipe.

Determine: A. Thrust force on the pipe bend
B. Allowable soil bearing pressure
C. Area of thrust block required

## Design example 5-Thrust block (continued)

Solution: A. Thrust force on the pipe bend
From figure 52-14, the thrust force on a bend may be estimated by:

$$
\mathrm{T}=2 \mathrm{PA} \sin \left(\frac{\theta}{2}\right)=2 \times 50 \times \frac{\pi \times 12^{2}}{4} \sin \left(\frac{90}{2}\right)=7,993 \mathrm{lb}
$$

B. Allowable soil bearing pressure

The depth to the center of the thrust block is

$$
\mathrm{h}+\frac{\mathrm{D}_{\mathrm{o}}}{2}=4+\frac{12}{12 \times 2}=4.5 \mathrm{ft}
$$

From table 52-6 the allowable bearing capacity for silty clay soil at a depth of 4 feet is $950 \mathrm{lb} / \mathrm{ft}^{2}$ and $1,200 \mathrm{lb} / \mathrm{ft}^{2}$ at 5 . The allowable bearing capacity at 4.5 feet may be determined by an average.

$$
\frac{950+1,200}{2}=1,075 \mathrm{lb} / \mathrm{ft}^{2}
$$

C. Area of thrust block required

From equation 52-75, the area of the thrust block required is:

$$
\mathrm{A}_{\mathrm{T}}=\frac{\mathrm{T}}{\mathrm{q}_{\mathrm{all}}}=\frac{7,993}{1,075}=7.43 \mathrm{ft}^{2}
$$

Conclusion: The thrust block should be a minimum of 7.43 square feet. A block 2.75 feet by 2.75 feet would be sufficient to resist the thrust force at the 90 degree bends.

## Design Example 6 - Longitudinal Bending

Problem: An 8-inch diameter HDPE pipe will be installed for an irrigation pipeline system. The alignment of the pipe requires a change of direction. It is desired to accomplish the change of direction by using the allowable longitudinal bending of the pipe. The pipe will have an internal pressure (including surge pressure) in the pipe of 80 pounds per square inch.


Assumptions: 1. The pipe material will be PE3408.
2. Since this is pressure pipe, it is fusion welded.
3. The pipe meets ASTM D 3035 and has a SDR of 21 to provide a pressure rating of $80 \mathrm{lb} / \mathrm{m}^{2}$.
4. The modulus of elasticity of the HDPE is $110,000 \mathrm{lb} / \mathrm{in}^{2}$.

Determine: A. Allowable bending stress for the pipe
B. Minimum bending radius of the pipe

## Solution: A. Allowable bending stress

From table $52 \mathrm{C}-1$, the hydrostatic design basis (HDB) is $1,600 \mathrm{lb} / \mathrm{in}^{2}$.
From section 636.5208, the allowable bending stress is

$$
\begin{aligned}
& \mathrm{S}_{\text {ball }}=\frac{\left(\mathrm{HDB}-\frac{\mathrm{HDB}}{2}\right) \times \mathrm{T}_{\mathrm{f}}}{\mathrm{FS}} \\
& \mathrm{~S}_{\text {ball }}=\frac{\left(1,600-\frac{1,600}{2}\right) \times 1}{2.0}=400 \mathrm{lb} / \mathrm{in}^{2}
\end{aligned}
$$

Design example 6-Longitudinal bending (continued)
B. Minimum bending radius

From equation 52-76, the minimum bending radius is

$$
\mathrm{R}_{\mathrm{b}}=\frac{\mathrm{ED}_{\mathrm{o}}}{2 \mathrm{~S}_{\text {ball }}}
$$

From appendix 52 C , table $52 \mathrm{C}-5$, the $\mathrm{D}_{\mathrm{o}}=8.625$ inches

$$
\mathrm{R}_{\mathrm{b}}=\frac{110,000 \times 8.625}{2 \times 400}=1,185 \mathrm{in}=98.8 \mathrm{ft}
$$

Conclusion: The minimum longitudinal bending radius of the HDPE pipe made of PE3408 material with an SDR of 21 is 99 feet.

## Design Example 7 - Aboveground Pipe

Problem: A 12-inch diameter PVC irrigation water supply pipe will be supported on concrete saddles. The pipe will have an internal pressure (including surge pressure) in the pipe of 60 pounds per square inch. It is desired to space a saddle support every 10 feet with the pipe restrained at both ends. The temperature of the water will vary by 30 degrees Fahrenheit.


Assumptions: 1. The pipe material will be PVC2112 with HDB of $2,500 \mathrm{lb} / \mathrm{in}^{2}$.
2. Since this is pressure pipe, the joints are solvent cemented.
3. The pipe meets ASTM D 2241 and requires a SDR of 41 to provide a pressure rating of $60 \mathrm{lb} / \mathrm{in}^{2}$ or greater.
4. The modulus of elasticity of the PVC is $400,000 \mathrm{lb} / \mathrm{in}^{2}$, and long-term modulus of elasticity is $110,000 \mathrm{lb} / \mathrm{in}^{2}$.
5. Conservatively assume density of PVC is equal to that of water.
6. The saddle angle will be 120 degrees.

Determine: A. Maximum theoretical deflection and allowable deflection
B. Hoop stress caused by internal pressure
C. Bending stress because of unsupported length
D. Localized stress at the saddle
E. Stress caused by temperature change
F. Total stress at the saddle support
G. Allowable stress in the pipe wall

Solution: A. Maximum theoretical deflection and allowable deflection From table 52C-2, a 12-inch diameter, SDR 41 PVC pipe has a $D_{o}=12.240$ inches and $\mathrm{t}=0.299$ inch.

From equation 52-64, the theoretical maximum deflection is

$$
\mathrm{y}=\frac{0.0130 \times \mathrm{w} \times \mathrm{L}_{\text {span }}^{4}}{\mathrm{E}_{\text {long }} \times \mathrm{I}}
$$

$\mathrm{w}=$ weight of pipe filled with liquid

$$
\mathrm{w}=\frac{\pi \mathrm{D}_{\mathrm{o}}{ }^{2}}{4} \times \gamma_{\mathrm{w}}=\frac{\pi \times 12.24^{2}}{4} \times \frac{62.4}{12^{3}}=4.25 \mathrm{lb} / \mathrm{in}
$$

## Design example 7-Aboveground pipe (continued)

$$
\begin{aligned}
& \mathrm{I}=\frac{\pi}{64}\left(\mathrm{D}_{\mathrm{o}}{ }^{4}-\mathrm{D}_{\mathrm{i}}^{4}\right)=\frac{\pi}{64}\left[12.24^{4}-(12.24-2(0.299))^{4}\right]=199 \mathrm{in}^{4} \\
& \mathrm{y}=\frac{0.0130 \times \mathrm{w} \times \mathrm{L}^{4}}{\mathrm{E}_{\text {long }} \times \mathrm{I}}=\frac{0.0130 \times 4.25 \times(10 \times 12)^{4}}{140,000 \times 199}=0.41 \mathrm{in}
\end{aligned}
$$

The maximum recommended deflection for PVC pipe is 0.50 percent of the span:

$$
0.005 \times(10 \times 12)=0.60 \text { in }<0.41 \text { in }
$$

B. Hoop stress from internal pressure

From equation 52-66, the hoop stress from internal pressure is

$$
\mathrm{S}_{\mathrm{p}}=\frac{\mathrm{P} \times \mathrm{D}_{\mathrm{i}}}{2 \times \mathrm{t}}=\frac{60 \times(12.24-2(0.299))}{2 \times 0.299}=1,168 \mathrm{lb} / \mathrm{in}^{2}
$$

C. Bending stress caused by unsupported length

From equation 52-62, the bending stress caused by unsupported length is

$$
\mathrm{S}_{\mathrm{b}}=\frac{0.0625 \mathrm{wL}_{\text {span }}^{2} \mathrm{D}_{\mathrm{o}}}{\mathrm{I}}=\frac{0.0625 \times 4.25 \times\left[(10 \times 12)^{2} \times 12.24\right]}{199}=235 \mathrm{lb} / \mathrm{in}^{2}
$$

D. Localized stress at the saddle

From equation 52-67, the localized stress at the saddle is

$$
\begin{aligned}
& S_{1}=k_{\text {support }} \frac{R_{\text {support }}}{t^{2}} \ln \left(\frac{R_{o}}{t}\right) \\
& \mathrm{k}_{\text {sup port }}=0.02-0.00012(\beta-90)=0.02-0.00012(120-90)=0.0164 \\
& \mathrm{R}_{\text {sup port }}=\frac{\mathrm{wL}_{\text {span }}}{2}=\frac{4.25 \times(10 \times 12)}{2}=255 \mathrm{lb} \\
& \mathrm{~S}_{1}=0.0164 \frac{255}{0.299^{2}} \ln \left(\frac{\frac{12.24}{2}}{0.299}\right)=141 \mathrm{lb} / \mathrm{in}^{2}
\end{aligned}
$$

E. Stress caused by temperature change

From section 636.5205 and table 52-5:

$$
\mathrm{S}_{\mathrm{EC}}=\mathrm{E} \alpha \Delta \mathrm{~T}=400,000 \times .00003 \times 30=360 \mathrm{lb} / \mathrm{in}^{2}
$$

Design example 7-Aboveground pipe (continued)
F. Total stress at the saddle support from equation 52-69

$$
\mathrm{S}_{\mathrm{T}}=\mathrm{vS}_{\mathrm{p}}+\mathrm{S}_{\mathrm{b}}+\mathrm{S}_{1}+\mathrm{S}_{\mathrm{EC}}=0.38 \times 1,168+235+141+360=1,179 \mathrm{lb} / \mathrm{in}^{2}
$$

G. Allowable stress in the pipe wall

From section 636.5206(e), allowable stress in the pipe wall is

$$
\begin{aligned}
& \mathrm{S}_{\mathrm{all}}=\frac{\mathrm{HDB} \times \mathrm{T}_{f}}{\mathrm{FS}}=\frac{2,500 \times 1}{2}=1,250 \mathrm{lb} / \mathrm{in}^{2} \\
& 1,250 \mathrm{lb} / \mathrm{in}^{2}>1,179 \mathrm{lb} / \mathrm{in}^{2}
\end{aligned}
$$

Conclusion: A PVC pipe of PVC 2112 with SDR of 41 will span 10 feet with an acceptable allowable deflection and allowable stress in the pipe wall.

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National Engineering Handbook

## Design Example 8 - Plastic Pipe Siphon

Problem: A 10-inch diameter PVC plastic irrigation pipe (PIP) with SDR of 51 will be installed for an irrigation pipeline system. The pipe will be buried under 2 feet of soil. The line acts as a siphon with a vacuum pressure of 7 pounds per square inch. The excavation will be backfilled and slightly compacted to approximately 85 percent of the Standard Proctor with CL soils that have less than 25 percent coarse particles.


Assumptions: 1. The pipe has an outside diameter of 10.2 inches and thickness of 0.2 inch, from table 52C-2.
2. The PVC pipe will be PVC 1120.
3. PVC has a short-term modulus of elasticity of 400,000 pounds per square inch and a long-term modulus of elasticity of 140,000 pounds per square inch. The long-term value will be used for buckling since the loads and vacuum pressure are permanent.
4. Assume unit weight of soil $=100$ pounds per cubic foot.
5. Slightly compacted CL soils, $\mathrm{E}^{\prime}=200$ pounds per square inch.
6. Deflection lag factor for soil loads, $\mathrm{D}_{\mathrm{L}}=1.5$.

Determine: A. Soil pressure on the pipe
B. Percent deflection of the pipe caused by soil and vacuum pressure
C. Allowable buckling pressure
D. Reduced allowable buckling pressure

Solution: A. Soil pressure on the pipe From equation 52-17

$$
\begin{aligned}
& \mathrm{P}_{\mathrm{S}}=\gamma_{\mathrm{s}} \times \mathrm{h} \\
& 100 \times 2=200 \mathrm{lb} / \mathrm{ft}^{2} \\
& \mathrm{P}_{\mathrm{v}}=7 \mathrm{lb} / \mathrm{in}=7 \times 144=1,008 \mathrm{lb} / \mathrm{ft}^{2}
\end{aligned}
$$

## Design example 8-Plastic pipe siphon (continued)

B. Percent deflection of the pipe from equation $52-29$

$$
\begin{aligned}
& \frac{\% \Delta X}{D}=\frac{\left(\mathrm{D}_{\mathrm{L}} \mathrm{P}_{\mathrm{s}}+\mathrm{P}_{\mathrm{L}}+\mathrm{P}_{\mathrm{v}}\right)\left(\frac{1}{144}\right) \mathrm{k}(100)}{\left[\left(\frac{2 \mathrm{E}}{3(\mathrm{SDR}-1)^{3}}\right)+0.061 \mathrm{E}^{\prime}\right]} \\
& \frac{\% \Delta \mathrm{X}}{\mathrm{D}}=\frac{(1.5 \times 200+0+1,008)\left(\frac{1}{144}\right)(0.1)(100)}{\left[\left(\frac{2(400,000)}{3(51-1)^{3}}\right)+0.061(200)\right]} \\
& \frac{\% \Delta X}{\mathrm{D}}=6.33 \%
\end{aligned}
$$

C. Allowable buckling pressure

From equation 52-33

$$
q_{a}=\frac{1}{F S}\left(32 R_{w} B^{\prime} E^{\prime} \frac{E_{\text {Long }} I_{p w}}{D_{o}{ }^{3}}\right)^{\frac{1}{2}}
$$

where:

$$
\begin{aligned}
& \left(\frac{\mathrm{h}}{\left(\frac{\mathrm{D}_{\mathrm{o}}}{12}\right)^{2}}\right)=\frac{2}{\frac{12}{10.2}}=2.4 \geq 2 \text { so F.S. } 2.5 \\
& \mathrm{R}_{\mathrm{w}}=1.0 \\
& \mathrm{~B}^{\prime}=\frac{4\left(\mathrm{~h}^{2}+\left(\frac{\mathrm{D}_{\mathrm{o}}}{12}\right) \mathrm{h}\right)}{1.5\left(2 \mathrm{~h}+\frac{\mathrm{D}_{\mathrm{o}}}{12}\right)^{2}}=\frac{4\left[2^{2}+\left(\frac{10.2}{12}\right) 2\right]}{1.5\left(2^{2}+10.2\right)^{2}}=0.640 \\
& \mathrm{I}_{\mathrm{pw}}=\frac{\mathrm{t}^{3}}{12}=\frac{0.2^{3}}{12}=0.00067 \mathrm{in}^{4} \\
& \mathrm{q}_{\mathrm{a}}=\frac{1}{2.5}\left[32(1.0)(0.646)(200) \frac{(140,000)(0.00067)}{(10.20)^{3}}\right]^{\frac{1}{2}} \\
& =7.64 \mathrm{lb} / \mathrm{in}^{2}
\end{aligned}
$$

## Design example 8-Plastic pipe siphon (continued)

D. Reduced allowable buckling pressure

From equation 52-34, the reduction factor for the allowable buckling pressure caused by deflection of the pipe is

$$
\begin{aligned}
\mathrm{C} & =\left[\frac{\left(1-\frac{\% \Delta \mathrm{X}}{\mathrm{D}} \frac{1}{100}\right)}{\left(1+\frac{\% \Delta \mathrm{X}}{\mathrm{D}} \frac{1}{100}\right)^{2}}\right]^{3} \\
& =\left[\frac{\left(1-6.33 \frac{1}{100}\right)}{\left(1+6.33 \frac{1}{100}\right)^{2}}\right]^{3} \\
& =0.57
\end{aligned}
$$

The reduced allowable buckling pressure caused by the deflected pipe is

$$
\begin{aligned}
\mathrm{q}_{\mathrm{a}} \mathrm{C} & =7.64 \times 0.57 \\
& =4.4 \mathrm{lb} / \mathrm{in}^{2}<\mathrm{P}_{\mathrm{s}}+\mathrm{P}_{\mathrm{v}} \\
\mathrm{P}_{\mathrm{s}}+\mathrm{P}_{\mathrm{v}} & =1.38+7 \\
\mathrm{P}_{\mathrm{s}}+\mathrm{P}_{\mathrm{v}} & =8.38 \mathrm{lb} / \mathrm{in}^{2}>4.4 \mathrm{lb} / \mathrm{in}^{2} \quad \text { Not O.K. }
\end{aligned}
$$

The PVC PIP with an SDR of 51 and backfilled as assumed does not provide adequate resistance to buckling. A higher quality backfill or pipe with a lower SDR (thicker wall) should be investigated. Try an SDR of 41 with $\mathrm{t}=0.299$ inch, from table 52C-2.

B1. Percent deflection of the SDR 41 pipe
From equation 52-29

$$
\begin{aligned}
& \frac{\% \Delta \mathrm{X}}{\mathrm{D}}=\frac{(1.5 \times 200+0+1,008)\left(\frac{1}{144 \mathrm{~s}}\right)(0.1)(100)}{\left[\left(\frac{2(400,000)}{3(41-1)^{3}}\right)+0.061(200)\right]} \\
& \frac{\% \Delta \mathrm{X}}{\mathrm{D}}=5.54 \%
\end{aligned}
$$

## Design example 8-Plastic pipe siphon (continued)

C1. Allowable buckling pressure of SDR 41 pipe

$$
\begin{aligned}
\mathrm{I}_{\mathrm{pw}} & =\frac{\mathrm{t}^{3}}{12} \\
& =\frac{0.299^{3}}{12} \\
& =0.00223 \mathrm{in}^{4} \\
\mathrm{q}_{\mathrm{a}} & =\frac{1}{2.5}\left[32(1.0)(0.646)(200) \frac{(140,000)(0.00223)}{(10.20)^{3}}\right]^{\frac{1}{2}} \\
& =13.9 \mathrm{lb} / \mathrm{in}^{2}
\end{aligned}
$$

D1. Reduced allowable buckling pressure of SDR 41 pipe
The reduction factor for the allowable buckling pressure from the deflection of the pipe is

$$
\mathrm{C}=\left[\frac{\left(1-5.54 \frac{1}{100}\right)}{\left(1+5.54 \frac{1}{100}\right)^{2}}\right]^{3}=0.61
$$

The reduced allowable buckling pressure caused by the deflected pipe is

$$
\begin{aligned}
\mathrm{q}_{\mathrm{a}} \mathrm{C} & =13.95 \times 0.61 \\
& =8.5 \mathrm{lb} / \mathrm{in}^{2}>\mathrm{P}_{\mathrm{s}}+\mathrm{P}_{\mathrm{v}} \\
\mathrm{P}_{\mathrm{s}}+\mathrm{P}_{\mathrm{v}} & =1.38+7 \\
& =8.38 \mathrm{lb} / \mathrm{in}^{2}<8.5 \quad \text { O.K. }
\end{aligned}
$$

Conclusion: The PVC PIP with an SDR of 51 and backfilled as assumed does not provide adequate resistance to buckling. Using a lower SDR of 41 provides adequate resistance to buckling.

## Design Example 9-Plastic Pipe During Construction

Problem: An 18-inch diameter HDPE pipe will be installed as an outlet pipe in an earthen dam. Heavy construction equipment with wheel loads up to 16,000 pounds will be allowed to traverse the pipe once 2 feet of fill has been placed over the top of the pipe. The top of the pipe will be 10 feet below the top of the completed dam. The dam will be constructed of an SC material compacted to 90 percent of the standard Proctor density.


Assumptions: 1. The pipe is outside diameter controlled.
2. The HDPE pipe will be PE 3408 with a Hydrostatic Design Basis of $1,600 \mathrm{lb} / \mathrm{in}^{2}$ (see app. 52C)
3. Assume unit weight of soil $=120 \mathrm{lb} / \mathrm{ft}^{3}$
4. Since the pipe will be installed in an embankment dam of SC soil, $E^{\prime}=400 \mathrm{lb} / \mathrm{in}^{2}$.
5. The allowable deflection is 5 percent.

Determine: A. External soil and wheel load during construction
B. Required wall area for external load during construction
C. Deflection during construction
D. Allowable buckling during construction
E. Strain during construction
F. External soil load upon completion of the dam
G. Required wall area for completed external load
H. Deflection upon completion of the dam
I. Allowable buckling upon completion of the dam
J. Strain upon completion of the dam

## Design example 9-Plastic pipe during construction (continued)

## Solution: <br> A. External soil and wheel load during construction

From equation 52-17, the soil pressure due to 2 feet of soil is

$$
\begin{aligned}
\mathrm{P}_{\mathrm{s}} & =\gamma_{\mathrm{s}} \times \mathrm{h} \\
& =120 \times 2 \\
& =240 \mathrm{lb} / \mathrm{ft}^{2}
\end{aligned}
$$

Wheel loading: From table 52C-5 of appendix 52C, an 18 -inch PE pipe with a SDR of 17 has a thickness, $\mathrm{t}=1.059 \mathrm{in}$. From section 636.5203 (b) and equations 52-19 and 52-21:

$$
\mathrm{D}_{\mathrm{o}}-\mathrm{t}<2.67 \mathrm{~h} \times 12
$$

Since

$$
\begin{aligned}
& 18-1.161<2.67(2) \times 12 \\
& 16.84<64.1 \\
& \mathrm{~W}_{\mathrm{L}}=\frac{0.48 \mathrm{P}_{\mathrm{L}} \mathrm{I}_{\mathrm{f}}\left(\frac{\mathrm{D}_{\mathrm{o}}-\mathrm{t}}{12}\right)^{2}}{2.67 \mathrm{~h}^{3}}\left[\frac{2.67 \mathrm{~h}}{\left(\frac{\mathrm{D}_{\mathrm{o}}-\mathrm{t}}{12}\right)}-0.5\right]
\end{aligned}
$$

Since the depth of cover is 2.0 , the $\mathrm{I}_{\mathrm{f}}$ is 1.2 .

$$
\begin{aligned}
& \mathrm{W}_{\mathrm{L}}=\frac{0.48(16,000)(1.2)\left(\frac{18-1.059}{12}\right)^{2}}{2.67(2)^{3}}\left[\frac{2.67(2)}{\left(\frac{18-1.059}{12}\right)}-0.5\right] \\
& W_{L}=2,822 \mathrm{lbs} . / \mathrm{ft} \text { of pipe } \\
& \mathrm{P}_{\mathrm{W}}=\frac{\mathrm{W}_{\mathrm{L}}}{\left(\frac{\mathrm{D}_{\mathrm{o}}}{12}\right)} \\
&=\frac{2,822}{\frac{18}{12}}=1,881 \mathrm{lb} / \mathrm{ft}^{2}
\end{aligned}
$$

Design Pressure :

$$
\begin{aligned}
\mathrm{P} & =\mathrm{P}_{\mathrm{S}}+\mathrm{P}_{\mathrm{W}}+\mathrm{P}_{\mathrm{v}} \\
& =240+1,881+0=2,121 \mathrm{lb} / \mathrm{ft}^{2}
\end{aligned}
$$

## Design example 9-Plastic pipe during construction (continued)

From equation 52-26:
Thrust:

$$
\begin{aligned}
\mathrm{T}_{\mathrm{pw}} & =\frac{\mathrm{P} \times\left(\frac{\mathrm{D}_{\mathrm{o}}}{12}\right)}{2} \\
\mathrm{~T}_{\mathrm{pw}} & =\frac{2,121 \times \frac{18}{12}}{2} \\
\mathrm{~T}_{\mathrm{pw}} & =1,591 \mathrm{lb} / \mathrm{ft} \text { of pipe }
\end{aligned}
$$

B. Required wall area for external load during construction From equation 52-27:

$$
\begin{aligned}
& \mathrm{A}_{\mathrm{pw}}=\frac{\frac{\mathrm{T}_{\mathrm{pw}}}{12}}{\sigma} \\
& \mathrm{~A}_{\mathrm{pw}}=\frac{\frac{1,591}{12}}{800}, \sigma=800{\mathrm{lb} / \mathrm{in}^{2} \text { from appendix } 52 \mathrm{C}, \text { table } 52 \mathrm{C}-1}_{\mathrm{A}_{\mathrm{pw}}=0.166 \mathrm{in}^{2} / \mathrm{in}}
\end{aligned}
$$

Area of an 18-inch pipe with SDR of 17 using equation 52-28

$$
\begin{aligned}
& \mathrm{A}_{\mathrm{pw}}=\frac{\left(\mathrm{D}_{\mathrm{o}}-\mathrm{D}_{\mathrm{i}}\right)}{2} \text { or } \mathrm{t} \\
& \mathrm{~A}_{\mathrm{pw}}=\mathrm{t} \\
& \mathrm{~A}_{\mathrm{pw}}=1.059 \mathrm{in}^{2} / \mathrm{in}>0.166 \mathrm{in}^{2} / \mathrm{in} \quad \text { O.K. }
\end{aligned}
$$

C. Deflection during construction:

From equation 52-29, the percent deflection for solid wall pipe is:

$$
\begin{aligned}
& \frac{\% \Delta X}{D}=\frac{\left(\mathrm{D}_{\mathrm{L}} \mathrm{P}_{\mathrm{S}}+\mathrm{P}_{\mathrm{w}}+\mathrm{P}_{\mathrm{V}}\right)\left(\frac{1}{144}\right) \mathrm{K}(100)}{\left[\left(\frac{2 \mathrm{E}}{3(\mathrm{SDR}-1)^{3}}\right)+0.061 \mathrm{E}^{\prime}\right]} \\
& \frac{\% \Delta \mathrm{X}}{\mathrm{D}}=\frac{(1.5(240)+1881+0)\left(\frac{1}{144}\right)(0.1)(100)}{\left[\left(\frac{2(110,000)}{3(17-1)^{3}}\right)+0.061(400)\right]} \\
& \frac{\% \Delta X}{\mathrm{D}}=3.67 \%<\left(\frac{\% \Delta}{\mathrm{D}}\right)_{\text {allowable }}=5 \% \quad \text { O.K. }
\end{aligned}
$$

## Design example 9-Plastic pipe during construction-(continued)

D. Allowable buckling pressure during construction:

From equation 52-33 using the short-term modulus of elasticity since the wheel loads are short and intermittent:

$$
\begin{aligned}
& \mathrm{q}_{\mathrm{a}}=\frac{1}{\mathrm{FS}}\left(32 \mathrm{R}_{\mathrm{w}} \mathrm{~B}^{\prime} \mathrm{E}^{\prime} \frac{\mathrm{EI}}{\mathrm{D}^{3}}\right)^{1 / 2} \\
& \text { where: } \frac{\mathrm{h}}{\left(\frac{\mathrm{D}_{\mathrm{o}}}{12}\right)}=\frac{2}{\left(\frac{18}{12}\right)}=1.3<2 \\
& \text { F.S. }=3.0 \\
& \mathrm{Rw}=1.0 \\
& \mathrm{~B}^{\prime}=\frac{4\left(\mathrm{~h}^{2}+\left(\frac{\mathrm{D}_{0}}{12}\right) \mathrm{h}\right)}{1.5\left(2 \mathrm{~h}+\frac{\mathrm{D}_{\mathrm{o}}}{12}\right)^{2}}=\frac{4\left(2^{2}+\left(\frac{18}{12}\right) 2\right)}{1.5\left(2 * 2+\frac{18}{12}\right)^{2}}=0.617 \\
& \mathrm{I}_{\mathrm{pw}}=\frac{\mathrm{t}^{3}}{12}=\frac{1.059^{3}}{12}=0.099 \mathrm{in}^{4} / \mathrm{in} \\
& \mathrm{q}_{\mathrm{a}}=\frac{1}{3.0}\left(32(1.0)(0.617)(400) \frac{(110,000)(0.099)}{(18)^{3}}\right)^{\frac{1}{2}} \\
& =40.5 \mathrm{lb} / \mathrm{in}^{2}=5.829 \mathrm{lb} / \mathrm{ft}^{2}
\end{aligned}
$$

From equation 52-34, the reduction factor for the allowable buckling pressure from the deflection of the pipe is:

$$
\begin{aligned}
& \mathrm{C}=\left[\frac{\left(1-\frac{\% \Delta \mathrm{X}}{\mathrm{D}} \frac{1}{100}\right)}{\left(1+\frac{\% \Delta \mathrm{X}}{\mathrm{D}} \frac{1}{100}\right)^{2}}\right]^{3} \\
& \mathrm{C}=\left[\frac{\left(1-3.67 \frac{1}{100}\right)}{\left(1+3.67 \frac{1}{100}\right)^{2}}\right]^{3} \\
& \mathrm{C}=0.72
\end{aligned}
$$

The reduced allowable buckling pressure is

$$
\mathrm{q}_{\mathrm{a}} \mathrm{C}=5,829 \times 0.72=4,197 \mathrm{lb} / \mathrm{ft}^{2}>\mathrm{P}=2,121 \mathrm{lb} / \mathrm{ft}^{2} \quad \text { O.K. }
$$

Design example 9-Plastic pipe during construction (continued)
E. Strain during construction

From equation 52-36, the hoop strain due to external load is:

$$
\varepsilon_{\mathrm{h}}=\frac{\frac{\mathrm{P}}{144} \mathrm{D}_{\mathrm{m}}}{2 \mathrm{tE}}=\frac{\frac{\mathrm{P}}{144} \times(18-1.059)}{2 \times 1.059 \times 110,000}=0.011 \mathrm{in} / \mathrm{in}
$$

From equation 52-37, the maximum strain due to ring bending is:

$$
\varepsilon_{\mathrm{f}}=\frac{1}{\operatorname{SDR}}\left(\frac{\frac{3 \Delta Y}{D_{M}}}{1-2 \frac{\Delta Y}{D_{M}}}\right)=\frac{1}{17}\left(\frac{3 \times 0.0367}{1-2(0.0367)}\right)=0.007 \mathrm{in} / \mathrm{in}
$$

From equation 52-37, the combined strain is:

$$
\varepsilon=\varepsilon_{\mathrm{f}} \pm \varepsilon_{\mathrm{h}}
$$

Since the hoop strain is due to external load it is subtracted.

$$
\begin{aligned}
& \varepsilon=0.007-0.0011=0.006 \\
& \varepsilon=0.006<\varepsilon_{\text {all }}=5 \%=0.05 \quad \text { O.K. }
\end{aligned}
$$

F. External soil load upon completion of the dam

From equation 52-17, the soil pressure due to 10 feet of soil is

$$
\begin{aligned}
\mathrm{P}_{\mathrm{s}} & =\gamma_{\mathrm{s}} \times \mathrm{h} \\
& =120 \times 10=1,200 \mathrm{lb} / \mathrm{ft}^{2}
\end{aligned}
$$

Design Pressure :

$$
\begin{aligned}
\mathrm{P} & =\mathrm{P}_{\mathrm{s}}+\mathrm{P}_{\mathrm{w}}+\mathrm{P}_{\mathrm{v}} \\
& =1,200+0+0=1,200 \mathrm{lb} / \mathrm{ft}^{2}
\end{aligned}
$$

From equation 52-26:
Thrust: $\mathrm{T}_{\mathrm{pw}}=\frac{\mathrm{P} \times\left(\frac{\mathrm{D}_{\mathrm{o}}}{12}\right)}{2}$

$$
\begin{aligned}
\mathrm{T}_{\mathrm{pw}} & =\frac{1,200 \times \frac{18}{12}}{2} \\
\mathrm{~T}_{\mathrm{pw}} & =900 \mathrm{lb} / \mathrm{ft} \text { of pipe }
\end{aligned}
$$

## Design example 9-Plastic pipe during construction (continued)

G. Required wall area for completed external load:

From equation 52-27:

$$
\begin{aligned}
& \mathrm{A}_{\mathrm{pw}}=\frac{\frac{\mathrm{T}_{\mathrm{pw}}}{12}}{\sigma} \\
& \mathrm{~A}_{\mathrm{pw}}=\frac{900}{\frac{92}{800}}, \sigma=800 \mathrm{lb} / \mathrm{in}^{2} \text { frm appendix } 52 \mathrm{C} \text {, table } 52 \mathrm{C}-1 \\
& \mathrm{~A}_{\mathrm{pw}}=0.094 \mathrm{in}^{2} / \mathrm{in}
\end{aligned}
$$

Area of an 18 -inch pipe with SDR of 17 using equation 52-28:

$$
\begin{aligned}
& \mathrm{A}_{\mathrm{pw}}=\frac{\left(\mathrm{D}_{\mathrm{o}}-\mathrm{D}_{\mathrm{i}}\right)}{2} \\
& \mathrm{~A}_{\mathrm{pw}}=\mathrm{t} \\
& \mathrm{~A}_{\mathrm{pw}}=1.059 \mathrm{in}^{2} / \mathrm{in}>0.094 \mathrm{in}^{2} / \mathrm{in}
\end{aligned}
$$

О.к.
H. Deflection upon completion of the dam:

From equation 52-29, the percent deflection for solid wall pipe is:

$$
\begin{aligned}
& \frac{\% \Delta X}{D}=\frac{\left(\mathrm{D}_{\mathrm{L}} \mathrm{P}_{\mathrm{S}}+\mathrm{P}_{\mathrm{w}}+\mathrm{P}_{\mathrm{V}}\right)\left(\frac{1}{144}\right) \mathrm{K}(100)}{\left[\left(\frac{2 \mathrm{E}}{3(\mathrm{SDR}-1)^{3}}\right)+0.061 \mathrm{E}^{\prime}\right]} \\
& \frac{\% \Delta \mathrm{X}}{\mathrm{D}}=\frac{(1.5(1200)+0+0)\left(\frac{1}{144}\right)(0.1)(100)}{\left[\left(\frac{2(110,000)}{3(17-1)^{3}}\right)+0.061(400)\right]} \\
& \frac{\% \Delta X}{\mathrm{D}}=2.95 \%<\left(\frac{\% \Delta \mathrm{X}}{\mathrm{D}}\right)_{\text {allowable }}=5 \% \quad \text { O.K. }
\end{aligned}
$$

## Design example 9-Plastic pipe during construction (continued)

I. Allowable buckling pressure upon completion of the dam:

From equation 52-33 using the long term modulus of elasticity since the soil load is a permanent load.

$$
\mathrm{q}_{\mathrm{a}}=\frac{1}{\mathrm{FS}}\left(32 \mathrm{R}_{\mathrm{w}} \mathrm{~B}^{\prime} \mathrm{E}^{\prime} \frac{\mathrm{EI}}{\mathrm{D}^{3}}\right)^{1 / 2}
$$

where:

$$
\begin{aligned}
& \frac{\mathrm{h}}{\left(\frac{\mathrm{D}_{\mathrm{o}}}{12}\right)}=\frac{10}{\left(\frac{18}{12}\right)}=6.66 \geq 2 \\
& \text { F.S. }=2.5 \\
& \mathrm{Rw}=1.0 \\
& \mathrm{~B}^{\prime}=\frac{4\left(\mathrm{~h}^{2}+\left(\frac{\mathrm{D}_{0}}{12}\right) \mathrm{h}\right)}{1.5\left(2 \mathrm{~h}+\frac{\mathrm{D}_{\mathrm{o}}}{12}\right)^{2}}=\frac{4\left(10^{2}+\left(\frac{18}{12}\right) 10\right)}{1.5\left(2 * 10+\frac{18}{12}\right)^{2}}=0.663 \\
& \mathrm{I}_{\mathrm{pw}}=\frac{\mathrm{t}^{3}}{12}=\frac{1.059^{3}}{12}=0.099 \mathrm{in}^{4} / \mathrm{in} \\
& \mathrm{q}_{\mathrm{a}}=\frac{1}{2.5}\left(32(1.0)(0.663)(400) \frac{(22,000)(0.099)}{(18)^{3}}\right)^{1 / 2} \\
& =22.5 \mathrm{lb} / \mathrm{in}^{2}=3,242 \mathrm{lb} / \mathrm{ft}^{2}
\end{aligned}
$$

From equation 52-34, the reduction factor for the allowable buckling pressure from the deflection of the pipe is:

$$
\begin{aligned}
& \mathrm{C}=\left[\frac{\left(1-\frac{\% \Delta \mathrm{X}}{\mathrm{D}} \frac{1}{100}\right)}{\left(1+\frac{\% \Delta \mathrm{X}}{\mathrm{D}} \frac{1}{100}\right)^{2}}\right]^{3} \\
& \mathrm{C}=\left[\frac{\left(1-2.95 \frac{1}{100}\right)}{\left(1+2.95 \frac{1}{100}\right)^{2}}\right]^{3} \\
& \mathrm{C}=0.77
\end{aligned}
$$

The reduced allowable buckling pressure is

$$
\mathrm{q}_{\mathrm{a}} \mathrm{C}=3,242 \times 0.77=2,496 \mathrm{lb} / \mathrm{ft}^{2}>\mathrm{P}=1,200 \mathrm{lb} / \mathrm{ft}^{2} \quad \text { O.K. }
$$

## Design example 9-Plastic pipe during construction (continued)

J. Strain upon completion of the dam

From equation 52-36, the hoop strain due to external load is:

$$
\varepsilon_{\mathrm{h}}=\frac{\frac{\mathrm{P}}{144} \mathrm{D}_{\mathrm{m}}}{2 \mathrm{tE}}=\frac{\frac{1,200}{144} \times(18-1.059)}{2 \times 1.059 \times 110,000}=0.0006 \mathrm{in} / \mathrm{in}
$$

From equation 52-37, the maximum strain due to ring bending is:

$$
\varepsilon_{\mathrm{f}}=\frac{1}{\operatorname{SDR}}\left(\frac{\frac{3 \Delta Y}{D_{M}}}{1-2 \frac{\Delta Y}{D_{M}}}\right)=\frac{1}{17}=0.005 \mathrm{in} / \mathrm{in}
$$

From equation 52-37, the combined strain is:

$$
\varepsilon=\varepsilon_{\mathrm{f}} \pm \varepsilon_{\mathrm{h}}
$$

Since the hoop strain is due to external load it is subtracted.

$$
\begin{aligned}
& \varepsilon=0.005-0.0006=0.005 \\
& \varepsilon=0.005<\varepsilon_{\text {all }}=5 \%=0.05 \quad \text { O.K. }
\end{aligned}
$$

Conclusion: An HDPE pipe of PE3408 with an SDR of 17 is acceptable for both the construction loads and final soil loads. NOTE: The construction loads are the most critical.

## Appendix 52C

## Material Properties, Pressure Ratings, and Pipe Dimensions for Plastic Pipe

(Note: The source of the information in this appendix is subject to periodic updating. The source documents should be referenced for any updated information.)

Table 52C-1 Hydrostatic design basis, allowable long-term compressive stress, short-term hoop strength, and designation of plastic pipe
$\left.\begin{array}{lcccl}\hline \text { Plastic pipe material } & \begin{array}{c}\text { Hydrostatic } \\ \text { design basis }\end{array} & \begin{array}{c}\text { Allowable } \\ \text { long.term } \\ \text { compressive } \\ \text { stress } \\ (\text { (b/in² })\end{array} & \begin{array}{c}\text { Short-term } \\ \text { hoop strength }\end{array} & \text { Designation } \\ (\text { (bb/in² })\end{array}\right]$

[^2]| Appendix 52C | MaterialProperties,PressureRatings,andPipe <br> Dimensions for Plastic Pipe |
| :--- | :--- |

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Table 52C-2 PVC plastic irrigation pipe (PIP)

| Nominal <br> -- pipe size <br> (in) | $\begin{aligned} & \text { SDR/ } \\ & \text { pressure } \\ & \text { head } \end{aligned}$ |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | $\begin{aligned} & 1120 \\ & 1220 \end{aligned}$ | 2116 | 2112 | 2110 | $\underset{\text { (in) }}{\operatorname{minimum}}$ | tolerance <br> (in) | average (in) | $\begin{gathered} -\ldots .+ \text { tol } \\ \text { average } \\ \text { OD (in) } \end{gathered}$ | max \& $\min$ (in) |
| 4 | 50 ft | 22 |  |  |  | 0.065 | +0.020 | 4.134 | 0.009 | 0.050 |
|  | 81 | 50 | 40 | 30 | 25 | 0.065 | +0.020 |  |  |  |
|  | 51 | 80 | 63 | 50 | 40 | 0.081 | +0.020 |  |  |  |
|  | 41 | 100 | 80 | 63 | 50 | 0.101 | +0.020 |  |  |  |
|  | 32.5 | 125 | 100 | 80 | 63 | 0.127 | +0.020 |  |  |  |
|  | 26 | 160 | 125 | 100 | 80 | 0.159 | +0.020 |  |  |  |
| 6 | 50 ft | 22 |  |  |  | 0.070 | +0.020 | 6.140 | 0.011 | 0.050 |
|  | 100 ft | 44 |  |  |  | 0.070 | +0.020 |  |  |  |
|  | 81 | 50 | 40 | 30 | 25 | 0.076 | +0.020 |  |  |  |
|  | 51 | 80 | 63 | 50 | 40 | 0.120 | +0.020 |  |  |  |
|  | 41 | 100 | 80 | 63 | 50 | 0.150 | +0.020 |  |  |  |
|  | 32.5 | 125 | 100 | 80 | 63 | 0.189 | +0.023 |  |  |  |
|  | 26 | 160 | 125 | 100 | 80 | 0.236 | +0.028 |  |  |  |
| 8 | 50 ft . | 22 |  |  |  | 0.080 | +0.020 | 8.160 | 0.015 | 0.075 |
|  | 100 ft | 44 |  |  |  | 0.087 | +0.020 |  |  |  |
|  | 81 | 50 | 40 | 30 | 25 | 0.101 | +0.020 |  |  |  |
|  | 51 | 80 | 63 | 50 | 40 | 0.160 | +0.020 |  |  |  |
|  | 41 | 100 | 80 | 63 | 50 | 0.199 | +0.024 |  |  |  |
|  | 32.5 | 125 | 100 | 80 | 63 | 0.251 | +0.031 |  |  |  |
|  | 26 | 160 | 125 | 100 | 80 | 0.314 | $+0.038$ |  |  |  |
| 10 | 50 ft | 22 |  |  |  | 0.100 | +0.020 | 10.200 | 0.015 | 0.075 |
|  | 100 ft | 44 |  |  |  | 0.109 | +0.020 |  |  |  |
|  | 81 | 50 | 40 | 30 | 25 | 0.126 | +0.020 |  |  |  |
|  | 51 | 80 | 63 | 50 | 40 | 0.200 | +0.024 |  |  |  |
|  | 41 | 100 | 80 | 63 | 50 | 0.240 | $+0.030$ |  |  |  |
|  | 32.5 | 125 | 100 | 80 | 63 | 0.314 | +0.038 |  |  |  |
|  | 26 | 160 | 125 | 100 | 80 | 0.392 | +0.047 |  |  |  |
| 12 | 50 ft | 22 |  |  |  | 0.120 | +0.020 | 12.240 | 0.015 | 0.070 |
|  | 100 ft | 44 |  |  |  | 0.131 | +0.020 |  |  |  |
|  | 81 | 50 | 40 | 30 | 25 | 0.151 | +0.020 |  |  |  |
|  | 51 | 80 | 63 | 50 | 40 | 0.240 | +0.029 |  |  |  |
|  | 41 | 100 | 80 | 63 | 50 | 0.299 | +0.036 |  |  |  |
|  | 32.5 | 125 | 100 | 80 | 63 | 0.377 | +0.045 |  |  |  |
|  | 26 | 160 | 125 | 100 | 80 | 0.471 | $+0.056$ |  |  |  |
| 14 | 51 |  | 63 | 50 |  | 0.280 | $+0.034$ | 14.280 | 0.015 | 0.075 |
|  | 41 | 100 | 80 | 63 | 50 | 0.348 | $+0.042$ |  |  |  |
|  | 32.5 | 125 | 100 | 80 | 63 | 0.439 | +0.053 |  |  |  |
|  | 26 | 160 | 125 | 100 | 80 | 0.549 | $+0.066$ |  |  |  |


| Appendix 52C | MaterialProperties,PressureRatings,andPipe <br> Dimensions for Plastic Pipe |
| :--- | :--- |

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Table 52C-2 PVC plastic irrigation pipe (PIP)—Continued

| Nominal -- pipe size (in) | SDR/ pressure head | PVC pressure rating $\left(\mathrm{lb} / \mathrm{in}^{2}\right)----------------------$ - - - Dimension and tolerance |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | $\begin{aligned} & 1120 \\ & 1220 \end{aligned}$ | 2116 | 2112 | 2110 | minimum <br> (in) | tolerance (in) | average (in) | $\begin{gathered} -- \pm \text { to } \\ \text { average } \\ \text { OD (in) } \end{gathered}$ |  <br> $\min$ (in) |
| 15 | 50 ft | 22 |  |  |  | 0.150 | +0.020 | 15.3 | 0.016 | 0.075 |
|  | 100 ft | 44 |  |  |  | 0.164 | +0.020 |  |  |  |
|  | 81 | 50 | 40 | 30 | 25 | 0.189 | +0.023 |  |  |  |
|  | 51 | 80 | 63 | 50 | 40 | 0.300 | +0.042 |  |  |  |
|  | 41 | 100 | 80 | 63 | 50 | 0.373 | +0.052 |  |  |  |
|  | 32.5 | 125 | 100 | 80 | 63 | 0.437 | +0.052 |  |  |  |
|  | 32.5 | 125 | 100 | 80 | 63 | 0.471 | +0.056 |  |  |  |
|  | 26 | 160 | 125 | 100 | 80 | 0.588 | +0.070 |  |  |  |
|  | 21 | 200 | 160 | 125 | 100 | 0.728 | +0.087 |  |  |  |
| 18 | 100 ft | 44 |  |  |  | 0.200 | +0.024 | 18.701 | 0.020 | 0.075 |
|  | 81 | 50 | 40 | 30 | 25 | 0.231 | +0.028 |  |  |  |
|  | 51 | 80 | 63 | 50 | 40 | 0.366 | +0.051 |  |  |  |
|  | 41 | 100 | 80 | 63 | 50 | 0.456 | +0.064 |  |  |  |
|  | 32.5 | 125 | 100 | 80 | 63 | 0.534 | +0.064 |  |  |  |
|  | 32.5 | 125 | 100 | 80 | 63 | 0.575 | +0.069 |  |  |  |
|  | 26 | 160 | 125 | 100 | 80 | 0.719 | +0.086 |  |  |  |
| 21 | 100 ft | 44 |  |  |  | 0.236 | +0.028 | 22.047 | 0.025 | 0.075 |
|  | 81 | 50 | 40 | 30 | 25 | 0.272 | +0.033 |  |  |  |
|  | 51 | 80 | 63 | 50 | 40 | 0.432 | +0.060 |  |  |  |
|  | 41 | 100 | 80 | 63 | 50 | 0.538 | +0.075 |  |  |  |
|  | 32.5 | 125 | 100 | 80 | 63 | 0.630 | +0.076 |  |  |  |
|  | 32.5 | 125 | 100 | 80 | 63 | 0.678 | +0.081 |  |  |  |
|  | 26 | 160 | 125 | 100 | 80 | 0.848 | +0.102 |  |  |  |
| 24 | 100 ft | 44 |  |  |  | 0.266 | +0.032 | 24.803 | 0.032 | 0.075 |
|  | 81 | 50 | 40 | 30 | 25 | 0.306 | +0.037 |  |  |  |
|  | 51 | 80 | 63 | 50 | 40 | 0.486 | +0.068 |  |  |  |
|  | 41 | 100 | 80 | 63 | 50 | 0.605 | +0.085 |  |  |  |
|  | 32.5 | 100 | 80 | 63 | 50 | 0.709 | +0.085 |  |  |  |
|  | 32.5 | 125 | 100 | 80 | 63 | 0.763 | +0.092 |  |  |  |
|  | 26 | 160 | 125 | 100 | 80 | 0.954 | 0.115 |  |  |  |
| 27 | 51 | 80 | 63 | 50 | 40 | 0.548 | +0.077 | 27.953 | 0.038 | 0.075 |
|  | 41 | 100 | 80 | 63 | 50 | 0.682 | +0.095 |  |  |  |
|  | 32.5 | 125 | 100 | 80 | 63 | 0.799 | +0.096 |  |  |  |
|  | 32.5 | 125 | 100 | 80 | 63 | 0.860 | +0.103 |  |  |  |
|  | 26 | 160 | 125 | 100 | 80 | 1.075 | +0.129 |  |  |  |

Source: ASTM D 2241 and ASAE S376.2
Note: PIP pipe sizes in the source documents were developed from Soil Conservation Service Practice Standards 430DD and 430EE, whic have been rescinded.

## Appendix 52C

## MaterialProperties,PressureRatings,andPipe Dimensions for Plastic Pipe

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Table 52C-3 PVC and ABS thermoplastic pipe (SDR-PR)-(IPS) (nonthreaded)

| Nominal pipe size (in) | SDR | --- PVC pressure rating (lb/in²) -- |  |  |  | Wall thickness --- Dimension and tolerance -------- -- |  |  |  |  | -- - ABS pressure rating $\left(\mathrm{lb} / \mathrm{in}^{2}\right)-$-- |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | $\begin{aligned} & 1120 \\ & 1220 \\ & 2120 \end{aligned}$ | 2116 | 2112 | 2110 | min. (in) | tolerance (in) | average <br> (in) | $\begin{aligned} & \text { tole } \\ & \text { avg OD } \\ & \text { (in) } \end{aligned}$ | erance max \& $\min (i n)$ | 1316 | 2112 | 1210 | 1208 |
| 1/8 | 13.5 | 315 | 250 | 200 | 160 | 0.060 | +0.020 | 0.405 | 0.004 | 0.008 | 250 | 200 | 160 | 125 |
| 1/4 | 13.5 | 315 | 250 | 200 | 160 | 0.060 | +0.020 | 0.540 | 0.004 | 0.008 | 250 | 200 | 160 | 125 |
| 3/8 | 13.5 | 315 | 250 | 200 | 160 | 0.060 | +0.020 | 0.675 | 0.004 | 0.008 | 250 | 200 | 160 | 125 |
| 1/2 | 17 | 250 | 200 | 160 | 125 | 0.060 | +0.020 | 0.840 | 0.004 | 0.008 | 200 | 160 | 125 | 100 |
|  | 13.5 | 315 | 250 | 200 | 160 | 0.062 | +0.020 |  |  | 0.008 | 250 | 200 | 160 | 125 |
| $3 / 4$ | 21 | 200 | 160 | 125 | 100 | 0.060 | +0.020 | 1.050 | 0.004 | 0.015 | 160 | 125 | 100 | 80 |
|  | 17 | 250 | 200 | 160 | 125 | 0.062 | +0.020 |  |  | 0.010 | 200 | 160 | 125 | 100 |
|  | 13.5 | 315 | 250 | 200 | 160 | 0.078 | +0.020 |  |  | 0.010 | 250 | 200 | 160 | 125 |
| 1 | 26 | 160 | 125 | 100 | 80 | 0.060 | +0.020 | 1.315 | 0.005 | 0.015 | 125 | 100 | 80 |  |
|  | 21 | 200 | 160 | 125 | 100 | 0.063 | +0.020 |  |  | 0.015 | 160 | 125 | 100 | 80 |
|  | 17 | 250 | 200 | 160 | 125 | 0.077 | +0.020 |  |  | 0.010 | 200 | 160 | 125 | 100 |
|  | 13.5 | 315 | 250 | 200 | 160 | 0.097 | +0.020 |  |  | 0.010 | 250 | 200 | 160 | 125 |
| $11 / 4$ | 32.5 | 125 | 100 | 80 | 63 | 0.060 | +0.020 | 1.660 | 0.005 | 0.015 |  |  |  |  |
|  | 26 | 160 | 125 | 100 | 80 | 0.064 | +0.020 |  |  | 0.015 | 125 | 100 | 80 |  |
|  | 21 | 200 | 160 | 125 | 100 | 0.079 | +0.020 |  |  | 0.015 | 160 | 125 | 100 | 80 |
|  | 17 | 250 | 200 | 160 | 125 | 0.098 | +0.020 |  |  | 0.012 | 200 | 160 | 125 | 100 |
|  | 13.5 | 315 | 250 | 200 | 160 | 0.123 | +0.020 |  |  | 0.012 | 250 | 200 | 160 | 125 |
| $11 / 2$ | 32.5 | 125 | 100 | 80 | 63 | 0.060 | +0.020 | 1.900 | 0.006 | 0.030 |  |  |  |  |
|  | 26 | 160 | 125 | 100 | 80 | 0.073 | +0.020 |  |  | 0.030 | 125 | 100 | 80 |  |
|  | 21 | 200 | 160 | 125 | 100 | 0.090 | +0.020 |  |  | 0.030 | 160 | 125 | 100 | 80 |
|  | 17 | 250 | 200 | 160 | 125 | 0.112 | +0.020 |  |  | 0.012 | 200 | 160 | 125 | 100 |
|  | 13.5 | 315 | 250 | 200 | 160 | 0.141 | +0.020 |  |  | 0.012 | 250 | 200 | 160 | 125 |
| 2 | 32.5 | 125 | 100 | 80 | 63 | 0.073 | +0.020 | 2.375 | 0.006 | 0.030 |  |  |  |  |
|  | 26 | 160 | 125 | 100 | 80 | 0.091 | +0.020 |  |  | 0.030 | 125 | 100 | 80 |  |
|  | 21 | 200 | 160 | 125 | 100 | 0.113 | $+0.020$ |  |  | 0.030 | 160 | 125 | 100 | 80 |
|  | 17 | 250 | 200 | 160 | 125 | 0.140 | $+0.020$ |  |  | 0.012 | 200 | 160 | 125 | 100 |
|  | 13.5 | 315 | 250 | 200 | 160 | 0.176 | +0.020 |  |  | 0.012 | 250 | 200 | 160 | 125 |
| $21 / 2$ | 32.5 | 125 | 100 | 80 | 63 | 0.088 | +0.020 | 2.875 | 0.007 | 0.030 |  |  |  |  |
|  | 26 | 160 | 125 | 100 | 80 | 0.110 | $+0.020$ |  |  | 0.030 | 125 | 100 | 80 |  |
|  | 21 | 200 | 160 | 125 | 100 | 0.137 | +0.020 |  |  | 0.030 | 160 | 125 | 100 | 80 |
|  | 17 | 250 | 200 | 160 | 125 | 0.169 | $+0.020$ |  |  | 0.015 | 200 | 160 | 125 | 100 |
|  | 13.5 | 315 | 250 | 200 | 160 | 0.213 | $+0.026$ |  |  | 0.015 | 250 | 200 | 160 | 125 |
| 3 | 41 | 100 | 80 | 63 | 50 | 0.085 | +0.020 | 3.500 | 0.008 | 0.030 |  |  |  |  |
|  | 32.5 | 125 | 100 | 80 | 63 | 0.108 | +0.020 |  |  | 0.030 |  |  |  |  |
|  | 26 | 160 | 125 | 100 | 80 | 0.135 | $+0.020$ |  |  | 0.030 | 125 | 100 | 80 |  |
|  | 21 | 200 | 160 | 125 | 100 | 0.167 | $+0.020$ |  |  | 0.030 | 160 | 125 | 100 | 80 |
|  | 17 | 250 | 200 | 160 | 125 | 0.206 | $+0.025$ |  |  | 0.015 | 200 | 160 | 125 | 100 |
|  | 13.5 | 315 | 250 | 200 | 160 | 0.259 | $+0.031$ |  |  | 0.015 | 250 | 200 | 160 | 125 |


$\overline{\text { Appendix 52C }} \quad$| MaterialProperties,PressureRatings,andPipe |
| :--- |
| Dimensions for Plastic Pipe |

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Table 52C-3 PVC and ABS thermoplastic pipe (SDR-PR)-(IPS) (nonthreaded)—Continued

| Nominal pipe size (in) | SDR | --- PVC pressure rating $\left(\mathrm{lb} / \mathrm{in}^{2}\right)$-- - |  |  |  | -------- Dimension and tolerance ------ - - <br> Wall thickness ---- Outside diameter --- |  |  |  |  | - - - ABS pressure rating ( $\mathrm{lb} / \mathrm{in}^{2}$ ) -- - <br> --------- Material ----------- |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
|  |  | $\begin{aligned} & 1120 \\ & 1220 \\ & 2120 \end{aligned}$ | 2116 | 2112 | 2110 | min. (in) | tolerance (in) | average (in) | $\begin{gathered} \text { tole } \\ \underset{\text { avg OD }}{\text { (in) }} \end{gathered}$ | rance $\max \&$ $\min (i n)$ | 1316 | 2112 | 1210 | 1208 |
| $31 / 2$ | 41 | 100 | 80 | 63 | 50 | 0.098 | +0.020 | 4.000 | 0.008 | 0.050 |  |  |  |  |
|  | 32.5 | 125 | 100 | 80 | 63 | 0.123 | +0.020 |  |  | 0.050 |  |  |  |  |
|  | 26 | 160 | 125 | 100 | 80 | 0.154 | +0.020 |  |  | 0.050 | 125 | 100 | 80 |  |
|  | 21 | 200 | 160 | 125 | 100 | 0.190 | +0.023 |  |  | 0.050 | 160 | 125 | 100 | 80 |
|  | 17 | 250 | 200 | 160 | 125 | 0.235 | +0.028 |  |  | 0.015 | 200 | 160 | 125 | 100 |
|  | 13.5 | 315 | 250 | 200 | 160 | 0.296 | +0.036 |  |  | 0.015 | 250 | 200 | 160 | 125 |
| 4 | 64 | 63 | 50 |  |  | 0.070 | $+0.020$ | 4.500 | 0.009 | 0.050 |  |  |  |  |
|  | 41 | 100 | 80 | 63 | 50 | 0.110 | +0.020 |  |  | 0.050 |  |  |  |  |
|  | 32.5 | 125 | 100 | 80 | 63 | 0.138 | +0.020 |  |  | 0.050 |  |  |  |  |
|  | 26 | 160 | 125 | 100 | 80 | 0.173 | +0.020 |  |  | 0.050 | 125 | 100 | 80 |  |
|  | 21 | 200 | 160 | 125 | 100 | 0.214 | +0.026 |  |  | 0.050 | 160 | 125 | 100 | 80 |
|  | 17 | 250 | 200 | 160 | 125 | 0.265 | +0.032 |  |  | 0.015 | 200 | 160 | 125 | 100 |
|  | 13.5 | 315 | 250 | 200 | 160 | 0.333 | +0.040 |  |  | 0.015 | 250 | 200 | 160 | 125 |
| 5 | 64 | 63 | 50 |  |  | 0.087 | +0.020 | 5.563 | 0.010 | 0.050 |  |  |  |  |
|  | 41 | 100 | 80 | 63 | 50 | 0.136 | +0.020 |  |  | 0.050 |  |  |  |  |
|  | 32.5 | 125 | 100 | 80 | 63 | 0.171 | +0.021 |  |  | 0.050 |  |  |  |  |
|  | 26 | 160 | 125 | 100 | 80 | 0.214 | +0.027 |  |  | 0.050 | 125 | 100 | 80 |  |
|  | 21 | 200 | 160 | 125 | 100 | 0.265 | +0.032 |  |  | 0.050 | 160 | 125 | 100 | 80 |
|  | 17 | 250 | 200 | 160 | 125 | 0.327 | +0.039 |  |  | 0.030 | 200 | 160 | 125 | 100 |
|  | 13.5 | 315 | 250 | 200 | 160 | 0.412 | +0.049 |  |  | 0.030 | 250 | 200 | 160 | 125 |
| 6 | 64 | 63 | 50 |  |  | 0.104 | +0.020 | 6.625 | 0.011 | 0.050 |  |  |  |  |
|  | 41 | 100 | 80 | 63 | 50 | 0.162 | +0.020 |  |  | 0.050 |  |  |  |  |
|  | 32.5 | 125 | 100 | 80 | 63 | 0.204 | +0.024 |  |  | 0.050 |  |  |  |  |
|  | 26 | 160 | 125 | 100 | 80 | 0.255 | +0.031 |  |  | 0.050 | 125 | 100 | 80 |  |
|  | 21 | 200 | 160 | 125 | 100 | 0.316 | +0.038 |  |  | 0.050 | 160 | 125 | 100 | 80 |
|  | 17 | 250 | 200 | 160 | 125 | 0.390 | +0.047 |  |  | 0.035 | 200 | 160 | 125 | 100 |
|  | 13.5 | 315 | 250 | 200 | 160 | 0.491 | +0.059 |  |  | 0.035 | 250 | 200 | 160 | 125 |
| 8 | 64 | 63 | 50 |  |  | 0.135 | +0.020 | 8.625 | 0.015 | 0.075 |  |  |  |  |
|  | 41 | 100 | 80 | 63 | 50 | 0.210 | +0.025 |  |  | 0.075 |  |  |  |  |
|  | 32.5 | 125 | 100 | 80 | 63 | 0.265 | +0.032 |  |  | 0.075 |  |  |  |  |
|  | 26 | 160 | 125 | 100 | 80 | 0.332 | +0.040 |  |  | 0.075 | 125 | 100 | 80 |  |
|  | 21 | 200 | 160 | 125 | 100 | 0.410 | +0.049 |  |  | 0.045 | 160 | 125 | 100 | 80 |
|  | 17 | 250 | 200 | 160 | 125 | 0.508 | +0.061 |  |  | 0.045 |  |  |  |  |
| 10 | 64 | 63 | 50 |  |  | 0.168 | +0.020 | 10.750 | 0.015 | 0.075 |  |  |  |  |
|  | 41 | 100 | 80 | 63 | 50 | 0.262 | +0.031 |  |  | 0.075 |  |  |  |  |
|  | 32.5 | 125 | 100 | 80 | 63 | 0.331 | +0.040 |  |  | 0.075 |  |  |  |  |
|  | 26 | 160 | 125 | 100 | 80 | 0.413 | +0.050 |  |  | 0.075 | 125 | 100 | 80 |  |
|  | 21 | 200 | 160 | 125 | 100 | 0.511 | +0.061 |  |  | 0.050 | 160 | 125 | 100 | 80 |
|  | 17 | 250 | 200 | 160 | 125 | 0.632 | +0.076 |  |  | 0.050 |  |  |  |  |


$\overline{\text { Appendix 52C }} \quad$| MaterialProperties,PressureRatings,andPipe |
| :--- |
| Dimensions for Plastic Pipe |

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Table 52C-3 PVC and ABS thermoplastic pipe (SDR-PR)-(IPS) (nonthreaded)—Continued

| Nominal | SDR | --- PVC | ressur | ting (1 | $\left.{ }^{2}\right)$--- |  | Dimensi | n and tol | rance |  | -- - AB | ressu | ating | $\mathrm{n}^{2}$ ) |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| e size |  |  |  |  |  |  | kn | Out | side |  |  | Mat |  |  |
| (in) |  | 1120 | 2116 | 2112 | 2110 |  |  | average | $\pm$ tole | rance | 1316 | 2112 | 1210 | 1208 |
|  |  | 1220 |  |  |  | min. (in) | toler- | (in) | $\operatorname{avg} \mathrm{OD}$ | $\max \&$ |  |  |  |  |
|  |  | 2120 |  |  |  |  | ance (in) |  | (in) | min (in) |  |  |  |  |
| 12 | 64 | 63 | 50 |  |  | 0.199 | +0.024 | 12.750 | 0.015 | 0.075 |  |  |  |  |
|  | 41 | 100 | 80 | 63 | 50 | 0.311 | +0.037 |  |  | 0.075 |  |  |  |  |
|  | 32.5 | 125 | 100 | 80 | 63 | 0.392 | +0.047 |  |  | 0.075 |  |  |  |  |
|  | 26 | 160 | 125 | 100 | 80 | 0.490 | +0.059 |  |  | 0.075 | 125 | 100 | 80 |  |
|  | 21 | 200 | 160 | 125 | 100 | 0.606 | +0.073 |  |  | 0.075 | 160 | 125 | 100 | 80 |
|  | 17 | 250 | 200 | 160 | 125 | 0.750 | +0.090 |  |  | 0.060 |  |  |  |  |
| 14 | 41 | 100 | 80 | 63 | 50 | 0.341 | +0.048 | 14.000 | 0.015 | 0.100 |  |  |  |  |
|  | 32.5 | 125 | 100 | 80 | 63 | 0.430 | +0.052 |  |  | 0.100 |  |  |  |  |
|  | 26 | 160 | 125 | 100 | 80 | 0.538 | +0.064 |  |  | 0.100 |  |  |  |  |
|  | 21 | 200 | 160 | 125 | 100 | 0.666 | +0.080 |  |  | 0.100 |  |  |  |  |
|  | 17 | 250 | 200 | 160 | 125 | 0.823 | +0.099 |  |  | 0.075 |  |  |  |  |
| 16 | 41 | 100 | 80 | 63 | 50 | 0.390 | +0.055 | 16.000 | 0.019 | 0.160 |  |  |  |  |
|  | 32.5 | 125 | 100 | 80 | 63 | 0.492 | +0.059 |  |  | 0.160 |  |  |  |  |
|  | 26 | 160 | 125 | 100 | 80 | 0.615 | +0.074 |  |  | 0.160 |  |  |  |  |
|  | 21 | 200 | 160 | 125 | 100 | 0.762 | +0.091 |  |  | 0.160 |  |  |  |  |
|  | 17 | 250 | 200 | 160 | 125 | 0.941 | +0.113 |  |  | 0.080 |  |  |  |  |
| 18 | 41 | 100 | 80 | 63 | 50 | 0.439 | +0.061 | 18.000 | 0.019 | 0.180 |  |  |  |  |
|  | 32.5 | 125 | 100 | 80 | 63 | 0.554 | +0.066 |  |  | 0.180 |  |  |  |  |
|  | 26 | 160 | 125 | 100 | 80 | 0.692 | +0.083 |  |  | 0.180 |  |  |  |  |
|  | 21 | 200 | 160 | 125 | 100 | 0.857 | +0.103 |  |  | 0.180 |  |  |  |  |
|  | 17 | 250 | 200 | 160 | 125 | 1.059 | +0.127 |  |  | 0.090 |  |  |  |  |
| 20 | 41 | 100 | 80 | 63 | 50 | 0.488 | +0.068 | 20.000 | 0.023 | 0.200 |  |  |  |  |
|  | 32.5 | 125 | 100 | 80 | 63 | 0.615 | +0.074 |  |  | 0.200 |  |  |  |  |
|  | 26 | 160 | 125 | 100 | 80 | 0.769 | +0.092 |  |  | 0.200 |  |  |  |  |
|  | 21 | 200 | 160 | 125 | 100 | 0.952 | +0.114 |  |  | 0.200 |  |  |  |  |
|  | 17 | 250 | 200 | 160 | 125 | 1.176 | +0.141 |  |  | 0.100 |  |  |  |  |
| 24 | 41 | 100 | 80 | 63 | 50 | 0.585 | +0.082 | 24.000 | 0.031 | 0.240 |  |  |  |  |
|  | 32.5 | 125 | 100 | 80 | 63 | 0.738 | +0.088 |  |  | 0.240 |  |  |  |  |
|  | 26 | 160 | 125 | 100 | 80 | 0.923 | +0.111 |  |  | 0.240 |  |  |  |  |
|  | 21 | 200 | 160 | 125 | 100 | 1.143 | +0.137 |  |  | 0.240 |  |  |  |  |
|  | 17 | 250 | 200 | 160 | 125 | 1.412 | +0.169 |  |  | 0.120 |  |  |  |  |
| 30 | 41 | 100 | 80 | 63 | 50 | 0.732 | +0.123 | 30.000 | 0.041 | 0.300 |  |  |  |  |
|  | 32.5 | 125 | 100 | 80 | 63 | 1.108 | +0.133 |  |  | 0.300 |  |  |  |  |
|  | 26 | 160 | 125 | 100 | 80 | 1.385 | +0.166 |  |  | 0.300 |  |  |  |  |
|  | 21 | 200 | 160 | 125 | 100 | 1.714 | +0.205 |  |  | 0.300 |  |  |  |  |
|  | 17 | 250 | 200 | 160 | 125 | 2.118 | +0.254 |  |  | 0.150 |  |  |  |  |

## MaterialProperties,PressureRatings,andPipe

Dimensions for Plastic Pipe

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Table 52C-3 PVC and ABS thermoplastic pipe (SDR-PR)-(IPS) (nonthreaded)—Continued

| Nominal pipe size (in) | SDR | --- PVC pressure rating $\left(\mathrm{lb} / \mathrm{in}^{2}\right)$--- |  |  |  | -- -- -- - - Dimension and tolerance - -- -- - - <br> Wall thickness ---- Outside diameter -- |  |  |  |  | -- - ABS pressure rating (lb/in ${ }^{2}$ ) -- - <br> --------- Material |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
|  |  | $\begin{aligned} & 1120 \\ & 1220 \\ & 2120 \end{aligned}$ | 2116 | 2112 | 2110 | min. (in) tolerance (in) |  | average <br> (in) |  <br> (in) $\quad \min (i n)$ |  | 1316 | 2112 | 1210 | 1208 |
| 36 | 41 | 100 | 80 | 63 | 50 | 0.878 | +0.123 | 36.000 | 0.050 | 0.360 |  |  |  |  |
|  | 32.5 | 125 | 100 | 80 | 63 | 1.108 | +0.133 |  |  | 0.360 |  |  |  |  |
|  | 26 | 160 | 125 | 100 | 80 | 1.385 | +0.166 |  |  | 0.360 |  |  |  |  |
|  | 21 | 200 | 160 | 125 | 100 | 1.714 | +0.205 |  |  | 0.360 |  |  |  |  |
|  | 17 | 250 | 200 | 160 | 125 | 2.118 | +0.254 |  |  | 0.180 |  |  |  |  |

Source: ASTM D 2241 and D 2282.

| $\overline{\text { Appendix 52C }}$ | MaterialProperties,PressureRatings,andPipe <br> Dimensions for Plastic Pipe | Part 636 <br> National Engineering Handbook |
| :--- | :--- | :--- |
|  |  |  |

Table 52C-4 Polyethylene plastic pipe (SIDR-PR)-I.D. controlled (nonthreaded)

| Nominal pipe size (in) | SIDR | -- PE pressure rating ( $\left(\mathrm{lb} / \mathrm{in}^{2}\right.$ ) --- |  |  |  | $\qquad$ Dimension and tolerance <br> -- - Wall thickness --- ---- Inside diameter -- -- |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  |  |  |  |  |  |  |  |  |
|  |  | 3408 | 3306 | 2305 | 1404 | min. (in) | $\begin{aligned} & \text { toler- } \\ & \text { ance (in) } \end{aligned}$ | average <br> (in) | $\pm$ tolerance |  |
|  |  |  | 3406 |  |  |  |  |  | + | - |
|  |  |  | 2306 |  |  |  |  |  | (in) | (in) |
|  |  |  | 2406 |  |  |  |  |  |  |  |
| 1/2 | 19 | 80 |  |  |  | 0.060 | +0.020 | 0.622 | 0.010 | 0.010 |
|  | 15 | 100 | 80 |  |  | 0.060 | +0.020 |  |  |  |
|  | 11.5 | 125 | 100 | 80 |  | 0.060 | +0.020 |  |  |  |
|  | 9 | 160 | 125 | 100 | 80 | 0.069 | +0.020 |  |  |  |
|  |  | 200 |  |  |  |  |  |  |  |  |
|  |  |  |  |  |  | 0.089 | +0.020 |  |  |  |
|  | 5.3 | 250 | 200 | 160 | 125 | 0.117 | +0.020 |  |  |  |
| $3 / 4$ | 19 | 80 |  |  |  | 0.060 | +0.020 | 0.824 | 0.010 | 0.015 |
|  | 15 | 100 | 80 |  |  | 0.060 | +0.020 |  |  |  |
|  | 11.5 | 125 | 100 | 80 |  | 0.072 | +0.020 |  |  |  |
|  | 9 | 160 | 125 | 100 | 80 | 0.092 | +0.020 |  |  |  |
|  | 7 | 200 | 160 | 125 | 100 | 0.118 | +0.020 |  |  |  |
|  | 5.3 | 250 | 200 | 160 | 125 | 0.155 | +0.020 |  |  |  |
| 1 | 19 | 80 |  |  |  | 0.060 | +0.020 | 1.049 | 0.010 | 0.020 |
|  | 15 | 100 | 80 |  |  | 0.070 | +0.020 |  |  |  |
|  | 11.5 | 125 | 100 | 80 |  | 0.091 | +0.020 |  |  |  |
|  | 9 | 160 | 125 | 100 | 80 | 0.117 | +0.020 |  |  |  |
|  | 7 | 200 | 160 | 125 | 100 | 0.150 | +0.020 |  |  |  |
|  | 5.3 | 250 | 200 | 160 | 125 | 0.198 | +0.024 |  |  |  |
| $11 / 4$ | 19 | 80 |  |  |  | 0.073 | +0.020 | 1.380 | 0.010 | 0.020 |
|  | 15 | 100 | 80 |  |  | 0.092 | +0.020 |  |  |  |
|  | 11.5 | 125 | 100 | 80 |  | 0.120 | +0.020 |  |  |  |
|  | 9 | 160 | 125 | 100 | 80 | 0.153 | +0.020 |  |  |  |
|  | 7 | 200 | 160 | 125 | 100 | 0.197 | +0.024 |  |  |  |
|  | 5.3 | 250 | 200 | 160 | 125 | 0.260 | +0.031 |  |  |  |
| $11 / 2$ | 19 | 80 |  |  |  | 0.085 | +0.020 | 0.230 | 0.015 | 0.020 |
|  | 15 | 100 | 80 |  |  | 0.107 | +0.020 |  |  |  |
|  | 11.5 | 125 | 100 | 80 |  | 0.140 | +0.020 |  |  |  |
|  | 9 | 160 | 125 | 100 | 80 | 0.179 | +0.020 |  |  |  |
|  | 7 | 200 | 160 | 125 | 100 | 0.230 | +0.028 |  |  |  |
|  | 5.3 | 250 | 200 | 160 | 125 | 0.304 | +0.036 |  |  |  |
| 2 | 19 | 80 |  |  |  | 0.109 | $+0.020$ | 2.067 | 0.015 | 0.020 |
|  | 15 | 100 | 80 |  |  | 0.138 | +0.020 |  |  |  |
|  | 11.5 | 125 | 100 | 80 |  | 0.180 | +0.022 |  |  |  |
|  | 9 | 160 | 125 | 100 | 80 | 0.230 | +0.028 |  |  |  |
|  | 7 | 200 | 160 | 125 | 100 | 0.295 | +0.035 |  |  |  |
|  | 5.3 | 250 | 200 | 160 | 125 | 0.390 | +0.047 |  |  |  |
| $21 / 2$ | 19 | 80 |  |  |  | 0.130 | +0.020 | 2.469 | 0.015 | 0.025 |
|  | 15 | 100 | 80 |  |  | 0.165 | +0.020 |  |  |  |
|  | 11.5 | 125 | 100 | 80 |  | 0.215 | +0.025 |  |  |  |

## MaterialProperties,PressureRatings,andPipe

Dimensions for Plastic Pipe

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Table 52C-4 Polyethylene plastic pipe (SIDR-PR)-I.D. controlled (nonthreaded)—Continued


[^3]$\overline{\text { Appendix 52C }} \quad$| MaterialProperties,PressureRatings,andPipe |
| :--- |
| Dimensions for Plastic Pipe |

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Table 52C-5 Polyethylene plastic pipe (SDR-PR)-O.D. controlled (IPS) (nonthreaded)

| Nominal pipe size (in) | SDR | --- PE pressure rating ( $\mathrm{lb} / \mathrm{in}^{2}$ ) --- |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  |  |  |  |  |  |  |  |  |
|  |  | 3408 | 3306 | 2305 | 1404 | min. (in) |  | average(in) | $\pm$ tolerance |  |
|  |  |  | 3406 |  |  |  | tolerance (in) |  |  |  |
|  |  |  | 2306 |  |  |  |  |  | (in) | (in) |
|  |  |  | 2406 |  |  |  |  |  |  |  |
| 1/2 | 32.5 | 51 | 40 | 32 | 25 | 0.062 | 0.020 | 0.840 | 0.004 | 0.004 |
|  | 26 | 64 | 50 | 40 | 32 | 0.062 | 0.020 |  |  |  |
|  | 21 | 80 | 63 | 50 | 40 | 0.062 | 0.020 |  |  |  |
|  | 17 | 100 | 79 | 63 | 50 | 0.062 | 0.020 |  |  |  |
|  | 15.5 | 110 | 87 | 69 | 55 | 0.062 | 0.020 |  |  |  |
|  | 13.5 | 128 | 100 | 80 | 64 | 0.062 | 0.020 |  |  |  |
|  | 11 | 160 | 126 | 100 | 80 | 0.076 | 0.020 |  |  |  |
|  | 9.3 | 193 | 152 | 120 | 96 | 0.090 | 0.020 |  |  |  |
|  | 9 | 200 | 158 | 125 | 100 | 0.093 | 0.020 |  |  |  |
|  | 7 | 267 | 210 | 167 | 133 | 0.120 | 0.020 |  |  |  |
| $3 / 4$ | 32.5 | 51 | 40 | 32 | 25 | 0.062 | 0.020 | 1.050 | 0.004 | 0.004 |
|  | 26 | 64 | 50 | 40 | 32 | 0.062 | 0.020 |  |  |  |
|  | 21 | 80 | 63 | 50 | 40 | 0.062 | 0.020 |  |  |  |
|  | 17 | 100 | 79 | 63 | 50 | 0.062 | 0.020 |  |  |  |
|  | 15.5 | 110 | 87 | 69 | 55 | 0.068 | 0.020 |  |  |  |
|  | 13.5 | 128 | 100 | 80 | 64 | 0.078 | 0.020 |  |  |  |
|  | 11 | 160 | 126 | 100 | 80 | 0.095 | 0.020 |  |  |  |
|  | 9.3 | 193 | 152 | 120 | 96 | 0.113 | 0.020 |  |  |  |
|  | 9 | 200 | 158 | 125 | 100 | 0.117 | 0.020 |  |  |  |
|  | 7 | 267 | 210 | 167 | 133 | 0.150 | 0.020 |  |  |  |
| 1 | 32.5 | 51 | 40 | 32 | 25 | 0.062 | 0.020 | 1.315 | 0.005 | 0.005 |
|  | 26 | 64 | 50 | 40 | 32 | 0.062 | 0.020 |  |  |  |
|  | 21 | 80 | 63 | 50 | 40 | 0.063 | 0.020 |  |  |  |
|  | 17 | 100 | 79 | 63 | 50 | 0.077 | 0.020 |  |  |  |
|  | 15.5 | 110 | 87 | 69 | 55 | 0.084 | 0.020 |  |  |  |
|  | 13.5 | 128 | 100 | 80 | 64 | 0.097 | 0.020 |  |  |  |
|  | 11 | 160 | 126 | 100 | 80 | 0.120 | 0.020 |  |  |  |
|  | 9.3 | 193 | 152 | 120 | 96 | 0.141 | 0.020 |  |  |  |
|  | 9 | 200 | 158 | 125 | 100 | 0.146 | 0.020 |  |  |  |
|  | 7 | 267 | 210 | 167 | 133 | 0.188 | 0.023 |  |  |  |
| $11 / 4$ | 32.5 | 51 | 40 | 32 | 25 | 0.062 | 0.020 | 1.660 | 0.005 | 0.005 |
|  | 26 | 64 | 50 | 40 | 32 | 0.064 | 0.020 |  |  |  |
|  | 21 | 80 | 63 | 50 | 40 | 0.079 | 0.020 |  |  |  |
|  | 17 | 100 | 79 | 63 | 50 | 0.098 | 0.020 |  |  |  |
|  | 15.5 | 110 | 87 | 69 | 55 | 0.107 | 0.020 |  |  |  |
|  | 13.5 | 128 | 100 | 80 | 64 | 0.123 | 0.020 |  |  |  |
|  | 11 | 160 | 126 | 100 | 80 | 0.151 | 0.020 |  |  |  |
|  | 9.3 | 193 | 152 | 120 | 96 | 0.178 | 0.021 |  |  |  |
|  | 9 | 200 | 158 | 125 | 100 | 0.184 | 0.022 |  |  |  |
|  | 7 | 267 | 210 | 167 | 133 | 0.237 | 0.028 |  |  |  |


| $\overline{\text { Appendix 52C }}$ | MaterialProperties,PressureRatings,andPipe <br> Dimensions for Plastic Pipe | Part 636 <br> National Engineering Handbook |
| :--- | :--- | :--- |

Table 52C-5 Polyethylene plastic pipe (SDR-PR)-O.D. controlled (IPS) (nonthreaded)—Continued

| Nominal pipe size (in) | SDR | --- PE pressure rating (lb/in ${ }^{2}$ ) <br> ----------- Material |  |  |  | ------------ Dimension and tolerance ---------- - <br> -- - Wall thickness --- -- - Outside diameter --- |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | 3408 | 3306 | 2305 | 1404 | min. (in) |  | average <br> (in) |  |  |
|  |  |  | 3406 |  |  |  | tolerance (in) |  | + | - |
|  |  |  | 2306 |  |  |  |  |  | (in) | (in) |
|  |  |  | 2406 |  |  |  |  |  |  |  |
| $11 / 2$ | 32.5 | 51 | 40 | 32 | 25 | 0.062 | 0.020 | 1.900 | 0.006 | 0.006 |
|  | 26 | 64 | 50 | 40 | 32 | 0.073 | 0.020 |  |  |  |
|  | 21 | 80 | 63 | 50 | 40 | 0.090 | 0.020 |  |  |  |
|  | 17 | 100 | 79 | 63 | 50 | 0.112 | 0.020 |  |  |  |
|  | 15.5 | 110 | 87 | 69 | 55 | 0.123 | 0.020 |  |  |  |
|  | 13.5 | 128 | 100 | 80 | 64 | 0.141 | 0.020 |  |  |  |
|  | 11 | 160 | 126 | 100 | 80 | 0.173 | 0.021 |  |  |  |
|  | 9.3 | 193 | 152 | 120 | 96 | 0.204 | 0.024 |  |  |  |
|  | 9 | 200 | 158 | 125 | 100 | 0.211 | 0.025 |  |  |  |
|  | 7 | 267 | 210 | 167 | 133 | 0.271 | 0.033 |  |  |  |
| 2 | 32.5 | 51 | 40 | 32 | 25 | 0.073 | 0.020 | 2.375 | 0.006 | 0.006 |
|  | 26 | 64 | 50 | 40 | 32 | 0.091 | 0.020 |  |  |  |
|  | 21 | 80 | 63 | 50 | 40 | 0.113 | 0.020 |  |  |  |
|  | 17 | 100 | 79 | 63 | 50 | 0.140 | 0.020 |  |  |  |
|  | 15.5 | 110 | 87 | 69 | 55 | 0.153 | 0.020 |  |  |  |
|  | 13.5 | 128 | 100 | 80 | 64 | 0.176 | 0.021 |  |  |  |
|  | 11 | 160 | 126 | 100 | 80 | 0.216 | 0.026 |  |  |  |
|  | 9.3 | 193 | 152 | 120 | 96 | 0.255 | 0.031 |  |  |  |
|  | 9 | 200 | 158 | 125 | 100 | 0.264 | 0.032 |  |  |  |
|  | 7 | 267 | 210 | 167 | 133 | 0.339 | 0.041 |  |  |  |
| 3 | 32.5 | 51 | 40 | 32 | 25 | 0.108 | 0.020 | 3.500 | 0.008 | 0.008 |
|  | 26 | 64 | 50 | 40 | 32 | 0.135 | 0.020 |  |  |  |
|  | 21 | 80 | 63 | 50 | 40 | 0.167 | 0.020 |  |  |  |
|  | 17 | 100 | 79 | 63 | 50 | 0.206 | 0.025 |  |  |  |
|  | 15.5 | 110 | 87 | 69 | 55 | 0.226 | 0.027 |  |  |  |
|  | 13.5 | 128 | 100 | 80 | 64 | 0.259 | 0.031 |  |  |  |
|  | 11 | 160 | 126 | 100 | 80 | 0.318 | 0.038 |  |  |  |
|  | 9.3 | 193 | 152 | 120 | 96 | 0.376 | 0.045 |  |  |  |
|  | 9 | 200 | 158 | 125 | 100 | 0.389 | 0.047 |  |  |  |
|  | 7 | 267 | 210 | 167 | 133 | 0.500 | 0.060 |  |  |  |
| 4 | 32.5 | 51 | 40 | 32 | 25 | 0.138 | 0.020 | 4.500 | 0.009 | 0.009 |
|  | 26 | 64 | 50 | 40 | 32 | 0.173 | 0.021 |  |  |  |
|  | 21 | 80 | 63 | 50 | 40 | 0.214 | 0.026 |  |  |  |
|  | 17 | 100 | 79 | 63 | 50 | 0.265 | 0.032 |  |  |  |
|  | 15.5 | 110 | 87 | 69 | 55 | 0.290 | 0.035 |  |  |  |
|  | 13.5 | 128 | 100 | 80 | 64 | 0.333 | 0.040 |  |  |  |
|  | 11 | 160 | 126 | 100 | 80 | 0.409 | 0.049 |  |  |  |
|  | 9.3 | 193 | 152 | 120 | 96 | 0.484 | 0.058 |  |  |  |
|  | 9 | 200 | 158 | 125 | 100 | 0.500 | 0.060 |  |  |  |
|  | 7 | 267 | 210 | 167 | 133 | 0.643 | 0.077 |  |  |  |


$\overline{\text { Appendix 52C }} \quad$| MaterialProperties,PressureRatings,andPipe |
| :--- |
| Dimensions for Plastic Pipe |

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Table 52C-5 Polyethylene plastic pipe (SDR-PR)-O.D. controlled (IPS) (nonthreaded)—Continued

| Nominal pipe size (in) | SDR | -- - PE pressure rating ( $\left(\mathrm{lb} / \mathrm{in}^{2}\right.$ ) -- - <br> ---------- Material ---------- |  |  |  | ------------ Dimension and tolerance ---------- - <br> -- - Wall thickness --- --- Outside diameter --average $\pm$ tolerance |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | 3408 | 3306 | 2305 | 1404 |  |  |  |  |  |
|  |  |  | 3406 |  |  | min. (in) | tolerance (in) | (in) | (in) | (in) |
|  |  |  | 2306 |  |  |  |  |  |  |  |
|  |  |  | 2406 |  |  |  |  |  |  |  |
| 5 | 32.5 | 51 | 40 | 32 | 25 | 0.171 | 0.021 |  |  |  |
|  | 26 | 64 | 50 | 40 | 32 | 0.214 | 0.026 |  |  |  |
|  | 21 | 80 | 63 | 50 | 40 | 0.265 | 0.032 |  |  |  |
|  | 17 | 100 | 79 | 63 | 50 | 0.327 | 0.039 |  |  |  |
|  | 15.5 | 110 | 87 | 69 | 55 | 0.359 | 0.043 |  |  |  |
|  | 13.5 | 128 | 100 | 80 | 64 | 0.412 | 0.049 |  |  |  |
|  | 11 | 160 | 126 | 100 | 80 | 0.506 | 0.061 |  |  |  |
|  | 9.3 | 193 | 152 | 120 | 96 | 0.598 | 0.072 |  |  |  |
|  | 9 | 200 | 158 | 125 | 100 | 0.618 | 0.074 |  |  |  |
|  | 7 | 267 | 210 | 167 | 133 | 0.795 | 0.095 |  |  |  |
| 6 | 32.5 | 51 | 40 | 32 | 25 | 0.204 | 0.024 | 6.625 | 0.011 | 0.011 |
|  | 26 | 64 | 50 | 40 | 32 | 0.255 | 0.031 |  |  |  |
|  | 21 | 80 | 63 | 50 | 40 | 0.315 | 0.038 |  |  |  |
|  | 17 | 100 | 79 | 63 | 50 | 0.390 | 0.047 |  |  |  |
|  | 15.5 | 110 | 87 | 69 | 55 | 0.427 | 0.051 |  |  |  |
|  | 13.5 | 128 | 100 | 80 | 64 | 0.491 | 0.059 |  |  |  |
|  | 11 | 160 | 126 | 100 | 80 | 0.602 | 0.072 |  |  |  |
|  | 9.3 | 193 | 152 | 120 | 96 | 0.712 | 0.085 |  |  |  |
|  | 9 | 200 | 158 | 125 | 100 | 0.736 | 0.088 |  |  |  |
|  | 7 | 267 | 210 | 167 | 133 | 0.946 | 0.114 |  |  |  |
| 8 | 32.5 | 51 | 40 | 32 | 25 | 0.265 | 0.032 | 8.625 | 0.013 | 0.013 |
|  | 26 | 64 | 50 | 40 | 32 | 0.332 | 0.040 |  |  |  |
|  | 21 | 80 | 63 | 50 | 40 | 0.411 | 0.049 |  |  |  |
|  | 17 | 100 | 79 | 63 | 50 | 0.507 | 0.061 |  |  |  |
|  | 15.5 | 110 | 87 | 69 | 55 | 0.556 | 0.067 |  |  |  |
|  | 13.5 | 128 | 100 | 80 | 64 | 0.639 | 0.077 |  |  |  |
|  | 11 | 160 | 126 | 100 | 80 | 0.784 | 0.094 |  |  |  |
|  | 9.3 | 193 | 152 | 120 | 96 | 0.927 | 0.111 |  |  |  |
|  | 9 | 200 | 158 | 125 | 100 | 0.958 | 0.115 |  |  |  |
|  | 7 | 267 | 210 | 167 | 133 | 1.232 | 0.147 |  |  |  |
| 10 | 32.5 | 51 | 40 | 32 | 25 | 0.331 | 0.040 | 10.750 | 0.015 | 0.015 |
|  | 26 | 64 | 50 | 40 | 32 | 0.413 | 0.050 |  |  |  |
|  | 21 | 80 | 63 | 50 | 40 | 0.512 | 0.061 |  |  |  |
|  | 17 | 100 | 79 | 63 | 50 | 0.632 | 0.076 |  |  |  |
|  | 15.5 | 110 | 87 | 69 | 55 | 0.694 | 0.083 |  |  |  |
|  | 13.5 | 128 | 100 | 80 | 64 | 0.796 | 0.096 |  |  |  |
|  | 11 | 160 | 126 | 100 | 80 | 0.977 | 0.117 |  |  |  |
|  | 9.3 | 193 | 152 | 120 | 96 | 1.156 | 0.139 |  |  |  |
|  | 9 | 200 | 158 | 125 | 100 | 1.194 | 0.143 |  |  |  |
|  | 7 | 267 | 210 | 167 | 133 | 1.536 | 0.184 |  |  |  |


| $\overline{\text { Appendix 52C }}$ | MaterialProperties,PressureRatings,andPipe <br> Dimensions for Plastic Pipe | Part 636 <br> National Engineering Handbook |
| :--- | :--- | :--- |

Table 52C-5 Polyethylene plastic pipe (SDR-PR)-O.D. controlled (IPS) (nonthreaded)—Continued

| Nominal pipe size (in) | SDR | --- PE pressure rating (lb/in ${ }^{2}$ ) <br> ----------- Material |  |  |  | ------------ Dimension and tolerance ---------- - <br> -- - Wall thickness --- -- - Outside diameter --- |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | 3408 | 3306 | 2305 | 1404 |  |  | average <br> (in) |  |  |
|  |  |  | 3406 |  |  | min. (in) | tolerance (in) |  | + | - |
|  |  |  | 2306 |  |  |  |  |  | (in) | (in) |
|  |  |  | 2406 |  |  |  |  |  |  |  |
| 12 | 32.5 | 51 | 40 | 32 | 25 | 0.392 | 0.047 | 12.750 | 0.017 | 0.017 |
|  | 26 | 64 | 50 | 40 | 32 | 0.490 | 0.059 |  |  |  |
|  | 21 | 80 | 63 | 50 | 40 | 0.607 | 0.073 |  |  |  |
|  | 17 | 100 | 79 | 63 | 50 | 0.750 | 0.090 |  |  |  |
|  | 15.5 | 110 | 87 | 69 | 55 | 0.823 | 0.099 |  |  |  |
|  | 13.5 | 128 | 100 | 80 | 64 | 0.944 | 0.113 |  |  |  |
|  | 11 | 160 | 126 | 100 | 80 | 1.159 | 0.139 |  |  |  |
|  | 9.3 | 193 | 152 | 120 | 96 | 1.371 | 0.165 |  |  |  |
|  | 9 | 200 | 158 | 125 | 100 | 1.417 | 0.170 |  |  |  |
|  | 7 | 267 | 210 | 167 | 133 | 1.821 | 0.219 |  |  |  |
| 14 | 32.5 | 51 | 40 | 32 | 25 | 0.431 | 0.052 | 14.000 | 0.063 | 0.063 |
|  | 26 | 64 | 50 | 40 | 32 | 0.538 | 0.065 |  |  |  |
|  | 21 | 80 | 63 | 50 | 40 | 0.667 | 0.080 |  |  |  |
|  | 17 | 100 | 79 | 63 | 50 | 0.824 | 0.099 |  |  |  |
|  | 15.5 | 110 | 87 | 69 | 55 | 0.903 | 0.108 |  |  |  |
|  | 13.5 | 128 | 100 | 80 | 64 | 1.037 | 0.124 |  |  |  |
|  | 11 | 160 | 126 | 100 | 80 | 1.273 | 0.153 |  |  |  |
|  | 9.3 | 193 | 152 | 120 | 96 | 1.505 | 0.181 |  |  |  |
|  | 9 | 200 | 158 | 125 | 100 | 1.556 | 0.187 |  |  |  |
|  | 7 | 267 | 210 | 167 | 133 | 2.000 | 0.240 |  |  |  |
| 16 | 32.5 | 51 | 40 | 32 | 25 | 0.492 | 0.059 | 16.000 | 0.072 | 0.072 |
|  | 26 | 64 | 50 | 40 | 32 | 0.615 | 0.074 |  |  |  |
|  | 21 | 80 | 63 | 50 | 40 | 0.762 | 0.091 |  |  |  |
|  | 17 | 100 | 79 | 63 | 50 | 0.941 | 0.113 |  |  |  |
|  | 15.5 | 110 | 87 | 69 | 55 | 1.032 | 0.124 |  |  |  |
|  | 13.5 | 128 | 100 | 80 | 64 | 1.185 | 0.142 |  |  |  |
|  | 11 | 160 | 126 | 100 | 80 | 1.455 | 0.175 |  |  |  |
|  | 9.3 | 193 | 152 | 120 | 96 | 1.720 | 0.206 |  |  |  |
|  | 9 | 200 | 158 | 125 | 100 | 1.778 | 0.213 |  |  |  |
|  | 7 | 267 | 210 | 167 | 133 | 2.286 | 0.274 |  |  |  |
| 18 | 32.5 | 51 | 40 | 32 | 25 | 0.554 | 0.066 | 18.000 | 0.081 | 0.081 |
|  | 26 | 64 | 50 | 40 | 32 | 0.692 | 0.083 |  |  |  |
|  | 21 | 80 | 63 | 50 | 40 | 0.857 | 0.103 |  |  |  |
|  | 17 | 100 | 79 | 63 | 50 | 1.059 | 0.127 |  |  |  |
|  | 15.5 | 110 | 87 | 69 | 55 | 1.161 | 0.139 |  |  |  |
|  | 13.5 | 128 | 100 | 80 | 64 | 1.333 | 0.160 |  |  |  |
|  | 11 | 160 | 126 | 100 | 80 | 1.636 | 0.196 |  |  |  |
|  | 9.3 | 193 | 152 | 120 | 96 | 1.935 | 0.232 |  |  |  |
|  | 9 | 200 | 158 | 125 | 100 | 2.000 | 0.240 |  |  |  |
|  | 7 | 267 | 210 | 167 | 133 | 2.571 | 0.309 |  |  |  |


$\overline{\text { Appendix 52C }} \quad$| MaterialProperties,PressureRatings,andPipe |
| :--- |
| Dimensions for Plastic Pipe |

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Table 52C-5 Polyethylene plastic pipe (SDR-PR)-O.D. controlled (IPS) (nonthreaded)—Continued

| Nominal pipe size (in) | SDR | -- - PE pressure rating ( $\mathrm{lb} / \mathrm{in}^{2}$ ) -- |  |  |  | ------------ Dimension and tolerance ----------- <br> -- - Wall thickness -- -- - Outside diameter --- - |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  |  |  |  |  |  |  |  |  |
|  |  | 3408 | 3306 | 2305 | 1404 | min. (in) | $\begin{aligned} & \text { toler- } \\ & \text { ance (in) } \end{aligned}$ | $\underset{\text { (in) }}{\substack{\text { average }}}$ | $\pm$ tolerance |  |
|  |  |  | 3406 |  |  |  |  |  | ${ }_{\text {+ }}+$ | - |
|  |  |  | 2306 |  |  |  |  |  | (in) | (in) |
|  |  |  | 2406 |  |  |  |  |  |  |  |
| 20 | 32.5 | 51 | 40 | 32 | 25 | 0.615 | 0.074 | 20.000 | 0.090 | 0.090 |
|  | 26 | 64 | 50 | 40 | 32 | 0.769 | 0.092 |  |  |  |
|  | 21 | 80 | 63 | 50 | 40 | 0.952 | 0.114 |  |  |  |
|  | 17 | 100 | 79 | 63 | 50 | 1.176 | 0.141 |  |  |  |
|  | 15.5 | 110 | 87 | 69 | 55 | 1.290 | 0.155 |  |  |  |
|  | 13.5 | 128 | 100 | 80 | 64 | 1.481 | 0.178 |  |  |  |
|  | 11 | 160 | 126 | 100 | 80 | 1.818 | 0.218 |  |  |  |
|  | 9.3 | 193 | 152 | 120 | 96 | 2.151 | 0.258 |  |  |  |
|  | 9 | 200 | 158 | 125 | 100 | 2.222 | 0.267 |  |  |  |
|  | 7 | 267 | 210 | 167 | 133 | 2.857 | 0.343 |  |  |  |
| 22 | 32.5 | 51 | 40 | 32 | 25 | 0.677 | 0.089 | 22.000 | 0.099 | 0.099 |
|  | 26 | 64 | 50 | 40 | 32 | 0.846 | 0.102 |  |  |  |
|  | 21 | 80 | 63 | 50 | 40 | 1.048 | 0.126 |  |  |  |
|  | 17 | 100 | 79 | 63 | 50 | 1.294 | 0.155 |  |  |  |
|  | 15.5 | 110 | 87 | 69 | 55 | 1.419 | 0.170 |  |  |  |
|  | 13.5 | 128 | 100 | 80 | 64 | 1.630 | 0.196 |  |  |  |
|  | 11 | 160 | 126 | 100 | 80 | 2.000 | 0.240 |  |  |  |
|  | 9.3 | 193 | 152 | 120 | 96 | 2.366 | 0.284 |  |  |  |
|  | 9 | 200 | 158 | 125 | 100 | 2.444 | 0.293 |  |  |  |
|  | 7 | 267 | 210 | 167 | 133 | 3.143 | 0.377 |  |  |  |
| 24 | 32.5 | 51 | 40 | 32 | 25 | 0.738 | 0.089 | 24.000 | 0.108 | 0.108 |
|  | 26 | 64 | 50 | 40 | 32 | 0.923 | 0.111 |  |  |  |
|  | 21 | 80 | 63 | 50 | 40 | 1.143 | 0.137 |  |  |  |
|  | 17 | 100 | 79 | 63 | 50 | 1.412 | 0.169 |  |  |  |
|  | 15.5 | 110 | 87 | 69 | 55 | 1.548 | 0.186 |  |  |  |
|  | 13.5 | 128 | 100 | 80 | 64 | 1.778 | 0.213 |  |  |  |
|  | 11 | 160 | 126 | 100 | 80 | 2.182 | 0.262 |  |  |  |
|  | 9.3 | 193 | 152 | 120 | 96 | 2.581 | 0.310 |  |  |  |
|  | 9 | 200 | 158 | 125 | 100 | 2.667 | 0.320 |  |  |  |
|  | 7 | 267 | 210 | 167 | 133 | 3.429 | 0.411 |  |  |  |

Source: ASTM D 3035

$\overline{\text { Appendix 52C }} \quad$| MaterialProperties,PressureRatings,andPipe |
| :--- |
| Dimensions for Plastic Pipe |

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Table 52C-6a PVC schedule 40, 80, and 120 and ABS schedule 40, and 80 plastic pipe (unthreaded)

| Nominal pipe size (in) | Sch. | -- PVC pressure rating $\left(\mathrm{lb} / \mathrm{in}^{2}\right)$-- - |  |  |  | -- -- - - - - Dimension and tolerance ------ - - <br> Wall thickness ---- Outside diameter --- |  |  |  |  | -- - ABS pressure rating $\left(\mathrm{lb} / \mathrm{in}^{2}\right)$-- - <br> ---------- Material |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | 1120 | 2116 | 2112 | 2110 |  |  |  |  |  | 1316 | 2112 | 1210 | 1208 |
|  |  | $\begin{aligned} & 1220 \\ & 2120 \end{aligned}$ |  |  |  | min. (in) | tolerance (in) | (in) | avg OD <br> (in) | $\max \&$ $\min$ (in) |  |  |  |  |
| 1/8 | 40 | 810 | 650 | 500 | 400 | 0.068 | +0.020 | 0.405 | 0.004 | 0.008 | 650 | 500 | 400 | 320 |
|  | 80 | 1230 | 980 | 770 | 610 | 0.095 | +0.020 |  |  |  | 980 |  |  |  |
| 1/4 | 40 | 780 | 620 | 490 | 390 | 0.088 | +0.020 | 0.540 | 0.004 | 0.008 | 620 | 490 | 390 | 310 |
|  | 80 | 1130 | 900 | 710 | 570 | 0.119 | +0.020 |  |  |  | 900 |  |  |  |
| 3/8 | 40 | 620 | 500 | 390 | 310 | 0.091 | $+0.020$ | 0.675 | 0.004 | 0.008 | 500 | 390 | 310 | 250 |
|  | 80 | 920 | 730 | 570 | 460 | 0.126 | +0.020 |  |  |  | 730 |  |  |  |
| 1/2 | 40 | 600 | 480 | 370 | 300 | 0.109 | +0.020 | 0.840 | 0.004 | 0.008 | 480 | 370 | 300 | 240 |
|  | 80 | 850 | 680 | 530 | 420 | 0.147 | +0.020 |  |  |  | 680 | 530 | 420 | 340 |
|  | 120 | 1010 | 810 | 630 | 510 | 0.170 | +0.020 |  |  |  |  |  |  |  |
| $3 / 4$ | 40 | 480 | 390 | 300 | 240 | 0.113 | +0.020 | 1.050 | 0.004 | 0.010 | 390 | 300 | 240 | 190 |
|  | 80 | 690 | 550 | 430 | 340 | 0.154 | +0.020 |  |  |  | 550 | 430 | 340 | 280 |
|  | 120 | 770 | 620 | 480 | 390 | 0.170 | +0.020 |  |  |  |  |  |  |  |
| 1 | 40 | 450 | 360 | 280 | 220 | 0.133 | +0.020 | 1.315 | 0.005 | 0.010 | 360 | 280 | 220 | 180 |
|  | 80 | 630 | 500 | 390 | 320 | 0.179 | +0.021 |  |  |  | 500 | 390 | 320 | 250 |
|  | 120 | 720 | 570 | 450 | 360 | 0.200 | +0.024 |  |  |  |  |  |  |  |
| $11 / 4$ | 40 | 370 | 290 | 230 | 180 | 0.140 | +0.020 | 1.660 | 0.005 | 0.012 | 290 | 230 | 180 | 150 |
|  | 80 | 520 | 420 | 320 | 260 | 0.191 | +0.023 |  |  |  | 420 | 330 | 260 | 210 |
|  | 120 | 600 | 480 | 370 | 300 | 0.215 | +0.026 |  |  |  |  |  |  |  |
| $11 / 2$ | 40 | 330 | 260 | 210 | 170 | 0.145 | $+0.020$ | 1.900 | 0.006 | 0.012 | 260 | 210 | 170 | 130 |
|  | 80 | 470 | 380 | 290 | 240 | 0.200 | +0.024 |  |  |  | 380 | 290 | 240 | 190 |
|  | 120 | 540 | 430 | 340 | 270 | 0.225 | +0.027 |  |  |  |  |  |  |  |
| 2 | 40 | 280 | 220 | 170 | 140 | 0.154 | +0.020 | 2.375 | 0.006 | 0.012 | 220 | 170 | 140 | 110 |
|  | 80 | 400 | 320 | 250 | 200 | 0.218 | +0.026 |  |  |  | 320 | 250 | 200 | 160 |
|  | 120 | 470 | 380 | 290 | 240 | 0.250 | +0.030 |  |  |  |  |  |  |  |
| $21 / 2$ | 40 | 300 | 240 | 190 | 150 | 0.203 | $+0.024$ | 2.875 | 0.007 | 0.015 | 240 | 190 | 150 | 120 |
|  | 80 | 420 | 340 | 260 | 210 | 0.276 | +0.033 |  |  |  | 340 | 270 | 210 | 170 |
|  | 120 | 470 | 370 | 290 | 230 | 0.300 | +0.036 |  |  |  |  |  |  |  |
| 3 | 40 | 260 | 210 | 160 | 130 | 0.216 | +0.026 | 3.500 | 0.008 | 0.015 | 210 | 160 | 130 | 100 |
|  | 80 | 370 | 300 | 230 | 190 | 0.300 | +0.036 |  |  |  | 300 | 230 | 190 | 150 |
|  | 120 | 440 | 360 | 280 | 220 | 0.350 | +0.042 |  |  |  |  |  |  |  |
| $31 / 2$ | 40 | 240 | 190 | 150 | 120 | 0.226 | $+0.027$ | 4.000 | 0.008 | 0.050 | 190 | 150 | 120 | 90 |
|  | 80 | 350 | 280 | 220 | 170 | 0.318 | +0.038 |  |  | 0.015 | 280 | 220 | 170 | 140 |
|  | 120 | 380 | 310 | 240 | 190 | 0.350 | +0.042 |  |  | 0.015 |  |  |  |  |
| 4 | 40 | 220 | 180 | 140 | 110 | 0.237 | +0.028 | 4.500 | 0.009 | 0.050 | 180 | 140 | 110 | 90 |
|  | 80 | 320 | 260 | 200 | 160 | 0.337 | +0.040 |  |  | 0.015 | 260 | 200 | 160 | 130 |
|  | 120 | 430 | 340 | 270 | 220 | 0.437 | +0.052 |  |  | 0.015 |  |  |  |  |


$\overline{\text { Appendix 52C }} \quad$| MaterialProperties,PressureRatings,andPipe |
| :--- |
| Dimensions for Plastic Pipe |

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Table 52C-6a PVC schedule 40, 80, and 120 and ABS schedule 40, and 80 plastic pipe (unthreaded)—Continued


Source: ASTM D 1785 for PVC and D 1527 for ABS.

MaterialProperties,PressureRatings,andPipe
Dimensions for Plastic Pipe

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Table 52C-6b PE schedule 40 and 80 plastic pipe (unthreaded)

| Nominal (in) <br> (in) | Sch. | $\begin{gathered} \text { PE pressure rating } \\ \binom{\text { (binin } \left.{ }^{2}\right)}{\text { Material }} \end{gathered}$ |  |  | D2104 Inside diameter |  |  | Dimension and tolerance |  |  |  |  | PE pressure rating (lb/in ${ }^{2}$ ) |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  |  |  | Wall thickness | D 2447Outside diameter |  |  |  |  | Iaterial |
|  |  |  |  |  |  |  |  | average <br> (in) | $\pm$ tolerance |  |  | min. | tolerance | average | $\pm$ tole |  | 2306 |
|  |  | 2406 |  |  | + (in) | - (in) | (in) |  | (in) |  | (in) | + (in) | - (in) | 2406 |  |  |
|  |  | 3306 |  |  |  |  |  |  |  |  |  |  | 3306 |  |  |
|  |  | 3406 |  |  |  |  |  |  |  |  |  |  | 3406 |  |  |
| 1/2 | 40 | 190 | 150 | 120 | 0.622 | 0.010 | 0.010 | 0.109 | +0.020 | 0.840 | 0.004 | 0.004 | 188 | 149 | 119 |
|  | 80 |  |  |  |  |  |  | 0.147 | +0.020 |  |  |  | 267 | 212 | 170 |
| $3 / 4$ | 40 | 150 | 120 | 100 | 0.824 | 0.010 | 0.015 | 0.113 | +0.020 | 1.050 | 0.004 | 0.004 | 152 | 120 | 96 |
|  | 80 |  |  |  |  |  |  | 0.154 | +0.020 |  |  |  | 217 | 172 | 137 |
| 1 | 40 | 140 | 110 | 90 | 1.049 | 0.010 | 0.020 | 0.133 | +0.020 | 1.315 | 0.005 | 0.005 | 142 | 113 | 90 |
|  | 80 |  |  |  |  |  |  | 0.179 | +0.021 |  |  |  | 199 | 158 | 126 |
| 11/4 | 40 | 120 | 90 | 70 | 1.380 | 0.010 | 0.020 | 0.140 | +0.020 | 1.660 | 0.005 | 0.005 | 116 | 92 | 74 |
|  | 80 |  |  |  |  |  |  | 0.191 | +0.023 |  |  |  | 164 | 130 | 104 |
| 11/2 | 40 | 100 | 80 | 70 | 1.610 | 0.015 | 0.020 | 0.145 | +0.020 | 1.900 | 0.006 | 0.006 | 104 | 83 | 66 |
|  | 80 |  |  |  |  |  |  | 0.200 | +0.024 |  |  |  | 148 | 118 | 94 |
| 2 | 40 | 90 | 70 | 60 | 2.067 | 0.015 | 0.020 | 0.154 | +0.020 | 2.375 | 0.006 | 0.006 | 87 | 69 | 55 |
|  | 80 |  |  |  |  |  |  | 0.218 | +0.026 |  |  |  | 127 | 101 | 81 |
| $21 / 2$ | 40 | 100 | 80 | 60 | 2.469 | 0.015 | 0.025 | 0.203 | +0.024 | 2.875 | 0.007 | 0.007 | 96 | 76 | 61 |
|  | 80 |  |  |  |  |  |  | 0.276 | +0.033 |  |  |  | 134 | 106 | 85 |
| 3 | 40 | 80 | 70 | 50 | 3.068 | 0.015 | 0.030 | 0.216 | +0.026 | 3.500 | 0.008 | 0.008 | 83 | 66 | 53 |
|  | 80 |  |  |  |  |  |  | 0.300 | +0.036 |  |  |  | 118 | 94 | 75 |
| $31 / 2$ | 40 |  |  |  |  |  |  | 0.226 | +0.027 | 4.000 | 0.008 | 0.008 | 75 | 60 | 50 |
|  | 80 |  |  |  |  |  |  | 0.318 | +0.038 |  |  |  | 109 | 86 | 69 |
| 4 | 40 | 70 | 60 | NPR | 4.026 | 0.015 | 0.035 | 0.237 | +0.028 | 4.500 | 0.009 | 0.009 | 70 | 55 | NPR |
|  | 80 |  |  |  |  |  |  | 0.337 | +0.040 |  |  |  | 102 | 81 | 65 |
| 5 | 40 |  |  |  |  |  |  | 0.258 | +0.031 | 5.563 | 0.010 | 0.010 | 61 | 50 | NPR |
|  | 80 |  |  |  |  |  |  | 0.375 | +0.045 |  |  |  | 91 | 72 | 58 |
| 6 | 40 | 60 | NPR | NPR | 6.065 | 0.020 | 0.035 | 0.280 | +0.034 | 6.625 | 0.011 | 0.011 | 55 | NPR | NPR |
|  | 80 |  |  |  |  |  |  | 0.432 | +0.052 |  |  |  | 88 | 70 | 56 |
| 8 | 40 |  |  |  |  |  |  | 0.322 | +0.039 | 8.625 | 0.015 | 0.015 | 50 | NPR | NPR |
| 10 | 40 |  |  |  |  |  |  | 0.365 | +0.044 | 10.750 | 0.015 | 0.015 | NPR | NPR | NPR |
| 12 | 40 |  |  |  |  |  |  | 0.406 | +0.049 | 12.750 | 0.015 | 0.015 | NPR | NPR | NPR |

[^4]

[^5]MaterialProperties,PressureRatings,andPipe
Dimensions for Plastic Pipe

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Table 52C-8 PVC plastic pipe dimensions, pressure classes, SDR, and tolerances for iron pipe sizes

| Nominal pipe size (in) | $\begin{gathered} \text { class } \\ \left(\mathrm{lb} / \mathrm{in}^{2}\right) \end{gathered}$ |  | Outside diameter (in) average tolerance |  | Minimum wall thickness (in) minimum tolerance |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 4 | 100 | 25 | 4.80 | 0.009 | 0.192 | 0.023 |
|  | 150 | 18 |  |  | 0.267 | 0.032 |
|  | 200 | 14 |  |  | 0.343 | 0.041 |
| 6 | 100 | 25 | 6.90 | 0.011 | 0.276 | 0.033 |
|  | 150 | 18 |  |  | 0.383 | 0.046 |
|  | 200 | 14 |  |  | 0.493 | 0.059 |
| 8 | 100 | 25 | 9.05 | 0.015 | 0.362 | 0.043 |
|  | 150 | 18 |  |  | 0.503 | 0.060 |
|  | 200 | 14 |  |  | 0.646 | 0.078 |
| 10 | 100 | 25 | 11.10 | 0.015 | 0.444 | 0.053 |
|  | 150 | 18 |  |  | 0.617 | 0.074 |
|  | 200 | 14 |  |  | 0.793 | 0.095 |
| 12 | 100 | 25 | 13.20 | 0.015 | 0.528 | 0.063 |
|  | 150 | 18 |  |  | 0.733 | 0.088 |
|  | 200 | 14 |  |  | 0.943 | 0.113 |

Source: AWWA C900
Hydrostatic Design Stress $(H D S)=1,600 \mathrm{lb} / \mathrm{in}^{2}$

Table 52C-9 Polyethylene pipe, inside diameter based

| Nominal pipe size (in) | SIDR | $\begin{gathered} \text { Pressure class } \\ \text {-- Material -- } \end{gathered}$ |  | -------------- - Dimside diameter ------------------- -- |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | $\begin{aligned} & 2406 \\ & 3406 \end{aligned}$ | 3408 | minimum <br> (in) | $\begin{aligned} & \text {-- tole } \\ & -(\text { in) } \end{aligned}$ | $\begin{gathered} \text { ance -- - } \\ +(\text { in }) \end{gathered}$ | minimum | tolerance |
| 0.5 | 9 | 125 | 160 | 0.622 | 0.010 | 0.010 | 0.069 | +0.020 |
|  | 7 | 160 | 200 |  |  |  | 0.089 | +0.020 |
|  | 5.3 | 200 |  |  |  |  | 0.117 | +0.020 |
| 0.75 | 11.5 |  | 125 | 0.824 | 0.015 | 0.010 | 0.072 | +0.020 |
|  | 9 | 125 | 160 |  |  |  | 0.092 | +0.020 |
|  | 7 | 160 | 200 |  |  |  | 0.118 | +0.020 |
|  | 5.3 | 200 |  |  |  |  | 0.155 | +0.020 |
| 1 | 11.5 |  | 125 | 1.049 | 0.020 | 0.010 | 0.091 | +0.020 |
|  | 9 | 125 | 160 |  |  |  | 0.117 | +0.020 |
|  | 7 | 160 | 200 |  |  |  | 0.150 | +0.020 |
|  | 5.3 | 200 |  |  |  |  | 0.198 | +0.024 |

## Appendix 52C

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Table 52C-9 Polyethylene pipe, inside diameter based-Continued

| Nominal pipe size (in) | SIDR | $\begin{gathered} \text { Pressure class } \\ \text {-- Matial -- } \end{gathered}$ |  | ------------- - Dimension and tolerance ------------ - - |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  |  |  |  |  |  |  |
|  |  | 2406 | 3408 | minimum | --- tol | ance --- | minimum | tolerance |
|  |  | 3406 |  | (in) | - (in) | + (in) |  |  |
| 1.25 | 11.5 |  | 125 | 1.380 | 0.020 | 0.010 | 0.120 | +0.020 |
|  | 9 | 125 | 160 |  |  |  | 0.153 | +0.020 |
|  | 7 | 160 | 200 |  |  |  | 0.197 | +0.024 |
|  | 5.3 | 200 |  |  |  |  | 0.260 | +0.031 |
| 1.5 | 11.5 |  | 125 |  |  |  | 0.140 | +0.020 |
|  | 9 | 125 | 160 |  |  |  | 0.179 | +0.020 |
|  | 7 | 160 | 200 |  |  |  | 0.230 | +0.028 |
|  | 5.3 | 200 |  |  |  |  | 0.304 | +0.036 |
| 2 | 19 |  | 80 | 2.067 | 0.020 | 0.015 | 0.109 | +0.020 |
|  | 15 | 80 | 100 |  |  |  | 0.138 | +0.020 |
|  | 11.5 | 100 | 125 |  |  |  | 0.180 | +0.022 |
|  | 9 | 125 | 160 |  |  |  | 0.230 | +0.028 |
|  | 7 | 160 | 200 |  |  |  | 0.295 | +0.035 |
|  | 5.3 | 200 |  |  |  |  | 0.390 | +0.047 |
| 2.5 | 19 |  | 80 | 2.469 | 0.025 | 0.015 | 0.130 | +0.020 |
|  | 15 | 80 | 100 |  |  |  | 0.165 | +0.020 |
|  | 11.5 | 100 | 125 |  |  |  | 0.215 | +0.025 |
|  | 9 | 125 | 160 |  |  |  | 0.272 | +0.033 |
|  | 7 | 160 | 200 |  |  |  | $0.353$ | $+0.042$ |
|  | 5.3 | 200 |  |  |  |  | 0.466 | +0.056 |
| 3 | 19 |  | 80 | 3.068 | 0.030 | 0.015 | 0.161 | +0.020 |
|  | 15 | 80 | 100 |  |  |  | 0.205 | +0.020 |
|  | 11.5 | 100 | 125 |  |  |  | 0.267 | +0.032 |
|  | 9 | 125 | 160 |  |  |  | 0.341 | +0.041 |
|  | 7 | 160 | 200 |  |  |  | 0.438 | +0.053 |
|  | 5.3 | 200 |  |  |  |  | 0.579 | +0.069 |

Source: AWWA C 901

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Table 52C-10 Polyethylene pipe, outside diameter based

| Nominal pipe size (in) | SDR | Pressure class -- - - Material- |  | ------------- Dimension and tolerance -------------- <br> ----- - Outside diameter ------ -- Wall thickness -- |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  |  |  |  |  |  |  |
|  |  | $\begin{aligned} & 2406 \\ & 3406 \end{aligned}$ | 3408 | minimum <br> (in) | $\begin{aligned} & \text { - - - tole } \\ & \text { - (in) } \end{aligned}$ | $\begin{aligned} & \text { ance -- - } \\ & +(\text { in) } \end{aligned}$ | minimum | tolerance |
| 0.5 | 11 | 125 | 160 | 0.840 | 0.004 | 0.004 | 0.076 | +0.020 |
|  | 9 | 160 | 200 |  |  |  | 0.093 | +0.020 |
| 0.75 | 13.5 |  | 125 | 1.050 | 0.004 | 0.004 | 0.078 | +0.020 |
|  | 11 | 125 | 160 |  |  |  | 0.095 | +0.020 |
|  | 9 | 160 | 200 |  |  |  | 0.117 | +0.020 |
| 1 | 13.5 |  | 125 | 1.315 | 0.005 | 0.005 | 0.097 | +0.020 |
|  | 11 | 125 | 160 |  |  |  | 0.119 | +0.020 |
|  | 9 | 160 | 200 |  |  |  | 0.146 | +0.020 |
| 1.25 | 13.5 |  | 125 | 1.660 | 0.005 | 0.005 | 0.123 | +0.020 |
|  | 11 | 125 | 160 |  |  |  | 0.151 | +0.020 |
|  | 9 | 160 | 200 |  |  |  | 0.184 | +0.022 |
| 1.5 | 13.5 |  | 125 | 1.900 | 0.006 | 0.006 | 0.141 | +0.020 |
|  | 11 | 125 | 160 |  |  |  | 0.173 | +0.021 |
|  | 9 | 160 | 200 |  |  |  | 0.211 | +0.025 |
| 2 | 21 |  | 80 | 2.375 | 0.006 | 0.006 | 0.113 | +0.020 |
|  | 17 | 80 | 100 |  |  |  | 0.140 | +0.020 |
|  | 13.5 | 100 | 125 |  |  |  | 0.176 | +0.021 |
|  | 11 | 125 | 160 |  |  |  | 0.216 | +0.026 |
|  | 9 | 160 | 200 |  |  |  | 0.264 | +0.032 |
| 3 | 21 |  | 80 | 3.500 | 0.008 | 0.008 | 0.167 | +0.020 |
|  | 17 | 80 | 100 |  |  |  | 0.206 | +0.025 |
|  | 13.5 | 100 | 125 |  |  |  | 0.259 | +0.031 |
|  | 11 | 125 | 160 |  |  |  | 0.318 | +0.038 |
|  | 9 | 160 | 200 |  |  |  | 0.389 | +0.047 |

[^6]
## Appendix 52C

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Table 52C-11 PVC plastic pipe, iron pipe size (IPS) outside diameter

| Nominal pipe size (in) | SDR | Pressure rating (lb/in ${ }^{2}$ ) | Outside average | - Dimension and diameter (in) tolerance (-/+) | tolerance Wall thi minimum | ness (in) tolerance |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 14 | 41 | 100 | 14.000 | 0.015 | 0.341 | +0.048 |
|  | 32.5 | 125 |  |  | 0.430 | +0.052 |
|  | 26 | 160 |  |  | 0.538 | +0.064 |
|  | 21 | 200 |  |  | 0.666 | +0.080 |
| 16 | 41 | 100 | 16.000 | 0.019 | 0.390 | +0.055 |
|  | 32.5 | 125 |  |  | 0.492 | +0.059 |
|  | 26 | 160 |  |  | 0.615 | +0.074 |
|  | 21 | 200 |  |  | 0.762 | +0.091 |
| 18 | 41 | 100 | 18.000 | 0.019 | 0.439 | +0.061 |
|  | 32.5 | 125 |  |  | 0.554 | +0.066 |
|  | 26 | 160 |  |  | 0.692 | +0.083 |
|  | 21 | 200 |  |  | 0.857 | +0.103 |
| 20 | 41 | 100 | 20.000 | 0.023 | 0.488 | +0.068 |
|  | 32.5 | 125 |  |  | 0.615 | +0.074 |
|  | 26 | 160 |  |  | 0.769 | +0.092 |
|  | 21 | 200 |  |  | 0.952 | +0.114 |
| 24 | 41 | 100 | 24.000 | 0.031 | 0.585 | +0.082 |
|  | 32.5 | 125 |  |  | 0.738 | +0.088 |
|  | 26 | 160 |  |  | 0.923 | +0.111 |
|  | 21 | 200 |  |  | 1.143 | +0.137 |
| 30 | 41 | 100 | 30.000 | 0.041 | 0.732 | +0.102 |
|  | 32.5 | 125 |  |  | 0.923 | +0.111 |
|  | 26 | 160 |  |  | 1.154 | +0.138 |
|  | 21 | 200 |  |  | 1.428 | +0.171 |
| 36 | 41 | 100 | 36.000 | 0.050 | 0.878 | +0.123 |
|  | 32.5 | 125 |  |  | 1.108 | +0.133 |
|  | 26 | 160 |  |  | 1.385 | +0.166 |
|  | 21 | 200 |  |  | 1.714 | +0.205 |

Source: AWWA C 905
PVC material cell class 12454-B as defined by ASTM D 1784 with hydrostatic design basis of 4,000 pounds per square inch.

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Table 52C-12 PVC plastic pipe, ductile iron pipe size (IPS) outside diameter

| Nominal pipe size (in) | SDR | Pressure rating ( $\mathrm{lb} / \mathrm{in}^{2}$ ) |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 14 | 41 | 100 | 15.300 | 0.015 | 0.373 | +0.052 |
|  | 32.5 | 125 |  |  | 0.471 | +0.056 |
|  | 25 | 165 |  |  | 0.612 | +0.073 |
|  | 21 | 200 |  |  | 0.729 | +0.088 |
|  | 18 | 235 |  |  | 0.850 | +0.102 |
|  | 14 | 305 |  |  | 1.093 | +0.131 |
| 16 | 41 | 100 | 17.400 | 0.020 | 0.424 | +0.059 |
|  | 32.5 | 125 |  |  | 0.535 | +0.064 |
|  | 25 | 165 |  |  | 0.696 | +0.084 |
|  | 21 | 200 |  |  | 0.829 | +0.100 |
|  | 18 | 235 |  |  | 0.967 | +0.116 |
|  | 14 | 305 |  |  | 1.243 | +0.149 |
| 18 | 51 | 80 | 19.500 | 0.020 | 0.382 | +0.053 |
|  | 41 | 100 |  |  | 0.476 | +0.067 |
|  | 32.5 | 125 |  |  | 0.600 | +0.072 |
|  | 25 | 165 |  |  | 0.780 | +0.094 |
|  | 21 | 200 |  |  | 0.929 | +0.111 |
|  | 18 | 235 |  |  | 1.083 | +0.130 |
|  | 14 | 305 |  |  | 1.393 | +0.167 |
| 20 | 51 | 80 | 21.600 | 0.025 | 0.424 | +0.059 |
|  | 41 | 100 |  |  | 0.527 | +0.074 |
|  | 32.5 | 125 |  |  | 0.665 | +0.080 |
|  | 25 | 165 |  |  | 0.864 | +0.104 |
|  | 21 | 200 |  |  | 1.029 | +0.123 |
|  | 18 | 235 |  |  | 1.200 | +0.144 |
| 24 | 51 | 80 | 25.800 | 0.030 | 0.506 | +0.071 |
|  | 41 | 100 |  |  | 0.629 | +0.088 |
|  | 32.5 | 125 |  |  | 0.794 | +0.095 |
|  | 25 | 165 |  |  | 1.032 | +0.124 |
|  | 21 | 200 |  |  | 1.229 | +0.147 |
|  | 18 | 235 |  |  | 1.433 | +0.172 |
| 30 | 51 | 80 | 32.000 | 0.040 | 0.627 | +0.088 |
|  | 41 | 100 |  |  | 0.780 | +0.109 |
|  | 32.5 | 125 |  |  | 0.985 | +0.118 |
|  | 25 | 165 |  |  | 1.280 | +0.154 |
|  | 21 | 200 |  |  | 1.524 | +0.183 |
|  | 18 | 235 |  |  | 1.778 | +0.213 |


| $\overline{\text { Appendix 52C }}$ | MaterialProperties,PressureRatings,andPipe <br> Dimensions for Plastic Pipe | Part 636 | National Engineering Handbook |
| :--- | :--- | :--- | :--- |

Table 52C-12 PVC plastic pipe, ductile iron pipe size (IPS) outside diameter—Continued

| Nominal pipe size (in) | SDR | Pressure rating ( $\mathrm{lb} / \mathrm{in}^{2}$ ) |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 36 | 51 | 80 | 38.300 | 0.050 | 0.751 | +0.105 |
|  | 41 | 100 |  |  | 0.934 | +0.131 |
|  | 32.5 | 125 |  |  | 1.178 | +0.141 |
|  | 25 | 165 |  |  | 1.532 | +0.184 |
|  | 21 | 200 |  |  | 1.824 | +0.219 |
| 42 | 51 | 80 | 44.500 | 0.060 | 0.872 | +0.122 |
|  | 41 | 100 |  |  | 1.085 | +0.152 |
|  | 32.5 | 125 |  |  | 1.369 | +0.164 |
|  | 25 | 165 |  |  | 1.780 | +0.214 |
| 48 | 51 |  | 50.800 | 0.075 | 0.996 | +0.139 |
|  | 41 |  |  |  | 1.239 | +0.173 |
|  | 32.5 |  |  |  | 1.563 | +0.188 |
|  | 25 |  |  |  | 2.032 | +0.244 |

Source: AWWA C 905
PVC material Cell class 12454-B as defined by ASTM D 1784 with hydrostatic design basis of 4,000 pounds per square inch.

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Table 52C-13 Polyethylene pipe, iron pipe size outside diameter


| $\overline{\text { Appendix 52C }}$ | MaterialProperties,PressureRatings,andPipe <br> Dimensions for Plastic Pipe | Part 636 <br> National Engineering Handbook |
| :--- | :--- | :--- |
|  |  |  |

Table 52C-13 Polyethylene pipe, iron pipe size outside diameter-Continued

| Nominal pipe size (in) | SDR | Pressure class -- - Material -- |  | Dimension and tolerance |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  |  | -- Outside | diameter - - | Wall thickness |
|  |  | $\begin{aligned} & 2406 \\ & 3406 \end{aligned}$ | 3408 | minimum (in) | tolerance (-/+) | minimum <br> (in) |
| 8 | 32.5 | 40 | 51 | 8.625 | 0.039 | 0.265 |
|  | 26 | 50 | 64 |  |  | 0.332 |
|  | 21 | 63 | 80 |  |  | 0.411 |
|  | 17 | 78 | 100 |  |  | 0.507 |
|  | 15.5 | 86 | 110 |  |  | 0.556 |
|  | 13.5 | 100 | 128 |  |  | 0.639 |
|  | 11 | 125 | 160 |  |  | 0.784 |
|  | 9.3 | 151 | 193 |  |  | 0.927 |
|  | 9 | 156 | 200 |  |  | 0.958 |
|  | 7.3 | 198 | 254 |  |  | 1.182 |
| 10 | 32.5 | 40 | 51 | 10.75 | 0.048 | 0.331 |
|  | 26 | 50 | 64 |  |  | 0.413 |
|  | 21 | 63 | 80 |  |  | 0.512 |
|  | 17 | 78 | 100 |  |  | 0.632 |
|  | 15.5 | 86 | 110 |  |  | 0.694 |
|  | 13.5 | 100 | 128 |  |  | 0.796 |
|  | 11 | 125 | 160 |  |  | 0.977 |
|  | 9.3 | 151 | 193 |  |  | 1.156 |
|  | 9 | 156 | 200 |  |  | 1.194 |
|  | 7.3 | 198 | 254 |  |  | 1.473 |
| 12 | 32.5 | 40 | 51 | 12.75 | 0.057 | 0.392 |
|  | 26 | 50 | 64 |  |  | 0.490 |
|  | 21 | 63 | 80 |  |  | 0.607 |
|  | 17 | 78 | 100 |  |  | 0.750 |
|  | 15.5 | 86 | 110 |  |  | 0.823 |
|  | 13.5 | 100 | 128 |  |  | 0.944 |
|  | 11 | 125 | 160 |  |  | 1.159 |
|  | 9.3 | 151 | 193 |  |  | 1.371 |
|  | 9 | 156 | 200 |  |  | 1.417 |
|  | 7.3 | 198 | 254 |  |  | 1.747 |
| 13 | 32.5 | 40 | 51 | 13.375 | 0.060 | 0.412 |
|  | 26 | 50 | 64 |  |  | 0.515 |
|  | 21 | 63 | 80 |  |  | 0.638 |
|  | 17 | 78 | 100 |  |  | 0.788 |
|  | 15.5 | 86 | 110 |  |  | 0.863 |
|  | 13.5 | 100 | 128 |  |  | 0.991 |
|  | 11 | 125 | 160 |  |  | 1.216 |
|  | 9.3 | 151 | 193 |  |  | 1.438 |
|  | 9 | 156 | 200 |  |  | 1.486 |
|  | 7.3 | 198 | 254 |  |  | 1.832 |

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Table 52C-13 Polyethylene pipe, iron pipe size outside diameter-Continued

| Nominal pipe size (in) | SDR | $\begin{gathered} \text { Pressure class } \\ \text {---- Material ---- } \end{gathered}$ |  | Dimension and tolerance |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  |  | -- Outside | diameter -- | Wall thickness |
|  |  | $\begin{aligned} & 2406 \\ & 3406 \end{aligned}$ | 3408 | minimum (in) | tolerance (-/+) | minimum (in) |
| 14 | 32.5 | 40 | 51 | 14.000 | 0.063 | 0.431 |
|  | 26 | 50 | 64 |  |  | 0.538 |
|  | 21 | 63 | 80 |  |  | 0.667 |
|  | 17 | 78 | 100 |  |  | 0.824 |
|  | 15.5 | 86 | 110 |  |  | 0.903 |
|  | 13.5 | 100 | 128 |  |  | 1.037 |
|  | 11 | 125 | 160 |  |  | 1.273 |
|  | 9.3 | 151 | 193 |  |  | 1.505 |
|  | 9 | 156 | 200 |  |  | 1.556 |
|  | 7.3 | 198 | 254 |  |  | 1.918 |
| 16 | 32.5 | 40 | 51 | 16.000 | 0.072 | 0.492 |
|  | 26 | 50 | 64 |  |  | 0.615 |
|  | 21 | 63 | 80 |  |  | 0.762 |
|  | 17 | 78 | 100 |  |  | 0.941 |
|  | 15.5 | 86 | 110 |  |  | 1.032 |
|  | 13.5 | 100 | 128 |  |  | 1.185 |
|  | 11 | 125 | 160 |  |  | 1.455 |
|  | 9.3 | 151 | 193 |  |  | 1.720 |
|  | 9 | 156 | 200 |  |  | 1.778 |
|  | 7.3 | 198 | 254 |  |  | 2.192 |
| 18 | 32.5 | 40 | 51 | 18.000 | 0.081 | 0.554 |
|  | 26 | 50 | 64 |  |  | 0.692 |
|  | 21 | 63 | 80 |  |  | 0.857 |
|  | 17 | 78 | 100 |  |  | 1.059 |
|  | 15.5 | 86 | 110 |  |  | 1.161 |
|  | 13.5 | 100 | 128 |  |  | 1.333 |
|  | 11 | 125 | 160 |  |  | 1.636 |
|  | 9.3 | 151 | 193 |  |  | 1.935 |
|  | 9 | 156 | 200 |  |  | 2.000 |
|  | 7.3 | 198 | 254 |  |  | 2.466 |
| 20 | 32.5 | 40 | 51 | 20.000 | 0.090 | 0.615 |
|  | 26 | 50 | 64 |  |  | 0.769 |
|  | 21 | 63 | 80 |  |  | 0.952 |
|  | 17 | 78 | 100 |  |  | 1.176 |
|  | 15.5 | 86 | 110 |  |  | 1.290 |
|  | 13.5 | 100 | 128 |  |  | 1.481 |
|  | 11 | 125 | 160 |  |  | 1.818 |
|  | 9.3 | 151 | 193 |  |  | 2.151 |
|  | 9 | 156 | 200 |  |  | 2.222 |
|  | 7.3 | 198 | 254 |  |  | 2.740 |


| $\overline{\text { Appendix 52C }}$ | MaterialProperties,PressureRatings,andPipe <br> Dimensions for Plastic Pipe | Part 636 <br> National Engineering Handbook |
| :--- | :--- | :--- |
|  |  |  |

Table 52C-13 Polyethylene pipe, iron pipe size outside diameter-Continued

| Nominal pipe size (in) | SDR | Pressure class -- - Material -- |  | Dimension and tolerance |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  |  | -- Outside | diameter - - | Wall thickness |
|  |  | $\begin{aligned} & 2406 \\ & 3406 \end{aligned}$ | 3408 | minimum <br> (in) | tolerance (-/+) | minimum <br> (in) |
| 21.5 | 32.5 | 40 | 51 | 21.500 | 0.097 | 0.662 |
|  | 26 | 50 | 64 |  |  | 0.827 |
|  | 21 | 63 | 80 |  |  | 1.024 |
|  | 17 | 78 | 100 |  |  | 1.265 |
|  | 15.5 | 86 | 110 |  |  | 1.387 |
|  | 13.5 | 100 | 128 |  |  | 1.593 |
|  | 11 | 125 | 160 |  |  | 1.955 |
|  | 9.3 | 151 | 193 |  |  | 2.312 |
|  | 9 | 156 | 200 |  |  | 2.389 |
|  | 7.3 | 198 | 254 |  |  | 2.945 |
| 22 | 32.5 | 40 | 51 | 22.000 | 0.099 | 0.677 |
|  | 26 | 50 | 64 |  |  | 0.846 |
|  | 21 | 63 | 80 |  |  | 1.048 |
|  | 17 | 78 | 100 |  |  | 1.294 |
|  | 15.5 | 86 | 110 |  |  | 1.419 |
|  | 13.5 | 100 | 128 |  |  | 1.630 |
|  | 11 | 125 | 160 |  |  | 2.000 |
|  | 9.3 | 151 | 193 |  |  | 2.366 |
|  | 9 | 156 | 200 |  |  | 2.444 |
|  | 7.3 | 198 | 254 |  |  | 3.014 |
| 24 | 32.5 | 40 | 51 | 24.000 | 0.108 | 0.738 |
|  | 26 | 50 | 64 |  |  | 0.923 |
|  | 21 | 63 | 80 |  |  | 1.143 |
|  | 17 | 78 | 100 |  |  | 1.412 |
|  | 15.5 | 86 | 110 |  |  | 1.548 |
|  | 13.5 | 100 | 128 |  |  | 1.778 |
|  | 11 | 125 | 160 |  |  | 2.182 |
|  | 9.3 | 151 | 193 |  |  | 2.581 |
|  | 9 | 156 | 200 |  |  | 2.667 |
|  | 7.3 | 198 | 254 |  |  | 3.288 |
| 26 | 32.5 | 40 | 51 | 26.000 | 0.117 | 0.800 |
|  | 26 | 50 | 64 |  |  | 1.000 |
|  | 21 | 63 | 80 |  |  | 1.238 |
|  | 17 | 78 | 100 |  |  | 1.529 |
|  | 15.5 | 86 | 110 |  |  | 1.677 |
|  | 13.5 | 100 | 128 |  |  | 1.926 |
|  | 11 | 125 | 160 |  |  | 2.364 |
|  | 9.3 | 151 | 193 |  |  | 2.796 |
|  | 9 | 156 | 200 |  |  | 2.889 |
|  | 7.3 | 198 | 254 |  |  | 3.562 |

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Table 52C-13 Polyethylene pipe, iron pipe size outside diameter-Continued

| Nominal pipe size (in) | SDR | $\begin{gathered} \text { Pressure class } \\ \text {---- Material ---- } \end{gathered}$ |  | Dimension and tolerance |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  |  | -- Outside | diameter -- | Wall thickness |
|  |  | $\begin{aligned} & 2406 \\ & 3406 \end{aligned}$ | 3408 | $\underset{(\mathrm{in})}{\operatorname{minimum}}$ | tolerance (-/+) | minimum (in) |
| 28 | 32.5 | 40 | 51 | 28.000 | 0.126 | 0.862 |
|  | 26 | 50 | 64 |  |  | 1.077 |
|  | 21 | 63 | 80 |  |  | 1.333 |
|  | 17 | 78 | 100 |  |  | 1.647 |
|  | 15.5 | 86 | 110 |  |  | 1.806 |
|  | 13.5 | 100 | 128 |  |  | 2.074 |
|  | 11 | 125 | 160 |  |  | 2.545 |
|  | 9.3 | 151 | 193 |  |  | 3.011 |
|  | 9 | 156 | 200 |  |  | 3.111 |
|  | 7.3 | 198 | 254 |  |  | 3.836 |
| 32 | 32.5 | 40 | 51 | 32.000 | 0.144 | 0.985 |
|  | 26 | 50 | 64 |  |  | 1.231 |
|  | 21 | 63 | 80 |  |  | 1.524 |
|  | 17 | 78 | 100 |  |  | 1.882 |
|  | 15.5 | 86 | 110 |  |  | 2.065 |
|  | 13.5 | 100 | 128 |  |  | 2.370 |
|  | 11 | 125 | 160 |  |  | 2.909 |
|  | 9.3 | 151 | 193 |  |  | 3.441 |
|  | 9 | 156 | 200 |  |  | 3.566 |
|  | 7.3 | 198 | 254 |  |  | 4.384 |
| 34 | 32.5 | 40 | 51 | 34.000 | 0.153 | 1.046 |
|  | 26 | 50 | 64 |  |  | 1.308 |
|  | 21 | 63 | 80 |  |  | 1.619 |
|  | 17 | 78 | 100 |  |  | 2.000 |
|  | 15.5 | 86 | 110 |  |  | 2.194 |
|  | 13.5 | 100 | 128 |  |  | 2.519 |
|  | 11 | 125 | 160 |  |  | 3.091 |
|  | 9.3 | 151 | 193 |  |  | 3.656 |
|  | 9 | 156 | 200 |  |  | 3.778 |
|  | 7.3 | 198 | 254 |  |  | 4.658 |
| 36 | 32.5 | 40 | 51 | 36.000 | 0.162 | 1.108 |
|  | 26 | 50 | 64 |  |  | 1.385 |
|  | 21 | 63 | 80 |  |  | 1.714 |
|  | 17 | 78 | 100 |  |  | 2.118 |
|  | 15.5 | 86 | 110 |  |  | 2.323 |
|  | 13.5 | 100 | 128 |  |  | 2.667 |
|  | 11 | 125 | 160 |  |  | 3.273 |
|  | 9.3 | 151 | 193 |  |  | 3.871 |
|  | 9 | 156 | 200 |  |  | 4.000 |
|  | 7.3 | 198 | 254 |  |  | 4.932 |


| $\overline{\text { Appendix 52C }}$ | MaterialProperties,PressureRatings,andPipe <br> Dimensions for Plastic Pipe | Part 636 <br> National Engineering Handbook |
| :--- | :--- | :--- |
|  |  |  |

Table 52C-13 Polyethylene pipe, iron pipe size outside diameter-Continued

| Nominal pipe size (in) | SDR | $\begin{gathered} \text { Pressure class } \\ \text {--- - Material --- } \end{gathered}$ |  | Dimension and tolerance |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  |  | -- Outside | diameter -- | Wall thickness |
|  |  | $\begin{aligned} & 2406 \\ & 3406 \end{aligned}$ | 3408 | minimum (in) | tolerance $(-/+)$ | minimum (in) |
| 42 | 32.5 | 40 | 51 | 42.000 | 0.189 | 1.292 |
|  | 26 | 50 | 64 |  |  | 1.615 |
|  | 21 | 63 | 80 |  |  | 2.000 |
|  | 17 | 78 | 100 |  |  | 2.471 |
|  | 15.5 | 86 | 110 |  |  | 2.710 |
|  | 13.5 | 100 | 128 |  |  | 3.111 |
|  | 11 | 125 | 160 |  |  | 3.818 |
|  | 9.3 | 151 | 193 |  |  | 4.516 |
|  | 9 | 156 | 200 |  |  | 4.667 |
|  | 7.3 | 198 | 254 |  |  | 5.753 |
| 48 | 32.5 | 40 | 51 | 48.000 | 0.216 | 1.477 |
|  | 26 | 50 | 64 |  |  | 1.846 |
|  | 21 | 63 | 80 |  |  | 2.286 |
|  | 17 | 78 | 100 |  |  | 2.824 |
|  | 15.5 | 86 | 110 |  |  | 3.097 |
|  | 13.5 | 100 | 128 |  |  | 3.556 |
|  | 11 | 125 | 160 |  |  | 4.364 |
|  | 9.3 | 151 | 193 |  |  | 5.161 |
|  | 9 | 156 | 200 |  |  | 5.333 |
|  | 7.3 | 198 | 254 |  |  | 6.575 |
| 54 | 32.5 | 40 | 51 | 54.000 | 0.243 | 1.662 |
|  | 26 | 50 | 64 |  |  | 2.077 |
|  | 21 | 63 | 80 |  |  | 2.571 |
|  | 17 | 78 | 100 |  |  | 3.177 |
|  | 15.5 | 86 | 110 |  |  | 3.484 |
|  | 13.5 | 100 | 128 |  |  | 4.000 |
|  | 11 | 125 | 160 |  |  | 4.909 |
|  | 9.3 | 151 | 193 |  |  | 5.807 |
|  | $9$ | 156 | 200 |  |  | 6.000 |
|  | 7.3 | 198 | 254 |  |  | 7.397 |
| 63 | 32.5 | 40 | 51 | 63.000 | 0.284 | 1.938 |
|  | 26 | 50 | 64 |  |  | 2.423 |
|  | 21 | 63 | 80 |  |  | 3.000 |
|  | 17 | 78 | 100 |  |  | 3.706 |
|  | 15.5 | 86 | 110 |  |  | 4.065 |
|  | 13.5 | 100 | 128 |  |  | 4.667 |
|  | 11 | 125 | 160 |  |  | 5.727 |
|  | 9.3 | 151 | 193 |  |  | 6.774 |
|  | 9 | 156 | 200 |  |  | 7.000 |
|  | 7.3 | 198 | 254 |  |  | 8.630 |

Source: AWWA C 906

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Table 52C-14 Polyethylene pipe, ductile iron pipe size outside diameter

| Nominal pipe size (in) | SDR | $\begin{aligned} & \text { Pressure class } \\ & \text {--- Material---- } \end{aligned}$ |  | Dimension and tolerance |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  |  | -- Outside | diameter -- | Wall thickness |
|  |  | $\begin{aligned} & 2406 \\ & 3406 \end{aligned}$ | 3408 | minimum <br> (in) | tolerance $(-/+)$ | $\underset{(\text { in) }}{\operatorname{minimum}}$ |
| 4 | 32.5 | 40 | 51 | 4.800 | 0.022 | 0.148 |
|  | 26 | 50 | 64 |  |  | 0.185 |
|  | 21 | 63 | 80 |  |  | 0.229 |
|  | 17 | 78 | 100 |  |  | 0.282 |
|  | 15.5 | 86 | 110 |  |  | 0.310 |
|  | 13.5 | 100 | 128 |  |  | 0.356 |
|  | 11 | 125 | 160 |  |  | 0.436 |
|  | 9.3 | 151 | 193 |  |  | 0.516 |
|  | 9 | 156 | 200 |  |  | 0.533 |
|  | 7.3 | 198 | 254 |  |  | 0.658 |
| 6 | 32.5 | 40 | 51 | 6.900 | 0.031 | 0.212 |
|  | 26 | 50 | 64 |  |  | 0.265 |
|  | 21 | 63 | 80 |  |  | 0.329 |
|  | 17 | 78 | 100 |  |  | 0.406 |
|  | 15.5 | 86 | 110 |  |  | 0.445 |
|  | 13.5 | 100 | 128 |  |  | 0.511 |
|  | 11 | 125 | 160 |  |  | 0.627 |
|  | 9.3 | 151 | 193 |  |  | 0.742 |
|  | 9 | 156 | 200 |  |  | 0.787 |
|  | 7.3 | 198 | 254 |  |  | 0.945 |
| 8 | 32.5 | 40 | 51 | 9.050 | 0.041 | 0.278 |
|  | 26 | 50 | 64 |  |  | 0.348 |
|  | 21 | 63 | 80 |  |  | 0.431 |
|  | 17 | 78 | 100 |  |  | 0.532 |
|  | 15.5 | 86 | 110 |  |  | 0.584 |
|  | 13.5 | 100 | 128 |  |  | 0.670 |
|  | 11 | 125 | 160 |  |  | 0.823 |
|  | 9.3 | 151 | 193 |  |  | 0.973 |
|  | 9 | 156 | 200 |  |  | 1.006 |
|  | 7.3 | 198 | 254 |  |  | 1.240 |
| 10 | 32.5 | 40 | 51 | 11.100 | 0.050 | 0.342 |
|  | 26 | 50 | 64 |  |  | 0.427 |
|  | 21 | 63 | 80 |  |  | 0.529 |
|  | 17 | 78 | 100 |  |  | 0.653 |
|  | 15.5 | 86 | 110 |  |  | 0.716 |
|  | 13.5 | 100 | 128 |  |  | 0.822 |
|  | 11 | 125 | 160 |  |  | 1.009 |
|  | 9.3 | 151 | 193 |  |  | 1.194 |
|  | 9 | 156 | 200 |  |  | 1.233 |
|  | 7.3 | 198 | 254 |  |  | 1.521 |


| $\overline{\text { Appendix 52C }}$ | $\overline{\text { MaterialProperties,PressureRatings,andPipe }}$ | $\overline{\text { Part 636 }}$ |
| :--- | :--- | :--- |
|  | Dimensions for Plastic Pipe |  |

Table 52C-14 Polyethylene pipe, ductile iron pipe size outside diameter-Continued


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Dimensions for Plastic Pipe

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Table 52C-14 Polyethylene pipe, ductile iron pipe size outside diameter-Continued

| Nominal pipe size (in) | SDR | Pressure class <br> -- - Material - |  | Dimension and tolerance |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  |  | -- Outside | diameter -- | Wall thickness |
|  |  | $\begin{aligned} & 2406 \\ & 3406 \end{aligned}$ | 3408 | minimum <br> (in) | tolerance $(-/+)$ | minimum <br> (in) |
| 20 | 32.5 | 40 | 51 | 21.600 | 0.097 | 0.665 |
|  | 26 | 50 | 64 |  |  | 0.831 |
|  | 21 | 63 | 80 |  |  | 1.029 |
|  | 17 | 78 | 100 |  |  | 1.271 |
|  | 15.5 | 86 | 110 |  |  | 1.394 |
|  | 13.5 | 100 | 128 |  |  | 1.600 |
|  | 11 | 125 | 160 |  |  | 1.964 |
|  | 9.3 | 151 | 193 |  |  | 2.323 |
|  | 9 | 156 | 200 |  |  | 2.400 |
|  | 7.3 | 198 | 254 |  |  | 2.959 |
| 24 | 32.5 | 40 | 51 | 25.800 | 0.116 | 0.794 |
|  | 26 | 50 | 64 |  |  | 0.992 |
|  | 21 | 63 | 80 |  |  | 1.229 |
|  | 17 | 78 | 100 |  |  | 1.518 |
|  | 15.5 | 86 | 110 |  |  | 1.665 |
|  | 13.5 | 100 | 128 |  |  | 1.911 |
|  | 11 | 125 | 160 |  |  | 2.345 |
|  | 9.3 | 151 | 193 |  |  | 2.774 |
|  | 9 | 156 | 200 |  |  | 2.867 |
|  | 7.3 | 198 | 254 |  |  | 3.534 |
| 30 | 32.5 | 40 | 51 | 32.000 | 0.144 | 0.985 |
|  | 26 | 50 | 64 |  |  | 1.231 |
|  | 21 | 63 | 80 |  |  | 1.524 |
|  | 17 | 78 | 100 |  |  | 1.882 |
|  | 15.5 | 86 | 110 |  |  | 2.065 |
|  | 13.5 | 100 | 128 |  |  | 2.370 |
|  | 11 | 125 | 160 |  |  | 2.909 |
|  | 9.3 | 151 | 193 |  |  | 3.441 |
|  | 9 | 156 | 200 |  |  | 3.556 |
|  | 7.3 | 198 | 254 |  |  | 4.384 |
| 36 | 32.5 | 40 | 51 | 38.300 | 0.172 | 1.178 |
|  | 26 | 50 | 64 |  |  | 1.473 |
|  | 21 | 63 | 80 |  |  | 1.824 |
|  | 17 | 78 | 100 |  |  | 2.253 |
|  | 15.5 | 86 | 110 |  |  | 2.471 |
|  | 13.5 | 100 | 128 |  |  | 2.837 |
|  | 11 | 125 | 160 |  |  | 3.482 |
|  | 9.3 | 151 | 193 |  |  | 4.118 |
|  | 9 | 156 | 200 |  |  | 4.256 |
|  | 7.3 | 198 | 254 |  |  | 5.247 |

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Table 52C-14 Polyethylene pipe, ductile iron pipe size outside diameter-Continued

| Nominal pipe size (in) | SDR | $\begin{gathered} \text { Pressure class } \\ \text {--- Material -- } \end{gathered}$ |  | Dimension and tolerance |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  |  | -- Outside | diameter -- | Wall thickness |
|  |  | $\begin{aligned} & 2406 \\ & 3406 \end{aligned}$ | 3408 | minimum (in) | tolerance $(-/+)$ | minimum <br> (in) |
| 42 | 32.5 | 40 | 51 | 44.500 | 0.200 | 1.369 |
|  | 26 | 50 | 64 |  |  | 1.712 |
|  | 21 | 63 | 80 |  |  | 2.119 |
|  | 17 | 78 | 100 |  |  | 2.618 |
|  | 15.5 | 86 | 110 |  |  | 2.871 |
|  | 13.5 | 100 | 128 |  |  | 3.296 |
|  | 11 | 125 | 160 |  |  | 4.046 |
|  | 9.3 | 151 | 193 |  |  | 4.785 |
|  | 9 | 156 | 200 |  |  | 4.944 |
|  | 7.3 | 198 | 254 |  |  | 6.096 |
| 48 | 32.5 | 40 | 51 | 50.800 | 0.229 | 1.563 |
|  | 26 | 50 | 64 |  |  | 1.954 |
|  | 21 | 63 | 80 |  |  | 2.419 |
|  | 17 | 78 | 100 |  |  | 2.988 |
|  | 15.5 | 86 | 110 |  |  | 3.277 |
|  | 13.5 | 100 | 128 |  |  | 3.763 |
|  | 11 | 125 | 160 |  |  | 4.618 |
|  | 9.3 | 151 | 193 |  |  | 5.462 |
|  | 9 | 156 | 200 |  |  | 5.644 |
|  | 7.3 | 198 | 254 |  |  | 6.959 |
| 54 | 32.5 | 40 | 51 | 57.100 | 0.257 | 1.757 |
|  | 26 | 50 | 64 |  |  | 2.196 |
|  | 21 | 63 | 80 |  |  | 2.719 |
|  | 17 | 78 | 100 |  |  | 3.359 |
|  | 15.5 | 86 | 110 |  |  | 3.684 |
|  | 13.5 | 100 | 128 |  |  | 4.230 |
|  | 11 | 125 | 160 |  |  | 5.191 |
|  | 9.3 | 151 | 193 |  |  | 6.140 |
|  | 9 | 156 | 200 |  |  | 6.344 |
|  | 7.3 | 198 | 254 |  |  | 7.822 |

[^7]MaterialProperties,PressureRatings,andPipe
Dimensions for Plastic Pipe

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Table 52C-15 Type PSM PVC pipe

| Nominal pipe size (in) | Outside diameter (in) |  | --------- Minimum wall thickness (in) |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | average | tolerance | SDR 41 | SDR 35 | SDR 26 | SDR 23.5 |
| 4 | 4.215 | 0.009 |  | 0.120 | 0.162 | 0.178 |
| 6 | 6.275 | 0.011 | 0.153 | 0.180 | 0.241 | 0.265 |
| 8 | 8.400 | 0.012 | 0.205 | 0.240 | 0.323 |  |
| 9 | 9.440 | 0.014 | 0.230 |  |  |  |
| 10 | 10.500 | 0.015 | 0.256 | 0.300 | 0.404 |  |
| 12 | 12.500 | 0.018 | 0.305 | 0.360 | 0.481 |  |
| 15 | 15.300 | 0.023 | 0.375 | 0.437 | 0.588 |  |

## Source: ASTM D 3034

Note: PSM is not an abbreviation, but rather an arbitrary designation for a product having certain dimensions.

Table 52C-16 PVC large-diameter plastic pipe

| $\begin{array}{l}\text { Nominal } \\ \text { pipe size } \\ \text { (in) }\end{array}$ | Outside diameter (in) |  |
| :--- | :--- | :--- | :--- | :--- | :--- |
| average |  |  |
| tolerance |  |  |\(\left.\quad \begin{array}{c}Minimum wall thickness (in) <br>

cell class <br>
12454\end{array} \quad $$
\begin{array}{c}\text { Minimum pipe } \\
\text { cellass } \\
\text { sitifness } \\
\text { (lb/in2) }\end{array}
$$\right]\)

[^8]
## MaterialProperties,PressureRatings,andPipe <br> Dimensions for Plastic Pipe

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Table 52C-17 Smooth wall PVC plastic underdrain pipe

| Nominal <br> pipe size <br> (in) | Outside diameter (in) |  | Minimum wall thickness (in) <br> PS28 |  |
| :--- | :--- | :--- | :--- | :--- |
| 4 | 4.215 | 0.009 | 0.103 | 0.120 |
| PS46 |  |  |  |  |

Source: ASTM F 758
Note: $\quad$ PS = pipe stiffness

Table 52C-18 Type PS46 and PS115 PVC plastic pipe

| Nominal pipe size (in) | $\begin{gathered} \text { Pipe } \\ \text { stiffness } \\ \left(\mathrm{lb} / \mathrm{in}^{2}\right) \end{gathered}$ | Outside diameter (in) average tolerance |  | T-1 |  | - Wall th | ess (in | T-3 |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  |  | est. avg. | minimum | est. avg. | minimum | est. avg. | minimum |
| 4 | 46.000 | 4.215 | 0.009 | 0.114 | 0.107 | 0.111 | 0.104 | 0.108 | 0.102 |
|  | 115.000 |  |  | 0.152 | 0.143 | 0.148 | 0.139 | 0.144 | 0.135 |
| 6 | 46.000 | 6.275 | 0.011 | 0.170 | 0.160 | 0.165 | 0.155 | 0.161 | 0.151 |
|  | 115.000 |  |  | 0.226 | 0.214 | 0.220 | 0.207 | 0.215 | 0.202 |
| 8 | 46.000 | 8.400 | 0.012 | 0.227 | 0.213 | 0.221 | 0.208 | 0.216 | 0.203 |
|  | 115.000 |  |  | 0.302 | 0.284 | 0.294 | 0.276 | 0.287 | 0.270 |
| 10 | 46.000 | 10.500 | 0.015 | 0.284 | 0.267 | 0.276 | 0.259 | 0.270 | 0.254 |
|  | 115.000 |  |  | 0.378 | 0.355 | 0.363 | 0.341 | 0.359 | 0.337 |
| 12 | 46.000 | 12.500 | 0.018 | 0.338 | 0.318 | 0.329 | 0.309 | 0.321 | 0.302 |
|  | 115.000 |  |  | 0.450 | 0.423 | 0.438 | 0.414 | 0.428 | 0.402 |
| 15 | 46.000 | 15.300 | 0.023 | 0.414 | 0.389 | 0.403 | 0.379 | 0.393 | 0.369 |
|  | 115.000 |  |  | 0.548 | 0.515 | 0.536 | 0.504 | 0.523 | 0.492 |
| 18 | 46.000 | 18.700 | 0.028 | 0.507 | 0.477 | 0.494 | 0.464 | 0.482 | 0.452 |
|  | 115.000 |  |  | 0.673 | 0.633 | 0.655 | 0.616 | 0.640 | 0.602 |

[^9]MaterialProperties,PressureRatings,andPipe
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Table 52C-19 Open and dual wall PVC profile plastic pipe dimensions and tolerances

| Nominal pipe size (in) | Inside diameter (in) minimum tolerance |  | -- - Minimum wall thickness in waterway (in) -- $\underset{\text { open profile }}{\text { dual wall }}$ |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  |  | PS 46 | PS 10 | PS 46 |
| 4 | 3.939 | 0.034 |  | 0.030 |  | 0.022 |
| 6 | 5.875 | 0.049 |  | 0.045 |  | 0.025 |
| 8 | 7.863 | 0.053 |  | 0.060 |  | 0.035 |
| 10 | 9.825 | 0.067 |  | 0.070 |  | 0.045 |
| 12 | 11.687 | 0.085 |  | 0.085 |  | 0.058 |
| 15 | 14.303 | 0.116 |  | 0.105 |  | 0.077 |
| 18 | 17.510 | 0.195 | 0.040 | 0.130 | 0.070 | 0.084 |
| 21 | 20.656 | 0.200 | 0.085 | 0.160 | 0.070 | 0.095 |
| 24 | 23.412 | 0.204 | 0.105 | 0.180 | 0.070 | 0.110 |
| 27 | 26.371 | 0.209 | 0.115 | 0.205 | 0.070 | 0.120 |
| 30 | 29.388 | 0.220 | 0.130 | 0.235 | 0.085 | 0.130 |
| 33 | 32.405 | 0.227 | 0.150 | 0.260 | 0.095 | 0.150 |
| 36 | 35.370 | 0.235 | 0.165 | 0.290 | 0.105 | 0.155 |
| 39 | 38.380 | 0.245 | 0.195 | 0.315 | 0.120 | 0.200 |
| 42 | 41.370 | 0.255 | 0.215 | 0.345 | 0.130 | 0.200 |
| 45 | 44.365 | 0.265 | 0.225 | 0.370 | 0.145 | 0.200 |
| 48 | 47.355 | 0.285 | 0.230 | 0.400 | 0.160 | 0.200 |

[^10]
## Appendix 52C

## MaterialProperties,PressureRatings,andPipe <br> Dimensions for Plastic Pipe

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Table 52C-20 PVC corrugated pipe with smooth interior dimensions and tolerances

| Nominal <br> piep size <br> (in) | Pipe <br> stifness <br> (bb/in ) | Outside diameter (in) <br> average <br> tolerance | Inside diameter (in) <br> average <br> tolerance | Minimum wall thickness (in) <br> inner wall <br> outer wall <br> at valley |  |  |  |  |
| :--- | ---: | :--- | :--- | :--- | :--- | :--- | :--- | :--- |
| 4 | 46 | 4.300 | 0.009 | 3.950 | 0.011 | 0.022 | 0.018 | 0.028 |
| 6 | 46 | 6.420 | 0.011 | 5.909 | 0.015 | 0.025 | 0.022 | 0.032 |
| 8 | 46 | 8.600 | 0.012 | 7.881 | 0.018 | 0.035 | 0.030 | 0.045 |
|  | 115 |  |  |  |  | 0.037 | 0.050 | 0.048 |
| 10 | 46 | 10.786 | 0.015 | 9.846 | 0.021 | 0.045 | 0.036 | 0.055 |
|  | 115 |  |  |  |  | 0.046 | 0.052 | 0.065 |
| 12 | 46 | 12.795 | 0.018 | 11.715 | 0.028 | 0.058 | 0.049 | 0.072 |
|  | 115 |  |  |  |  | 0.070 | 0.068 | 0.091 |
| 15 | 46 | 15.658 | 0.023 | 14.338 | 0.035 | 0.077 | 0.055 | 0.092 |
|  | 115 |  |  |  |  | 0.092 | 0.088 | 0.118 |
| 18 | 46 | 19.152 | 0.028 | 17.552 | 0.042 | 0.084 | 0.067 | 0.103 |
| 21 | 46 | 22.630 | 0.033 | 20.705 | 0.049 | 0.095 | 0.073 | 0.110 |
| 24 | 46 | 25.580 | 0.039 | 23.469 | 0.057 | 0.110 | 0.085 | 0.123 |
| 27 | 46 | 28.860 | 0.049 | 26.440 | 0.069 | 0.120 | 0.091 | 0.137 |
| 30 | 46 | 32.150 | 0.059 | 29.469 | 0.081 | 0.130 | 0.105 | 0.147 |
| 36 | 46 | 38.740 | 0.079 | 35.475 | 0.105 | 0.150 | 0.125 | 0.171 |

[^11]MaterialProperties,PressureRatings,andPipe
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Table 52C-21 Open profile polyethylene pipe dimensions and tolerances

| Nominal pipe size (in) | Inside diameter (in) |  | Minimum wall thickness in pipe waterway (in) |  |  |  | Min. bell thickness (in) |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | average | tolerance | RSC 40 | RSC 63 | RSC 100 | RSC 160 |  |
| 18 | 18.00 | 0.38 | 0.18 | 0.18 | 0.18 | 0.22 | 0.7 |
| 21 | 21.00 | 0.38 | 0.18 | 0.18 | 0.18 | 0.24 | 0.7 |
| 24 | 24.00 | 0.38 | 0.18 | 0.18 | 0.22 | 0.24 | 0.7 |
| 27 | 27.00 | 0.38 | 0.18 | 0.18 | 0.24 | 0.24 | 0.7 |
| 30 | 30.00 | 0.38 | 0.18 | 0.22 | 0.24 | 0.26 | 0.7 |
| 33 | 33.00 | 0.38 | 0.18 | 0.24 | 0.24 | 0.30 | 0.95 |
| 36 | 36.00 | 0.38 | 0.18 | 0.24 | 0.26 | 0.30 | 1.05 |
| 42 | 42.00 | 0.42 | 0.24 | 0.24 | 0.30 | 0.38 | 1.15 |
| 48 | 48.00 | 0.48 | 0.24 | 0.26 | 0.30 | 0.38 | 1.25 |
| 54 | 54.00 | 0.54 | 0.24 | 0.30 | 0.38 | 0.42 | 1.25 |
| 60 | 60.00 | 0.60 | 0.26 | 0.30 | 0.38 | 0.52 | 1.3 |
| 66 | 66.00 | 0.66 | 0.30 | 0.38 | 0.42 | 0.67 | 1.3 |
| 72 | 72.00 | 0.72 | 0.30 | 0.38 | 0.42 | 0.90 | 1.3 |
| 78 | 78.00 | 0.78 | 0.30 | 0.38 | 0.52 | 0.90 | 1.35 |
| 84 | 84.00 | 0.84 | 0.38 | 0.42 | 0.67 | 0.90 | 1.35 |
| 90 | 90.00 | 0.90 | 0.38 | 0.42 | 0.90 | 0.95 | 1.35 |
| 96 | 96.00 | 0.96 | 0.38 | 0.52 | 0.90 | 0.95 | 1.35 |
| 108 | 108.00 | 1.08 | 0.42 | 0.67 | 0.90 | 0.95 | 1.35 |
| 120 | 120.00 | 1.20 | 0.52 | 0.67 | 0.90 | 0.95 | 1.35 |

Source: ASTM F 894
Appendix 52C

MaterialProperties,PressureRatings,andPipe
Dimensions for Plastic Pipe

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Table 52C-22 Closed profile polyethylene pipe dimensions and tolerances

| Nominal pipe size (in) | Inside dian average | neter (in) tolerance | Min. wall thickness in pipe waterway (in) | Min. bell thickness (in) |
| :---: | :---: | :---: | :---: | :---: |
| 10 | 10.00 | 0.38 | 0.18 | 0.5 |
| 12 | 12.00 | 0.38 | 0.18 | 0.5 |
| 15 | 15.00 | 0.38 | 0.18 | 0.5 |
| 18 | 18.00 | 0.38 | 0.18 | 0.5 |
| 21 | 21.00 | 0.38 | 0.18 | 0.5 |
| 24 | 24.00 | 0.38 | 0.18 | 0.5 |
| 27 | 27.00 | 0.38 | 0.18 | 0.5 |
| 30 | 30.00 | 0.38 | 0.18 | 0.5 |
| 33 | 33.00 | 0.38 | 0.18 | 0.5 |
| 36 | 36.00 | 0.38 | 0.18 | 0.5 |
| 40 | 40.00 | 0.38 | 0.18 | 0.5 |
| 42 | 42.00 | 0.42 | 0.18 | 0.5 |
| 48 | 48.00 | 0.48 | 0.18 | 0.5 |
| 54 | 54.00 | 0.54 | 0.18 | 0.5 |
| 60 | 60.00 | 0.60 | 0.18 | 0.6 |
| 66 | 66.00 | 0.66 | 0.18 | 0.6 |
| 72 | 72.00 | 0.72 | 0.18 | 0.6 |
| 78 | 78.00 | 0.78 | 0.18 | 0.6 |
| 84 | 84.00 | 0.84 | 0.18 | 0.7 |
| 90 | 90.00 | 0.90 | 0.18 | 0.7 |
| 96 | 96.00 | 0.96 | 0.18 | 0.7 |
| 108 | 108.00 | 1.08 | 0.18 | 0.7 |
| 120 | 120.00 | 1.20 | 0.18 | 0.8 |

## Appendix 52D

## Selection Properties of Corrugated and Spiral Rib Metal Pipe

Table 52D-1 Section properties of corrugated steel pipe

| Gage | Specified thickness (galvanized) (in) | ---------1-1/2" x 1/4" Corrugation - |  |  | ---------2-2/3" x 1/2" Corrugation --------- |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | $\begin{gathered} \text { Area of } \\ \text { section, } \mathrm{A}_{\mathrm{s}} \\ \left(\mathrm{in}^{2} / \mathrm{ft}\right) \end{gathered}$ | Moment of I, inertia (in $4 / \mathrm{in}$ ) | Radius of gyration, r (in) | Area of section, $\mathrm{A}_{\mathrm{s}}$ (in ${ }^{2} / \mathrm{ft}$ ) | Moment of I, inertia (in $4 / \mathrm{in}$ ) | Radius of gyration, r (in) |
| 20 | 0.040 | 0.456 | 0.000253 | 0.0816 | 0.465 | . 001122 | 0.1702 |
| 18 | 0.052 | 0.608 | 0.000343 | 0.0824 | 0.619 | . 001500 | 0.1707 |
| 16 | 0.064 | 0.761 | 0.000439 | 0.0832 | 0.775 | . 001892 | 0.1712 |
| 14 | 0.079 | 0.950 | 0.000566 | 0.0846 | 0.968 | . 002392 | 0.1721 |
| 12 | 0.109 | 1.333 | 0.000857 | 0.0879 | 1.356 | . 003425 | 0.1741 |
| 10 | 0.138 | 1.712 | 0.001205 | 0.0919 | 1.744 | . 004533 | 0.1766 |
| 8 | 0.168 | 2.098 | 0.001635 | 0.0967 | 2.133 | . 005725 | 0.1795 |
| Gage | Specified | -----------3" x $1^{\prime \prime}$ Corrugation ----------- |  |  | -----------5" x 1" Corrugation----------- |  |  |
|  | thickness (galvanized) (in) | $\begin{gathered} \text { Area of } \\ \text { section, } \mathrm{A}_{\mathrm{s}} \\ \left(\mathrm{in}^{2} / \mathrm{ft}\right) \end{gathered}$ | Moment of I, inertia (Ix10-3in ${ }^{4} / \mathrm{in}$ ) | Radius of gyration, r (in) | $\begin{gathered} \text { Area of } \\ \text { section, } \mathrm{A}_{\mathrm{s}} \\ \left(\mathrm{in}^{2} / \mathrm{ft}\right) \end{gathered}$ | $\begin{aligned} & \text { Moment of } \\ & \text { I, inertia } \\ & \left(\mathrm{I} \times 10^{-3} \mathrm{in} 4 / \mathrm{in}\right) \end{aligned}$ | Radius of gyration, r (in) |
| 18 | 0.052 | 0.711 | 0.006892 | 0.3410 |  |  |  |
| 16 | 0.064 | 0.890 | 0.008658 | 0.3417 | 0.794 | . 008850 | 0.3657 |
| 14 | 0.079 | 1.113 | 0.010883 | 0.3427 | 0.992 | . 011092 | 0.3663 |
| 12 | 0.109 | 1.560 | 0.015458 | 0.3448 | 1.390 | . 015550 | 0.3677 |
| 10 | 0.138 | 2.008 | 0.020175 | 0.3472 | 1.788 | . 020317 | 0.3693 |
| 8 | 0.168 | 2.458 | 0.025083 | 0.3499 | 2.186 | . 025092 | 0.3711 |

Source: ASTM A 796
AASHTO Standard Specifications for Highway Bridges

Table 52D-2 Ultimate longitudinal seam strength of riveted or spot welded corrugated steel pipe

| Gage | Seam strength ( $\mathrm{lb} / \mathrm{ft}$ of seam) |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | thickness |  |  |  |  | $3 \times 1$ "and |  | $3 \times 1$ " and |
|  | $\underset{\text { (in) }}{\text { (galvanized) }}$ | single | double | single | double | $5 \times 1$ " double |  | $5 \times 1 "$ |
| 16 | . 064 | 16,700 | 21,600 |  |  | 28,700 |  |  |
| 14 | 0.079 | 18,200 | 29,800 |  |  | 35,700 |  |  |
| 12 | 0.109 |  |  | 23,400 | 46,800 |  | 53,000 |  |
| 10 | 0.138 |  |  | 24,500 | 49,000 |  | 63,700 |  |
| 8 | 0.168 |  |  | 25,600 | 51,300 |  | 70,700 |  |

[^12]| Appendix 52D | Selection Properties of Corrugated and <br> Spiral Rib Metal Pipe |
| :--- | :--- |

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Table 52D-3 Section properties of corrugated aluminum pipe

| Gage | Specified thickness <br> (in) | --------- 1-1/2" x 1/4" Corrugation --------- |  |  | ---------2-2/3" x 1/2" Corrugation -------- - |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | $\begin{gathered} \text { Area of } \\ \text { section, } \mathrm{A}_{\mathrm{s}} \\ \left(\text { in }^{2} / \mathrm{ft}\right) \end{gathered}$ | Moment of I, inertia (in ${ }^{4} / \mathrm{in}$ ) | Radius of gyration, r (in) | $\begin{gathered} \text { Area of } \\ \text { section, } \mathrm{A}_{\mathrm{s}} \\ \left(\text { in }^{2} / \mathrm{ft}\right) \end{gathered}$ | Moment of I, inertia (in $4 / \mathrm{in}$ ) | Radius of gyration, r (in) |
| 18 | 0.048 | 0.608 | 0.000344 | 0.0824 |  |  |  |
| 16 | 0.060 | 0.761 | 0.000349 | 0.0832 | 0.775 | 0.001892 | 0.1712 |
| 14 | 0.075 | - | - | - | 0.968 | 0.002392 | 0.1721 |
| 12 | 0.105 | - | - | - | 1.356 | 0.003425 | 0.1741 |
| 10 | 0.135 | - | - | - | 1.745 | 0.004533 | 0.1766 |
| 8 | 0.164 | - | - | - | 2.130 | 0.005725 | 0.1795 |


| Gage | Specified thickness <br> (in) | ----------- 3" x 1" Corrugation ----------- - - |  |  | Area of section, $\mathrm{A}_{\mathrm{s}}$ (in ${ }^{2} / \mathrm{ft}$ ) | -- - 6" x 1" Corrugation |  | Radius of gyration, r (in) |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | $\begin{gathered} \text { Area of } \\ \text { section, } \mathrm{A}_{\mathrm{s}} \\ \left(\mathrm{in}^{2} / \mathrm{ft}\right) \end{gathered}$ | Moment of I, inertia (in ${ }^{4} / \mathrm{in}$ ) | Radius of gyration, r (in) |  | Effective area (in $\left.{ }^{2} / \mathrm{ft}\right)$ | Moment of I, inertia (in $4 / \mathrm{in}$ ) |  |
| 16 | 0.060 | 0.890 | 0.008659 | 0.3417 | 0.775 | 0.387 | 0.008505 | 0.3629 |
| 14 | 0.075 | 1.118 | 0.010883 | 0.3427 | 0.968 | 0.484 | 0.010631 | 0.3630 |
| 12 | 0.105 | 1.560 | 0.015459 | 0.3448 | 1.356 | 0.678 | 0.014340 | 0.3636 |
| 10 | 0.135 | 2.008 | 0.020183 | 0.3472 | 1.744 | 0.872 | 0.019319 | 0.3646 |
| 8 | 0.164 | 2.458 | 0.025091 | 0.3499 | 2.133 | 1.066 | 0.02376 | 0.3656 |

Source: ASTM B 790
AASHTO Standard Specification for Highway Bridges

Table 52D-4 Ultimate longitudinal seam strength of riveted corrugated aluminum pipe

| Gage | Seam strength ( $\mathrm{lb} / \mathrm{ft}$ of seam) |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | Specified thickness | $\begin{aligned} & ---5 / 16 \text { in rivets }------2 / 3 \times 1 / 2 \text { in ---- } \\ & ---2) \end{aligned}$ |  |  |  |  |  | $1 / 2$ in rivets <br> $3 \times 1$ in and $5 \times \operatorname{lin}$ double |
|  | (in) | single | double | single | double | $5 \times 1 \text { in }$ double |  |  |
| 16 | 0.064 | 9,000 | 14,000 |  |  | 16,500 |  |  |
| 14 | 0.075 | 9,000 | 18,000 |  |  | 20,500 |  |  |
| 12 | 0.105 |  |  | 15,600 | 31,500 |  | 28,000 |  |
| 10 | 0.135 |  |  | 16,200 | 33,000 |  | 42,000 |  |
| 8 | 0.164 |  |  | 16,800 | 34,000 |  | 54,500 |  |

Source: ASTM B 790
Selection Properties of Corrugated and
Spiral Rib Metal Pipe

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Table 52D-5 Section properties of spiral rib steel pipe

| Gage | Specified thickness (galvanized) (in) | --------- - 3/4" x 3/4" x 7-1/2" ------- - |  |  | ---------3/4" x $1^{\prime \prime}$ x 11-1/2" -------- |  |  | --------- - 3/4" x $1^{\prime \prime}$ x 8-1/2" -------- |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | $\begin{gathered} \text { Area of } \\ \text { section, } \mathrm{A}_{\mathrm{s}} \\ \left(\mathrm{in}^{2} / \mathrm{ft}\right) \end{gathered}$ | Moment of I, inertia (in $4 / \mathrm{in}$ ) | Radius of gyration, r (in) | $\begin{gathered} \text { Area of } \\ \text { section, } \mathrm{A}_{\mathrm{s}} \\ \left(\mathrm{in}^{2} / \mathrm{ft}\right) \end{gathered}$ | Moment of I, inertia (in $4 / \mathrm{in}$ ) | Radius of gyration, r (in) | $\begin{gathered} \text { Area of } \\ \text { section, } \mathrm{A}_{\mathrm{s}} \\ \left(\mathrm{in}^{2} / \mathrm{ft}\right) \end{gathered}$ | Moment of I, inertia (in $4 / \mathrm{in}$ ) | Radius of gyration, (in) |
| 16 | 0.064 | 0.509 | 0.002821 | 0.258 | 0.374 | 0.00458 | 0.383 | 0.499 | 0.005979 | 0.379 |
| 14 | 0.079 | 0.712 | 0.003701 | 0.25 | 0.524 | 0.00608 | 0.373 | 0.694 | 0.007913 | 0.37 |
| 12 | 0.109 | 1.184 | 0.005537 | 0.237 | 0.883 | 0.00926 | 0.355 | 1.149 | 0.011983 | 0.354 |
| 10 | 0.138 | 1.717 | 0.007433 | 0.228 |  |  |  |  |  |  |

Source: ASTM A 796

Table 52D-6 Section properites of spiral rib aluminum pipe

| Gage | Specified thickness <br> (in) | ---------3/4" x 3/4" x 7-1/2" ---------------3/4" x 1" x 11-1/2" ---------- |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | $\begin{gathered} \text { Area of } \\ \text { section, } \mathrm{A}_{\mathrm{s}} \\ \left(\mathrm{in}^{2} / \mathrm{ft}\right) \end{gathered}$ | Moment of I, inertia (in ${ }^{4} / \mathrm{in}$ ) | Radius of gyration, r (in) | $\begin{gathered} \text { Area of } \\ \text { section, } \mathrm{A}_{\mathrm{s}} \\ \left(\mathrm{in}^{2} / \mathrm{ft}\right) \end{gathered}$ | Moment of I, inertia (in ${ }^{4} / \mathrm{in}$ ) | Radius of gyration, r (in) |
| 16 | 0.06 | 0.415 | 0.002558 | 0.272 | 0.312 | 0.00408 | 0.396 |
| 14 | 0.075 | 0.569 | 0.003372 | 0.267 | 0.427 | 0.00545 | 0.391 |
| 12 | 0.105 | 0.914 | 0.005073 | 0.258 | 0.697 | 0.00839 | 0.38 |
| 10 | 0.135 | 1.29 | 0.006826 | 0.252 | 1.009 | 0.01148 | 0.369 |

## Appendix 52E

## Allowable Flexibility Factors of Corrugated and Spiral Rib Metal Pipe

Table 52E-1 Flexibility factor for corrugated metal pipe

| Depth of <br> corruga- <br> tion (in) | Material <br> thickness <br> (in) | steel |  |  | In trench <br> aluminum |
| :--- | :---: | :---: | :---: | :---: | :---: |
| $1 / 4$ | 0.060 | 0.043 | 0.031 | 0.043 | 0.031 |
|  | 0.075 | 0.043 | 0.061 | 0.043 | 0.061 |
|  | others | 0.043 | 0.092 | 0.043 | 0.092 |
|  | 0.060 | 0.060 | 0.031 | 0.043 | 0.031 |
| $1 / 2$ | 0.075 | 0.060 | 0.061 | 0.043 | 0.061 |
|  | others | 0.060 | 0.092 | 0.043 | 0.092 |
|  | all | 0.060 | 0.060 | 0.033 | 0.060 |
| 1 | all | 0.020 | - | 0.020 | - |
| 2 | all | - | 0.025 | - | 0.025 |
| $21 / 2$ | all | 0.020 | - | 0.020 | - |
| $51 / 2$ |  |  |  |  |  |

Source: ASTM A 796 and B 790

Table 52E-2 Flexibility factor for spiral rib metal pipe

| Profile <br> (in) | In trench w/compacted steel soil envelope aluminum |  | Flexibility f In trench w/ soil steel | $\begin{aligned} & \text { or (in/lbf) - } \\ & \text { compacted } \\ & \text { elope } \\ & \text { aluminum } \end{aligned}$ | Emb <br> steel | kment <br> aluminum |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $3 / 4 \times 3 / 4 \times 7-1 / 2$ | $0.367 \mathrm{I}^{1 / 3}$ | $0.600 \mathrm{I}^{1 / 3}$ | $0.263 \mathrm{I}^{1 / 3}$ | $0.420 \mathrm{I}^{1 / 3}$ | $0.217 \mathrm{I}^{1 / 3}$ | $0.340 \mathrm{I}^{1 / 3}$ |
| $3 / 4 \times 1 \times 8-1 / 2$ | $0.262 \mathrm{I}^{1 / 3}$ |  | $0.163 \mathrm{I}^{1 / 3}$ |  | $0.140 \mathrm{I}^{1 / 3}$ |  |
| $3 / 4 \times 1 \times 11-1 / 2$ | $0.220 \mathrm{I}^{1 / 3}$ | $0.310 \mathrm{I}^{1 / 3}$ | $0.163 \mathrm{I}^{1 / 3}$ | $0.215 \mathrm{I}^{1 / 3}$ | $0.140 \mathrm{I}^{1 / 3}$ | $0.175 \mathrm{I}^{1 / 3}$ |

[^13]
# Appendix 52F Nominal Thickness for Standard Pressure Classes of Ductile Iron Pipe 

Table 52F-1 Nominal thickness for standard pressure classes of ductile iron pipe and allowances for casting tolerance

| Size <br> in | Outside diameter, in (mm) | --- <br> $-\mathbf{1 5 0}$ | -- - - Nominal thickness, in (mm) <br> --------- Pressure class - |  |  |  | Casting tolerance, in (mm) |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  | 200 | 250 | 300 | 350 |  |
| 3 | 3.96 (100.6) | - | - | - | - | 0.25 (6.4) | 0.05 (1.3) |
| 4 | 4.80 (121.9) | - | - | - | - | 0.25 (6.4) | 0.05 (1.3) |
| 6 | 6.90 (175.3) | - | - | - | - | 0.25 (6.4) | 0.05 (1.3) |
| 8 | 9.05 (229.9) | - | - | - | - | 0.25 (6.4) | 0.05 (1.3) |
| 10 | 11.10 (281.9) | - | - | - | - | 0.26 (6.6) | 0.06 (1.5) |
| 12 | 13.20 (335.3) | - | - | - | - | 0.28 (7.1) | 0.06 (1.5) |
| 14 | 15.30 (388.6) | - | - | 0.28 (7.1) | 0.30 (7.6) | 0.31 (7.9) | 0.07 (1.8) |
| 16 | 17.40 (442.0) | - | - | 0.30 (7.6) | 0.32 (8.1) | 0.34 (8.6) | 0.07 (1.8) |
| 18 | 19.50 (495.3) | - | - | 0.31 (7.9) | 0.34 (8.6) | 0.36 (9.1) | 0.07 (1.8) |
| 20 | 21.60 (548.6) | - | - | 0.33 (8.4) | 0.36 (9.1) | 0.38 (9.7) | 0.07 (1.8) |
| 24 | 25.80 (655.3) | - | 0.33 (8.4) | 0.37 (9.4) | 0.40 (10.2) | 0.43 (10.9) | 0.07 (1.8) |
| 30 | 32.00 (812.8) | 0.34 (8.6) | 0.38 (9.7) | 0.42 (10.7) | 0.45 (11.4) | 0.49 (12.4) | 0.07 (1.8) |
| 36 | 38.30 (972.8) | 0.38 (9.7) | 0.42 (10.7) | 0.47 (11.9) | 0.51 (12.9) | 0.56 (14.2) | 0.07 (1.8) |
| 42 | 44.50 (1,130.3) | 0.41 (10.4) | 0.47 (11.9) | 0.52 (13.2) | 0.57 (14.5) | 0.63 (16.0) | 0.07 (1.8) |
| 48 | 50.80 (1,290.3) | 0.46 (11.7) | 0.52 (13.2) | 0.58 (14.7) | 0.64 (16.3) | 0.70 (17.8) | 0.08 (2.0) |
| 54 | 57.56 (1,450.3) | 0.51 (12.9) | 0.58 (14.7) | 0.65 (16.5) | 0.72 (18.3) | 0.79 (20.1) | 0.09 (2.3) |
| 60 | 61.61 (1,564.9) | 0.54 (13.7) | 0.61 (15.5) | 0.68 (17.3) | 0.76 (19.3) | 0.83 (21.1) | 0.09 (2.3) |
| 64 | 65.67 (1,668.0) | 0.56 (14.2) | 0.64 (16.3) | 0.72 (18.3) | 0.80 (20.3) | 0.87 (22.1) | 0.09 (2.3) |

Source: ASTM A 746


[^0]:    1/ Source ASCE Journal of Geotechnical Engineering Division, January 1977
    2/ LL = liquid limit

[^1]:    Source: AWWA, 2002

[^2]:    Source: ASTM D 1527, D 1785, D 2104, D 2239, D 2241, D 2282, and D 3035.

[^3]:    Source: ASTM D 2239

[^4]:    Source: ASTM D 2104 for inside diameter controlled and D 2447 for outside diameter controlled. NPR: Not Pressure Rated

[^5]:    Source: ASTM D 2737

[^6]:    Source: AWWA C 901

[^7]:    Source: AWWA C 906.

[^8]:    Source: ASTM F 679

    * Cast iron pipe size

[^9]:    Source: ASTM F 789
    T-1: Made with material that has modulus of 440,000 to $480,000 \mathrm{lb} / \mathrm{in}^{2}$.
    T-2: Made with material that has modulus of 480,000 to $520,000 \mathrm{lb} / \mathrm{in}^{2}$.
    T-3: Made with material that has modulus of 520,000 to $560,000 \mathrm{lb} / \mathrm{in}^{2}$.

[^10]:    Source: ASTM F 794

[^11]:    Source: ASTM F 949

[^12]:    Source: ASTM A 796.

[^13]:    Source: ASTM A 796 and B 790

